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DEVELOPMENT AND EXPERIMENTAL VERIFICATION OF PROCEDURES TO DETERMINE NONLINEAR LOAD-DEFLECTION CHARACTERISTICS OF HELICOPTER SUBSTRUCTURES SUBJECTED TO CRASH FORCES. VOLUME II. TEST DATA AND DESCRIPTION OF REFINED PROGRAM 'KRASH', INCLUDING A USER'S GUIDE AND SAMPLE CASE

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4. Continued.

TEST DATA AND DESCRIPTION OF REFINED PROGRAM "KRASH", INCLUDING A
USER'S GUIDE AND SAMPLE CASE

20. Continued

Studies using an existing 31 lumped mass model of the UH-1H helicopter are performed to determine the sensitivity of responses to changes in the load-deflection representation for the engine and transmission mounts, landing gear and fuselage. Simplified techniques are used to predict the load-deflection curve for the crushing of a segment of the lower fuselage under impact conditions. The predictions include elastic behavior, failure load and post-failure behavior. The structural segment selected for analysis and test is a section supported on four edges, representative of the transmission pylon support.

Twelve specimens were fabricated. The specimens are 46 inches long by 18 inches wide by 6.125 inches to 12.125 inches deep. The 6.125-inch-deep specimens are approximately half the size (except for thickness) and are varied in detail design (number of angles, spacing of angles, lightening holes). Static and dynamic tests were performed. The predicted load-deflection curves are compared to the test load-deflection curves and show good agreement with regard to peak failure load, failure point, energy absorbed, and shape. The results of the tests show that for this type of typical fuselage structure, static tests provide load-deflection data which is similar to data that can be obtained from dynamic tests, but more economically.

Program KRASH is refined to facilitate its use by designers. In particular, the input data is reordered, some inputs are standardized and more general load-deflection curve characteristics are possible. The capacity of the program is increased to 80 lumped masses, 100 internal beams and 120 load-deflection tables. The refined program was run to demonstrate capability to treat a three-dimensional impact velocity, mass penetration into an occupiable space, and simplified rotor blade contact. Specimen test data is also used to refine the 31 mass UH-1H model.

The analytical techniques developed herein are presented in the form of design charts, nomographs, curves, tables and equations and form the basis of a structural crashworthiness design manual. The design procedures are outlined in a step-by-step process including examples.

Volume II contains supporting analytical and test data and a literature matrix categorization. A description of refined program KRASH is provided which shows the new input-output format, a listing, and sample problems.

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EUSTIS DIRECTORATE POSITION STATEMENT

The Eustis Directorate technical monitor for this effort was Mr. G. T. Singley III of the Military Operations Technology Division.

The conclusions submitted by the contractor are considered to be valid.

The report is divided into two volumes. Volume I contains a description of the survey of technical publications, investigation of the sensitivity of the simulated structural response to load-deflection variations, sub-structure test program, refinement of KRASH, structural crashworthiness design procedures, and results obtained. Volume II contains abstracts of literature reviewed, supporting analytical and test data, a description of the refined KRASH computer program, and a user's guide for the computer program.

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INTRODUCTION

This volume contains test and analysis data in support of the detailed discussions presented in Volume I. The 60 technical reports and publications reviewed during the study are briefly summarized and then tabulated in a Literature Survey Subject Index. The test data is presented in the form of recorded load, deflection, strain and accelerometer readings versus scan (time). The refinements of program KRASH to facilitate its usage are described. Included in the writeup for program KRASH is the refined input-output format, user's guide, listing, and two sample problems. Also included in the section on program KRASH are the energy balance equations and the sample format for the energy data.

LITERATURE SURVEY AND SYNOPSIS

1. Wittlin, G., Gamon, M.A., EXPERIMENTAL PROGRAM FOR THE DEVELOPMENT OF IMPROVED HELICOPTER STRUCTURAL CRASHWORTHINESS ANALYTICAL AND DESIGN TECHNIQUES, Lockheed-California Company; USAAMRDL Technical Report 72-72A,-72B, Eustis Directorate, U.S. Army Air Mobility Research and Development Laboratory, Fort Eustis, Virginia, May 1973, AD 764985.

This report presents the results of a four-phase study to develop improved rotary-wing aircraft structural crashworthiness analytical and design techniques. A digital computer program, designated KRASH, was developed and shown to be capable of accurately predicting responses during a crash in which multidirectional forces are present. The program was verified using controlled test data obtained on the same program via a combined vertical-lateral velocity impact using a fully instrumented UH-1H helicopter. Parameter studies were performed to ascertain the effect on the response of the structure and the occupants due to changes in structural element load deflection characteristics. A consistent design approach is presented, and the results of the parameter study are used to illustrate its application in a crash analysis. The study also included a detailed accident investigation, a literature survey and evaluation, substructure test, and analysis.

2. Turnbow, J.W., Carroll, D.F., Haley, J.L., Jr., Robertson, S.N., CRASH SURVIVAL DESIGN GUIDE, Dynamic Science; USAAMRDL Technical Report 71-22, U.S. Army Air Mobility Research and Development Laboratory, Fort Eustis, Virginia, October 1971, AD 733358.

This report is a design guide that has been assembled to provide the engineer with an understanding of the basic problems associated with the development of crashworthy U.S. Army aircraft. Where possible, solutions to specific problems are indicated. In areas in which little design data are available, only the general philosophy appropriate to the problem solution is presented; the details of such solutions as well as the degree of crashworthiness to be achieved must be left, at present, to the ingenuity of the designer.

This guide presents, in a condensed form, the data, design techniques, and criteria that are presently available in eight areas: (1) aircraft crash kinematic and survival envelope, (2) airframe crashworthiness design criteria, (3) aircraft seat design criteria (crew and troop/passenger), (4) restraint system design criteria (crew, troop/passenger, and cargo), (5) occupant environment design criteria, (6) aircraft ancillary equipment stowage design criteria, (7) emergency escape provisions, and (8) postcrash fire design criteria.

It is intended that both airframe and component designers and manufacturers use this guide to extend the "region of survivability" in aircraft accidents to a maximum level.

3. Reed, William H., Avery, James P., Ph.D, PRINCIPLES FOR IMPROVING STRUCTURAL CRASHWORTHINESS FOR STOL AND CTOL AIRCRAFT, Aviation Safety Engineering and Research; USAAVLABS Technical Report 66-39, U.S. Army Aviation Materiel Laboratories, Fort Eustis, Virginia, June 1966, AD 637133.

In this report the crash behavior of aircraft structures is investigated. The investigation begins with the definition of two indices of crashworthiness of basic aircraft structures and the analysis of the influence of several general types of structural modifications upon these two indices. This analysis, using fundamental principles of mechanics, contains several simplifying assumptions, which are explained as they are introduced.

Design concepts to improve the ability of the "protective container" to maintain living space for occupants during a crash or to attenuate the accelerations experienced by occupants during a crash are developed for crash conditions which are either primarily longitudinal in nature or primarily vertical in nature. Analytical methods are then provided to show how and when to apply these design concepts to any particular aircraft. Principles are presented which are considered to be suitable for use during design of new aircraft as well as modifications of existing aircraft.

The results are presented from three full-scale crash tests of small twin-engine airplanes which were conducted as a part of this investigation.

Among the pertinent conclusions of the report are: (1) improvements in crashworthiness can be achieved via minor changes in structural design or modification of existing structure, (2) vertical and longitudinal impact environments offer significantly different problems in designing for improved crashworthiness, and (3) analysis of aircraft behavior is hampered by the lack of adequate knowledge of the relationships which apply to the determination of the reaction force which decelerates the aircraft upon contact with the ground.

4. Greer, D.L., et al, DESIGN STUDY AND MODEL STRUCTURES TEST PROGRAM TO IMPROVE FUSELAGE CRASHWORTHINESS, General Dynamics, Convair; FAA Report DS-67-20, Federal Aviation Administration, Washington, D.C., October 1967, AD 666816.

This report presents the results of a study to evaluate methods to improve crashworthiness by retaining transport cabin integrity during crash impact loadings.

The study includes analyses of the effects of strengthening, redistribution of bending material, and incorporation of energy dissipating features on the ability of the fuselage to provide a protective shell

around the occupants. Analytical results were substantiated by a test program.

The test program included compression tests of plate-stringer panels and drop tests of representative fuselage structure. Tests were made on three 100-inch-diameter cylindrical sections dropped axially, four segments of 100-inch-diameter cylinders dropped laterally, and a structurally complete nose section of a jet transport dropped in a 10-degree nose-down attitude.

The requirement for a plastically deforming structure is important for both axial and vertical collapse characteristics of a fuselage. Plastic collapse provides the most efficient energy-absorbing capability and also reduces the possibility of excessive tearing or complete disintegration of the structure.

Both energy-absorbing capacity and failure mode are important for vertical crushing conditions. The recommended manner of reinforcement for the fuselage lower frame segments strengthens the bottom center-line portion and the floor beam to frame area. No significant weight or cost penalty is involved since the crash requirement reinforcement occurs at the most critical areas for existing design conditions.

5. Saczalski, K.J., and Park, K.C., TRANSIENT RESPONSE OF INELASTICALLY CONSTRAINED RIGID BODY SYSTEMS, to appear in the Journal of Engineering for Industry, A.S.M.E. Transactions, 1974.

An energy rate balance is employed to develop the incremental equations of motion for a shock-loaded, inelastically constrained, rigid-body structural system. Lagrangian multipliers provide the coupling mechanism necessary to reduce the overall system of equations to a set of modified rigid-body equations which include the nonlinear geometric and structural material effects. Kinematic material hardening and a modified yield criteria are used. Examples illustrate the technique and are compared with experimental results.

6. Massonnet, C.E., and Save, M.A., PLASTIC ANALYSIS AND DESIGN, VOL. I, BEAMS AND FRAMES; Blusdell, New York, 1965.

This book deals with the analysis and design of beams and frames made of a ductile material on the basis of the ultimate load. With the second volume, which is concerned with more complicated structures such as plates and shells, it aims at giving a broad (if not exhaustive) coverage of plastic analysis and design methods. These methods essentially apply to mild steel structures, but may also be used, with adequate caution, for reinforced and prestressed concrete structures.

Presently, the first volume is the most important for practical applications. The so-called simple plastic theory of beams and frames is

presented in Volume II, which is now being finalized.

The material has been considered as an engineering problem and not as applied mathematics. Much attention and extensive treatment have been given to plastic buckling and to design of joints. The influence of details of construction (machining, drilling, punching, etc.) is also considered.

7. Roark, R.J., FORMULAS FOR STRESS AND STRAIN, McGraw-Hill, New York, 1965.

This book provides a compact, complete summary of the formulas, facts, and principles pertaining to strength of materials. It is intended primarily as a reference book and represents an attempt to meet what is believed to be a present need of the design engineer.

Presented are certain general principles. Included are brief descriptions of analytical and experimental methods of stress analysis and information concerning the behavior of material under stress. The behavior of structural elements under various conditions of loading is discussed, and extensive tables of formulas for the calculation of stress, strain, and strength are given. Derivations and detailed explanations are omitted. However, examples are included to illustrate the application of the various formulas and methods.

8. Ayre, Robert S., Shock and Vibration Handbook, Chap. 8, Vol. 1, TRANSIENT RESPONSE TO STEP AND PULSE FUNCTIONS, McGraw-Hill, 1961.

Hoppmann, W.K., Shock and Vibration Handbook, Chap. 9, Vol. 1, EFFECTS OF IMPACT ON STRUCTURES, McGraw-Hill, New York, 1961.

Chapter 8 deals briefly with methods of analysis for obtaining the response spectrum from the time history, and includes in graphical form certain significant spectra for various regular step- and pulse-type excitations. The usual concept of the response spectrum is based upon the single-degree-of-freedom system, usually considered linear and undamped although useful information sometimes can be obtained by introducing nonlinearity or damping. The single-degree-of-freedom system is considered to be subjected to the shock or transient vibration, and its response is determined.

The response spectrum is a graphical presentation of a reflected quantity in the response taken with reference to a quantity in the excitation. It is plotted as a function of a dimensionless parameter that includes the natural period of the responding system and a significant period of the excitation. The excitation may be defined in terms of various physical quantities, and the response spectrum likewise may depict various characteristics of the response.

Chapter 9 discusses a particular phenomenon in the general field of shock and vibration usually referred to as impact. An impact occurs when two or more bodies collide. An important characteristic of an impact is the generation of relatively large forces at points of contact for relatively short periods of time. Such forces sometimes are referred to as impulse-type forces.

Three general classes of impact are considered in this chapter: (1) impact between spheres or other rigid bodies, where a body is considered to be rigid if its dimensions are large relative to the wavelengths of the elastic stress waves in the body; (2) impact of a rigid body against a beam or plate that remains substantially elastic during the impact; and (3) impact involving yielding of structures.

9. Timoshenko, S. Woinowsky-Krieger, S., THEORY OF PLATES AND SHELLS, McGraw-Hill Publishers, New York, 1959.

This book deals with the three regions of plate and shell theory: (1) thin plates with small deflections; (2) thin plates with large deflections; (3) thick plates. The book considers problems with membrane stresses and the case with clamped edges. Simplifications are given for special cases of deformation to the shape of a developable surface. The thick plate theory presented considers the problem as a three-dimensional problem of elasticity.

10. Tulk, F.D., BUCKLING OF CIRCULAR CYLINDRICAL SHELLS UNDER DYNAMICALLY APPLIED AXIAL LOADS, UTIAS report 160, 1972.

A theoretical and experimental study was made of the buckling characteristics of perfect and imperfect circular cylindrical shells subjected to dynamic axial loading.

The tests were performed on a specially designed dynamic testing machine which was capable of producing controlled ramp-type loads at rates ranging from the quasi-static up to higher than 200,000 pounds/second. Ten shell specimens were tested, including two near-perfect shells, seven shells with axisymmetric sinusoidal imperfections of a variety of amplitudes and wavelengths, and one shell with quasi-random axisymmetric imperfections. The shells were produced from a photoelastic epoxy plastic using a spin-casting technique. The imperfection profiles were machined into the shell walls using a high-precision hydraulic tracing apparatus. For three of the shells with sinusoidal imperfections, imperfection profiles were cut on the inner surface alone; while for the remaining four shells with sinusoidal imperfections and for the shell with quasi-random imperfections, a special manufacturing procedure was adopted which produced shells with matching inner and outer profiles, thus providing effectively constant thickness walls.

Experimental data included dynamic buckling loads (124 data points), high-speed photographs of the buckling mode shapes, and observations of the dynamic stability of shells subjected to rapidly applied subcritical loads.

A mathematical model is developed to describe the dynamic behavior of perfect and imperfect shells. This model is based on the Donnell-von Karman compatibility and equilibrium equations and has a wall deflection function incorporating five separate modes of deflection. Close agreement between theory and experiment is found for both dynamic buckling strength and buckling mode shapes.

11. Stricklin, J.E., et al, LARGE DEFLECTION ELASTIC-PLASTIC DYNAMIC RESPONSE OF STIFFENED SHELLS OF REVOLUTION, TEES-RPT-72-25 and SLA-73-0128, 1972.

This report presents the formulation and check-out problems for a computer code DYNAPLAS, which analyzes the large deflection elastic-plastic dynamic response of stiffened shells of revolution. The formulation is by the finite element method, with finite differences being used for the evaluation of the pseudo forces due to material and geometric nonlinearities. Time integration is by the Houbolt method. The stiffness may be due to concentrated or distributed eccentric rings and spring supports at arbitrary angles around the circumference of the elements. Check-out problems include the comparison of solutions from DYNAPLAS with experimental and other computer calculations for rings, conical and cylindrical shells, and a curved panel. A hypothetical submarine including stiffeners and missile tube is studied under a combination of hydrostatic and dynamically applied asymmetrical pressure loadings.

12. Becker, H., and Gerard G., HANDBOOKS OF STRUCTURAL STABILITY, PARTS I-V, NACA Technical Notes 3781-3785, 1957.

The local buckling of stiffener sections and the buckling of plates with angle stiffeners are reviewed, and the results are summarized in charts and tables. Numerical values of buckling coefficients are presented for longitudinally compressed stiffener sections of various shapes, and for stiffened cylinders loaded in torsion. Although the data presented consists primarily of elastic-buckling coefficients, the effects of plasticity are discussed for a few special cases.

13. Skogh, J., Stern, P., POSTBUCKLING BEHAVIOR OF A SECTION REPRESENTATIVE OF THE B-1 AFT INTERMEDIATE FUSELAGE, Lockheed Palo Alto Research Laboratory, AFFDL-TR-73-63, Air Force Flight Dynamics Laboratory, Air Force Systems Command, Wright-Patterson Air Force Base, Ohio, May 1973.

A section of the B-1 aft intermediate fuselage consisting of a combination of flat and curved panels is analyzed for postbuckling strength under a combination of torque and axial loading. The analysis, which extends to load levels about ten times the load that produces the first buckle, was carried out rigorously by the use of the finite-difference computer code STAGS. The results show that the fuselage section does not collapse at the applied ultimate load. Shear stiffness values as a function of the applied load are calculated. These data can be used as inputs for a finite-element analysis of the fuselage section.

14. Atluri, S., PETROS 3: A FINITE-DIFFERENCE METHOD AND PROGRAM FOR THE CALCULATION OF LARGE ELASTIC-PLASTIC DYNAMICALLY-INDUCED DEFORMATIONS OF MULTILAYER VARIABLE THICKNESS SHELLS, BRL, U.S. Army Aberdeen Research & Development Center, Aberdeen Proving Ground, Maryland, Contract #DAAD05-68-C-0314, Nov. 1971.

The governing equations for the arbitrarily large-deformation elastic-plastic transient responses of variable-thickness, hard-bonded, multilayer, multimaterial, thin, Kirchhoff shells of any initial shape are formulated and solved by the finite-difference technique. The material is assumed to be initially isotropic and to exhibit elastic, strain-hardening, strain-rate-sensitive, and temperature-dependent behavior. The structure may be subjected to a variety of initial velocity distributions, transient mechanical loads, and/or transient thermal loads. These capabilities and features are contained in a computer program, PETROS 3, which has been applied to a variety of example problems.

Included is a FORTRAN IV listing and a description of PETROS 3 together with the data input and solution output for several example problems.

15. Haftka, R.T., A KOITER-TYPE METHOD FOR FINITE ELEMENT ANALYSIS OF NONLINEAR STRUCTURAL BEHAVIOR, AFFDL-TR-70-130, Vol. I, Air Force Flight Dynamics Laboratory, Wright-Patterson Air Force Base, Ohio, Nov. 1970.

Koiter's method for the asymptotic analysis of post-buckling behavior is reformulated in finite element notation for application to structures idealized by finite element models. Originally restricted to the analysis either of structures exhibiting bifurcation buckling or of slightly imperfect versions of such structures, Koiter's method is therein adapted to a more general class of structures exhibiting the more common snap-through (limit point) type of buckling. This adaptation of Koiter's method is referred to as the Modified Structure method. It is accomplished by modification of the actual energy functional to create a hypothetical modified structure having a strictly linear pre-buckling path along which buckling must be of

the bifurcation type. The analysis of the actual structure is then accomplished by application of Koiter's method through consideration of the actual structure as an imperfect version of the modified structure.

In this way, the Modified Structure method operates within the theoretical framework established by Koiter, and the effects of prebuckling nonlinearity are approximated asymptotically. Various levels of approximation are considered. Additionally, the use of the Modified Structure method, in conjunction with direct methods of nonlinear analysis, is examined. A highly accurate finite element representation is employed in presenting a comprehensive numerical evaluation of the Modified Structure method of analysis on the basis of a number of planar frame problems. Collectively, these examples exhibit a broad spectrum of nonlinear behavior characteristics. Emphasis throughout is placed upon assessing the limitations and attributes of the Modified Structure method of analysis. Conclusions regarding applicability and performance emerge from detailed examination of the results obtained.

16. Stilwell, W.C., and Ball, R.E., A DIGITAL COMPUTER STUDY OF THE BUCKLING OF SHALLOW SPHERICAL CAPS AND TRUNCATED HEMISPHERES, NASA CR 1998, June 1972.

Several user-oriented digital computer programs for the static analysis of shells of revolution exist. A detailed discussion of most of these programs is given in Reference 1 of the report. Of particular interest here is the program developed by Ball (Reference 2) for the geometrically nonlinear analysis of arbitrarily loaded shells of revolution. This program is an equilibrium program; that is, it solves for the displacement and stress resultant fields for an arbitrary loading condition. Since geometric nonlinearities are included, the magnitude of load that leads to a condition of instability can be determined.

The utility of the program would be considerably enhanced if it could be used to determine bifurcation buckling loads and the behavior of the shell in the vicinity of the bifurcation load. This latter feature is often referred to as the imperfection sensitivity of the shell to the load. As a consequence, the objective of this study was to use the computer program to examine the buckling behavior of several shells subjected to axisymmetric and nearly axisymmetric loads. It was anticipated that an examination of the effects of the small asymmetric perturbations upon the stability of the shell would disclose the bifurcation buckling load and provide a quantitative evaluation of the imperfection sensitivity of the shell to the load.

17. Witmer, E.A., LARGE DYNAMIC DEFORMATIONS OF BEAMS, RINGS, PLATES AND SHELLS, AIAA Journal, Vol. I, No. 2, August 1963.

The axisymmetric responses of shells, plates, rings, and beams to impulsive or blast loading that produces large deformations involving both the elastic and plastic regions of material behavior are analyzed. A general numerical method that includes (1) elastic, (2) perfectly plastic, (3) elastic, strain-hardening, and/or (4) elastic, strain-hardening, strain-rate sensitive material behavior and large structural deflections is formulated and applied. In the timewise step-by-step numerical analysis, the increments in stress resultants and stress couples are determined by idealizing the shell thickness as consisting of n concentrated layers of materials separated by a material that cannot carry normal stresses but has infinite shear rigidity. The influences of the number of layers employed in the idealized model, as well as the aforementioned various types of material behavior, are demonstrated. Theoretical predictions of time-history responses and/or final structural deformations are compared with experimental data for impact-loaded spherical shells, for blast-loaded circular plates, and for explosively loaded circular rings and clamped beams.

18. Perrone, N., ON A SIMPLIFIED METHOD FOR SOLVING IMPULSIVELY LOADED STRUCTURES OF RATE-SENSITIVE MATERIALS, Office of Naval Research, Washington, D.C., Journal of Applied Mechanics, A.S.M.E., September 1965.

In an attempt to assess more completely rate-sensitive material effects, two fundamental structural elements are analyzed: a wire with an impulsively loaded end mass, and an impulsively loaded ring. The ring and wire are made of perfectly plastic, rate-sensitive materials. In each case, exact and approximate solutions are obtained for an exponential rate-sensitivity law. The results suggest that very good approximations to the exact solutions may be found by utilizing a rate-insensitive material with constant yield stress equal to the initial dynamic yield stress.

19. Bodner, S.R., and Symonds, P.S., EXPERIMENTAL AND THEORETICAL INVESTIGATION OF THE PLASTIC DEFORMATION OF CANTILEVER BEAMS SUBJECTED TO IMPULSIVE LOADING, Brown University, Journal of Applied Mechanics, December 1962.

The experimental techniques and the results obtained in a program to evaluate the assumptions of dynamic, rigid-plastic theory of beams are presented. The experiments use steel and aluminum-alloy cantilever beams subjected to either a rapid velocity change at the base or to an impulsive load at the tip. A rigid-plastic theory that includes the strain-rate dependence of the yield stress and geometry changes is outlined for the case of the tip impulsive loading. Predictions of this theory are in satisfactory agreement with the experimental results.

20. Hibbit, H.D., et al, A FINITE ELEMENT FORMULATION FOR PROBLEMS OF LARGE STRAIN AND LARGE DISPLACEMENT, Brown University, Int'l Journal of Solids and Structures, 1970, Vol 6, pp. 1069-1086.

An incremental and piecewise linear finite element theory is developed for the large-displacement, large-strain regime with particular reference to elastic-plastic behavior in metals. The resulting equations, though more complex, are in a similar form to those previously developed for large-displacement, small-strain problems, the only additional term being an initial load stiffness matrix which is dependent on current loads. This similarity in form means that existing non-linear general-purpose programs may easily be extended to include finite strains. A large-displacement, small-strain formulation (as applicable to problems of structural stability) is obtained from this theory by assuming that changes in length of line elements and relative rotation of orthogonal line elements are negligible compared to unity. The simplified equations are in essential agreement with previous formulations in the literature. The only difference which is observed is the persistence of the initial load stiffness matrix, which may be significant in some cases.

21. Toridio, T.G., and Khozeimeh, K., INELASTIC RESPONSE OF FRAMES TO DYNAMIC LOADS, Journal of Engineering Mechanics, June 1971.

A procedure of analysis is presented for determining the elastic-inelastic response of framed structures under dynamic loads. Application of Hamilton's principle in conjunction with the finite-element method leads to the basic dynamic equation of the system incorporating the plastic effects in the form of equivalent nodal forces. This approach also allows the more accurate treatment of the distributed mass of the element than the usual geometrical method of lumping.

22. McDaniel, T.J., DYNAMICS OF STIFFENED CYLINDRICAL SHELLS WITH SPATIALLY VARYING CURVATURE, University of Dayton Research Institute, Air Force Materials Laboratory Report AFML-TR-72-134, Air Force Systems Command, Wright-Patterson Air Force Base, Ohio, July 1972.

This report presents the results of a theoretical study of the effects of spatially varying curvature on the dynamics of cylindrical panels and stiffened cylindrical shell structures. The first section of the report contains a brief discussion of the general problem area. Following this discussion, the analytical techniques for solving constant-curvature cylindrical panels and stiffened cylindrical shell problems by a transfer matrix approach are reviewed. These techniques are found to apply directly to the varying curvature shell analysis, provided the transfer matrix for this type of shell can be obtained. An analytical approach to obtaining the transfer matrix for a shell with varying curvature is explored. A solution to the transfer matrix for a cylindrical shell with exponentially varying curvature is obtained. In a following section, the preparation of this solution to obtain numerical transfer matrix is discussed. Several approximate and numerical

procedures for obtaining a transfer matrix are explored. Finally, the dynamic responses of both single panels and stringer stiffened cylindrical structures with increasing and decreasing curvature are compared to similar structures with constant curvature.

23. Bendix Corp., Final Engineering Report, ENERGY ABSORBING CHARACTERISTICS OF CRUSHABLE ALUMINUM STRUCTURES IN A SPACE ENVIRONMENT, NASA-CR-65096, July 1965.

The research effort detailed in this report involves the basic objective of obtaining quantitative design data concerning the characteristics of aluminum honeycomb materials when used in high L/D ratio, crushable, energy-absorbing capsules. Nine configurations of honeycomb energy-absorbing capsules using alloy 5056 are evaluated. The nine basic configurations incorporate three cross-sectional shapes of high, medium, and low crush strength, each of which was fabricated with cell axes oriented at angles of 0, 15 and 30 degrees to the capsule longitudinal axis. The characteristics which are studied included specific energy, load onset rate, and rebound. Variations of these characteristics are investigated under controlled environmental conditions.

The capsules were subjected to both static and dynamic loads, impact velocities from 5 thru 20 feet per second, and impact weights varying from 760 thru 3750 pounds. The environmental extremes under which the specimens were tested spanned the temperature range from -260°F thru room temperature up to +300°F, and a vacuum of 3×10^{-1} TORR.

24. Kornhauser, M., STRUCTURAL EFFECTS OF IMPACT, Sparten Books, Inc., Baltimore, Md., 1964.

Mechanical impact has been treated traditionally, in the United States, as a subject which is variously appended to college textbooks on elasticity, strength of materials, engineering mechanics, or vibrations. The elasticity and strength of materials texts are generally concerned with local surface effects or with stress waves; the mechanics texts usually treat the whole-body motions on impact; and the vibrations texts are apt to consider impact a special case of unsteady vibrations and amplification factors. In the actual case of impact, all of these phenomena and effects come into play, and the significance of each effect must be emphasized relative to the purpose of the analysis.

In this book the emphasis is placed on go or no-go behavior, survivability, or failure. Loading and response must, of course, be analyzed. Nevertheless, wherever possible, the object is to permit estimates of failure directly in terms of the loading conditions. Impact is a complex process. Given the loading history of a structure (and assuming no interaction with the structure's response), it is, in general, impossible to trace the stress waves and their

reflections throughout the structure, resulting in vibrations and permanent set or failure. Many solutions of loading and response, as well as theories of failure in terms of material properties, are available in the literature for various idealized conditions and configurations. What is desired by the practicing engineer are some relatively simple approaches to prediction of failure. This book is intended as a start in this direction.

In attempting to provide engineering answers to some very complex problems, liberal use has been made of "engineering judgment" and idealizations. The soundness of each assumption is in proportion to the number of exact theoretical and experimental results available for very similar situations. For this reason it is anticipated that modifications of the approaches presented therein will be appropriate as the various disciplines produce more data.

The book is organized in three sections: "Loading Conditions", "Response and Failure of Structures", and "Effects of Impact." The section on response and failure is expository in nature, introducing the background theory and application of the sensitivity method of presenting and predicting inertial failure, as well as discussing low-speed and hypervelocity impact effects. Having accepted the approaches recommended in the second section, the practicing engineer may use the curves and tables of the first and third sections for prediction of impact effects in terms of the environmental input functions.

25. Fisher, L. J., Jr., LANDING-IMPACT-DISSIPATION SYSTEMS, NASA, Technical Note D-975, December 1961.

Analytical and experimental investigations are made to determine the landing-energy-dissipation characteristics for several types of earth-landing-impact systems having application to reentry vehicles. The areas of study are divided into three velocity regions: (1) those having primarily vertical velocity, (2) those having both moderate horizontal and moderate vertical velocity, and (3) those having primarily horizontal velocity. The impact systems discussed are braking rockets, gas-filled bags, frangible metal tubing, aluminum honeycomb, balsa wood, strain straps, and both skid and skid-rocker landings on hard-surface runways and on water.

The report states that it appears feasible to evaluate landing-gear systems for reentry vehicles by computational methods and free-body landing techniques with energy dissipation for an earth landing of such a vehicle. Some systems are more efficient than others, some package better than others, and a variety of promising systems are under study. Horizontal energy dissipation is simpler to deal with than vertical energy dissipation since translational friction is all that is involved; however, runout behavior becomes a factor. Vertical velocity can also be a big factor when high flight-path angles are

associated with even moderate horizontal velocities. High-speed landings are particularly a problem, especially high-speed water landings, and indications are that if large horizontal velocities are involved in hard-surface landings, a selected site will be required.

26. McGehee, J. R., A PRELIMINARY EXPERIMENTAL INVESTIGATION OF AN ENERGY-ABSORPTION PROCESS EMPLOYING FRANGIBLE METAL TUBING, NASA Technical Note D-1477, 1962.

A highly efficient energy-absorption process, employing frangible metal tubing as the working element, is investigated. A preliminary experimental investigation is conducted to determine the variation of the average fragmenting stress of 2024-T3 aluminum-alloy tubing with the pertinent parameters of this process. Limited tests were made to determine the feasibility of employing this process in a landing-gear system. A 1/5-scale model of a proposed manned spacecraft with a landing gear incorporating this process is employed in these tests.

The results of this investigation show that the fragmenting process produces a fluctuating force with displacement, but for a fixed set of parameters, the force about which the fluctuation occurs is approximately constant. A large force which occurs when the process is started with the unaltered tube seated symmetrically in the die can be reduced most effectively by tapering the wall thickness over a short length at the die end of the tube. The average fragmenting stress, for 2024-T3 aluminum-alloy tubing and the range of parameters investigated, appears to be independent of the ratio of wall thickness to tube diameter, but varies as the cube of the ratio of the wall thickness to the radius of the forming die. The fragmenting stress obtained at 12,000 inches per minute was about 60 percent higher than those obtained at 1 inch per minute. The 2024-T3 aluminum-alloy tubing, when fragmenting on a die at 90 percent of the yield stress, is capable of absorbing 31,000 foot-pounds of energy per pound of material. This energy-absorption capability is greater than that of the most frequently considered processes; for example, the crushing of balsa wood, aluminum honeycomb, or pressurized thin-walled metallic cylinders. Model tests, employing frangible tubing as the working element in the landing gear, indicate that this process is suitable for use in a load-alleviation application.

27. Kroell, C. K., A SIMPLE, EFFICIENT, ONE SHOT ENERGY ABSORBER, General Motors Research Laboratory, Warren, Michigan, Shock, Vibration and Associated Environments, Part III, Bulletin No. 30, 1962.

This paper describes a single-shot expendable energy absorber which has been developed at the General Motors Research Laboratory. The device is inherently simple and is characterized by a rectangular force-displacement relationship and high specific energy absorption capacity. Both a qualitative discussion of the mechanics of the

plastic deformation process involved and a graphical summary of the experimental performance data which have been collected to date are presented.

28. Weinberg, L.W.T., and Turnbow, J.W., Ph.D., SURVIVABILITY SEAT DESIGN DYNAMIC TEST PROGRAM, Aviation Safety Engineering and Research, USAAVIABS Technical Report 65-43, U.S. Army Aviation Materiel Laboratories, Fort Eustis, Virginia, July 1965, AD 621718.

This report presents the results of a series of dynamic tests conducted with four different concepts of experimental crew seats.

The experimental seats were designed and constructed by four helicopter manufacturers. The seats were designed to withstand static load factors equivalent to those recommended in TRECOM Technical Report 63-4, "Crew Seat Design Criteria for Army Aircraft", dated February 1963.

The design load factors recommended in the above-referenced report are as follows: longitudinal - 45G for 0.10 second; lateral - 45G for 0.10 second; and vertical - 25G for 0.10 second.

Special kits for small-arms ballistic protection were also designed and installed in the seats tested.

These seats were designed exclusively using static load factors. No previous testing was conducted by any seat manufacturer prior to the conduct of these tests.

The four seats were tested under four load conditions. Two of the conditions involved simultaneous half loads on the seats in two different seat positions, and two of the conditions involved full loads in two different seat positions.

Only one of the four seats tested withstood the loads imposed for all four conditions. Three of the seats failed and were damaged beyond economical repair when each was subjected to the first full-load test condition.

This report also includes a detailed description of an acceleration device which was specifically designed and fabricated for this series of tests.

29. Langhaar, H. L., THEORETICAL AND EXPERIMENTAL INVESTIGATIONS OF THIN-WEBBED PLATE-GIRDER BEAMS, Transactions of the ASME, October 1943.

A simple, semirational theory for the design of webs and flange rivets of thin-webbed rectangular plate-girder shear beams is presented. Calculations of shear loads to cause web rupture and flange rivet failure are compared with test data from 27 beams.

30. Perry, D. J., AIRCRAFT STRUCTURES, McGraw Hill Book Co., New York, 1950.

In this book an attempt is made to emphasize basic structural theory which will not change as new materials and new construction methods are developed. Most of the theory is applicable for any design requirements and for any materials. The design engineer may then supplement this theory with the detail design specifications and the material properties which are applicable to his particular airplane.

Heavy emphasis is placed on the application of the elementary principles of mechanics to the analysis of aircraft structures.

31. Jones, N., et al, THE DYNAMIC PLASTIC BEHAVIOR OF SHELLS, MIT Report 71-6, 1971.

A survey is made of the literature published previously on the inelastic behavior of shells subjected to dynamic loads. An experimental investigation is also being undertaken to examine the behavior of various cylindrical shell panels which are loaded with an impulse on the inner surface. The panels are fully clamped along the two longitudinal edges and free on the other two. The initial kinetic energy of the dynamic loads is sufficiently large to cause inelastic behavior and to produce maximum permanent transverse deflections of up to nearly twice the corresponding panel thickness. Tests are conducted on mild steel and aluminum 6061-T6 panels which have various thicknesses and included angle of 90° approximately.

32. Mitchell, B., THE DYNASORB ENERGY ABSORBER, Lockheed Report LR 16735, March 1963.

This report introduces a method of absorbing energy while maintaining a constant load level. The tubular load-carrying member can support its full load capability and still be almost totally consumed by rolling up one end. The energy curves presented are typical and show excellent load consistency with efficiencies as high as 450,000 in.-lb/lb.

33. Mitchell, B., SHOCK ABSORPTION WITH ONE SLOT TUBES, Lockheed Report LR 16369, June 1963.

The work covered in this report is a follow-on of an independent research program that developed a very efficient method of absorbing energy by load control and end roll-up of tubes. This study investigates 12 different materials and improves the control rings to give a smooth, nearly rectangular, energy curve. The load level can be set, by suitable design, at any desired point within the column or local

strength limits of the tube. The maximum efficiency attained here with a 2-inch-diameter by .049 wall 4130 steel tube heat treated to 200,000 psi is 600,000 in.-lb per lb of tube weight. This represents a mean compressive stress of 189,000 psi over the total 6 inches of travel.

34. Mitchell, B., DESIGN NOTES FOR THE DYNASORB ENERGY ABSORBER, Lockheed Report LR 17201, December 1963.

The Dynasorb Energy Unit, developed in the Lockheed Engineering Laboratory, is adaptable to many applications as a shock absorber or load limiter. Design procedures based on previously reported data are described, and several typical installations are illustrated.

This report is submitted in fulfillment of the reporting requirements of a 1963 Independent Development Project, "Energy Absorption Products."

35. Mitchell, B., ENERGY ABSORPTION AT HIGH SPEED VERTICAL LANDING, Lockheed Report LR 21023, November 1967.

This report describes the use of the Lockheed Dynasorb energy absorber in landing impact tests of a 3400-lb simulated vehicle structure. The structure, released from a helicopter and tethered by parachute, impacted at a velocity of 112 feet per second.

36. Perrone, N., RESPONSE OF RATE SENSITIVE FRAMES TO IMPULSIVE LOAD, ASME Journal of Applied Mechanics, February 1971, pp. 49-62.

A relatively convenient method is presented herein to determine the plastic response of rate-sensitive frameworks. The technique represents a significant extension of usual limit analysis type approaches as well as similar efforts applied to rate insensitive structures.

The method consists of assuming a modal deformation pattern for the structure in question, determination of the initial modal velocity utilizing the appropriate criteria, and integration in some form of the equations of motion after estimating the magnitude of the dynamic yield moments at the rate-sensitive plastic hinge locations. It is recommended that an effective hinge length of the order of three beam depths should suffice in estimating the strain rate magnitude. For the perfectly plastic case, the dynamic yield moment is assumed to be a constant with time.

A square portal frame made of a solid rectangular cross-section and loaded under a horizontal impulsive load is considered in some detail. It is shown that a critical initial velocity exists.

The effects of strain hardening and pulse load application are considered. Minor modifications of the impulsively loaded, perfectly plastic situation are necessary to accommodate these extensions.

The usual limitations should be noted. It is assumed that buckling will not occur, that the median surface or membrane effects are negligible, and that elastic effects are negligibly small relative to plastic flow because the former are omitted.

Portal frame experiments under high intensity loading similar to those conducted at Brown University on cantilever beams would be most welcome. Most of the experiments performed to date have been in a much lower load range where elastic and plastic effects are of comparable order. The true limiting strength of the framework could be more completely assessed only if larger loads are applied. Other tests have been reported, and hopefully these results will soon be available.

37. Jones, N., INFLUENCE OF STRAIN-HARDENING AND STRAIN-RATE SENSITIVITY ON THE PERMANENT DEFORMATION OF IMPULSIVELY LOADED RIGID PLASTIC BEAMS, International Journal of Mechanical Science, 1967, Vol. 9, pp. 777-796.

A simple method is presented for estimating the combined influence of strain-hardening and strain-rate sensitivity on the permanent deformation of rigid-plastic structures loaded dynamically. A study is made of the particular case of a beam supported at the ends by immovable frictionless pins and loaded with a uniform impulse. The results of this work indicate that, when stress-hardening or strain-rate sensitivity are considered, permanent deformations are experienced which are similar to those predicted by an analysis retaining both effects simultaneously.

38. Ni, C. M., IMPACT RESPONSE OF CURVED BOX BEAM-COLUMNS WITH LARGE GLOBAL AND LOCAL DEFORMATIONS, General Motors Research Laboratory, Warren, Michigan, AIAA Paper 73-401.

A numerical approach based on a lumped-mass model is developed for investigating the impact response of curved box beam-columns with large global and local deformations. An empirical formula which relates the changes of depth and bending angle of a beam cross-section is obtained to take into account the local deformations of the cross-sections. The strain-hardening and the strain-rate properties of the material are considered in this analysis. The correlation between the present analysis and test results is very good. The results obtained indicate that the strain-wave propagations due to the impact and the strain-rate sensitivity of the material play the key roles in increasing the energy-absorbing capacity of the structure when subjected to high-speed impact.

39. O'Bryan, T.C., and Hatch, H.G., Jr., LIMITED INVESTIGATION OF CRUSH-ABLE STRUCTURES FOR ACCELERATION PROTECTION OF OCCUPANTS OF VEHICLES AT LOW IMPACT SPEEDS, NASA Technical Note D-158, 1958.

A limited investigation is made to determine the characteristics of three materials to see how they can be applied for human protection against accelerations encountered at low impact speeds. As a result, if given man's physiological tolerance to abrupt acceleration, which has not yet been well-defined, an alleviation system can be designed.

Foamed plastics require considerable depth to provide a given stopping distance for impact alleviation, and their use will require some control of rebound. They can be made soft enough to obtain the low onset of acceleration that may be necessary for man where depth is not limited.

Aluminum honeycomb is an efficient material for impact load alleviation from the standpoint of usable material depth, and it exhibits very little rebound. The stiffness of the material results in a very high initial onset rate of acceleration. For many installations this may be controlled by reducing the initial loading area of contact to get the material to start failing.

40. Jones, Norman, THE INFLUENCE OF LARGE DEFLECTION ON THE BEHAVIOR OF RIGID-PLASTIC CYLINDRICAL SHELLS LOADED IMPULSIVELY, Journal of Applied Mechanics, ASME, June 1970, pp. 417-425.

In order to gain some insight into the importance and influence of finite deflections on the response of shells loaded dynamically, this article studies theoretically the behavior of a cylindrical shell subjected to a uniform axisymmetric impulsive pressure. The cylindrical shell is assumed to be made from a rigid, perfectly plastic material, and the external energy imparted to the shell is much greater than the total strain-energy which can be absorbed elastically. The results of the investigation indicate that geometry changes are important and should be retained when studying the behavior of cylindrical shells loaded dynamically.

41. D'Amato, R., STATIC POST-FAILURE STRUCTURAL CHARACTERISTICS OF MULTI-WEB BEAMS, WADC TR 59-112, February 1959.

The structural behavior of aircraft components is discussed in relationship to the general lethality problem, the concept of post-failure restoring force for built-up structures of multiweb beams is considered, and a relatively simple analysis is developed to compute the static post-failure behavior of multiweb beams in pure bending. An experimental investigation is conducted to examine the validity of the analysis and, within the range of parameters considered, agreement between experiment and theory is satisfactory. Both the theoretical analysis and the experimental results indicated that an arbitrary extrapolation of post-failure data on the basis of ultimate strength can be very misleading.

42. Rawlings, B. ENERGY ABSORPTION OF DYNAMICALLY AND STATICALLY TESTED MILD STEEL BEAMS UNDER CONDITIONS OF GROSS DEFORMATION, International Journal of Mechanical Science, Pergamon Press, Ltd., 1967, Vol. 9, pp. 633-649. Printed in Great Britain.

An account is given of dynamic and static tests on mild steel members deformed under conditions of gross geometry change. An evaluation is made of the rigid-plastic theory, taking account of change of geometry, and also the elastic-plastic theory, assuming deformation to occur in the static mode, in predicting the behavior of the members.

43. Thompson, J.E., VEHICLE CRUSH PREDICTION USING FINITE-ELEMENT TECHNIQUES, Chrysler Corp., SAE Paper 73-157, January 1973.

The principal objective of this investigation is to develop analytical tools in the form of computer programs which will permit the automobile designer to predict the crush characteristics of a given car structure due to forces generated in a variety of impact modes.

The predictive or control capability is embodied in two large computer programs. "TELSAP" forms, reduces, and inverts the vehicle structure mass matrix expressed relative to a datum coordinate system and writes the mass matrix and its inversion onto a file for reading by the "CRUSH" program. "CRUSH" is a general matrix structural analysis program which calculates the large, plastic, rate-sensitive response of an interconnected beam structure due to known dynamic boundary displacement inputs. The theoretical bases and assumptions employed in developing these programs are described along with a detailed discussion of how the automobile design engineer might use them to develop a new vehicle structure.

Experimental correlation with the computer models is given for a vehicle-to-vehicle 90-degree intersection collision between a special rigid moving barrier and an intermediate-size four-door sedan. The structural model is further correlated with a laboratory test of a clamped-clamped beam struck at its center by an impact pendulum. The correlation indicates general agreement between experimental and analytical results.

By using the computer programs developed in this investigation, the automobile designer is able to reduce the amount of testing required to prove his design, and is able to identify the benefits of a particular structural reinforcement with a minimum of development time and expense.

44. Burgmann, J. B., and Rawlings, B., DYNAMIC PLASTIC ANALYSIS OF PIN-JOINTED FRAMES, International Journal of Mechanical Science, Pergamon Press, 1968, Vol. 10, pp. 967-980, Printed in Great Britain. (Revised 30 July 1968).

The paper presents an analysis of a pin-jointed frame subjected to dynamic or impulsive overload, of sufficient magnitude to cause permanent deformation. Rigid-plastic behavior of tensile members is assumed and, as a first approximation, a similar behavior of compressive members is assumed, although modifications to account for other characteristics are also discussed.

The behavior is considered in terms of the kinematic conditions, dynamic conditions, and the load-deformation characteristics assumed for the members.

45. Martin, J.B., MODE APPROXIMATION FOR IMPULSIVELY LOADED STRUCTURES IN THE INELASTIC RANGE, Proceedings of the Southampton 1969 Civil Engineering Material Conference.

A convergence approximation technique, based on the uniqueness proof, is reviewed for impulsively loaded rigid plastic and rigid viscoplastic structures. Emphasis is given to the use of mode or quasi-mode solutions and their usefulness in establishing a general approximating procedure and in providing insight into the important aspects of the gross structural response.

46. Jensen, W.R., Flaby, W.E., and Prince, N., MATRIX ANALYSIS METHODS FOR ANISOTROPIC INELASTIC STRUCTURES, AFFDL-TR-65-220, April 1966.

Most aerospace structural materials exhibit some degree of anisotropic strain-hardening. During the past few years, several methods have appeared in the literature for introducing inelastic isotropic material behavior effects into existing matrix analysis procedures using the incremental theory of plasticity. A review is presented of these methods and a step-by-step routine known as the "Constant Strain" method is selected for the development of an anisotropic inelastic procedure.

47. Isaakson, G., Armen, H., Jr., and Pipko, A., DISCRETE ELEMENT METHODS FOR THE PLASTIC ANALYSIS OF STRUCTURES, NASA CR 803, October 1967.

This study deals with the extension of finite-element methods to provide analytical means for determining the failure loads of aeronautical structures. Two areas are considered as related to predicting failure loads: inelastic stress analysis in the presence of load cycling, and plastic buckling of the bifurcation type.

Finite-element inelastic stress analysis methods are extended to take into account the Bauschinger effect for biaxial stress states using a plasticity theory based on Ziegler's modification to Prager's kinematic hardening theory. This methodology is applied to several structures representative of aeronautical construction, including a notched plate, a shear lag specimen, and a swept wing. Good correlation is obtained between analytical and experimental results for

the strains at the root of the notched plate subjected to load cycling in the plastic range.

Finite-element buckling methods are also extended to consider plastic buckling using Stowell's formulation for implementing a deformation plasticity theory into the buckling theory. Sample calculations are carried out for the plastic buckling of a flat plate with various geometries and edge conditions.

48. Stricklin, J.A., et al, NONLINEAR DYNAMIC ANALYSIS OF SHELLS OF REVOLUTION BY MATRIX DISPLACEMENT METHOD, AIAA Journal, Vol. 9, No. 4, April 1971, p. 629.

A formulation and computer program is developed for the geometrically nonlinear dynamic analysis of shells of revolution under symmetric loads. The nonlinear strain energy expression is evaluated using linear functions for all displacements. Five different procedures are examined for solving the equations of equilibrium, with Houbolt's method proving to be the most suitable. Solutions are presented for the symmetrical and asymmetrical buckling of shallow caps under step pressure loadings and a wide variety of other problems, including some highly nonlinear ones.

49. Owens, R. H., and Symonds, P.S., PLASTIC DEFORMATIONS OF A FREE RING UNDER CONCENTRATED DYNAMIC LOADING, ASME Journal of Applied Mechanics, December 1955, p. 524.

A concentrated time-dependent force acts on an unsupported thin ring along a diameter. The problem considered in this paper is to determine the deformations of the ring when the force magnitudes are such that plastic strains occur which are large compared to the elastic strains. By neglecting elastic strains and assuming ideally plastic behavior, approximations to the final deformations of the ring are obtained. The analysis is developed for force pulses of arbitrary shape, but numerical results are obtained only in the special case of a rectangular force pulse. A criterion is stated for conditions when this type of analysis can be expected to provide satisfactory results.

50. Lee, H., and Symonds, P.S., LARGE PLASTIC DEFORMATIONS OF BEAM UNDER TRANSVERSE IMPACT, ASME Journal of Applied Mechanics, September 1952, p. 308.

A comparatively simple method of analysis is developed to determine the deformations in a beam subjected to lateral impact of such a magnitude that plastic strains occur which are large compared with elastic strains. A useful approximation to the motion then can be obtained by neglecting elastic strains and considering rigid-body motion of segments of the beam joined at plastic hinges where the

entire deformation takes place. A method of analyzing such a situation is described and applied to a beam subjected to central impact. The approximate final permanent deformation is obtained; this includes deformation during application of the load, and plastic flow which continues afterward when the kinetic energy of the motion generated by the impact is transformed into additional plastic deformation. A criterion is given for conditions when this type of theory can be expected to provide a satisfactory analysis. The method of solution provides an interesting analogy to the concept of static determinacy which has been used in the analysis of quasi-static plastic-flow problems.

51. Prager, W., A NEW METHOD OF ANALYZING STRESSES AND STRAINS IN WORK-HARDENING PLASTIC SOLIDS, ASME Journal of Applied Mechanics, December 1956, p. 493.

For work-hardening plastic solids, segment-wise linear yield conditions and the associated flow rules constitute a reasonable compromise between the mathematically convenient but physically unsound total stress-strain laws and the physically sound but mathematically inconvenient incremental laws. They allow total stress-strain laws to be used in the small, but retain the characteristic features of the incremental laws in the large. The use of a segmentwise linear yield condition and the associated flow rule is illustrated by the analysis of the bending moments and deflections of a simply supported circular plate that is made of a work-hardening material and subjected to a uniformly distributed transverse load.

52. Morino, L., Leech, J.W., and Witmer, E.A., AN IMPROVED NUMERICAL CALCULATION TECHNIQUE FOR LARGE ELASTIC-PLASTIC TRANSIENT DEFORMATIONS OF THIN SHELLS, Part 1, ASME Journal of Applied Mechanics, June 1971, p. 423.

In this paper, the governing differential relations which describe the large-deflection elastic-plastic dynamic responses of arbitrarily shaped thin Kirchhoff shells are given, including recent improvements. These relations are then cast into finite-difference form for numerical solutions. These finite-difference relations are employed in a computer program, PETROS 2, which has been applied to evaluate the "analysis improvements" by comparing PETROS 2 predictions with those of the earlier analysis and with experimental results.

53. Pifko, A. Issakson, A FINITE-ELEMENT METHOD FOR THE PLASTIC BUCKLING ANALYSIS OF PLATES, Grumman Aerospace Corporation, Bethpage, N.Y., AIAA Journal of Applied Mechanics, Vol. 7, No. 10, October 1969.

An existing finite-element technique for elastic buckling of plates is extended to include the case of plastic buckling. The Stowell theory for the plastic buckling of flat plates is used in conjunction with the finite-element technique. Application is made to rectangular

plates, and results are presented for a variety of boundary support conditions and several different edge loading conditions.

54. Armen, H., Jr., Pifko, A., and Levine, H.S., FINITE ELEMENT ANALYSIS OF STRUCTURES IN THE PLASTIC RANGE, NASA CR 1649, February 1971.

The present report is concerned with the development of finite-element methods for the treatment of the nonlinear behavior of complex structures. It represents an extension of a previous study reported in NASA Contractor's Report CR-803. The nonlinearity may be of two types - material nonlinearity associated with plastic deformation, and geometric nonlinearity associated with the changing geometry of the structure as it deforms - or it may involve a combination of the two. Effects due to creep and other time-dependent material properties are neglected.

The methods developed are applicable to loading conditions that cause membrane stress states or pure bending, or both in combination. The Prager-Ziegler kinematic hardening theory of plasticity is incorporated in the finite-element methods to allow for consideration of the plastic response of structures subjected to realistic loading conditions, including cyclic loadings that cause stress reversals into the plastic range. Ideally, plastic behavior is also included to provide capability for predicting the collapse load of structures. The plasticity theory is implemented in the finite-element analysis by using an incremental approach in conjunction with the initial strain concept, with plastic strains interpreted as initial strains.

The treatment of geometric nonlinearity requires use of an incremental technique in which the internal forces and configuration of the structure are continuously updated to account for its changing geometry.

The methods developed are applied to a number of sample structures. For membrane stress states alone, the analysis employs a triangular finite element in which stress and strain vary linearly. This element is used for the plastic analysis of a variety of structures characterized by regions of rapid stress variation and subjected to cyclic loading resulting in reversed plasticity. Comparisons of the results of the analysis and experimental data indicate good correlation.

Plastic analyses are also performed for a variety of beam and plate structures. These problems make use of refined rectangular and triangular finite elements. Among the problems considered are rectangular, circular, and annular plates with various boundary conditions. Once again, comparisons with results of other available analyses are favorable.

Problems of combined bending and stretching of plates are also considered. Results are obtained for rectangular and circular plates. Results for combined geometric and material nonlinearity are presented for beams and arches.

55. Mallett, R. H., AUTOMATED METHOD FOR THE LARGE DEFLECTION AND INSTABILITY ANALYSIS OF THREE-DIMENSIONAL TRUSS AND FRAME ASSEMBLIES, AFFDL-TR-66-102, December 1966.

The computer program presented in this report was developed to predict large deflection behavior of three-dimensional truss and frame assemblies. The solutions are obtained by the direct minimization of the total potential energy with respect to the displacement variables rather than by solving nonlinear matrix equations. Sample problems are presented to demonstrate the analysis capability developed. Instructions for the preparation of the input data and the FORTRAN IV source program listing are included.

56. Przemieniecki, J.S., MATRIX METHODS IN STRUCTURAL MECHANICS, AFFDL-TR-66-80, Conference held October 26-28, 1965.

The Conference on Matrix Methods in Structural Mechanics held at Wright-Patterson Air Force Base on 26-28 October 1965 was sponsored jointly by the Air Force Flight Dynamics Laboratory, Research and Technology Division, Air Force Systems Command, and the Air Force Institute of Technology, Air University. The purpose of the conference was to discuss the recent developments in the field of matrix methods of structural analysis and design of aerospace vehicles.

57. Berke, L., PROCEEDINGS OF THE SECOND CONFERENCE ON MATRIX METHODS IN STRUCTURAL MECHANICS, AFFDL-TR-68-150, Conference held October 15-17, 1968.

The Second Conference on Matrix Methods in Structural Mechanics, sponsored by the Air Force Institute of Technology (AFIT), Air University, and the Air Force Flight Dynamics Laboratory (AFFDL), Air Force Systems Command, was held on 15-17 October 1968. The purpose of the conference is to discuss the recent developments in matrix structural analysis and design of structural systems. This volume contains all the papers presented at the conference.

Forty papers were presented at the conference in seven sessions, entitled Structural Weight Optimization, Dynamics, General Elements, Curved Elements, Applications, General Methods, and Nonlinear Analysis. The papers covered practically all major aspects of recent research and development work on matrix methods in structural mechanics.

58. Symonds, P.S., SURVEY OF TECHNICAL METHODS OF ANALYSIS FOR PLASTIC DEFORMATION OF STRUCTURES UNDER DYNAMIC LOADING, BU/NSRDC/1-67, Brown University, Providence, R.I., 1967.

This survey attempts to make a critical study of methods described in the literature for the analysis of metal structures under dynamic loading to plastic deformation.

Analyses have now appeared in the literature of a considerable variety of structures of engineering interest. They include beams (under many conditions of loading, support, and materials), rings, arches, frames, (simple rectangular bents), plates (circular and rectangular), membranes (i.e., plates with deflections greatly exceeding the thickness), and shells (axially symmetric loading on cylinders, spheres, and spherical caps). Most of these have been obtained by a rigid-plastic type of analysis (in which strain rates are assumed zero unless a yield condition is satisfied). A few have been obtained by wholly numerical approaches of finite-difference type.

Experiments reported in the literature have, in most cases, shown that the actual permanent deflections are smaller than those predicted on the basis of plastic properties determined by quasi-static tests, the predictions often being in error by as much as 100 percent or more for mild steel, with smaller discrepancies for other metals, such as aluminum alloys or high-strength steels. Strengthening under conditions of rapid straining has been considered the principal cause of such discrepancies; when it has been possible to modify the analysis to take account of the increase of yield and flow stresses at high strain rates, much better agreement has, in most cases, been obtained.

59. Semonian, J.W., and Anderson, P.A., AN ANALYSIS OF THE STABILITY AND ULTIMATE BENDING STRENGTH OF MULTIWEB BEAMS WITH FORMED CHANNEL WEBS, NACA Technical Note TN 3232, 1954.

Design curves and procedures are presented for calculating the stresses for instability and failure of multiweb beams with formed-channel webs. The ultimate bending strength of this type of construction is shown to depend upon the deflectional stiffness of the web attachment flanges. A simple criterion is also given for predicting whether a multiweb beam with a given attachment-flange design will be susceptible to a wrinkling instability or will buckle as if the webs are integrally joined to the cover skins.

The criteria for predicting buckling and failure stresses are compared with experimental data. These criteria are sensitive to the offset, pitch, and diameter of the rivets used on the web attachment flanges, and the riveting specification is, therefore, emphasized as an important design consideration.

60. Semonian, J.W., and Peterson, J.P., AN ANALYSIS OF THE STABILITY AND ULTIMATE COMPRESSIVE STRENGTH OF SHORT SHEET-STRINGER PANELS WITH SPECIAL REFERENCE TO THE INFLUENCE OF THE RIVETED CONNECTION BETWEEN SHEET AND STRINGER, NACA Technical Report TR 1255, 1956.

A method of strength analysis of short sheet-stringer panels subjected to compression is presented which takes into account the effect that the riveted attachments between the plate and the stiffeners have on the strength of panels. An analysis of experimental data shows that panel strength is highly influenced by rivet pitch, diameter, and location and that the degree of influence for a given riveting depends on the panel configuration and panel material.

A

REPORT NUMBER		1	2	3	4	5	6	7	8	9	10	11	12	13
Specific Contents and Areas of Applicability	Elastic-Plastic Behavior	•			•	•	•		•	•	•	•	•	
	Elastic Behavior	•			•		•	•	•	•		•		
	Inelastic Behavior	•			•	•	•					•		•
	Plastic Deformation				•		•						•	
	Buckling				•		•	•			•		•	•
	Analytical Procedures						•	•	•					
	Linear Analysis	•					•	•	•					
	Nonlinear Analysis	•				•	•							
	Static Analysis									•				
	Dynamic Analysis	•		•	•			•	•	•	•			
	Analytical Programs	•			•							•		
	Experimental Data	•			•		•	•			•	•		
	Static Test	•			•									
	Dynamic Test	•		•	•	•					•			
	Load-Deflection Curves	•			•		•						•	
	Airframe Structure	•	•		•		•	•	•					
	Frame Structure					•	•	•						•
	Beam Elements	•			•	•	•	•	•					
	Columns, Bars, Tubes							•	•					•
	Plates, Shells, Cylinders	•			•		•		•	•	•	•	•	
	Composite Elements				•		•	•						
	Large Displacements	•				•				•		•		
	Design Procedures/Guidelines	•	•				•	•					•	
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	Inertia Effect	•							•					
	Strain Rate Effects					•								
	Energy Absorption	•	•	•	•									
	Load-Limiting Devices		•											
	Failure Modes	•			•					•			•	•

TABLE I. LITERATURE MATRIX CATEGORIZATION

[illegible]

CATEGORIZATION

27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	
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TEST DATA

DYNAMIC TEST INSTALLATION

The dynamic test setup uses existing components. The drop carriage assembly is the same one that was utilized during the drop test of the fuselage bumper, the results of which are reported in Reference 1. Between the test specimen support and the ground are installed six load cells (two each along the north and south edges of the support and one each along the east and west edge of the support). The load cells are used for tests 7 and 8. In the previous dynamic tests (4, 5, 6), the specimen support is grouted to the concrete slab ground. The installation provides sufficient free-fall clearance (14 ft) to perform 30-ft/sec impact tests. The setup has the flexibility of performing higher impact velocity drop tests by adding additional frames, thus increasing the available free-fall distance. Figure 1 shows a layout of the test installation and notes the various items and assemblies that form a part of the complete installation.

TEST SPECIMENS

A complete description of the test specimen overall dimensions, bulkhead and end beam web thicknesses, number and thickness of angles, design configuration, nominal weight, and type of test performed for each specimen is shown in Volume I, Table XVII. A complete set of drawings for the various design configurations is presented herein as described in Table II below:

Table II. SPECIMEN DRAWING IDENTIFICATION		
Figure	Applicable Specimen	Drawing X-16628A
2	1	-1
3	2,3,4	-2
4	5,6	-3
5	7,8,9	-4
6	10,11,12	-5

RECORDED SCAN (TIME) HISTORIES

A complete set of pertinent recorded test data is presented in Figures 7 through 80. Load-deflection curves for each of the test specimens are presented in Volume I in the section entitled Substructure Test Program.

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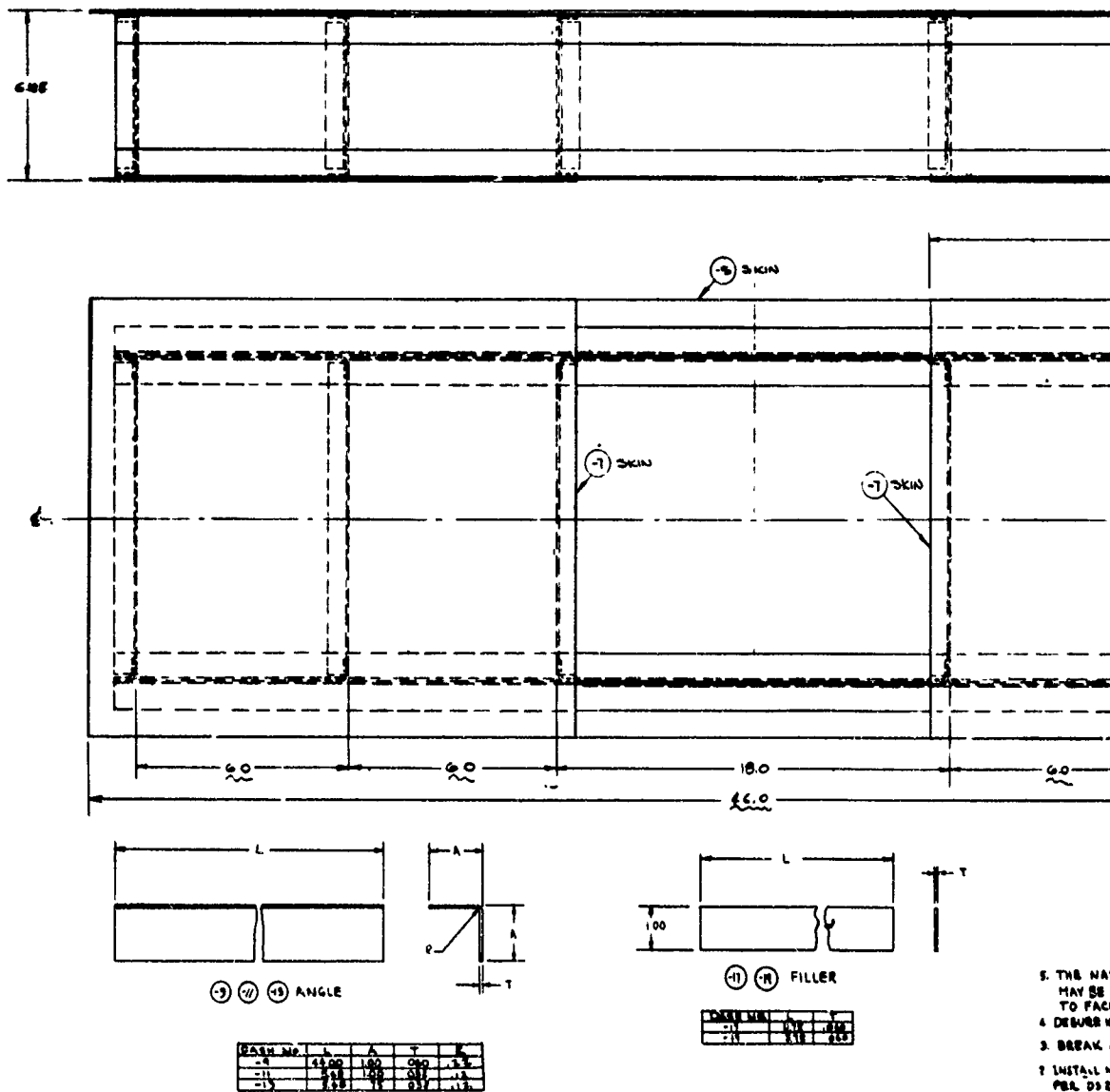
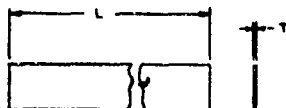
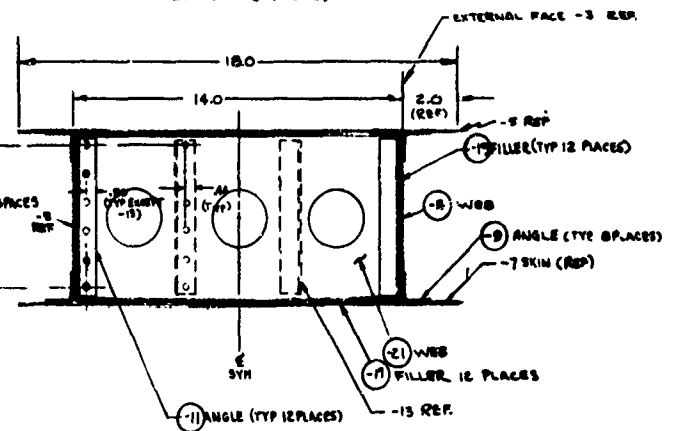


Figure 2. Drawing for Specimen 1.

[illegible][illegible]

Examen 1.

A

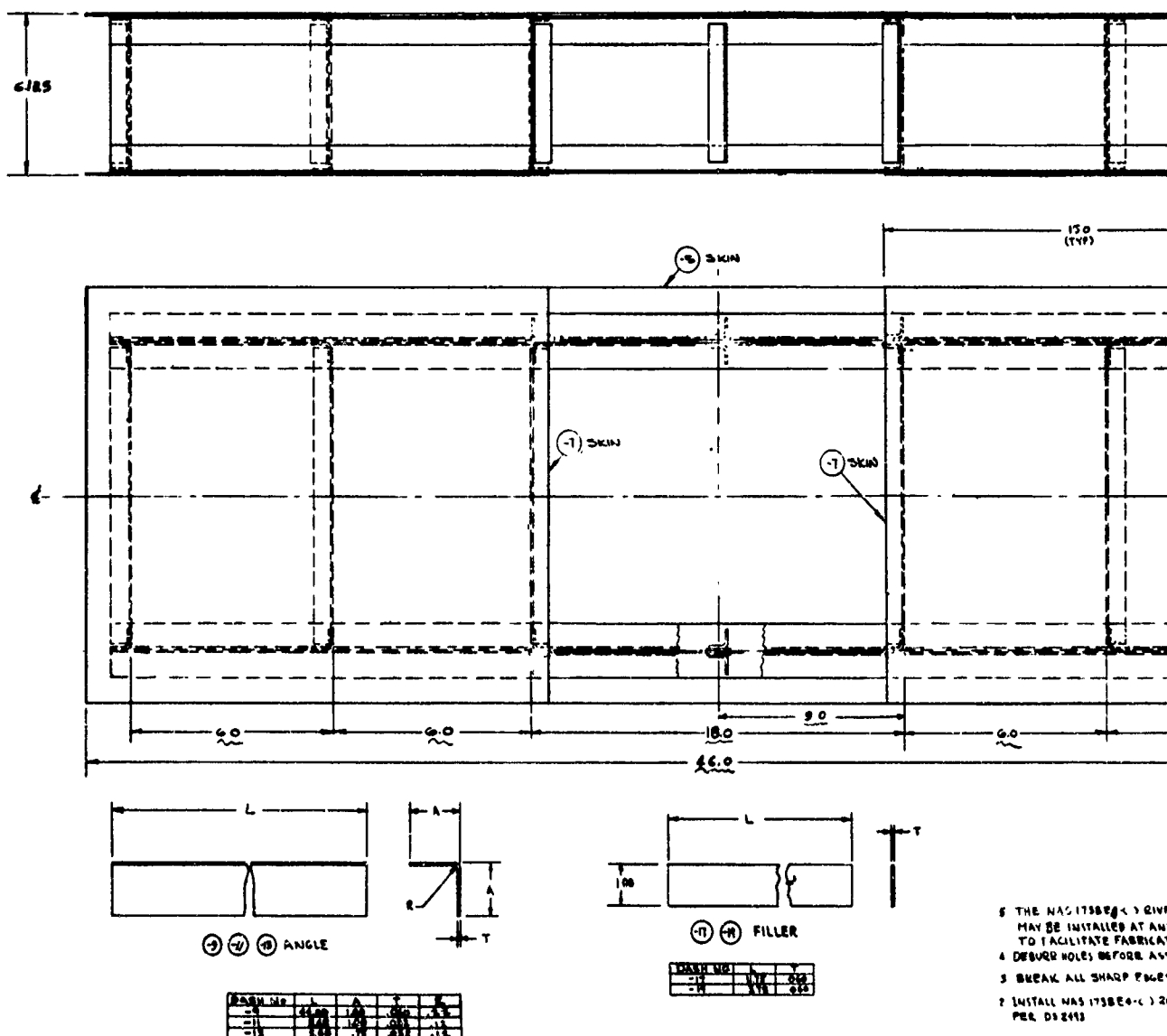


Figure 3. Drawing for Specimens 2, 3, and 4.

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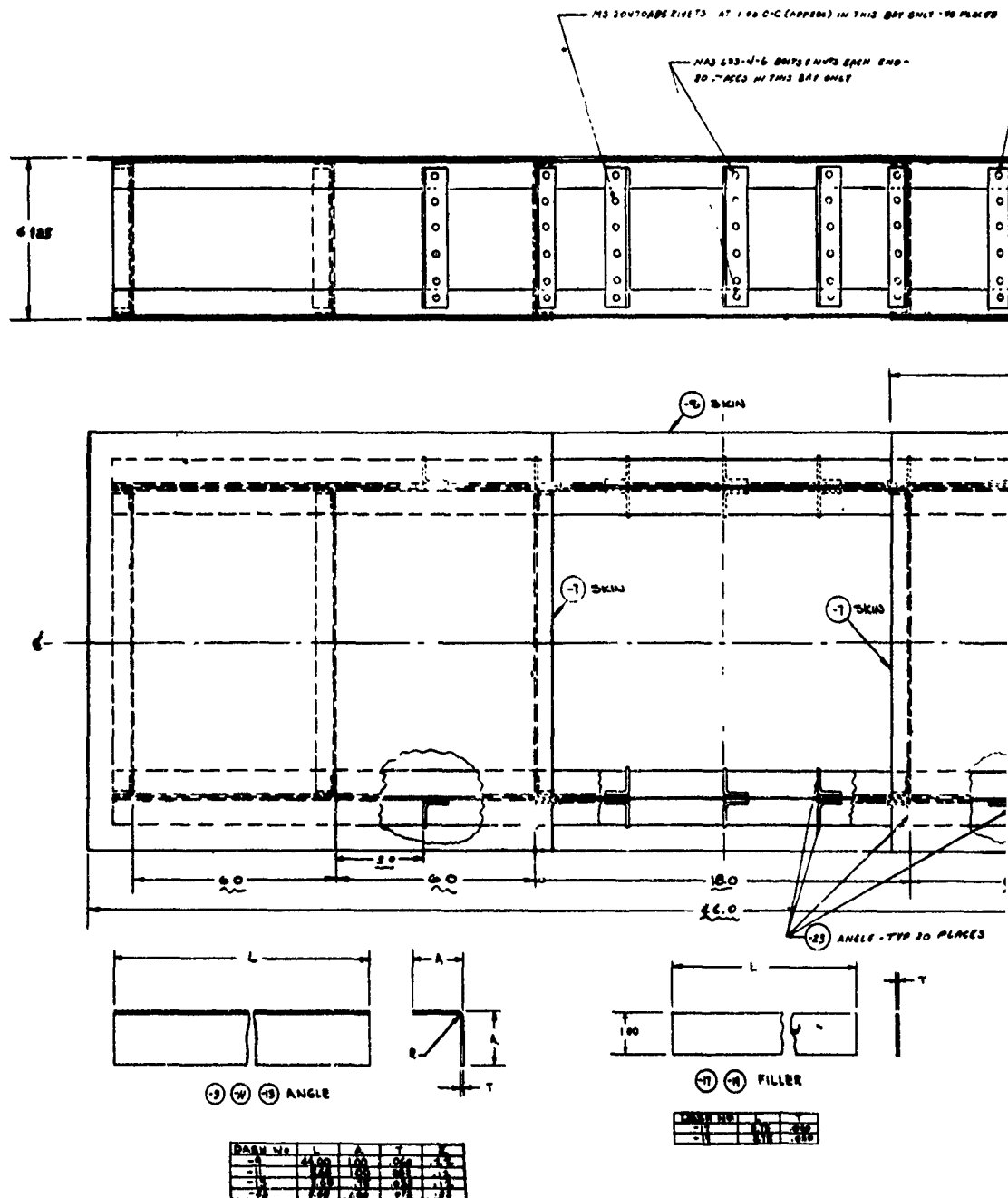
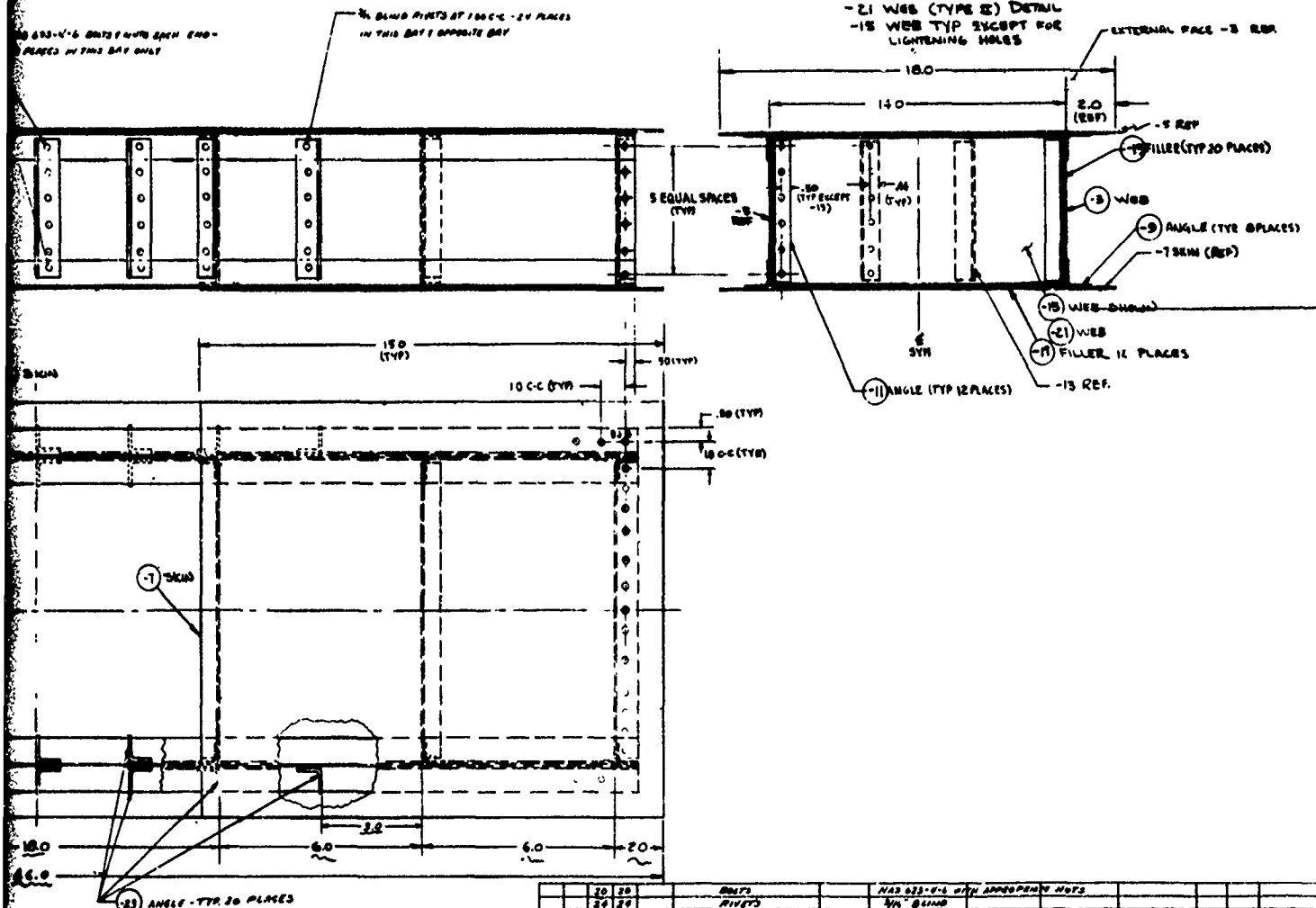


Figure 4. Drawing for Specimens 5 and 6.

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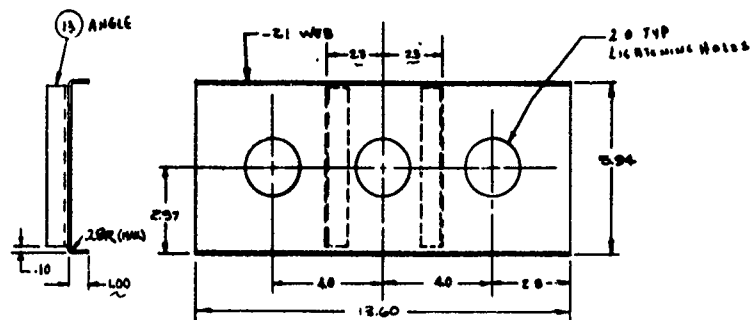
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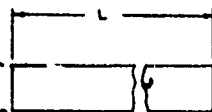
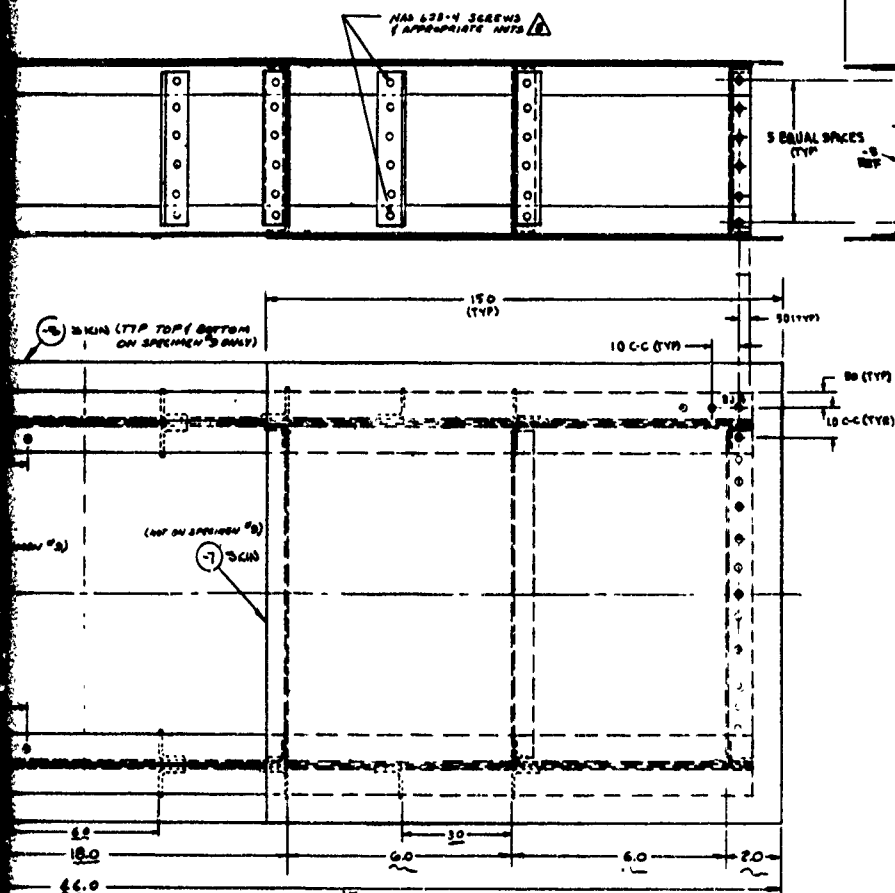
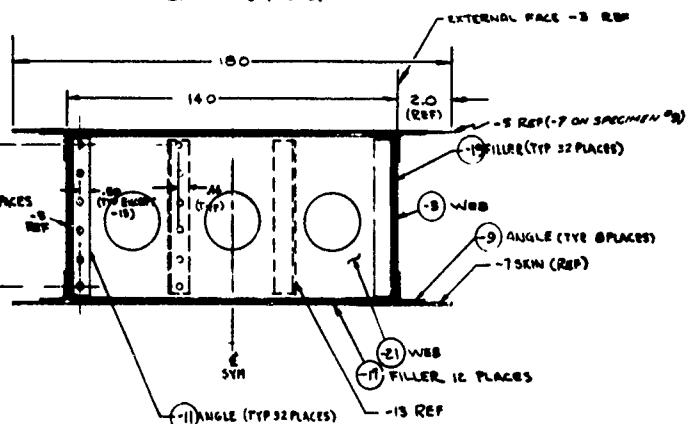
LIST OF MATERIALS OR PARTS LIST

5 and 6.

B



-21 WEB (TYPE 2) DETAIL



(7) (9) FILLER

CHAMFER	1/4"	1/8"	1/16"
1/4"	1/8"	1/16"	1/32"

△ ATTACHMENTS AT ENDS OF ALL "H" ANGLES SHALL BE HAS 622-4 SCREWS AND APPROPRIATE NUTS

△ HAS 1222B-1 RIVETS USED IN LOCATION NOTED FOR SPECIMEN #B
6. RIVETS THAT SEEM TO BE ADV. ALL OTHER RIVETS TO BE ADV.

5. REMOVED

4. DEBURR HOLES BEFORE ASSY

3. BREAK ALL SHARP EDGES

2. REMOVED

1. CDD BUSH NUMBER SHOWN NEXT HIGHER CONDUCTIVE OVER BUSH NUMBER OPPOSITE NOTES

QTY	NO	REV	DESCRIPTION	CODE	PART OR IDENTIFYING NO.	MATERIAL SPECIFICATION	MATERIAL OR NOTES	SIZE OR SHAPE	UNIT WT	100%	HEAT TREAT
10	10	10	RIVETS								
10	10	10	BOLTS								
10	10	10	RIVETS								
10	10	10	RIVETS								
10	10	10	WEB								
10	10	10	FILLER								
10	10	10	FILLER								
10	10	10	ANGLE								
10	10	10	ANGLE								
10	10	10	ANGLE								
10	10	10	SKIN								
10	10	10	SKIN								
10	10	10	WEB								
10	10	10	ASSY								

7, 8, and 9.

Technical drawing of a mechanical part, showing front and side views with dimensions and labels.

Front View Dimensions:

- Overall width: 13.60
- Overall height: 5.94
- Distance between hole centers: 4.0, 4.0, 2.5
- Distance from left edge to first hole center: 2.57
- Distance from right edge to last hole center: 2.5
- Distance between vertical dashed lines: 2.5, 2.5

Side View Dimensions:

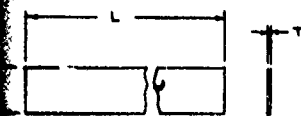
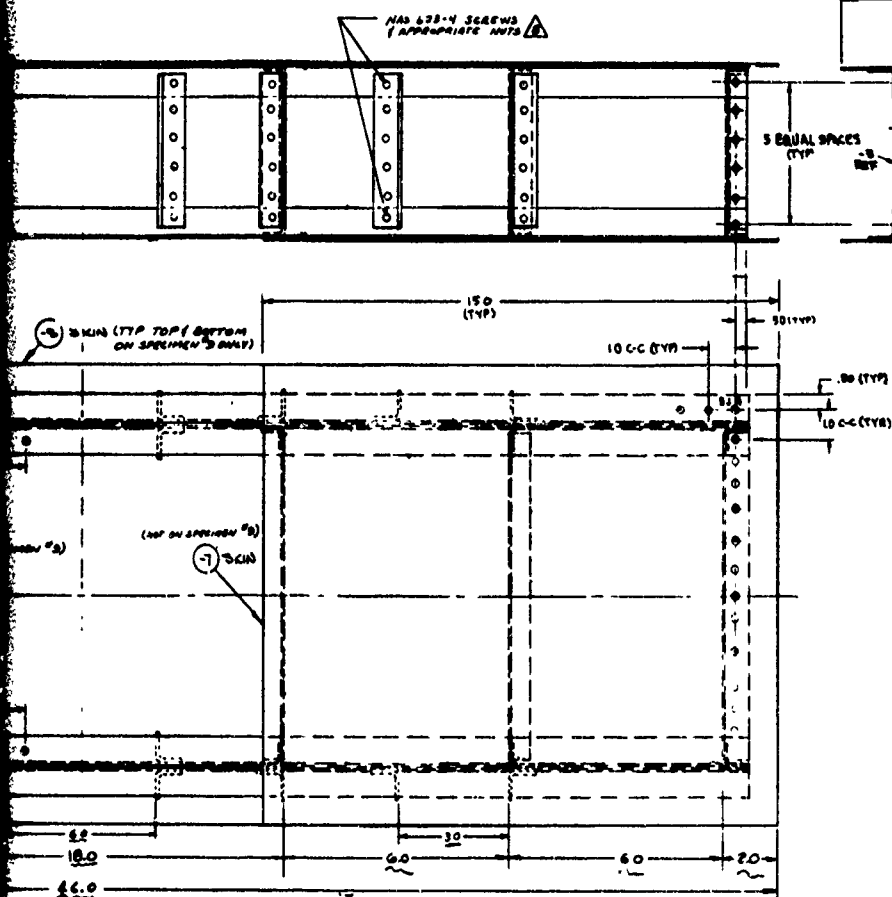
- Overall height: 1.10
- Distance from bottom edge to top edge: 2.94
- Distance from left edge to right edge: 1.00

Labels and Notes:

- 15 ANGLE (pointing to the top-left corner of the side view)
- 20 TYP 216 HORIZONTAL HOLES (pointing to the three circular holes in the front view)
- 21 WEB (pointing to the vertical dashed lines in the front view)
- 22 R (pointing to the rounded bottom-left corner of the side view)

Technical drawing of a rectangular specimen with dimensions and callouts:

- Overall width: 180
- Internal width: 140
- Top edge callout: EXTERNAL FACE - 3 REF
- Right edge callout: 2.0 (REF)
- Right edge callout: -5 REF (-7 ON SPECIMEN 90)
- Right edge callout: -10 FILLER (TYP 32 PLACES)
- Right edge callout: -3 WEB
- Right edge callout: -9 ANGLE (TYP 8 PLACES)
- Right edge callout: -7 SKIN (REF)
- Bottom edge callout: 54H
- Bottom edge callout: -21 WEB
- Bottom edge callout: -17 FILLER 12 PLACES
- Bottom edge callout: -11 ANGLE (TYP 32 PLACES)
- Bottom edge callout: -13 REF
- Left edge callout: -8 REF
- Left edge callout: -15
- Left edge callout: -11
- Left edge callout: -10
- Left edge callout: -9
- Left edge callout: -8
- Left edge callout: -7
- Left edge callout: -6
- Left edge callout: -5
- Left edge callout: -4
- Left edge callout: -3
- Left edge callout: -2
- Left edge callout: -1



(-7) (-R) FILLER

DATE	TIME	LOCATION	REMARKS
10/10/19	14:00	100m	100m
10/10/19	14:00	100m	100m
10/10/19	14:00	100m	100m

- ATTACHMENTS AT ENDS OF ALL "H" ANGLES SHALL BE NAB 633-W SCREWS AND APPROPRIATE NUTS

- △ HAS 13384-() RIVETS USED IN
LOCATION NOTED 'X' FOR SPECIMEN 'D'
6. RIVETS THAT SEEMS TO BE ADJ. ALL
OTHER RIVETS TO BE ADJ

- 5. REMARKS**

- 4 DEBURR HOLES BEFORE ASSY

- 3 BREAK ALL SHARP EDGES

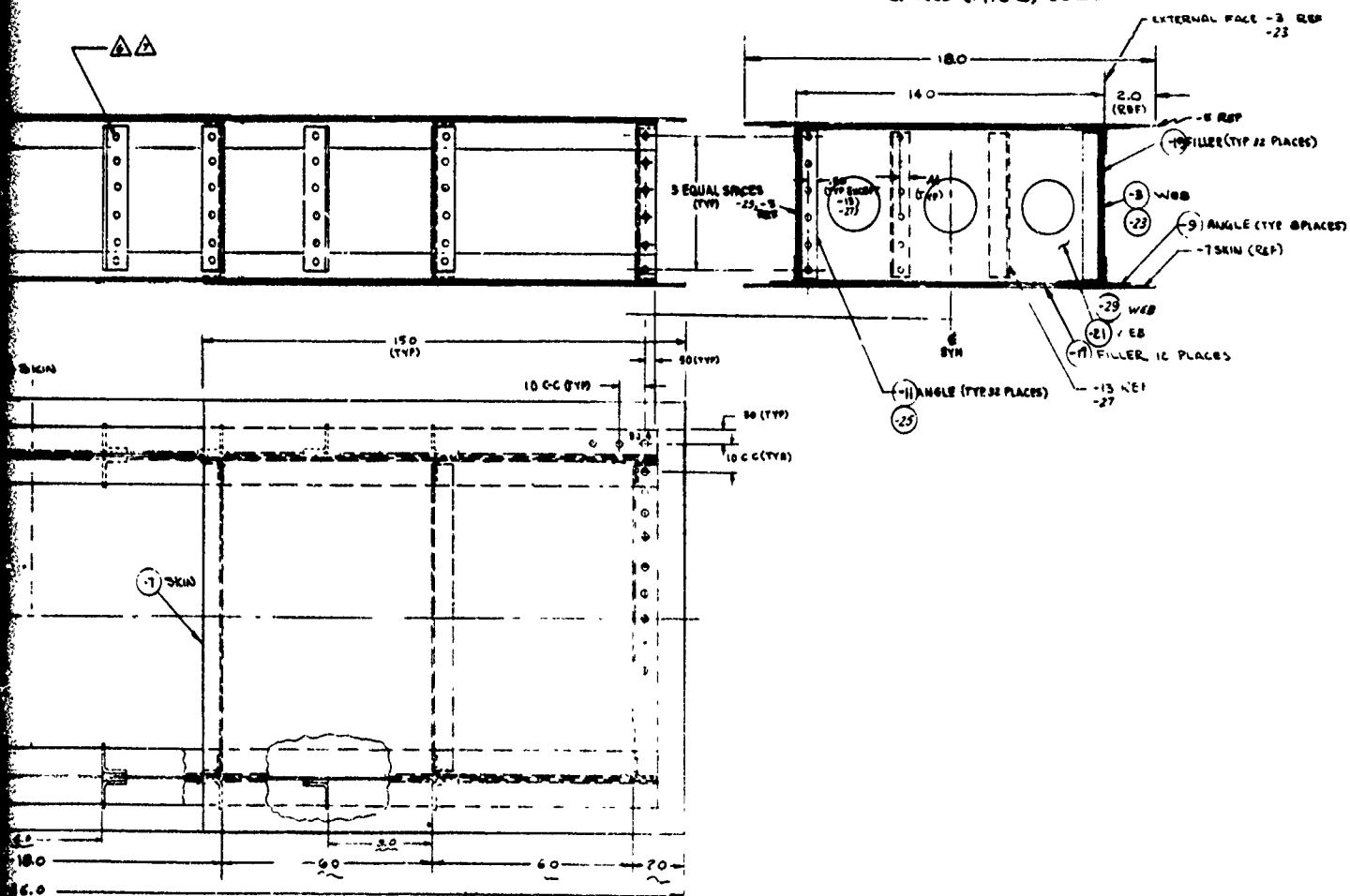
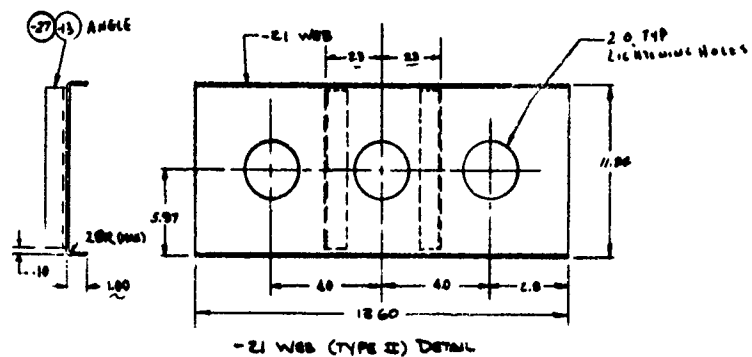
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7, 8, and 9.

[illegible]



7. SPECIMEN NO 11 17" IN HEIGHT AT
ENDS OF 28 ANGLES SHALL BE
HAS 823-4 SCREWS RIVETS THRU
SKINS SHALL BE ADV ALL OTHER
RIVETS SHALL BE AGT

△ SPECIMEN NOS 10112 RIVETS THRU
ENDS OF 11 ANGLES SHALL BE
CO 6 () ALL OTHER RIVETS
MAY BE ADS

5 REMOVED

4 DEBURR HOLE BEFORE ASSY

3 BREAK ALL SHARP EDGES

2. *Chlorophyll*

1. ODD DASH NUMBER SHOWN NEXT HIGHER
CONSECUTIVE EVEN DASH NUMBER OPPOSITE
NOTES

METHODS

① ② HILLER

[illegible]

The data in this section includes load, deflection, acceleration (tests 4-8) and strain versus scan plots. Table III shows the test data available in the section for each specimen.

The sampling rate for test 4 is 750 scans/second. The sampling rate for tests 5 through 8 is 1500 scans/second; thus for the dynamic tests the time in seconds can be obtained by dividing the oscilloscope scale (scan) by 750 for test 4 and by 1500 for tests 5 through 8. Positive strain is compression for tests 1, 2, 3, 9, 10, 11, 12 and tension for tests 4, 5, 6, 7 and 8.

Table III. TEST DATA IDENTIFICATION

Table III. TEST DATA IDENTIFICATION												
Data Item	Test Specimen											
	1	2	3	4	5	6	7	8	9	10	11	12
Load	X	X	X				X	X	X	X	X	X
Deflection	X	X	X	X	X	X	X	X	X	X	X	X
Strain Gage 1A	X	X	X	X	X	X	X	X	X	X	X	X
Strain Gage 1B	X	X	X	X	X	X	X	X	X	X	X	X
Strain Gage 2	X	X	X	X	X	X						
Strain Gage 3	X	X	X									
Strain Gage 4	X	X	X									
Strain Gage 5	X	X	X									
Strain Gage 6	X	X	X	X	X	X						
Acceleration				X	X	X	X	X				
Applicable Figures	7 thru 13	14 thru 20	21 thru 29	30 thru 38	39 thru 44	45 thru 50	51 thru 60	61 thru 68	69 thru 71	72 thru 74	75 thru 77	78 thru 80

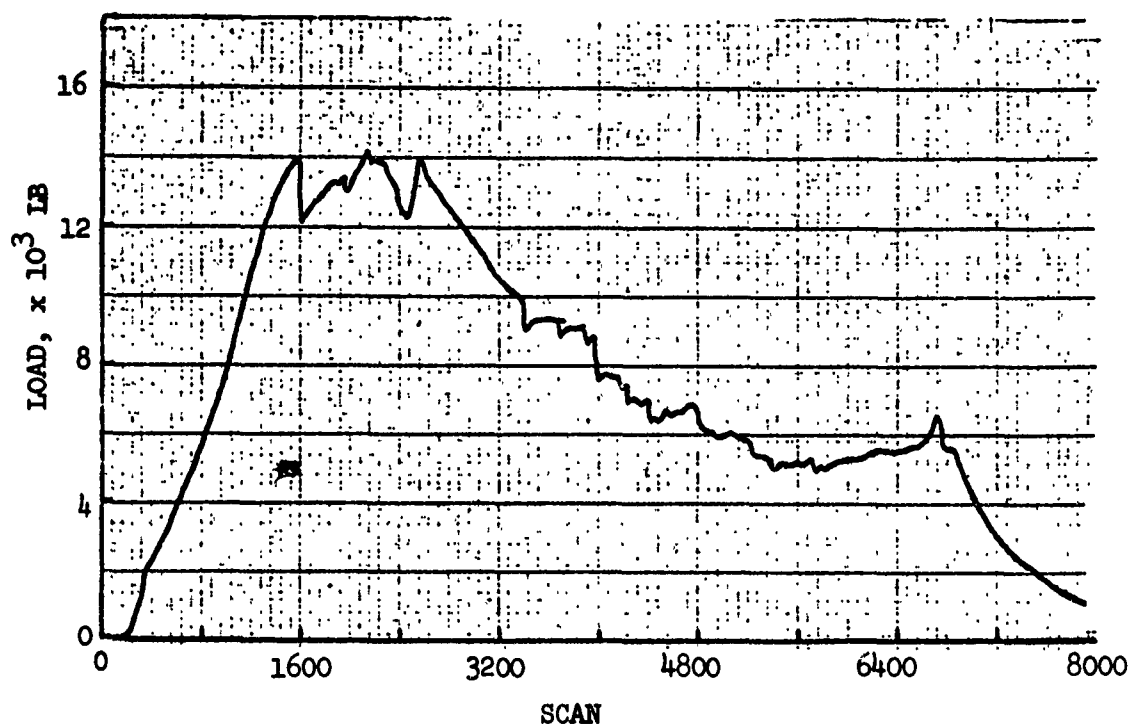


Figure 7. Load Versus Scan, Test Specimen 1.

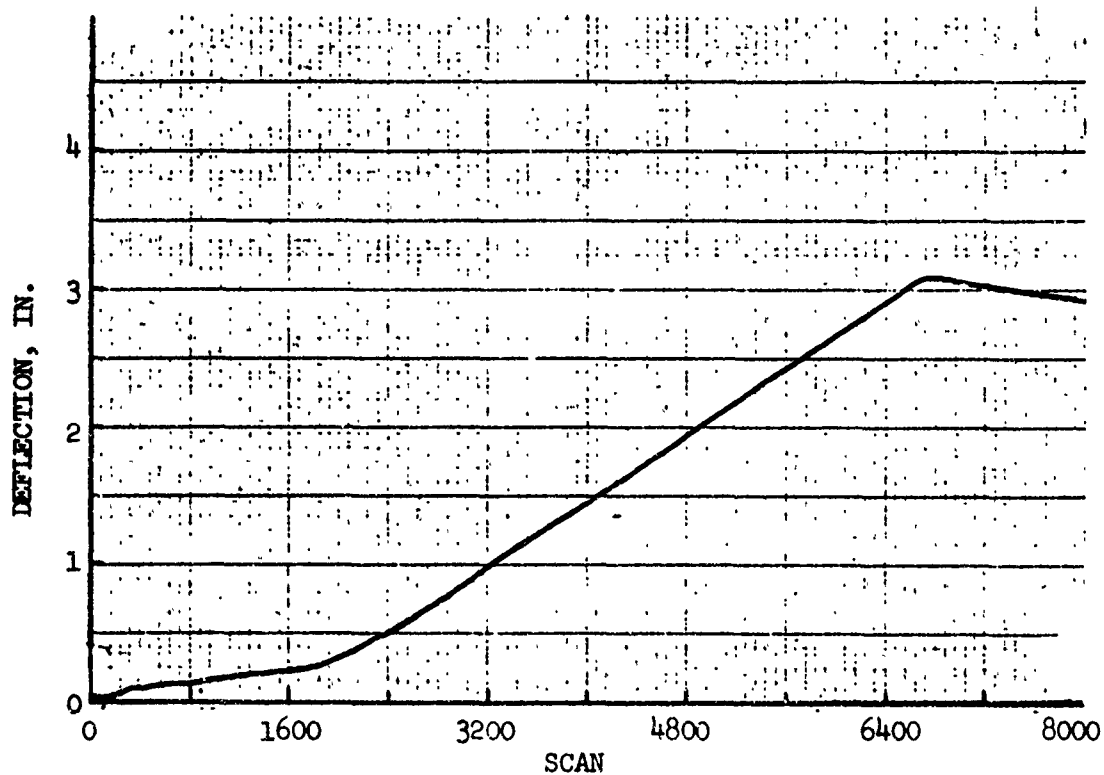


Figure 8. Deflection Versus Scan, Test Specimen 1.

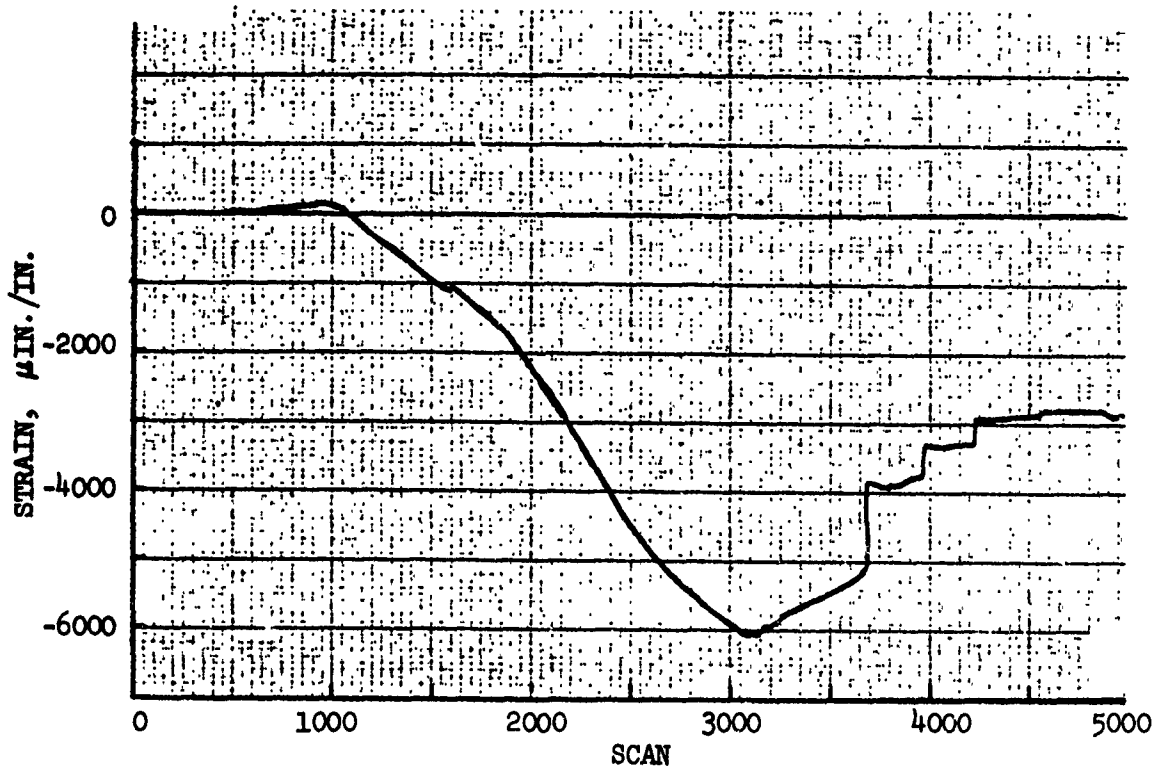


Figure 9. Strain Gage 1A Versus Scan, Test Specimen 1.

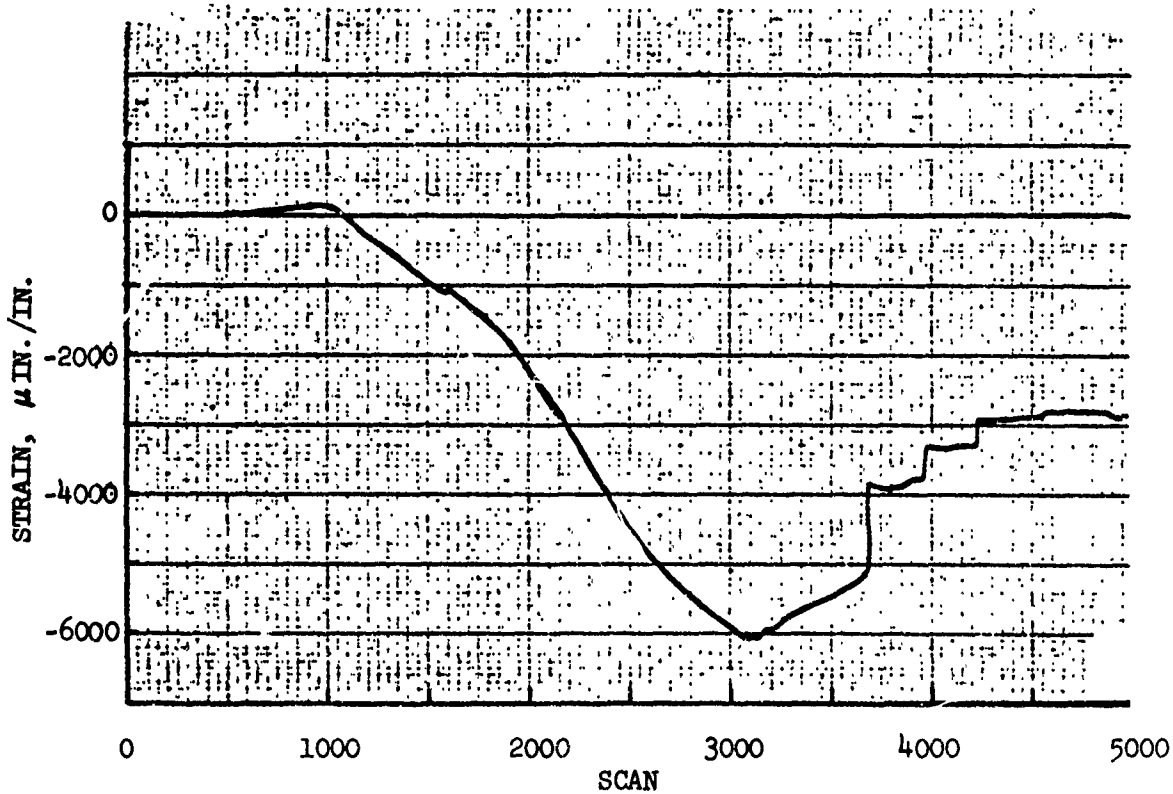


Figure 10. Strain Gage 1B Versus Scan, Test Specimen 1.

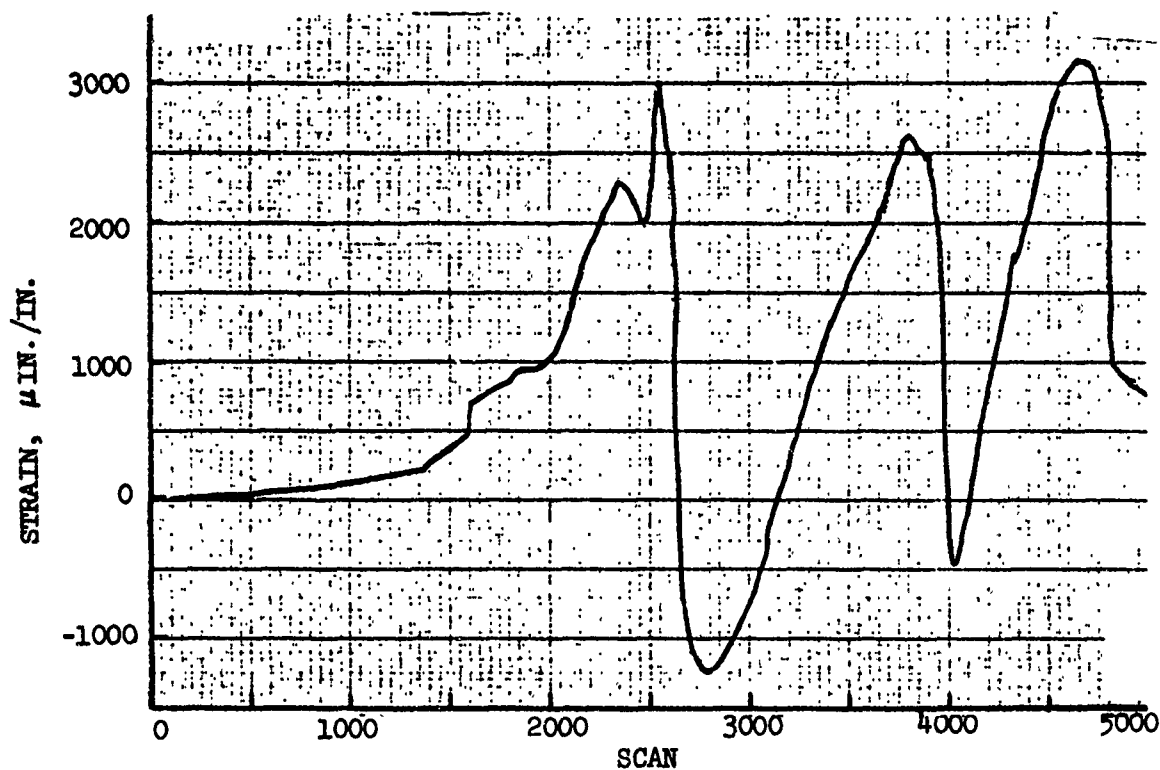


Figure 11. Strain Gage 2 Versus Scan, Test Specimen 1.

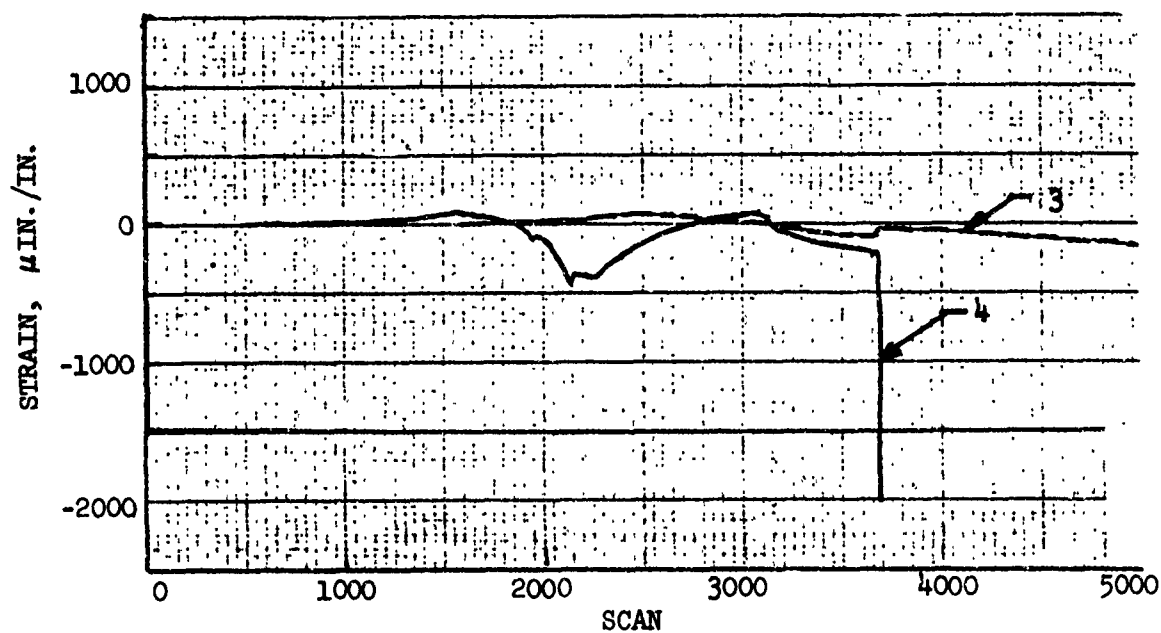


Figure 12. Strain Gages 3 and 4 Versus Scan, Test Specimen 1.

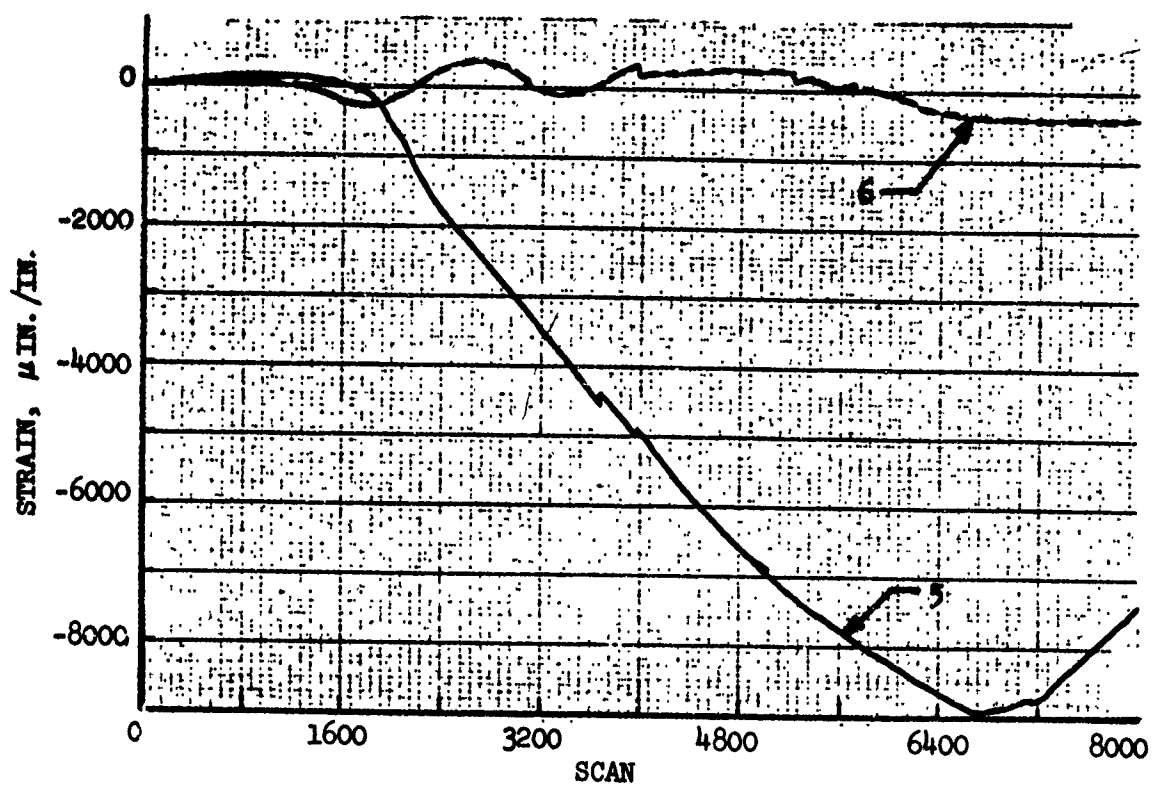


Figure 13. Strain Gages 5 and 6 Versus Scan, Test Specimen 1.

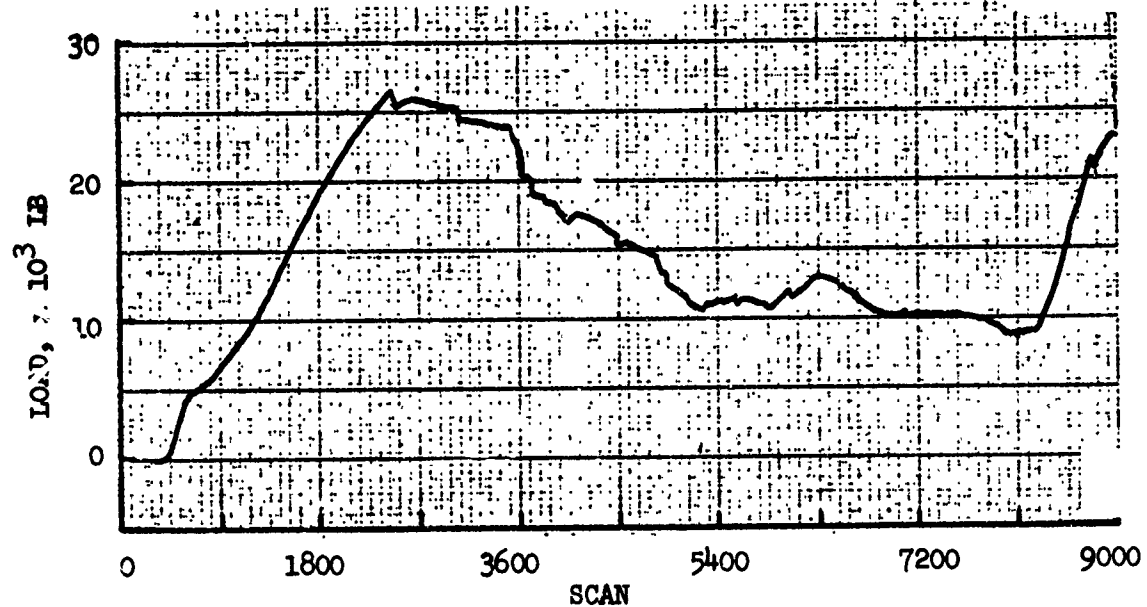


Figure 14. Load Versus Scan, Test Specimen 2.

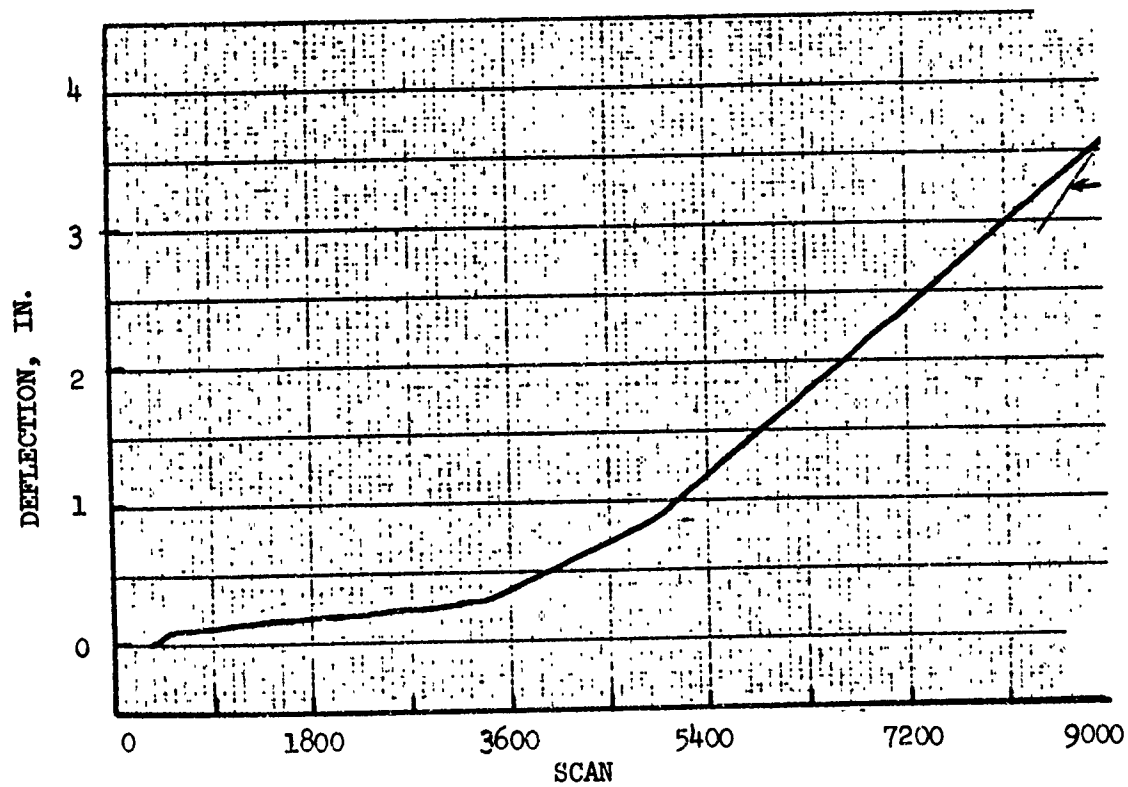


Figure 15. Deflection Versus Scan, Test Specimen 2.

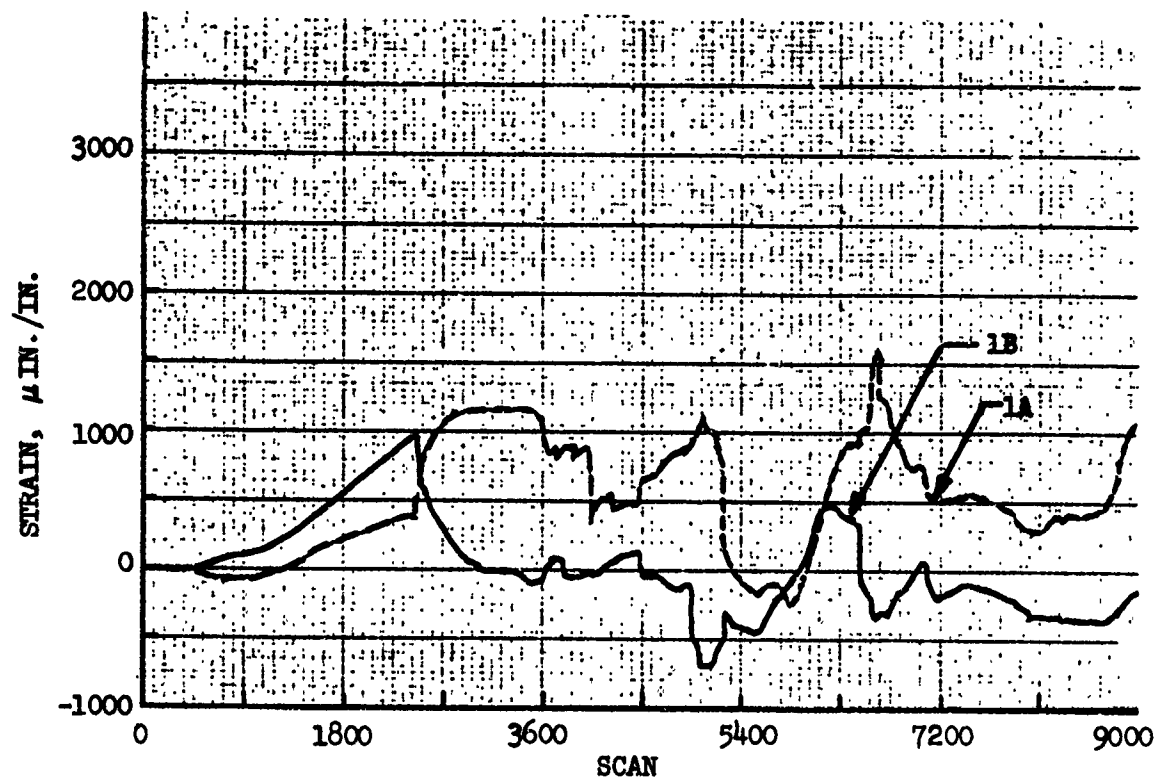


Figure 16. Strain Gages 1A and 1B Versus Scan, Test Specimen 2.

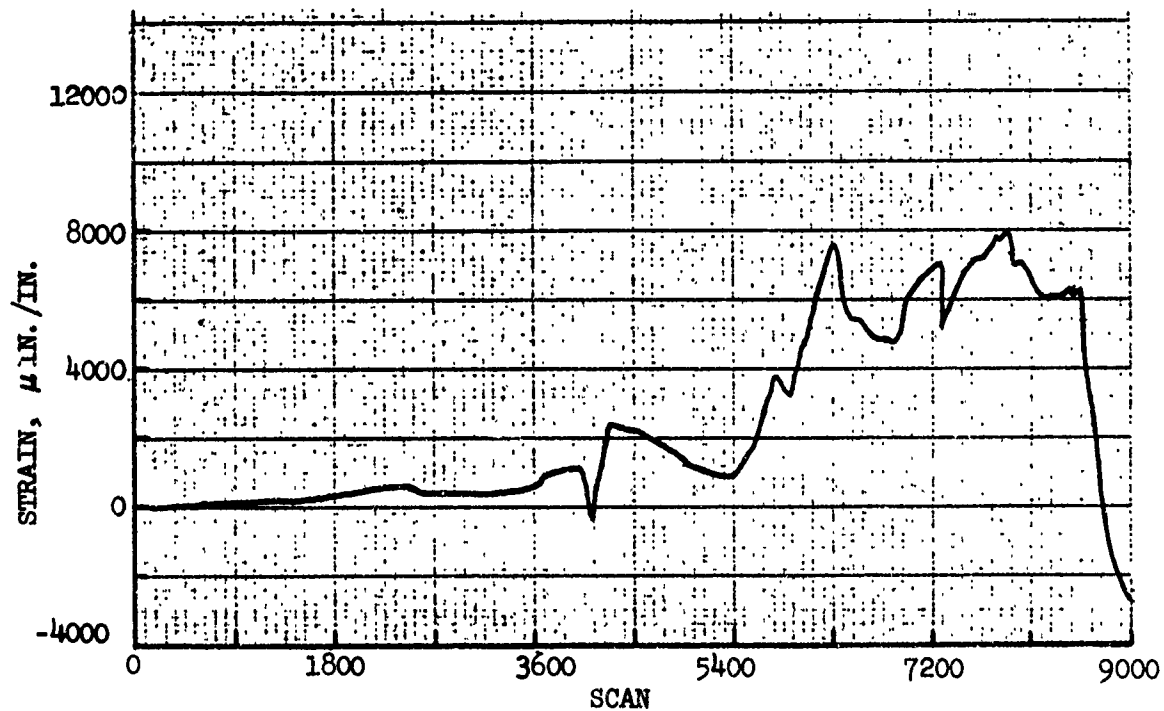


Figure 17. Strain Gage 2 Versus Scan, Test Specimen 2.

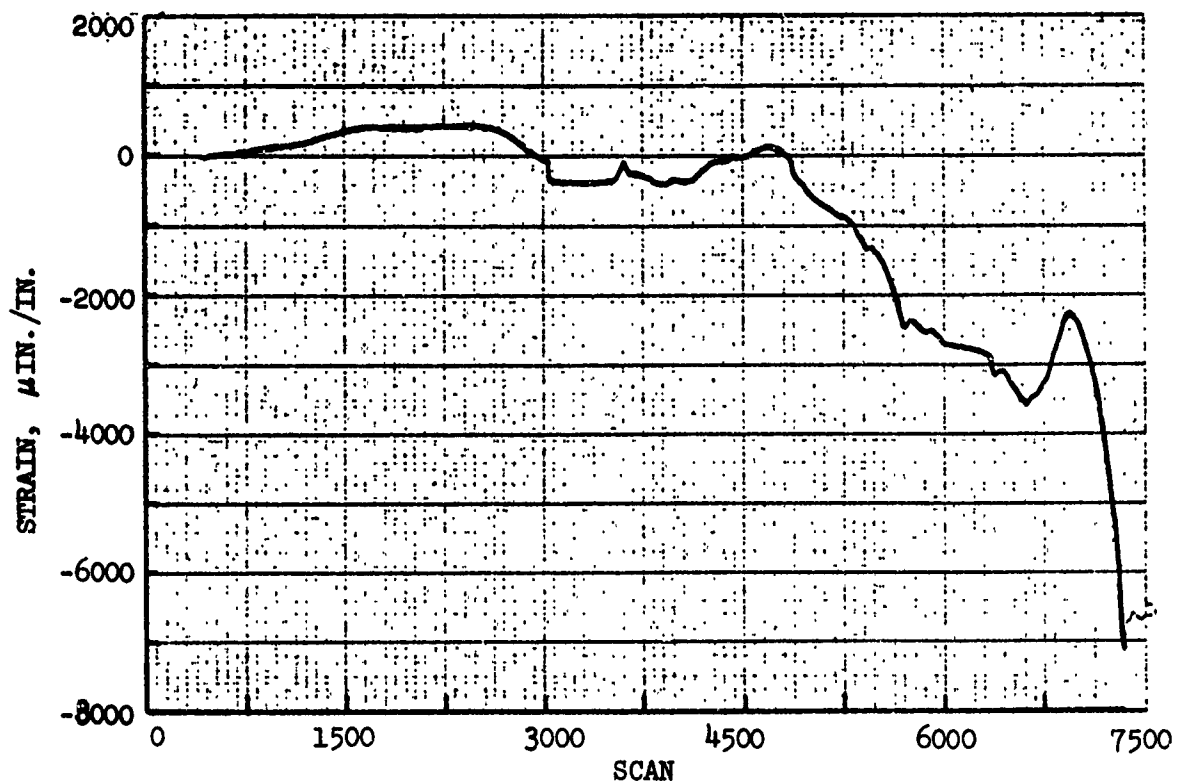


Figure 18. Strain Gage 3 Versus Scan, Test Specimen 2.

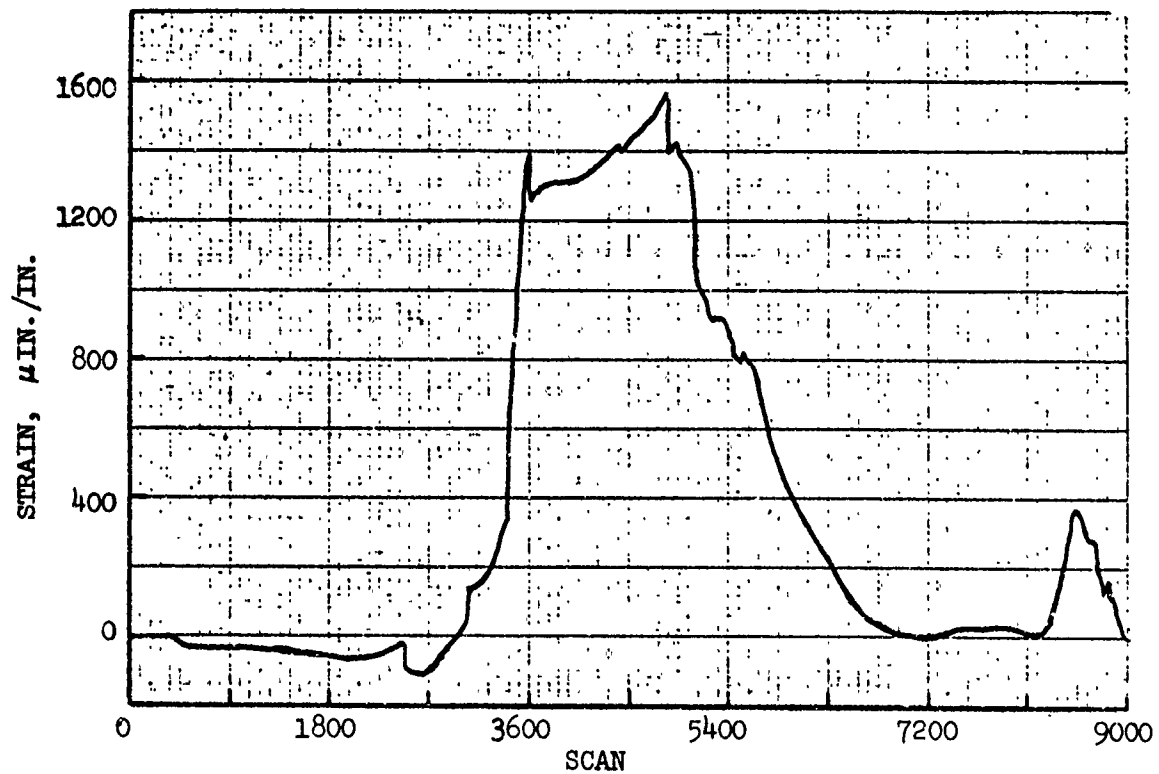


Figure 19. Strain Gage 4 Versus Scan, Test Specimen 2.

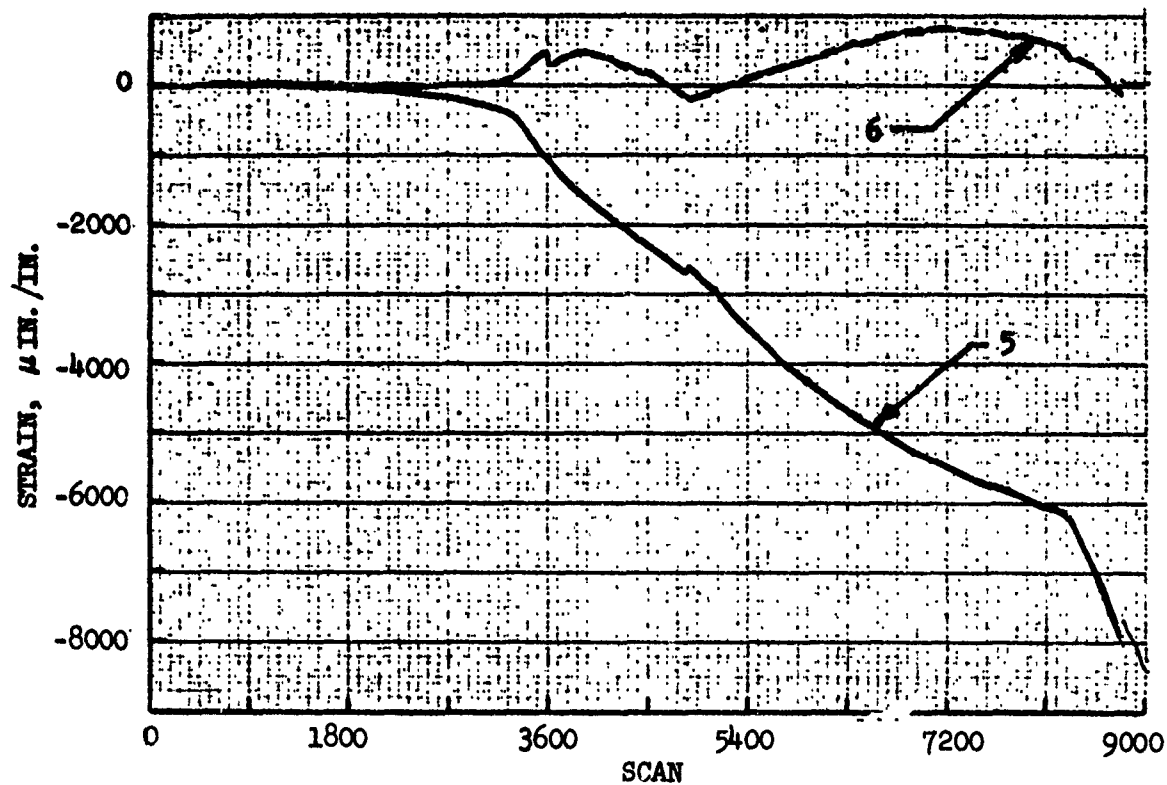


Figure 20. Strain Gages 5 and 6 Versus Scan, Test Specimen 2.

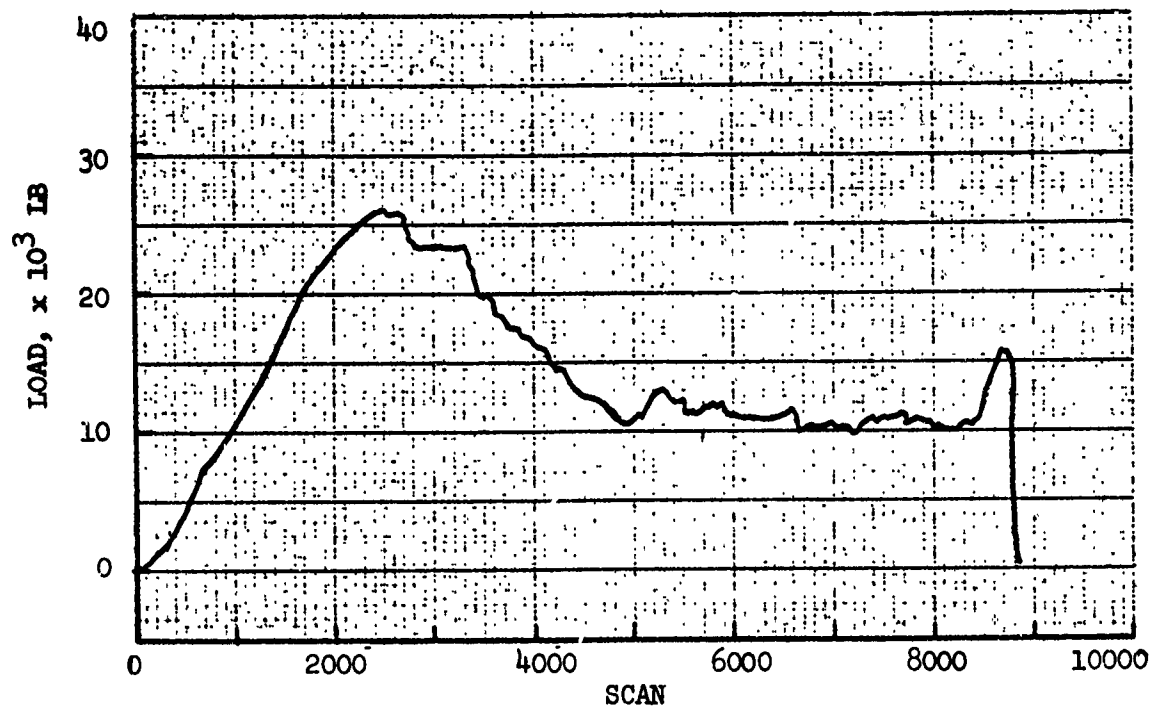


Figure 21. Load Versus Scan, Test Specimen 3.

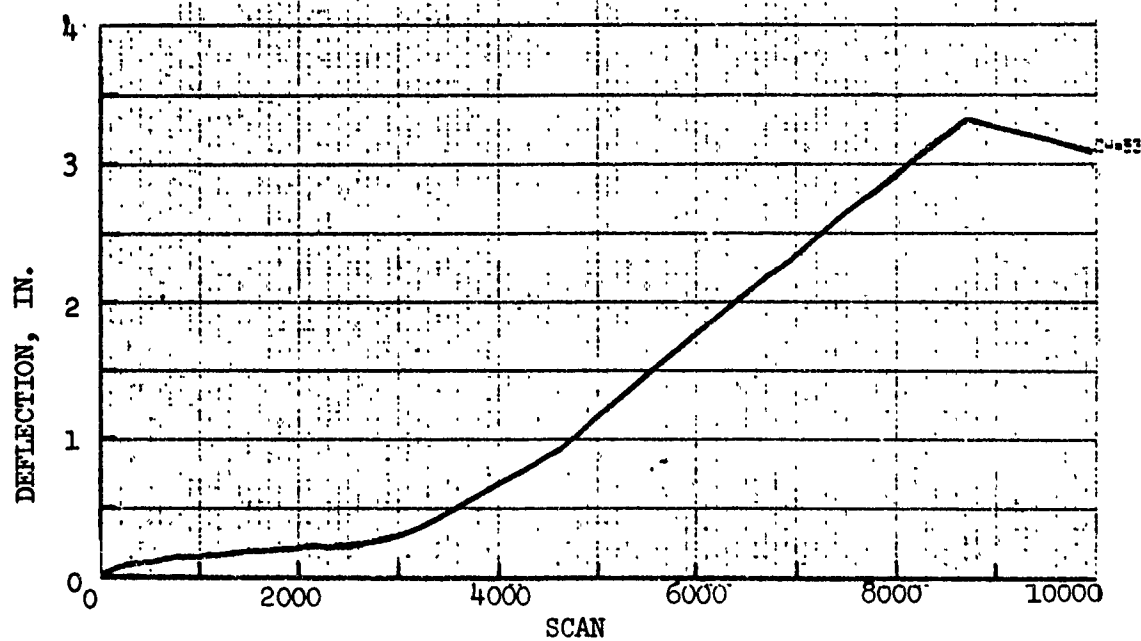


Figure 22. Deflection Versus Scan, Test Specimen 3.

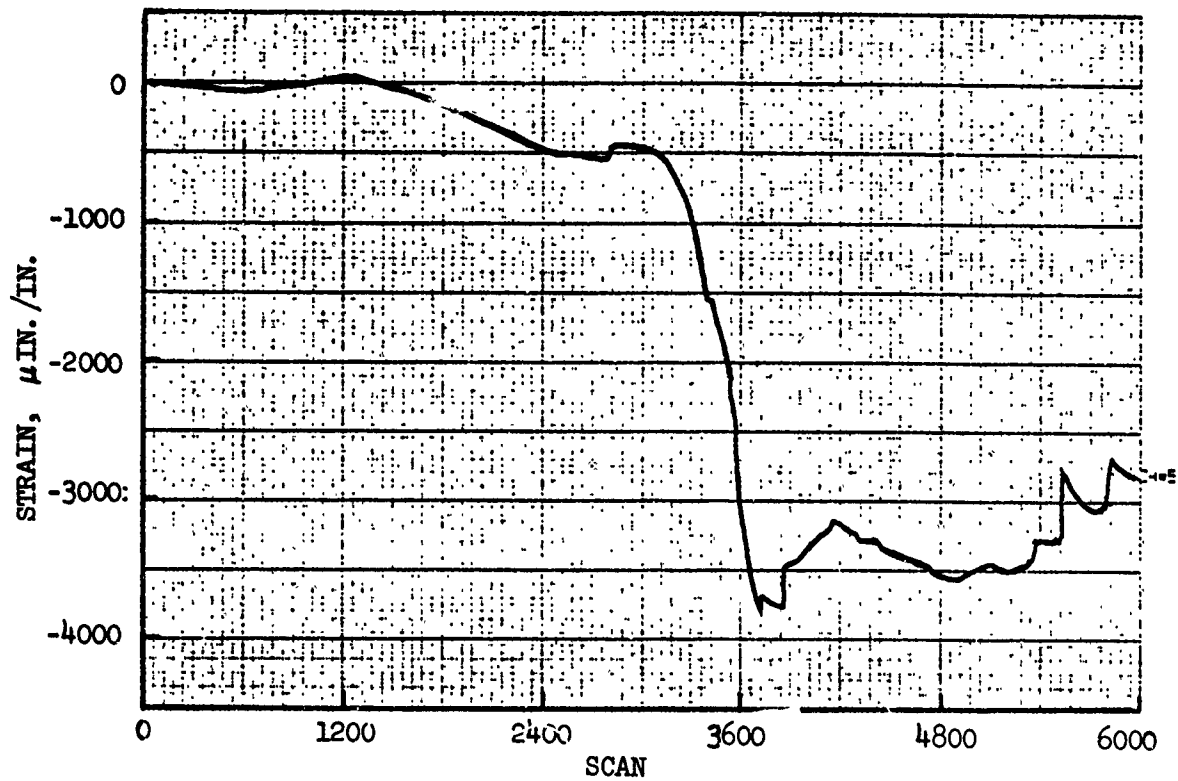


Figure 23. Strain Gage 1A Versus Scan, Test Specimen 3.

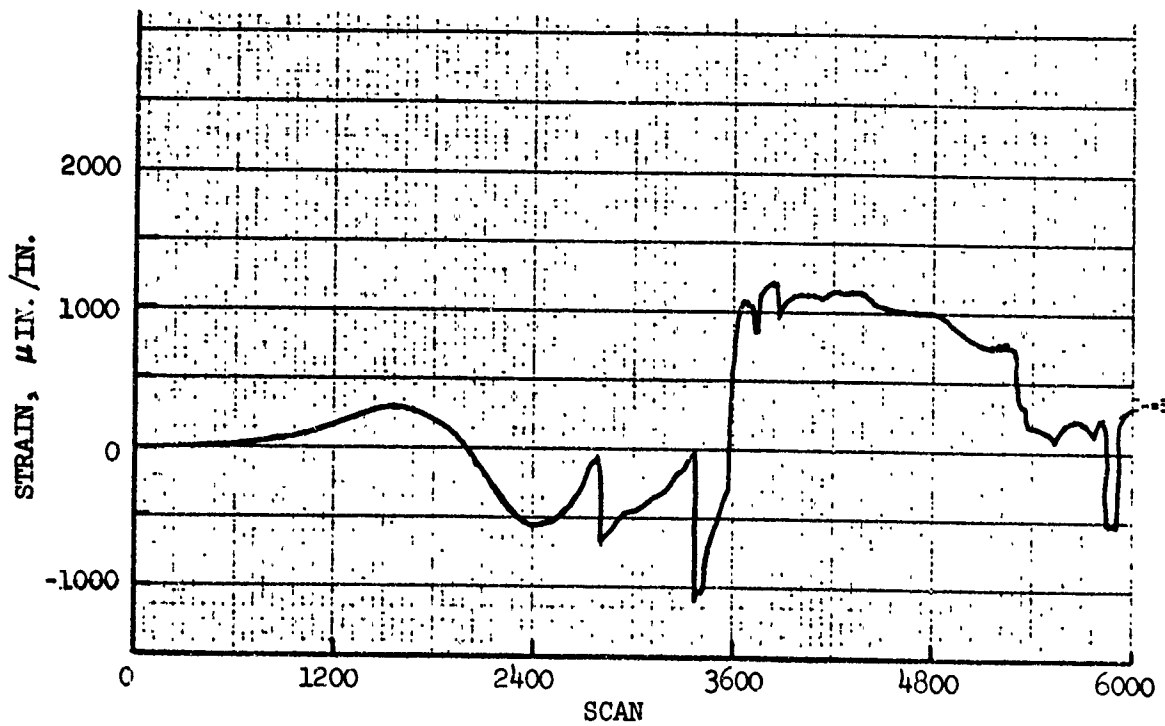


Figure 24. Strain Gage 1B Versus Scan, Test Specimen 3.

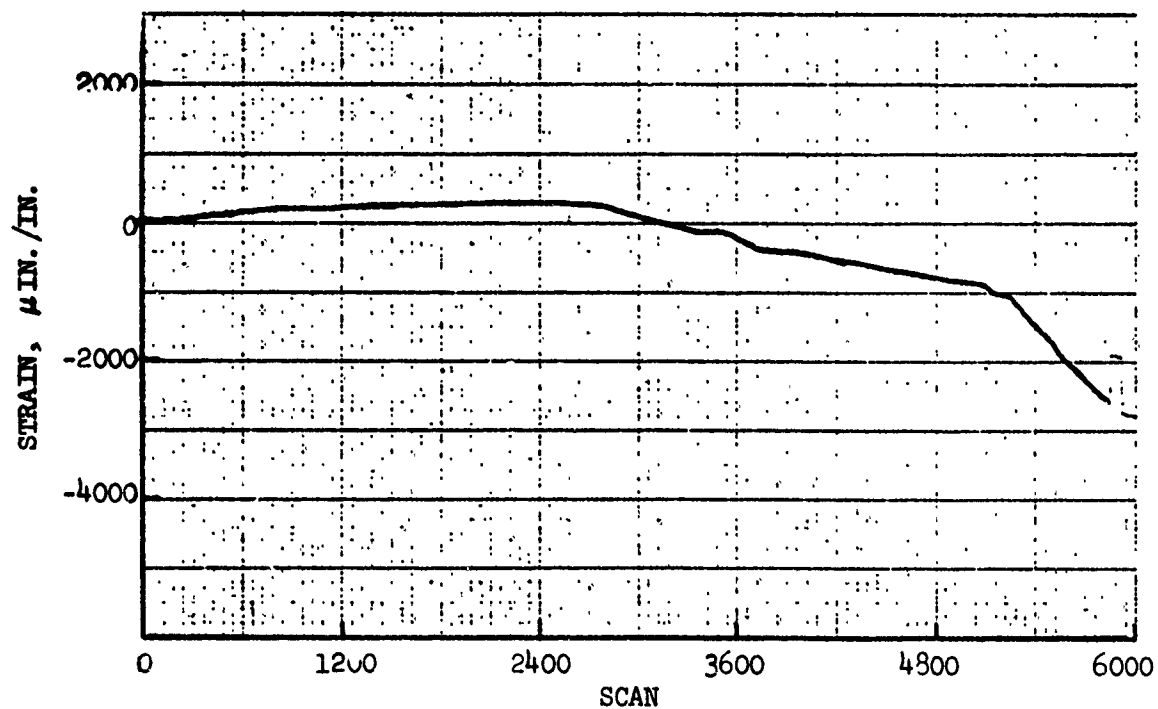


Figure 25. Strain Gage 2 Versus Scan Test Specimen 3.

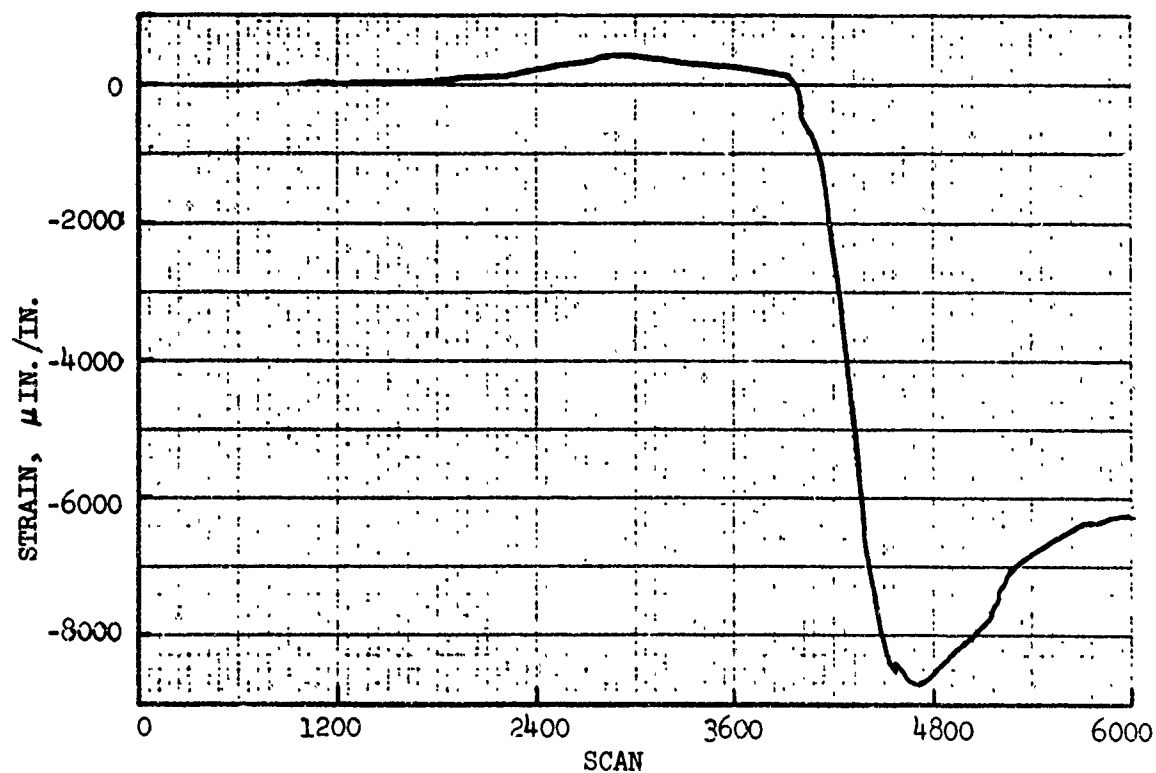


Figure 26. Strain Gage 3 Versus Scan, Test Specimen 3.

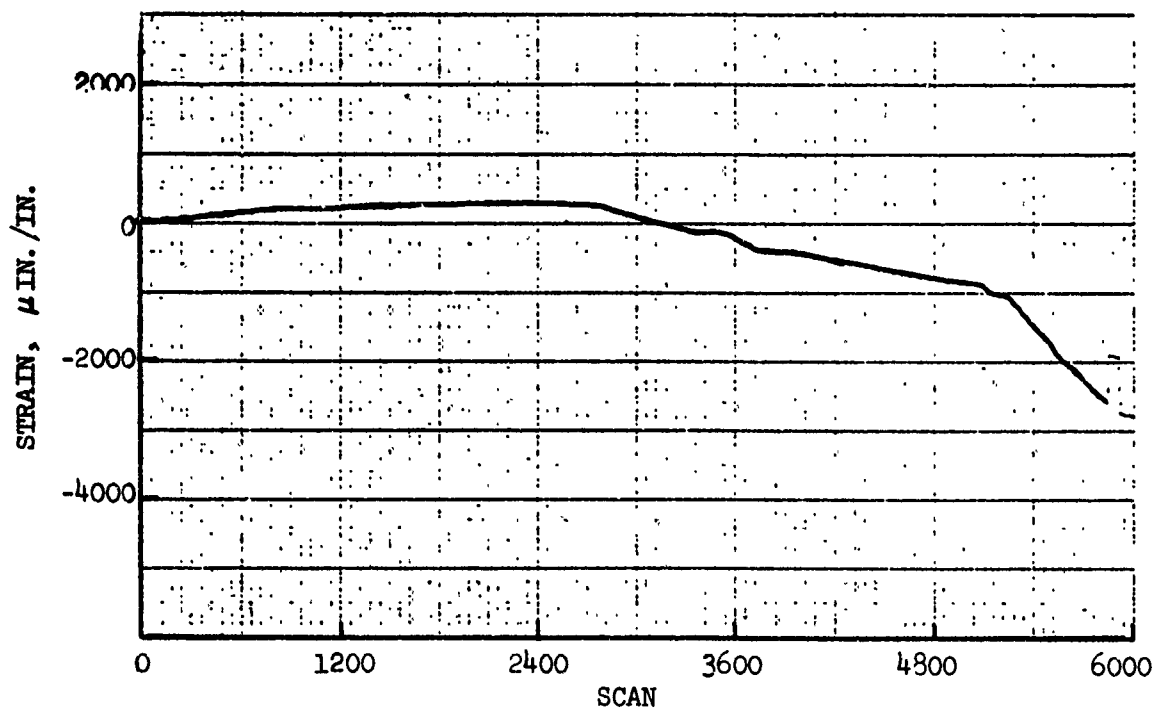


Figure 25. Strain Gage 2 Versus Scan, Test Specimen 3.

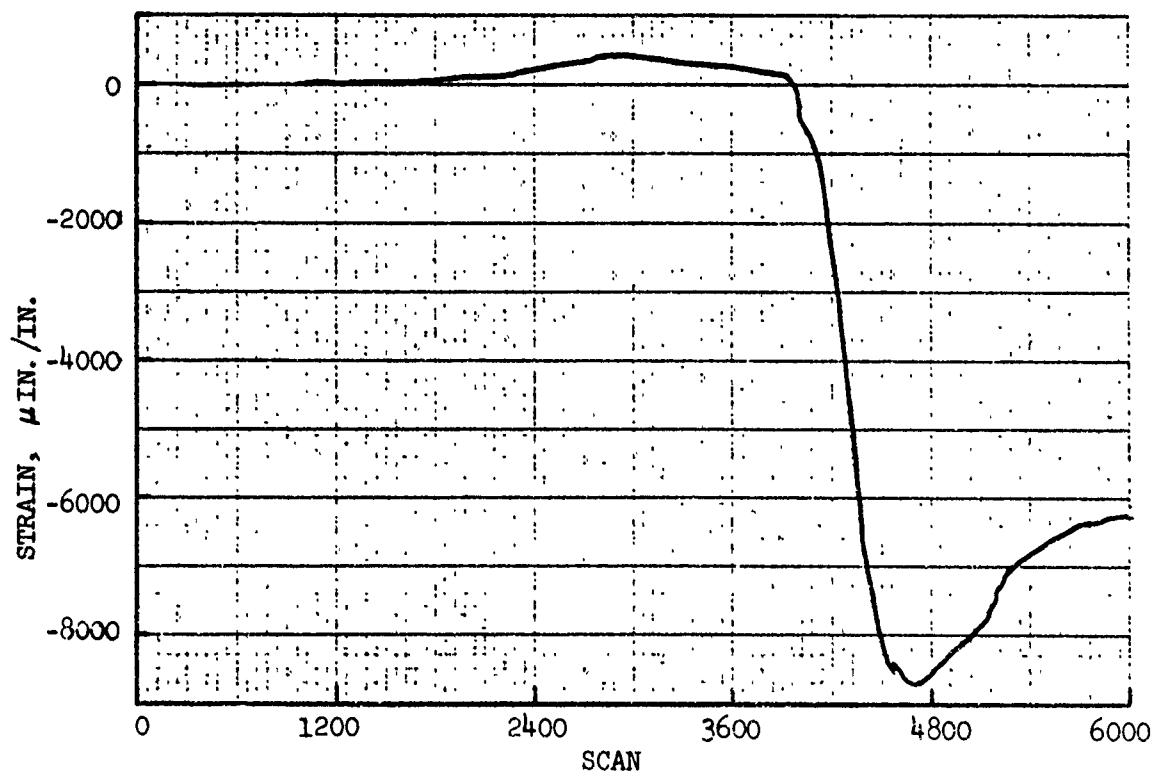


Figure 26. Strain Gage 3 Versus Scan, Test Specimen 3.

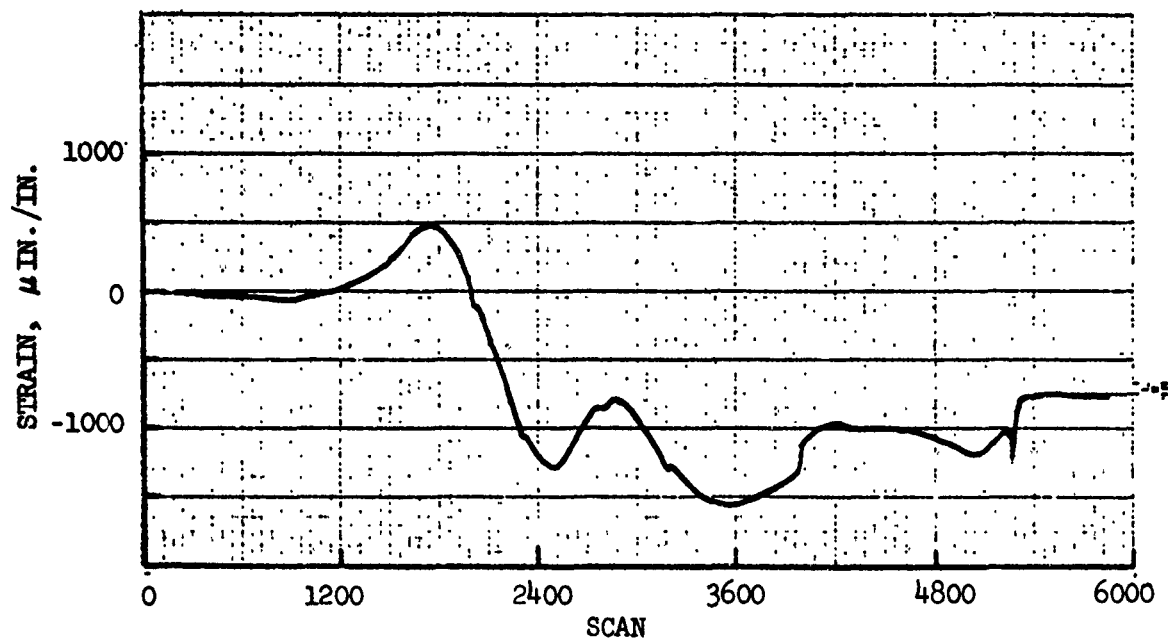


Figure 27. Strain Gage 4 Versus Scan, Test Specimen 3.

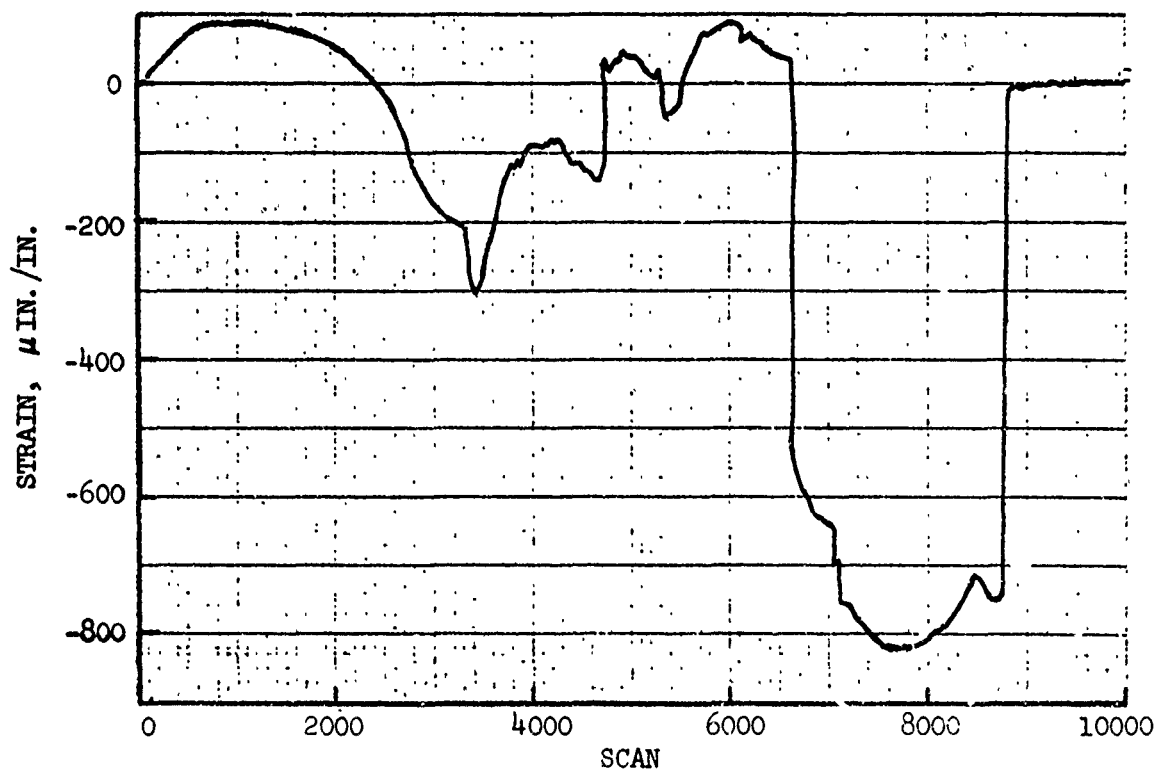


Figure 28. Strain Gage 5 Versus Scan, Test Specimen 3.

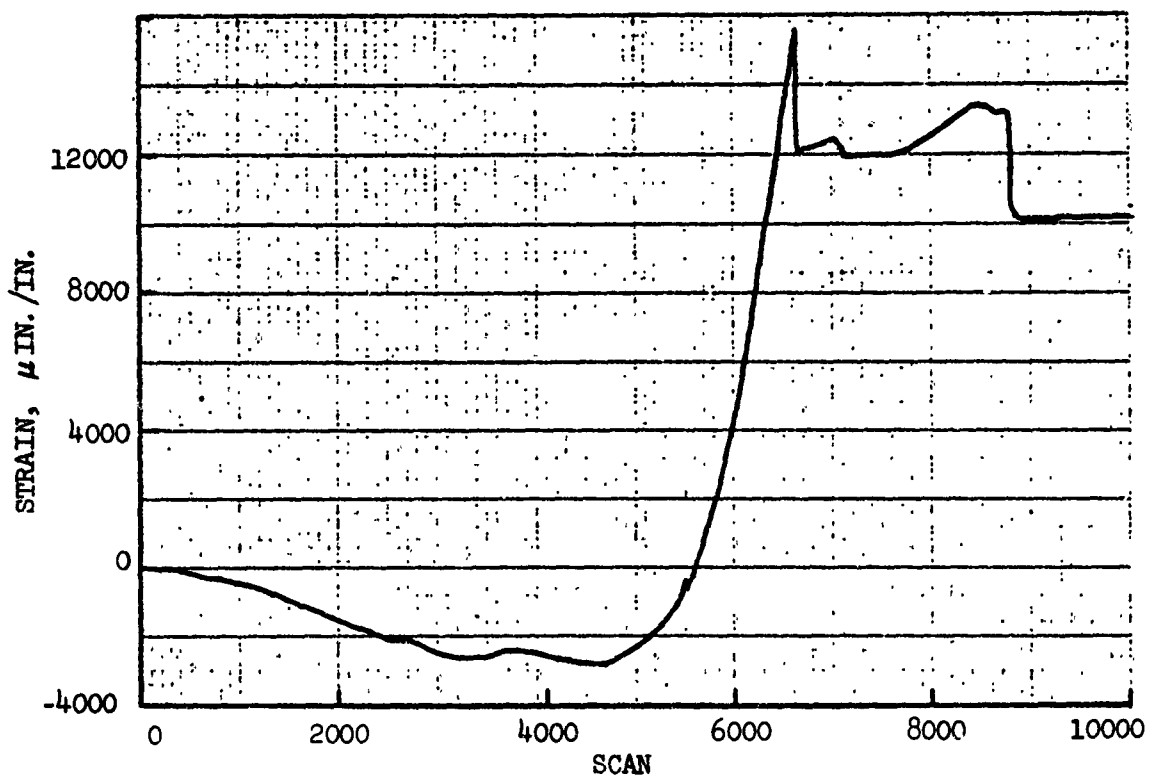


Figure 29. Strain Gage 6 Versus Scan, Test Specimen 3.

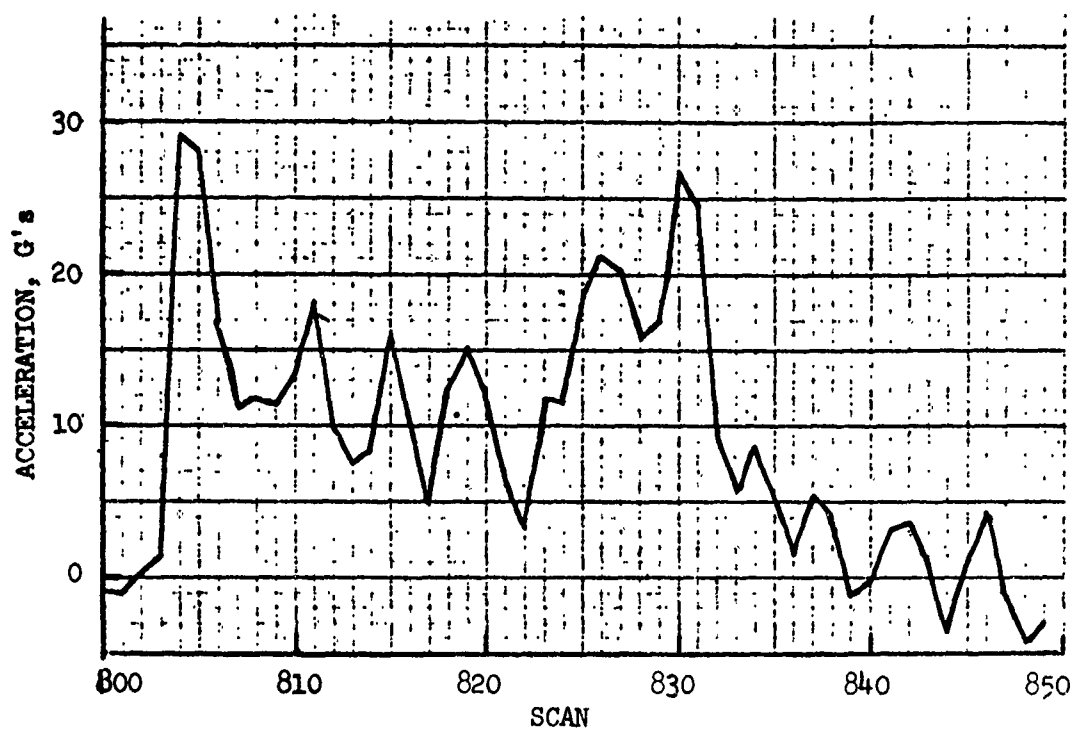


Figure 30. Acceleration Versus Scan, Test Specimen 4.

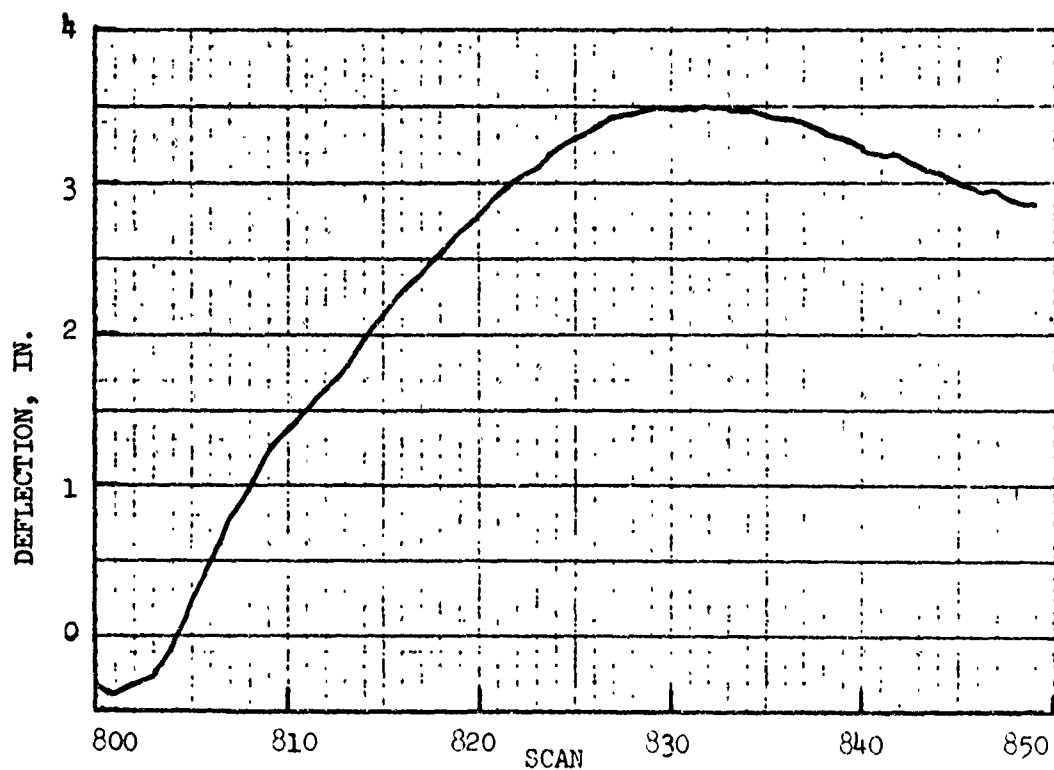


Figure 31. Deflection Versus Scan, Test Specimen 4.

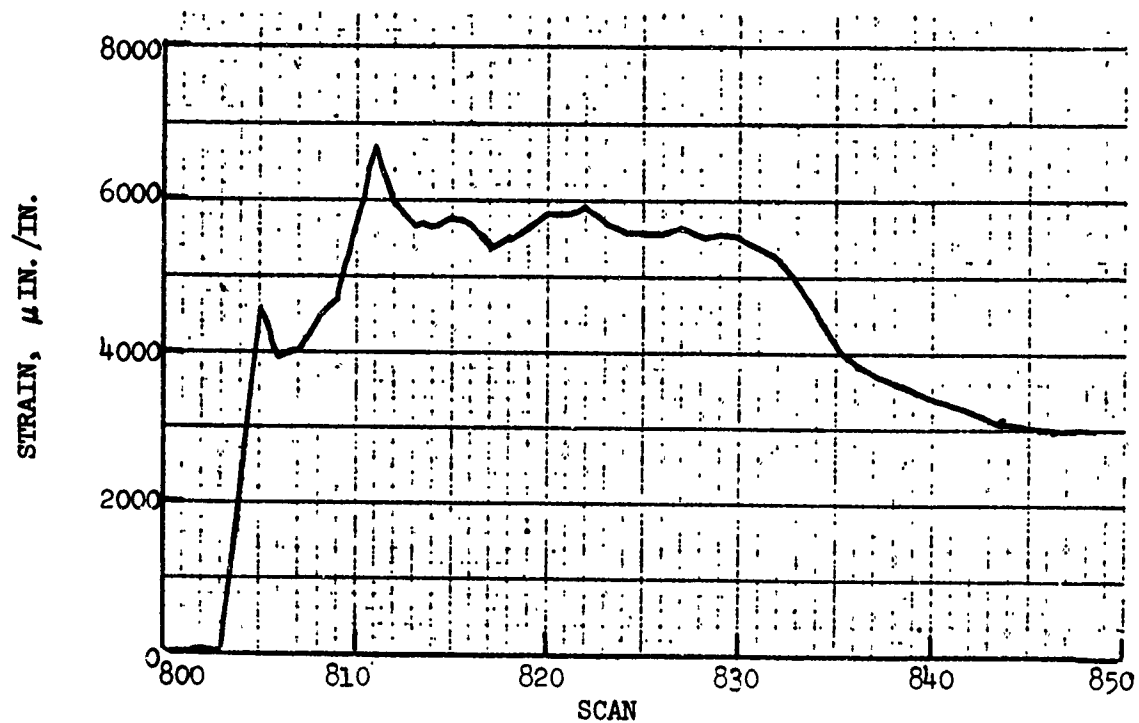
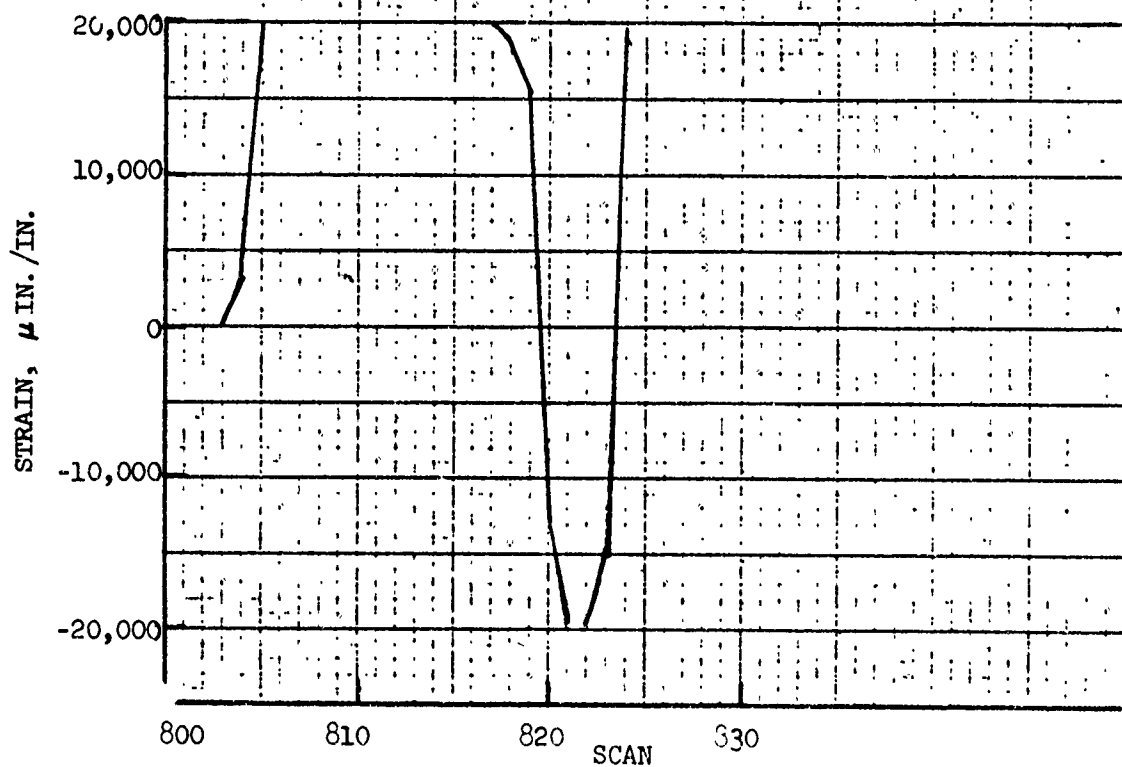


Figure 32. Strain Gage 1A Versus Scan, Test Specimen 4.



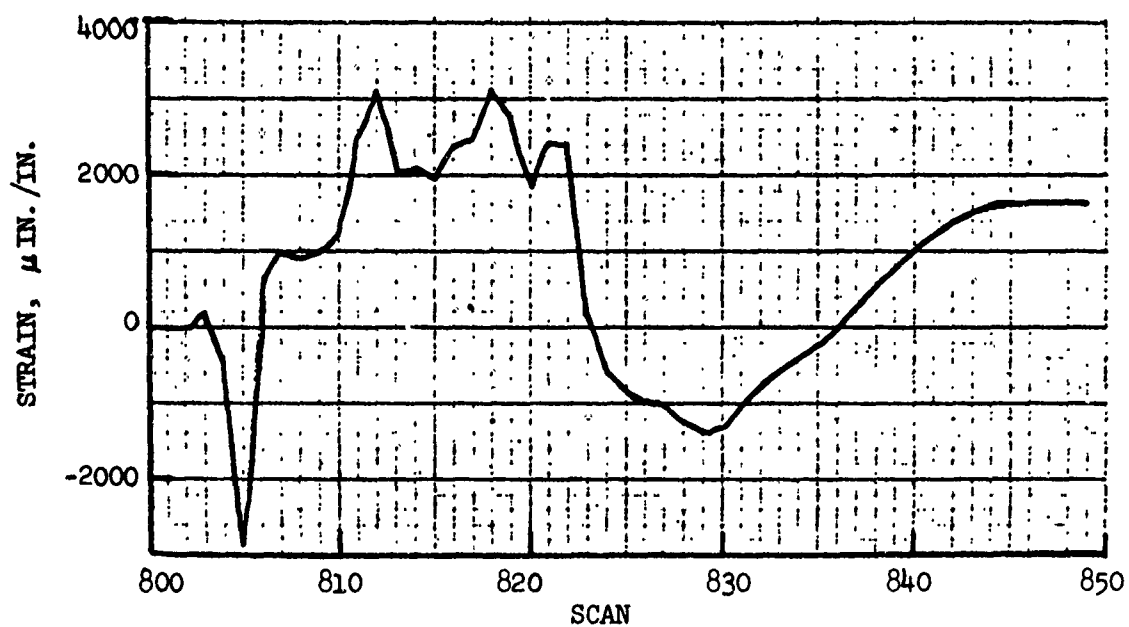


Figure 34. Strain Gage 2 Versus Scan, Test Specimen 4.

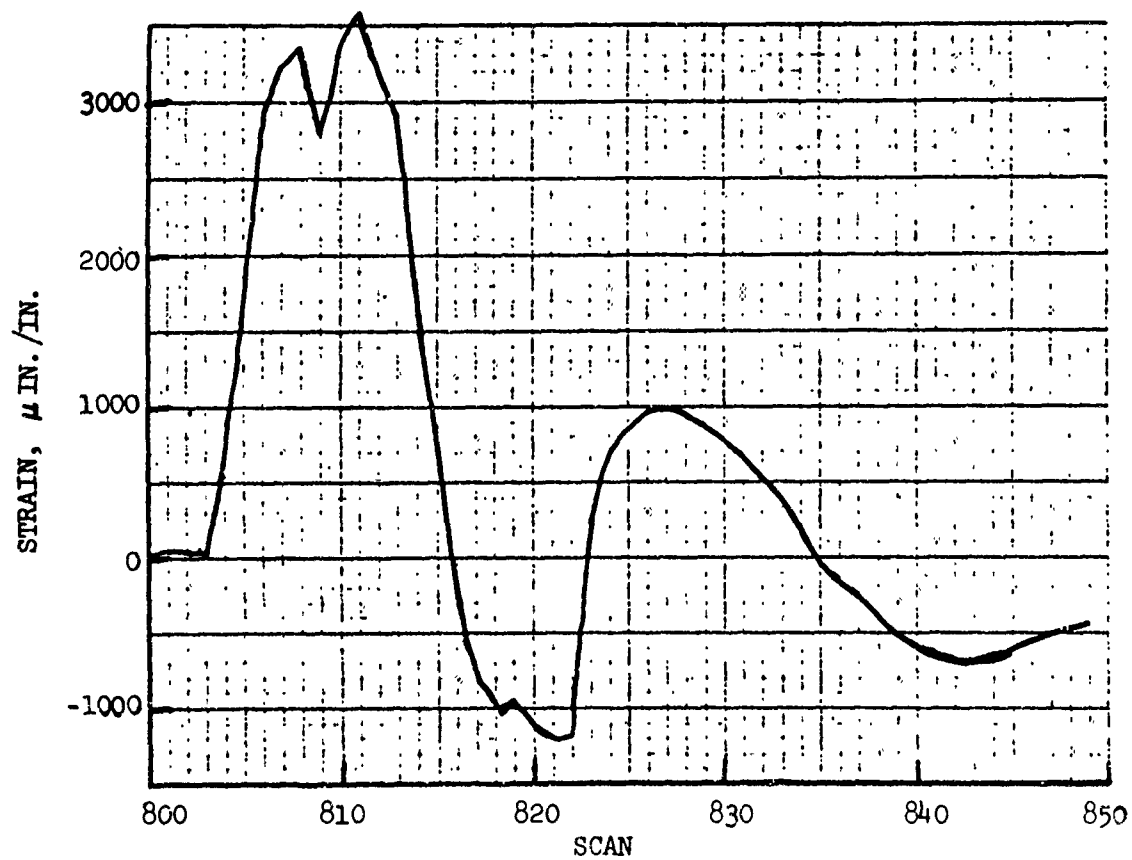


Figure 35. Strain Gage 3 Versus Scan, Test Specimen 4.

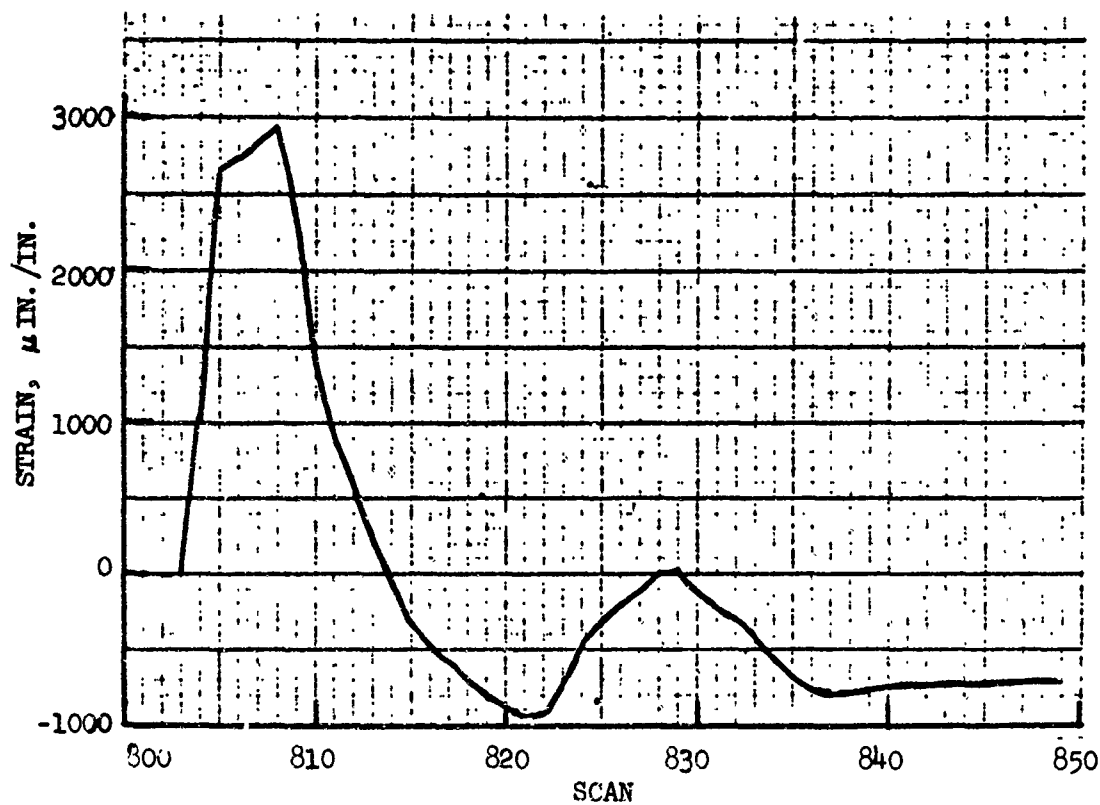


Figure 36. Strain Gage 4 Versus Scan, Test Specimen 4.

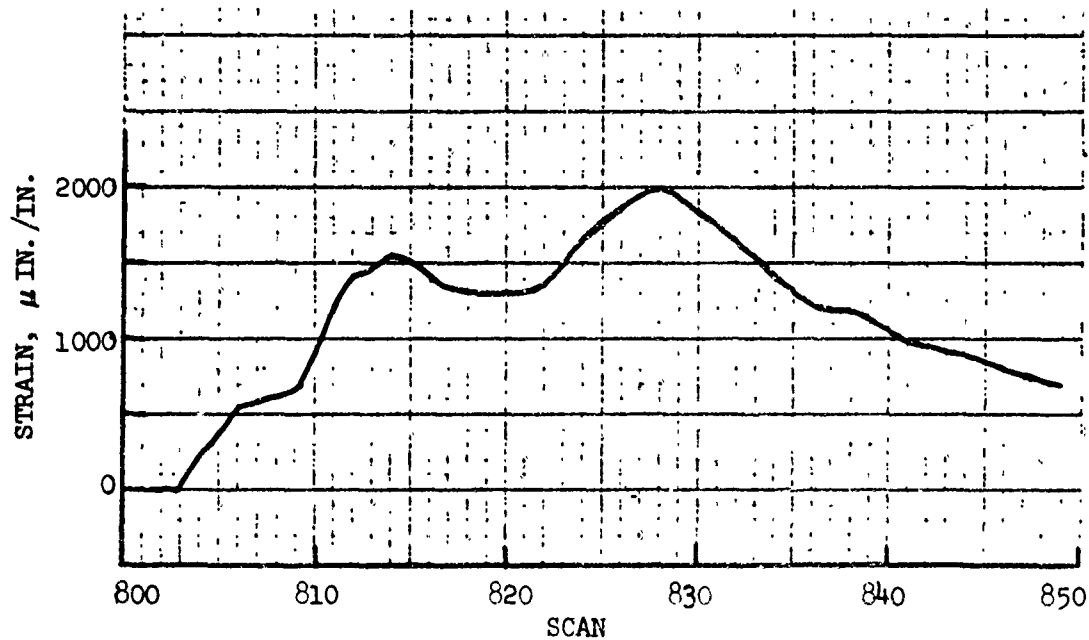


Figure 37. Strain Gage 5 Versus Scan, Test Specimen 4.

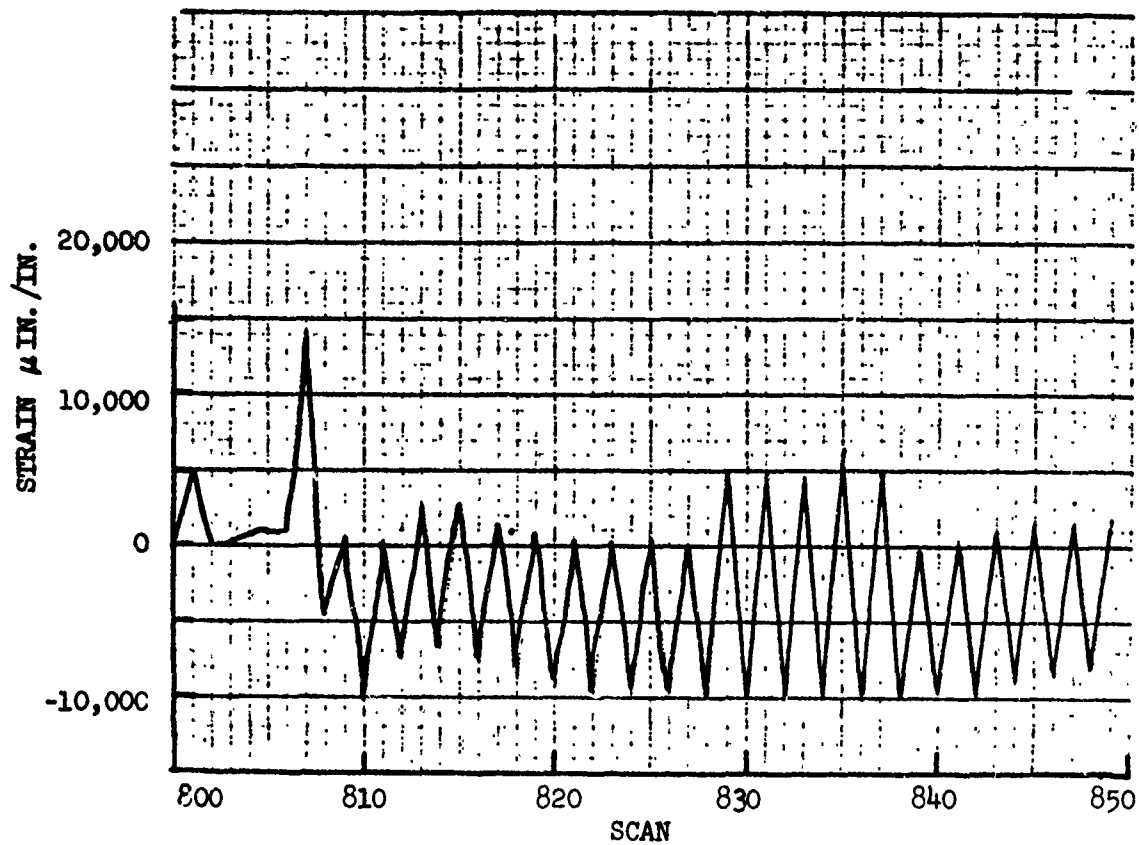


Figure 38. Strain Gage 6 Versus Scan, Test Specimen 4.

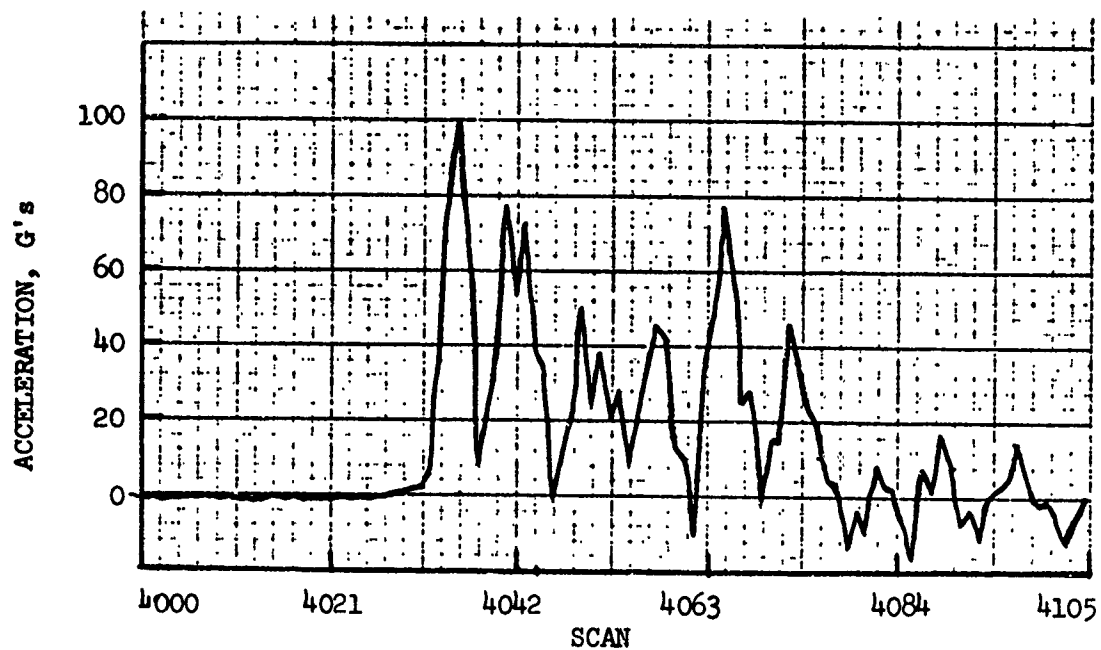


Figure 39. Acceleration Versus Scan, Test Specimen 5.

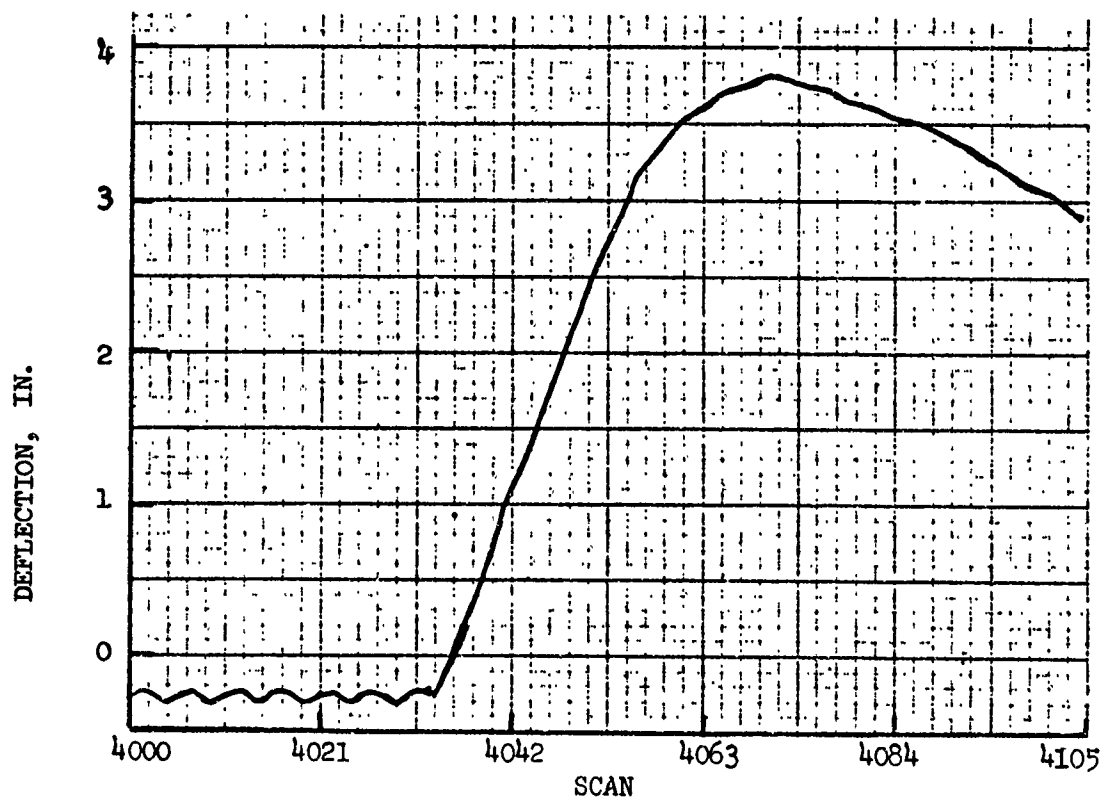


Figure 40. Deflection Versus Scan, Test Specimen 5.

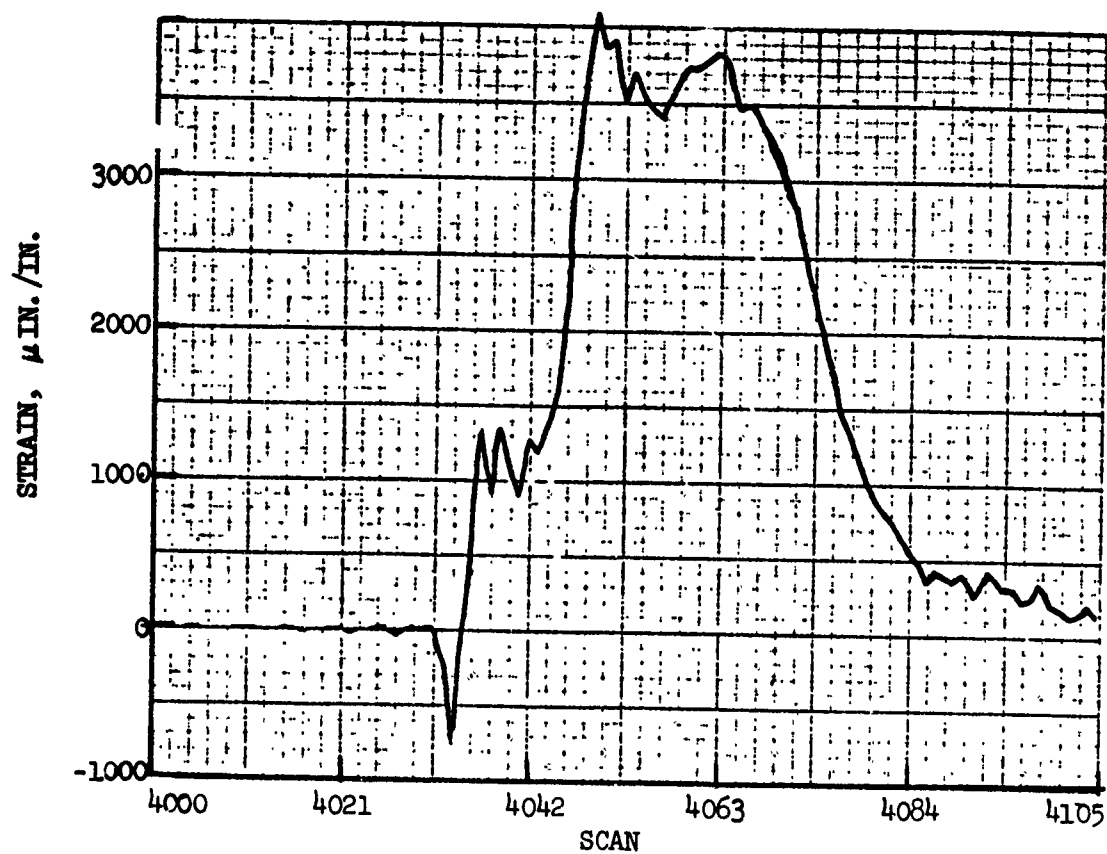


Figure 41. Strain Gage 1A Versus Scan, Test Specimen 5.

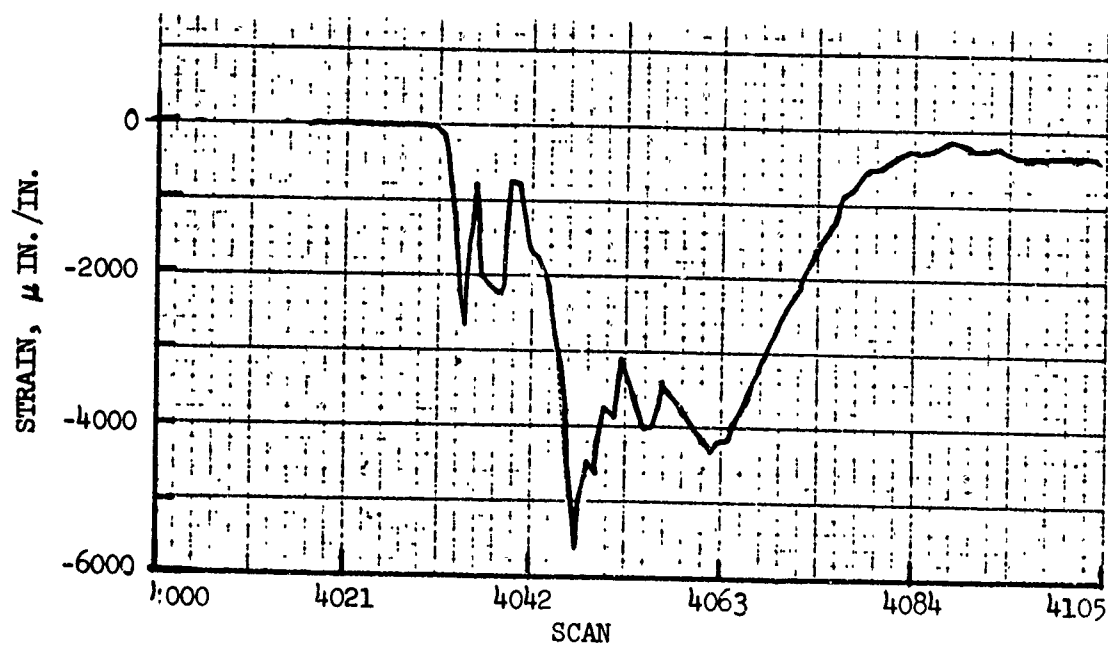


Figure 42. Strain Gage 1B Versus Scan, Test Specimen 5.

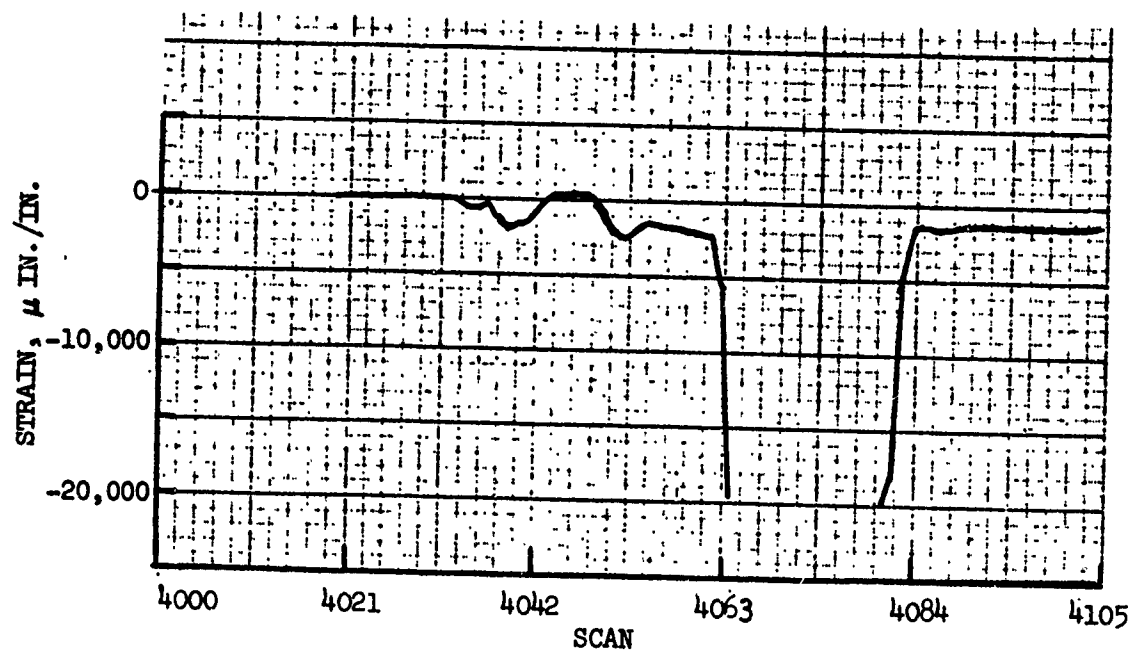


Figure 43. Strain Gage 2 Versus Scan, Test Specimen 5.

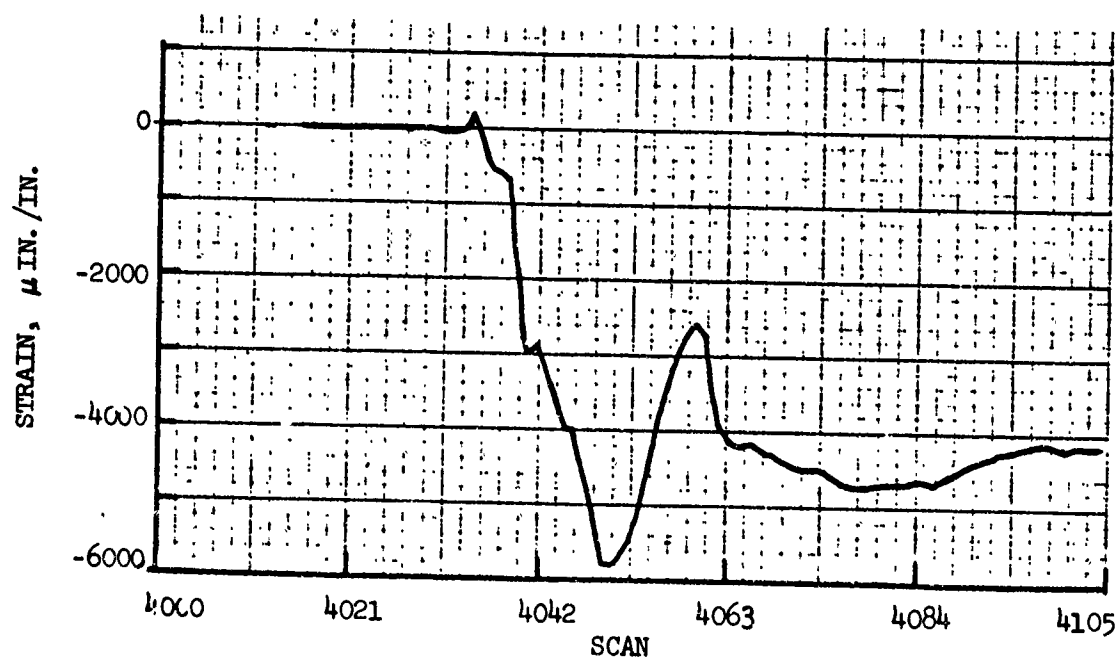


Figure 44. Strain Gage 4 Versus Scan, Test Specimen 5.

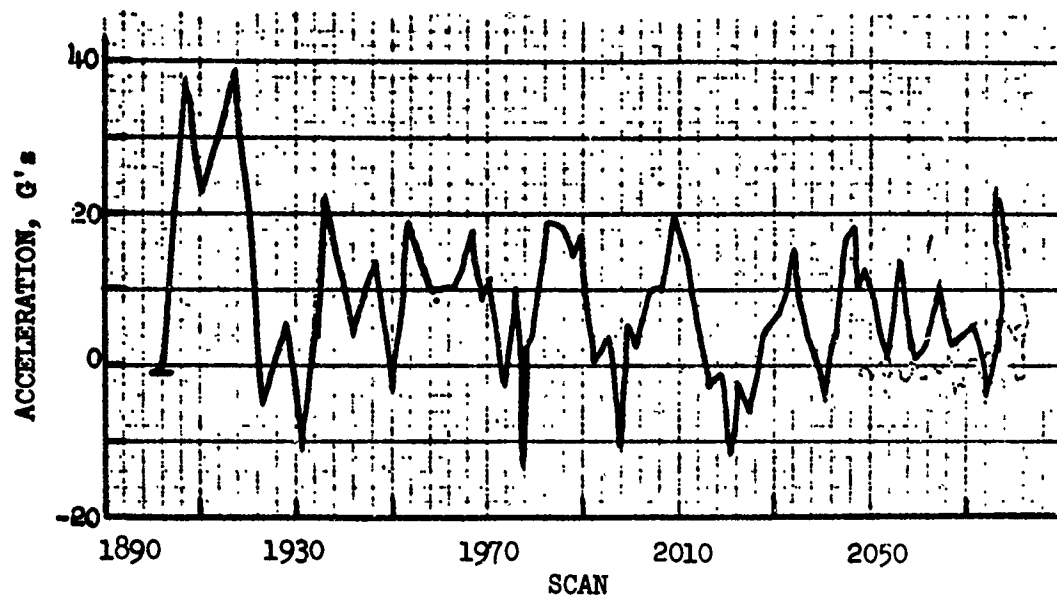


Figure 45. Acceleration Versus Scan, Test Specimen 6.

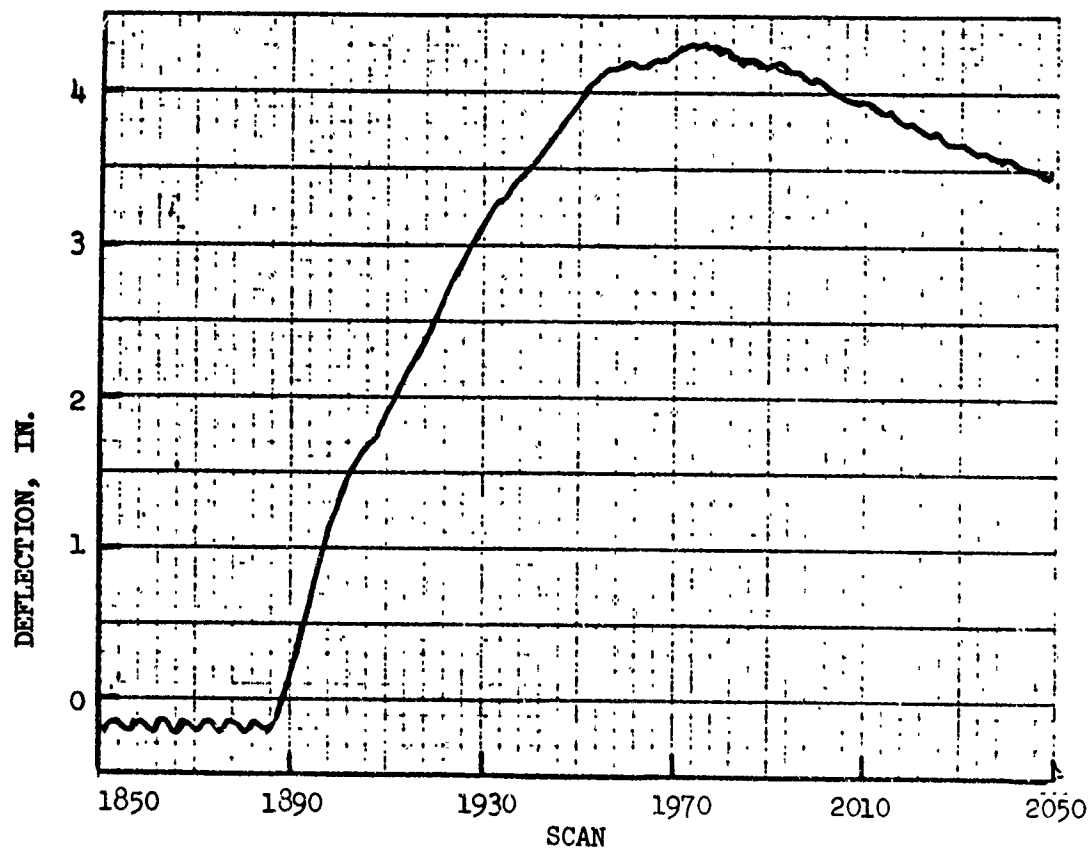


Figure 46. Deflection Versus Scan, Test Specimen 6.

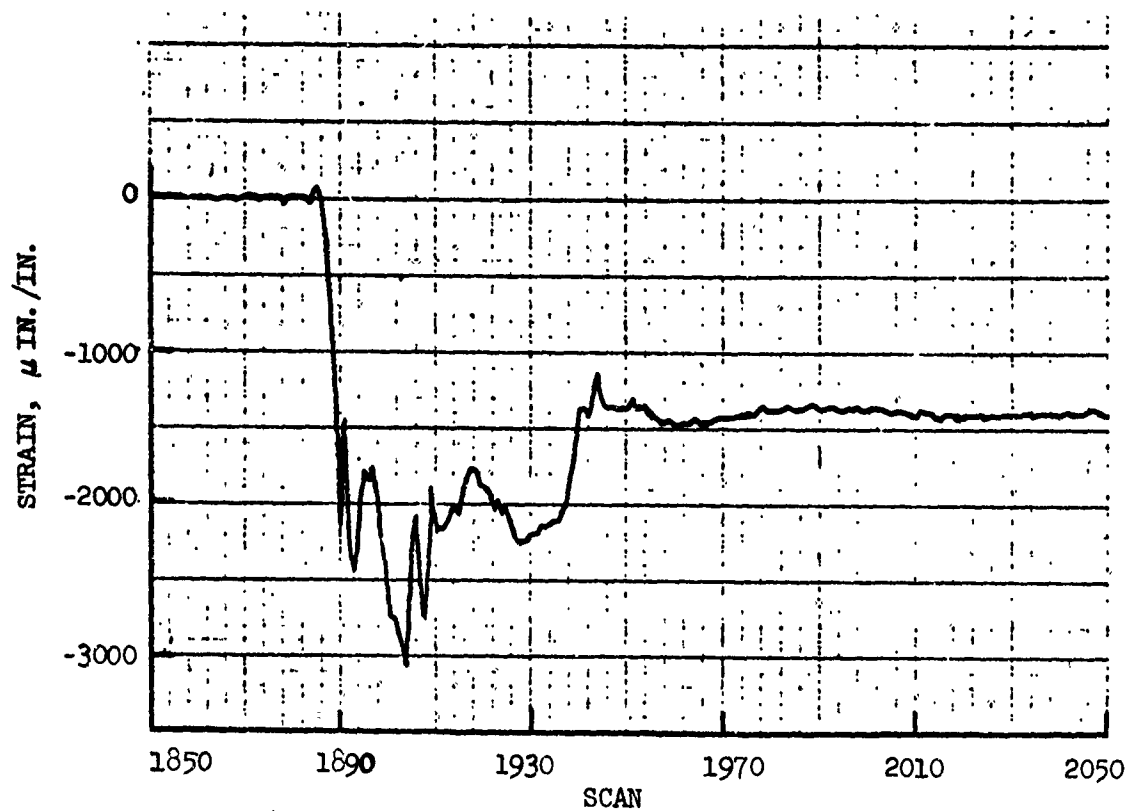


Figure 47. Strain Gage 1A Versus Scan, Test Specimen 6.

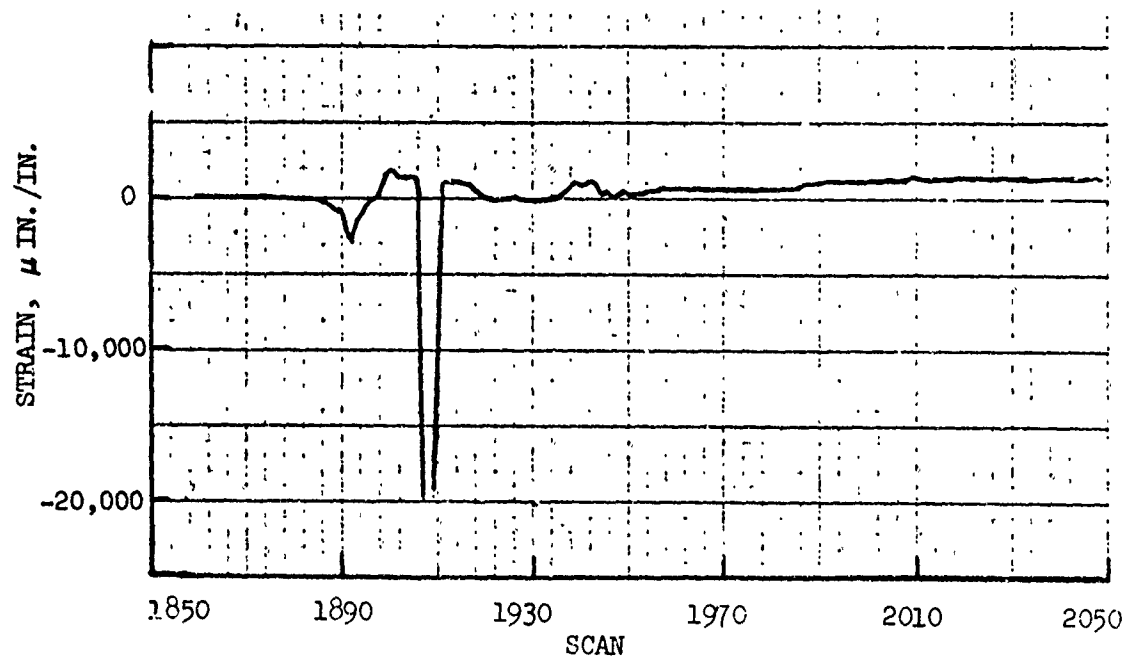


Figure 48. Strain Gage 1B Versus Scan, Test Specimen 6.

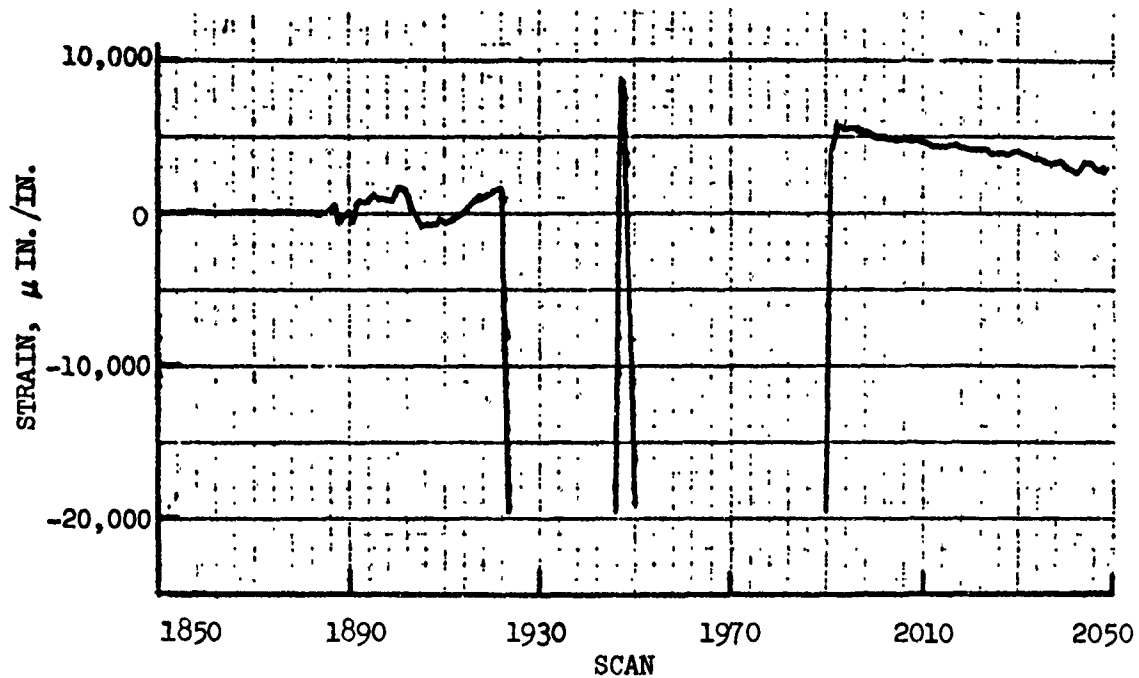


Figure 49. Strain Gage 2 Versus Scan, Test Specimen 6.

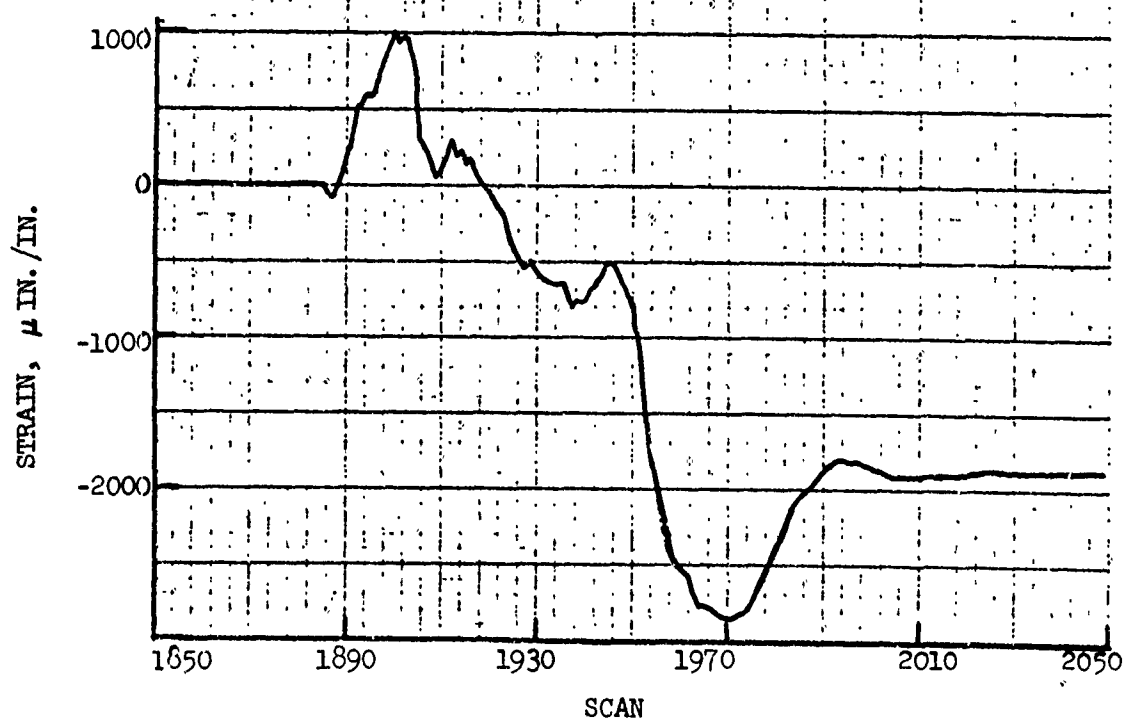


Figure 50. Strain Gage 6 Versus Scan, Test Specimen 6.

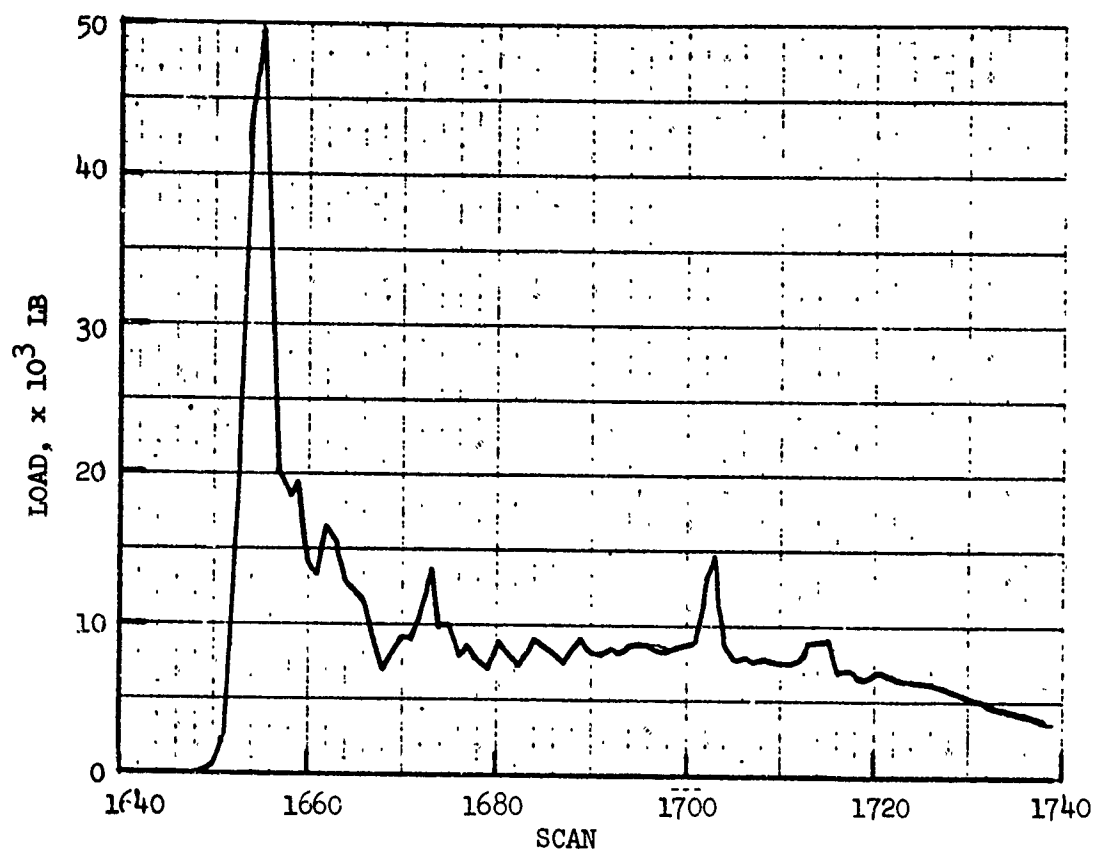


Figure 51. Load Cell (North) Versus Scan, Test Specimen 7.

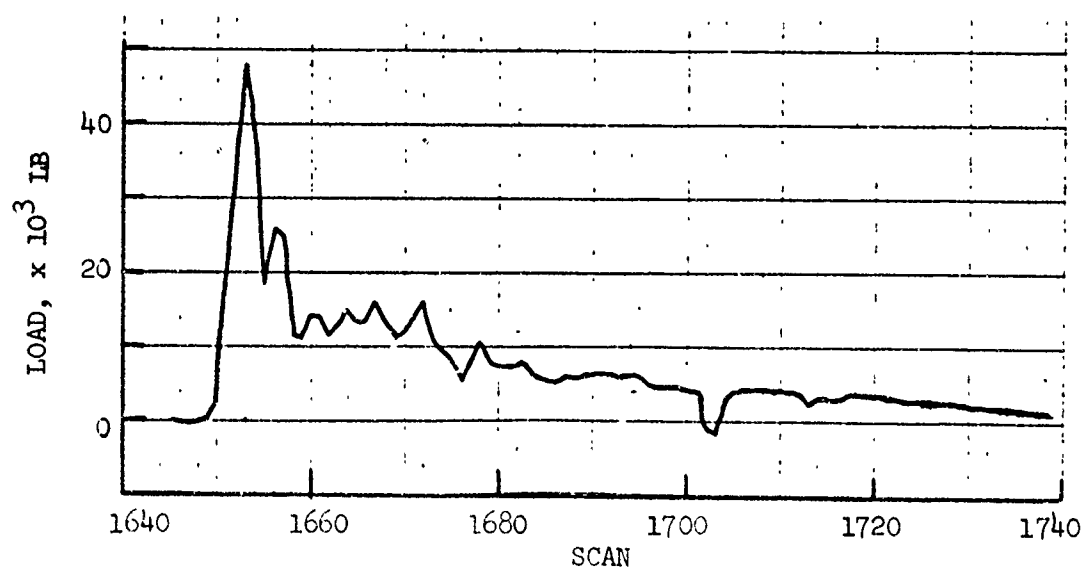


Figure 52. Load Cell (South) Versus Scan, Test Specimen 7.

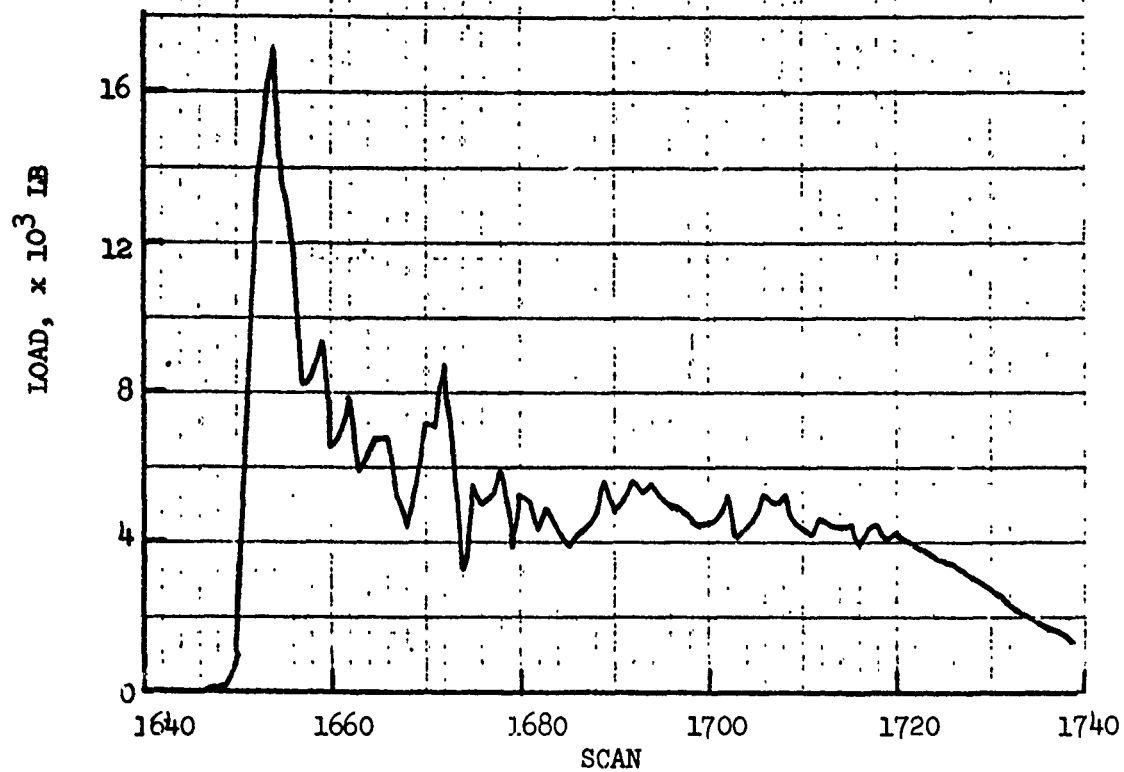


Figure 53. Load Cell (East) Versus Scan, Test Specimen 7.

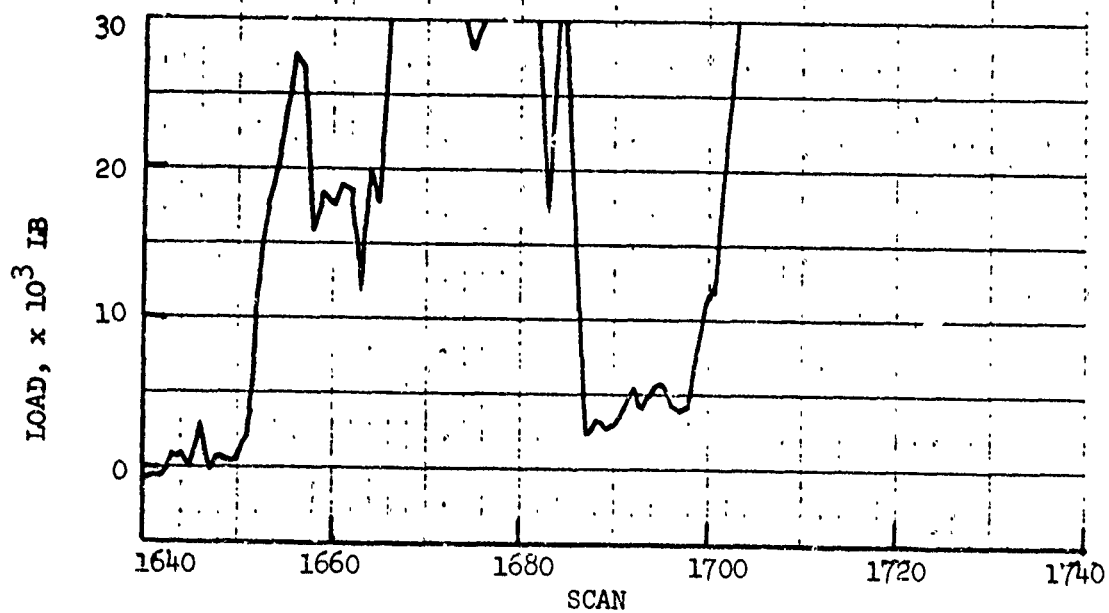


Figure 54. Load Cell (West) Versus Scan, Test Specimen 7.

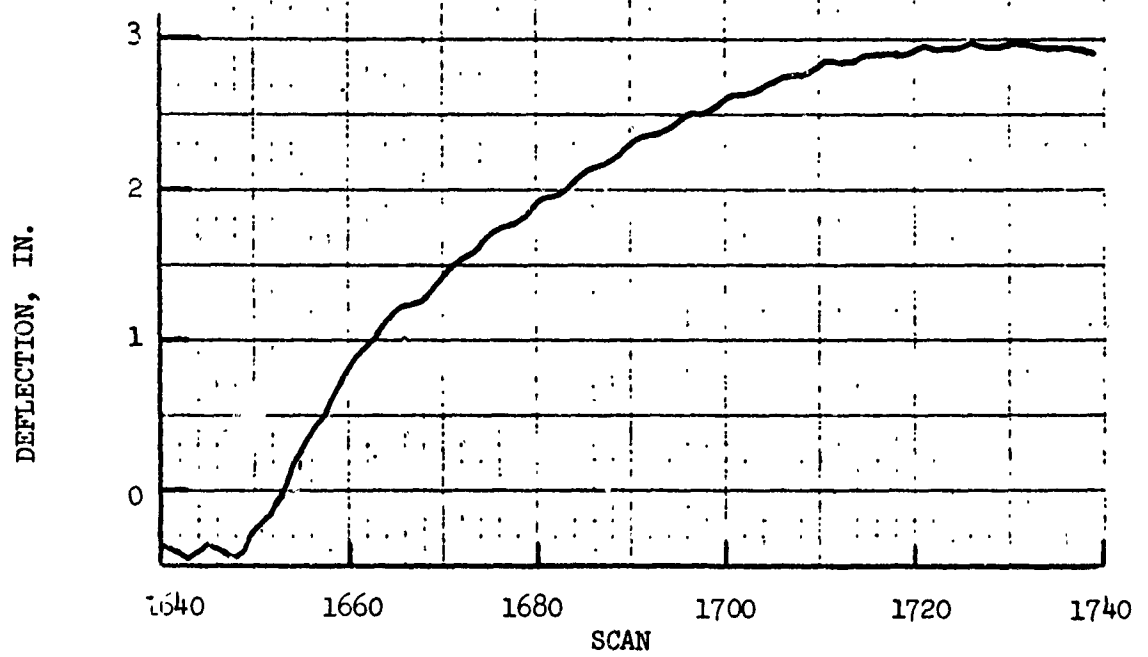


Figure 55. Deflection Versus Scan, Test Specimen 7.

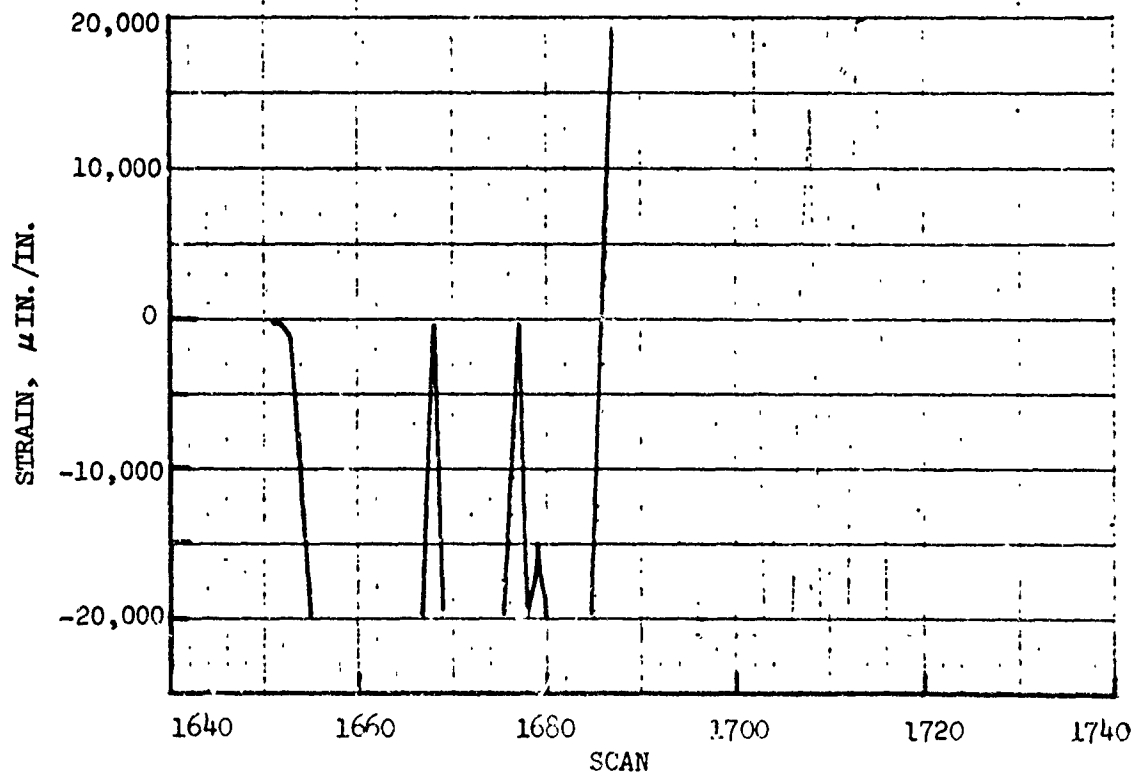


Figure 56. Strain Gage 1A Versus Scan, Test Specimen 7.

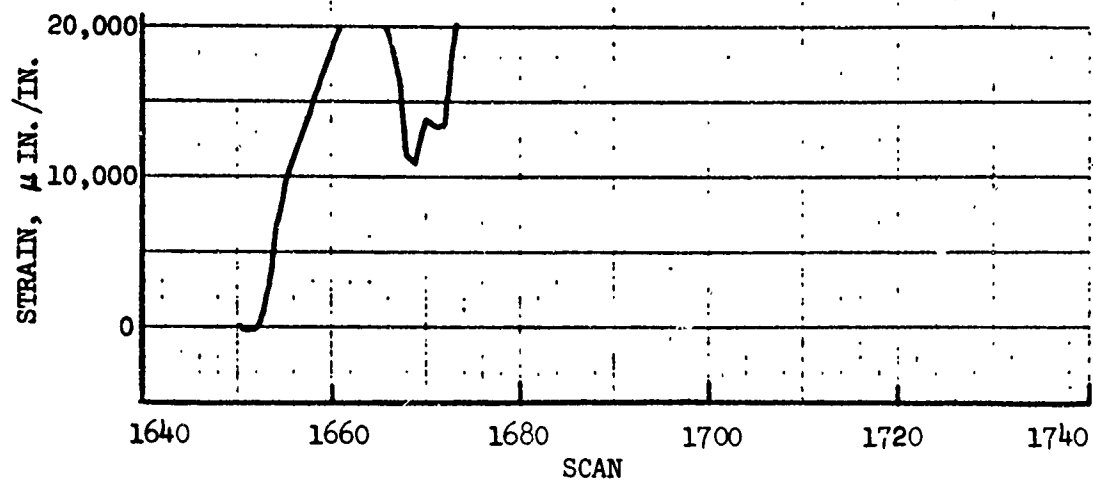


Figure 57. Strain Gage 1B Versus Scan, Test Specimen 7.

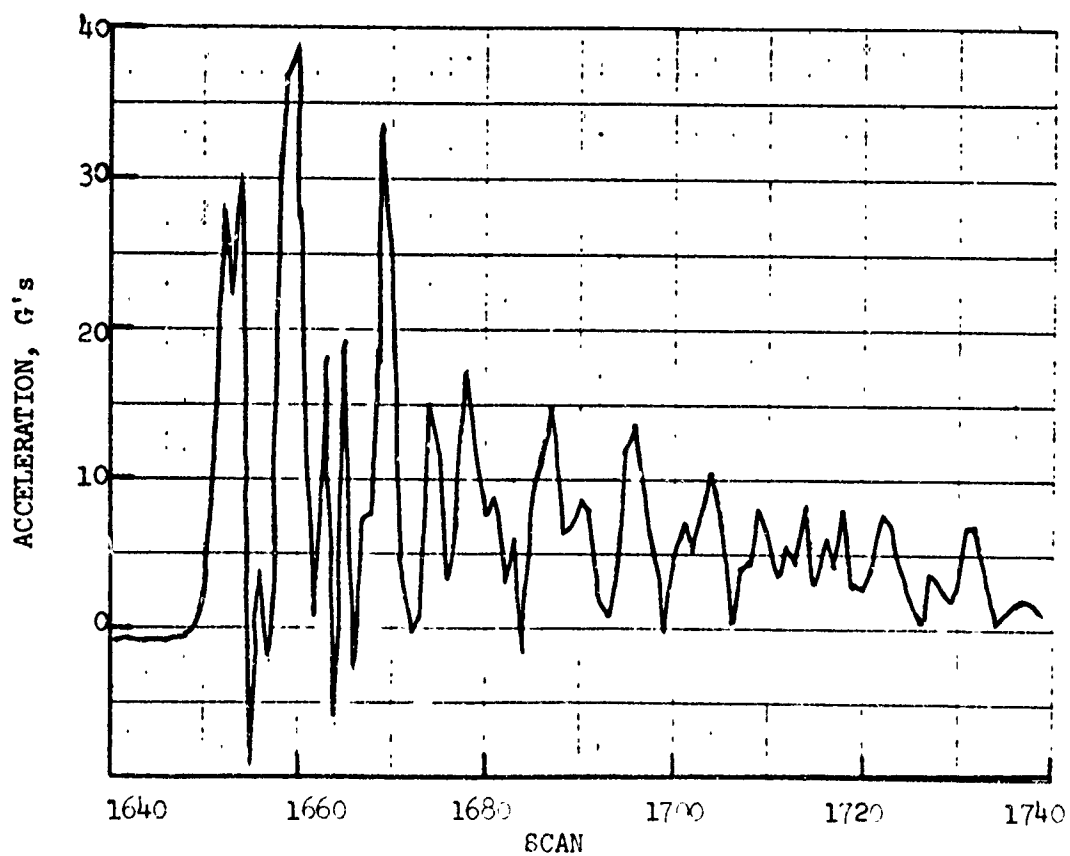


Figure 58. Acceleration (East) Versus Scan, Test Specimen 7.

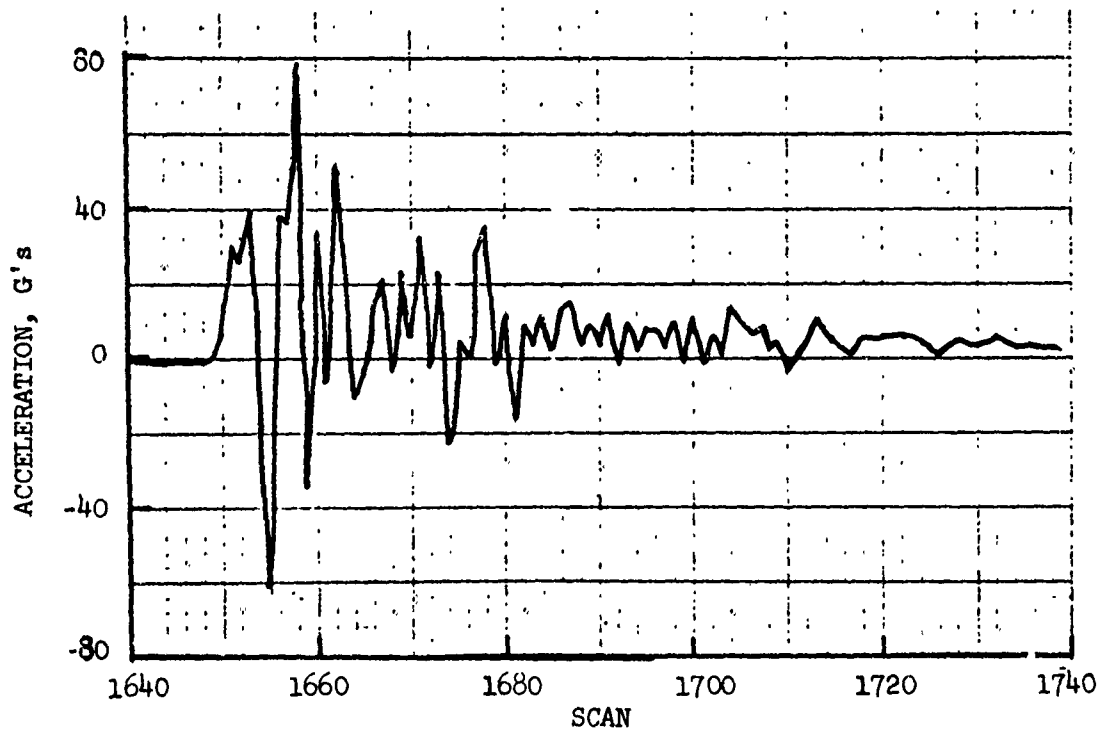


Figure 59. Acceleration (Center) Versus Scan, Test Specimen 7

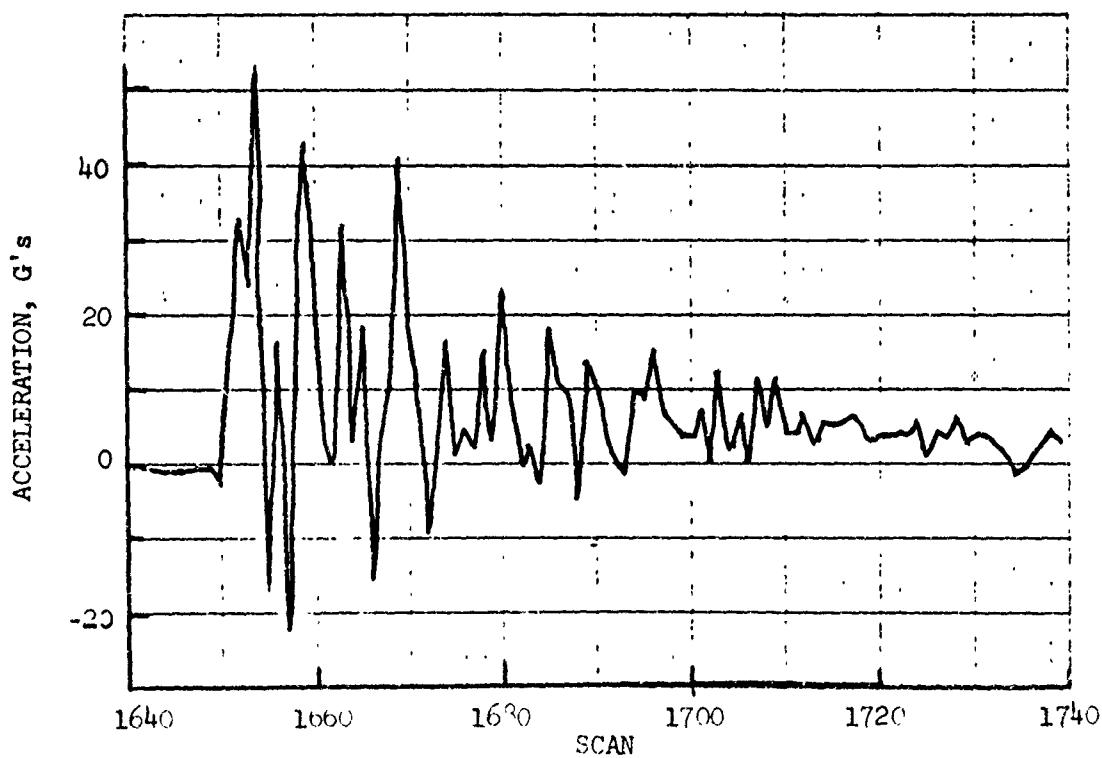


Figure 60. Acceleration (Left) Versus Scan, Test Specimen 7.

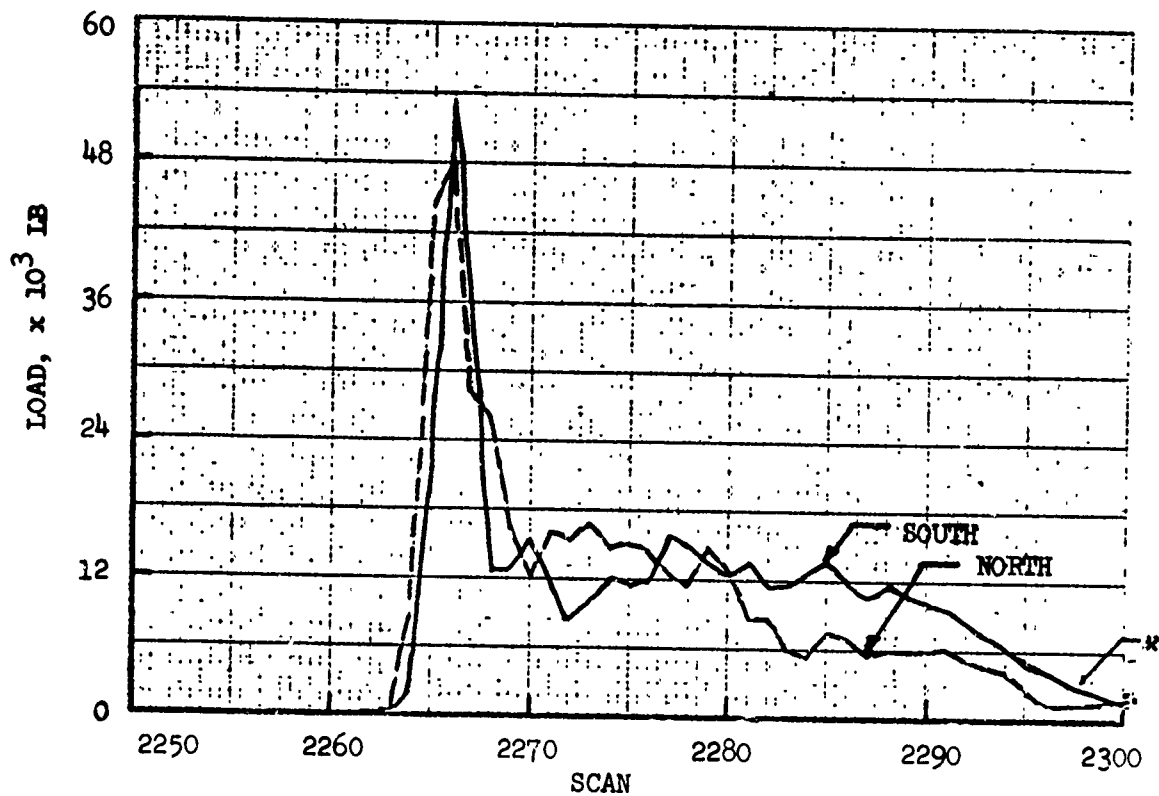


Figure 61. Load Cells (North and South) Versus Scan, Test Specimen 8.

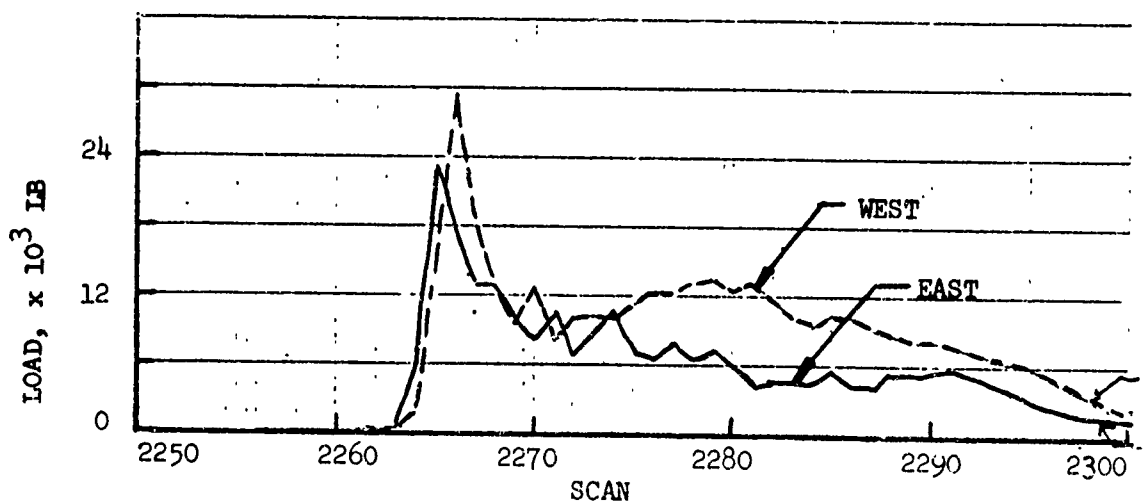


Figure 62. Load Cells (East and West) Versus Scan, Test Specimen 8.

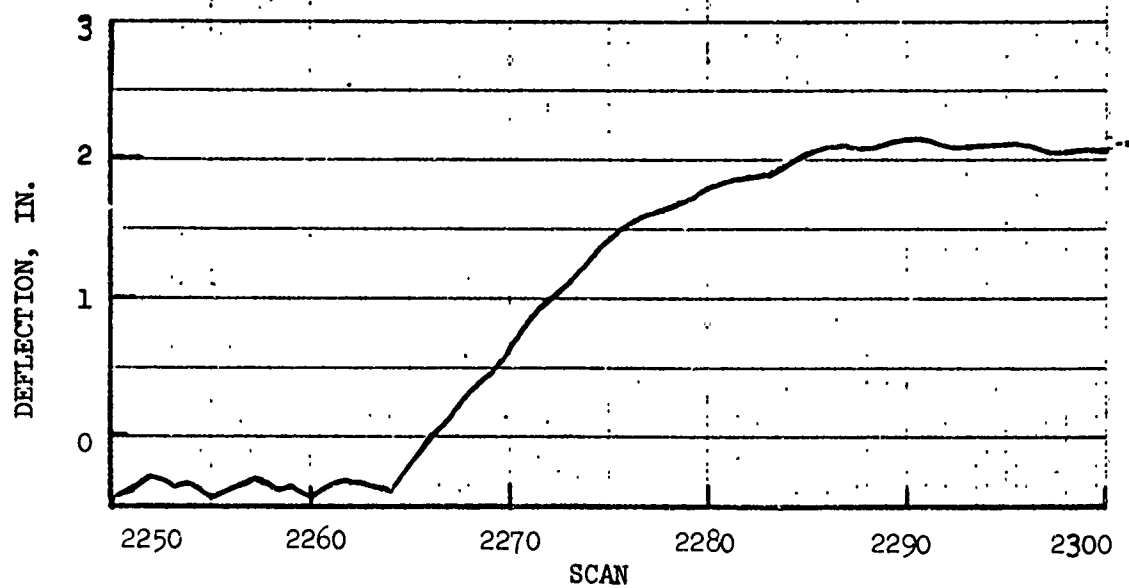


Figure 63. Deflection Versus Scan, Test Specimen 8.

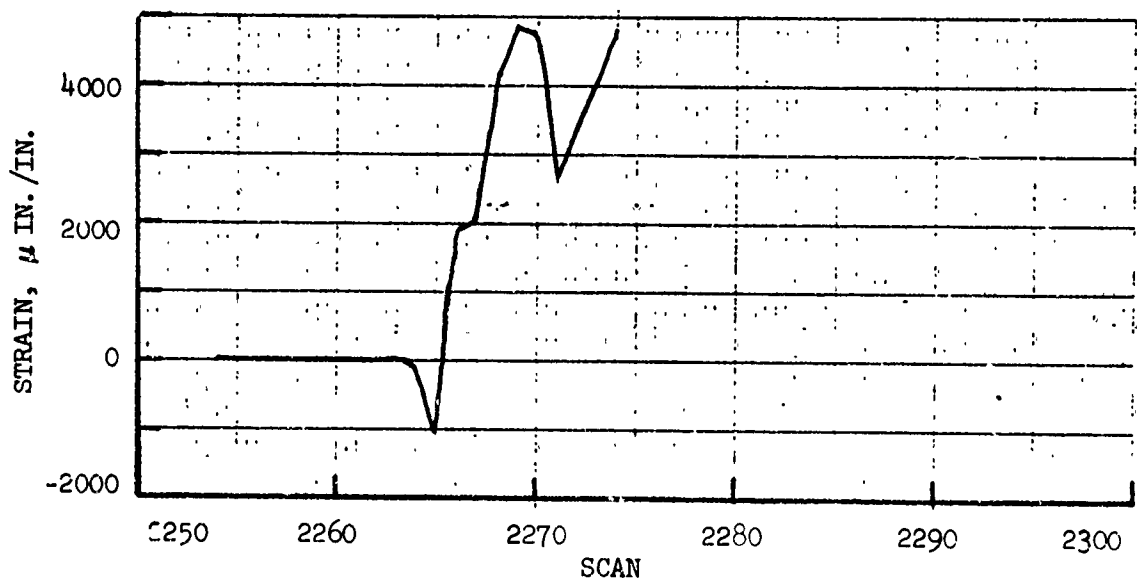


Figure 64. Strain Gage 1A Versus Scan, Test Specimen 8.

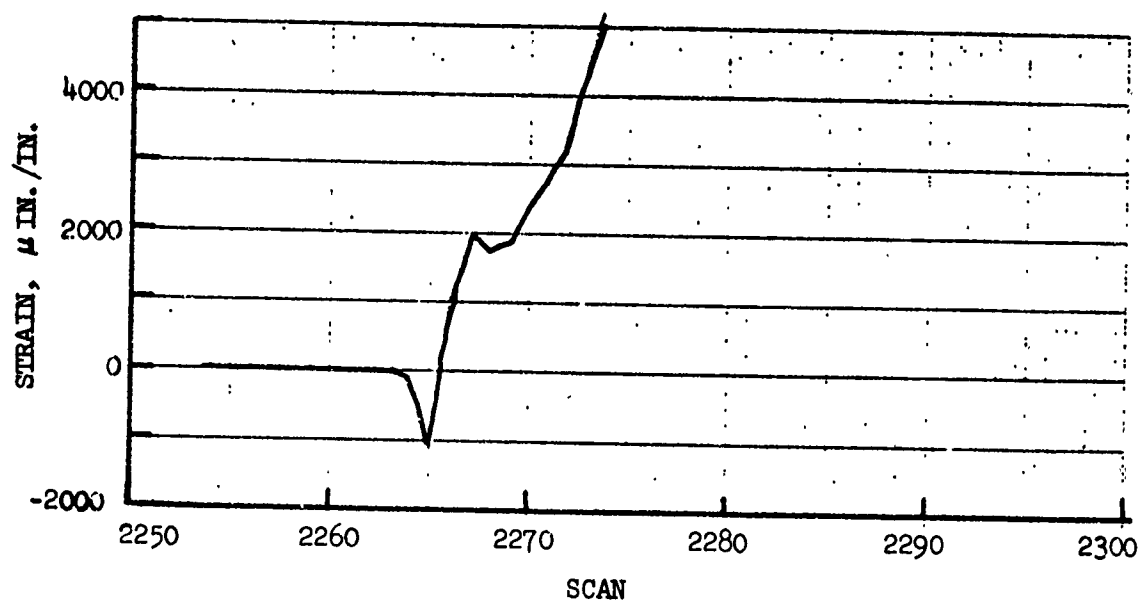


Figure 65. Strain Gage 1B Versus Scan, Test Specimen 8.

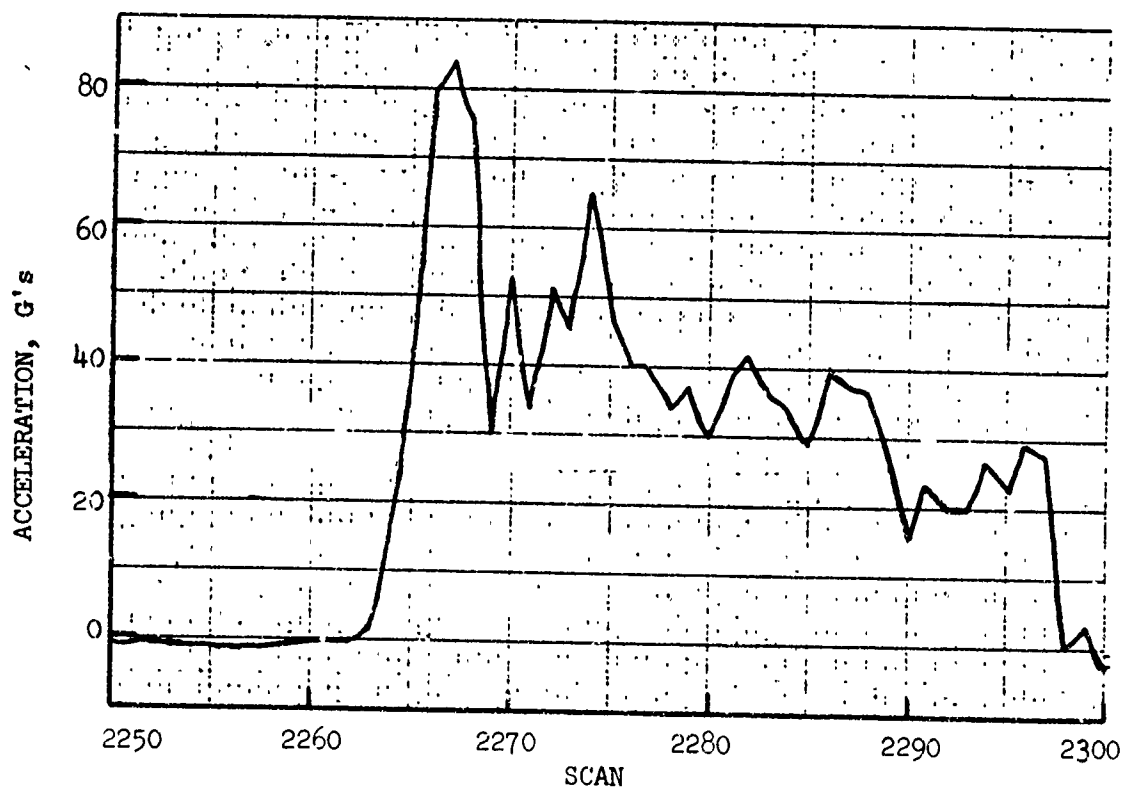


Figure 66. Acceleration (East) Versus Scan, Test Specimen 8.

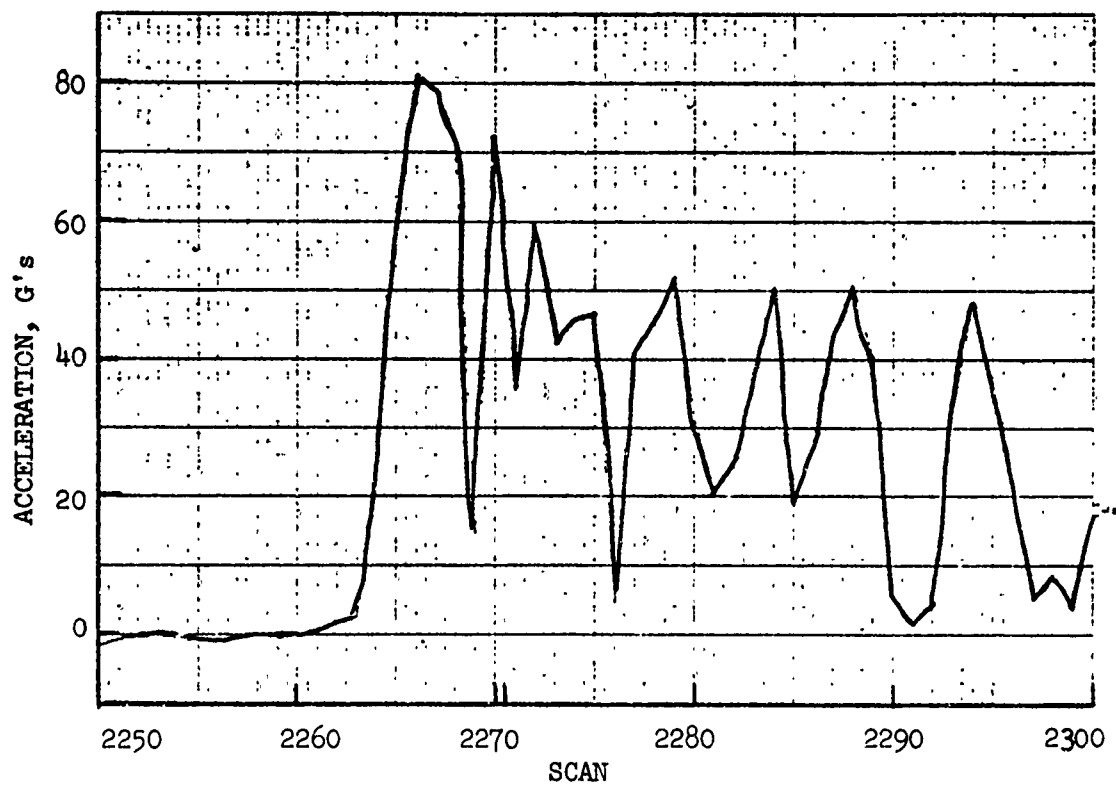


Figure 67. Acceleration (Center) Versus Scan, Test Specimen 8.

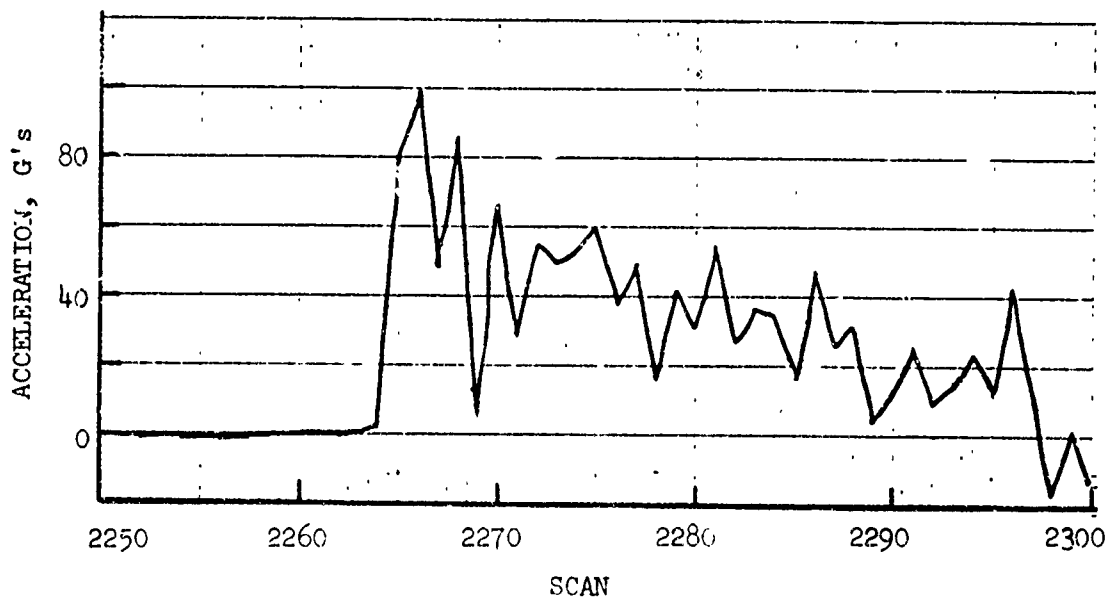


Figure 68. Acceleration (West) Versus Scan, Test Specimen 8.

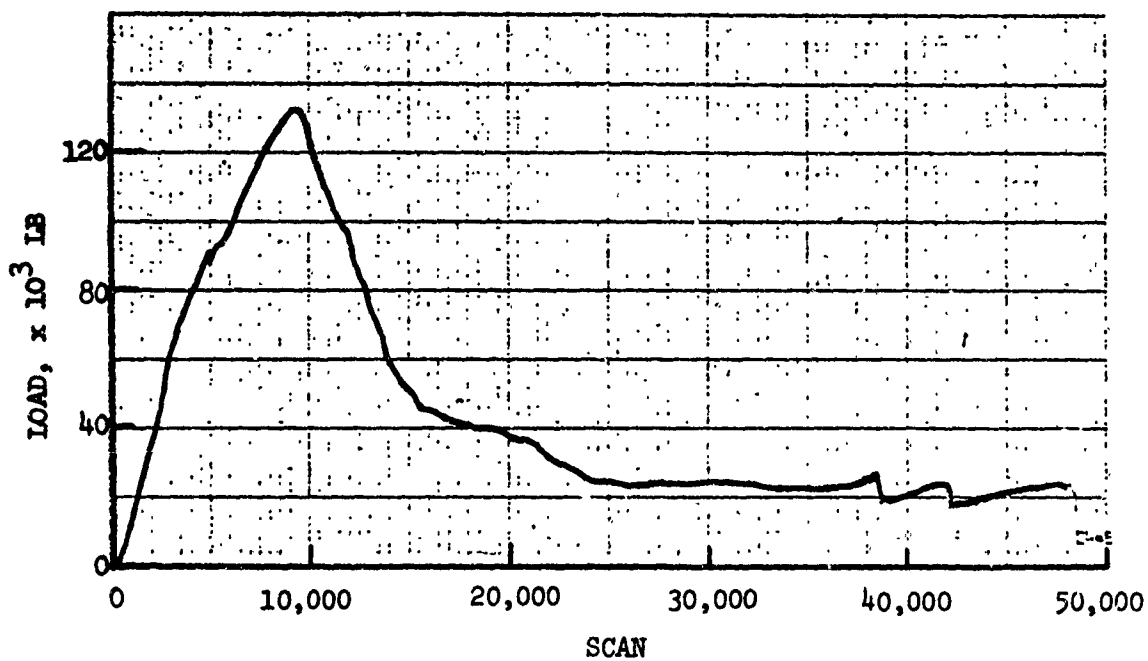


Figure 69. Load Versus Scan, Test Specimen 9.

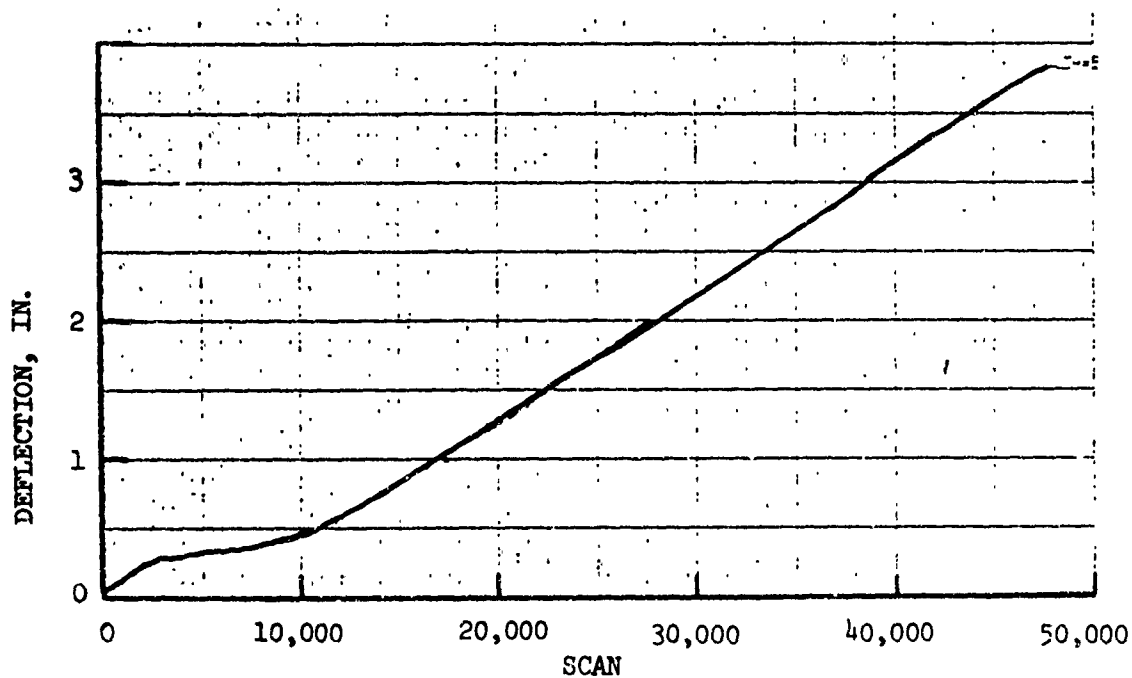


Figure 70. Deflection Versus Scan, Test Specimen 9.

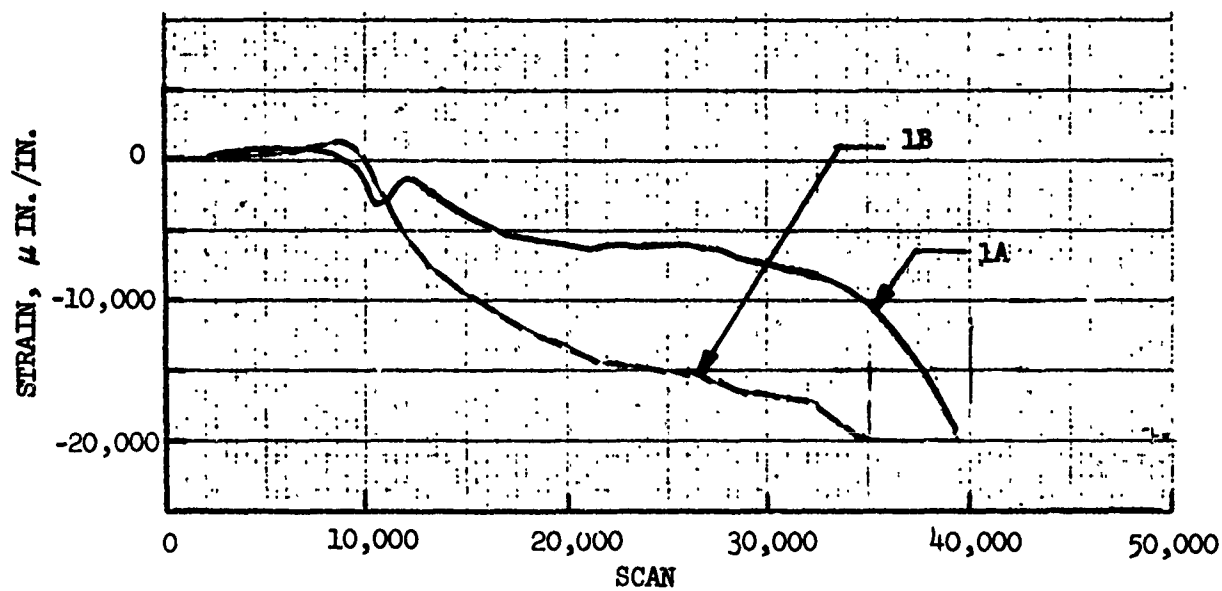


Figure 71. Strain Gages 1A and 1B Versus Scan, Test Specimen 9.

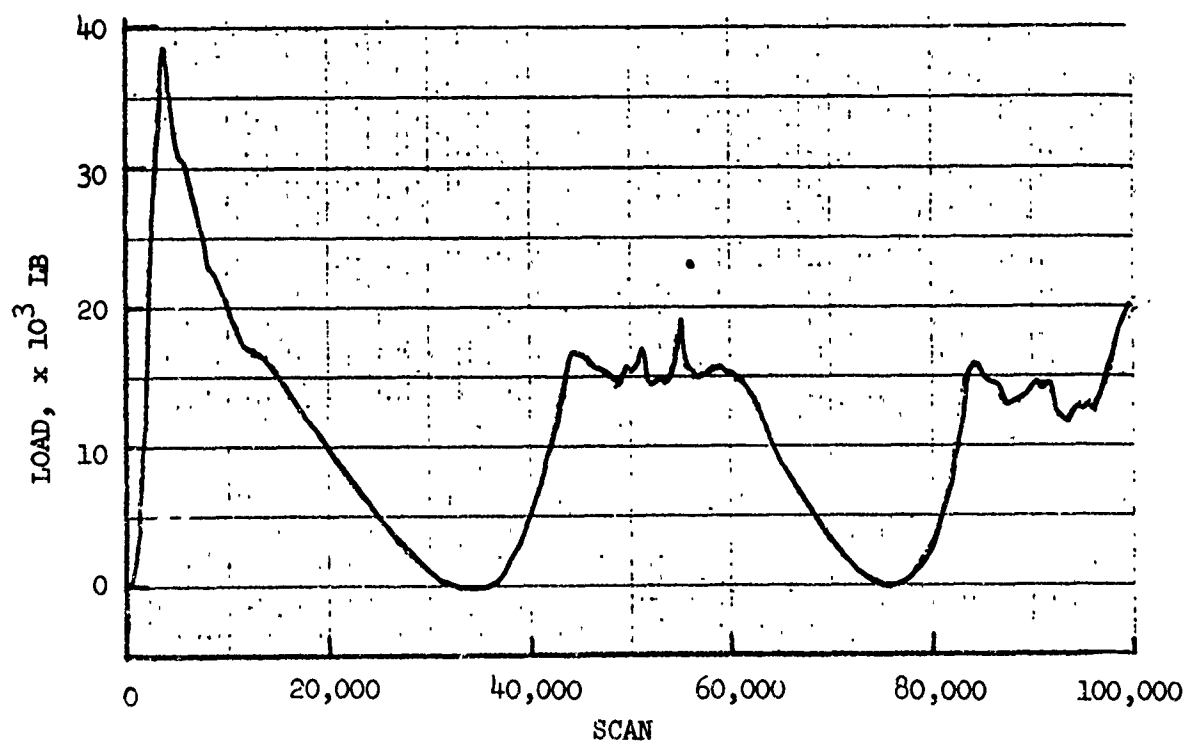


Figure 72. Load Versus Scan, Test Specimen 10.

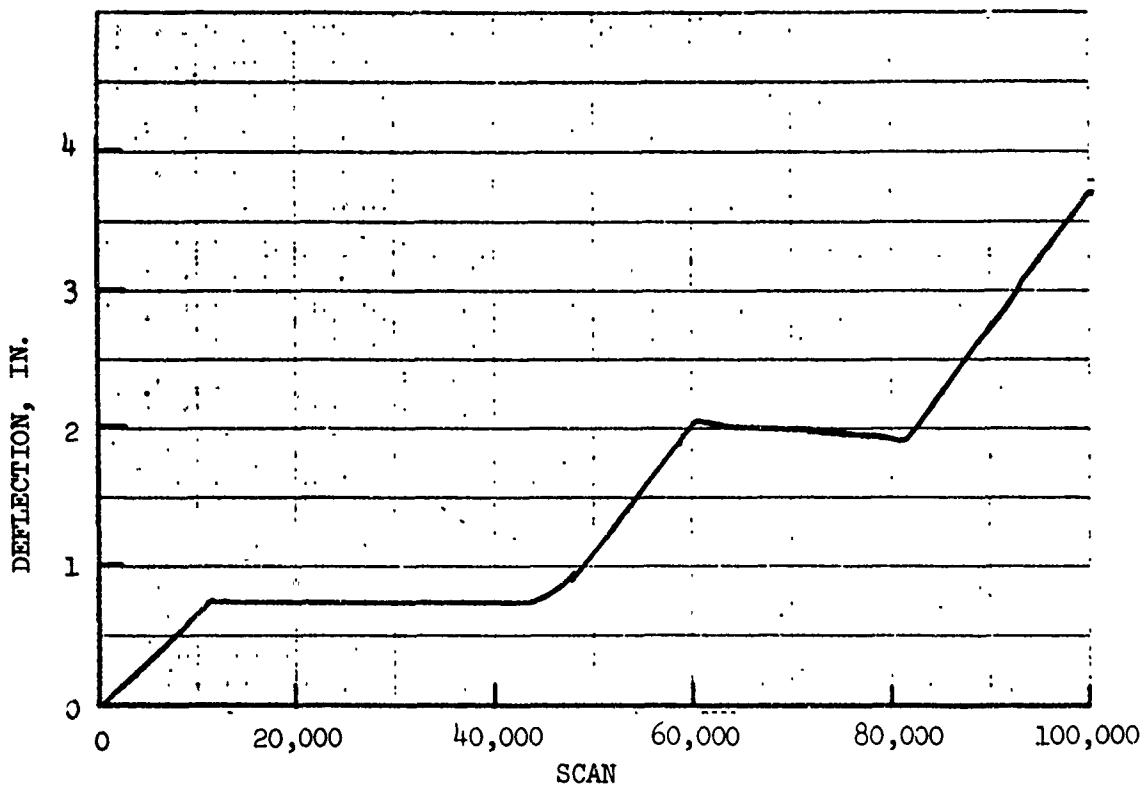


Figure 73. Deflection Versus Scan, Test Specimen 10.

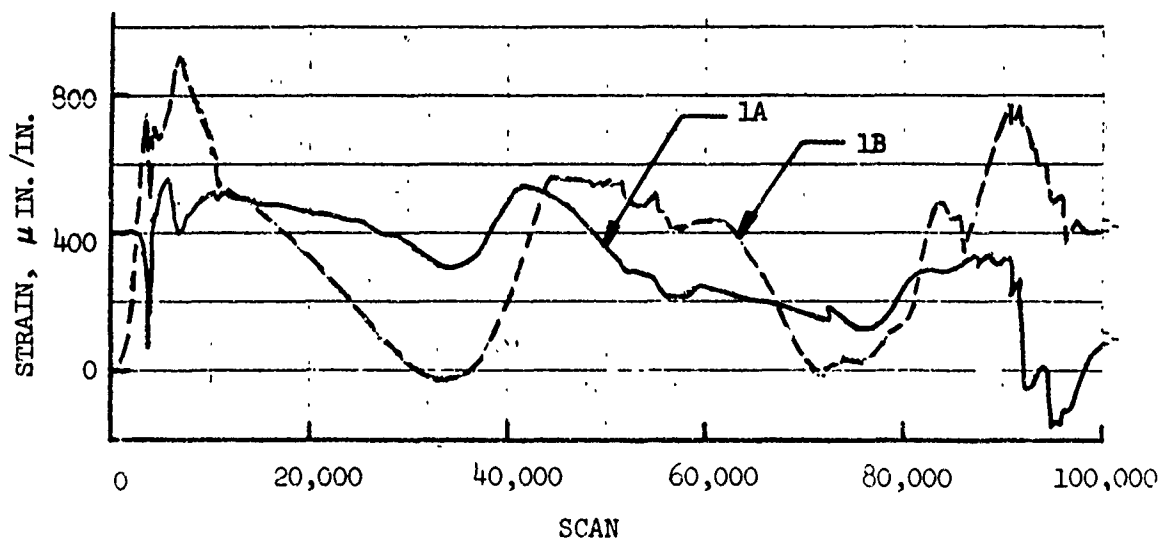


Figure 74. Strain Gages 1A and 1B Versus Scan, Test Specimen 10.

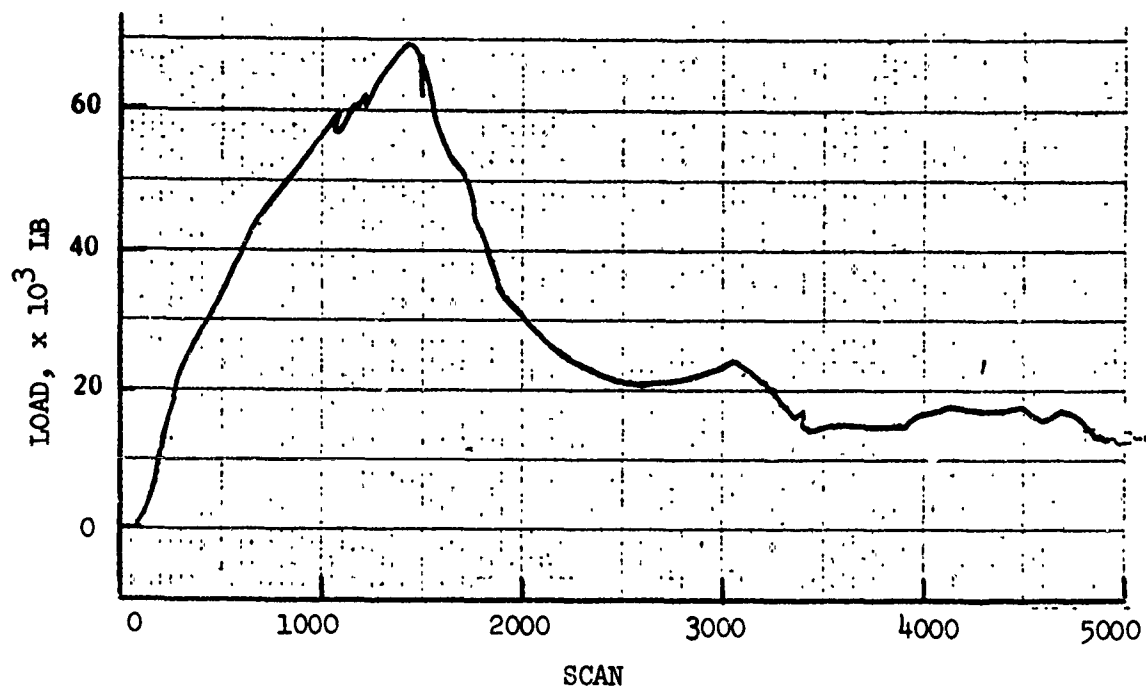


Figure 75. Load Versus Scan, Test Specimen 11.

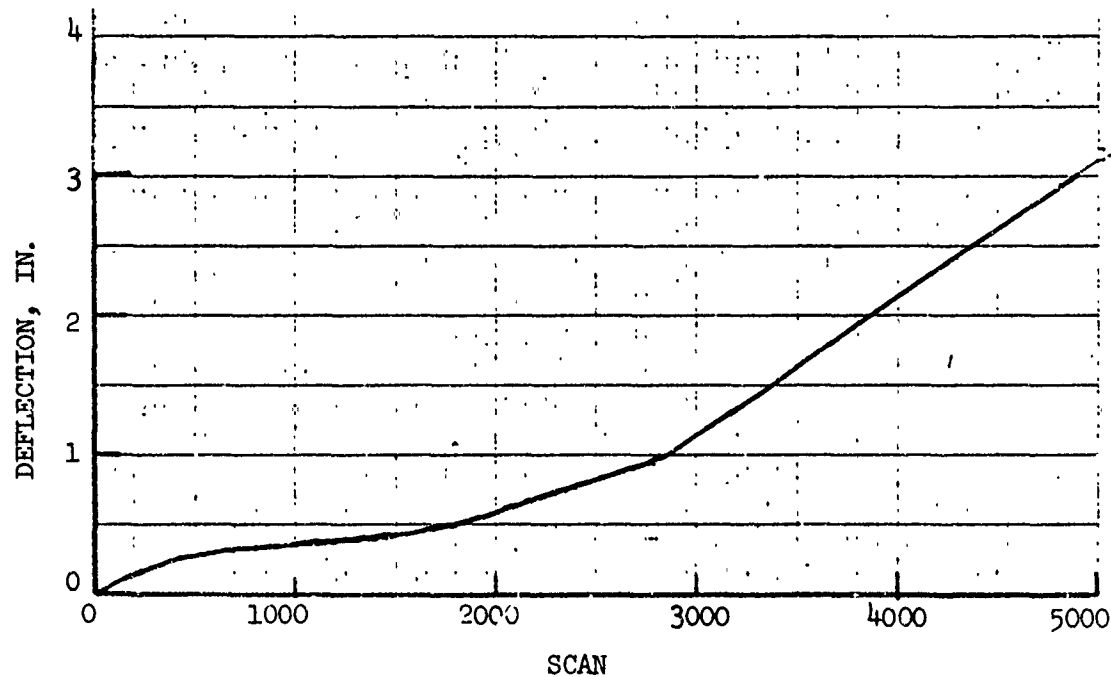


Figure 76. Deflection Versus Scan, Test Specimen 11.

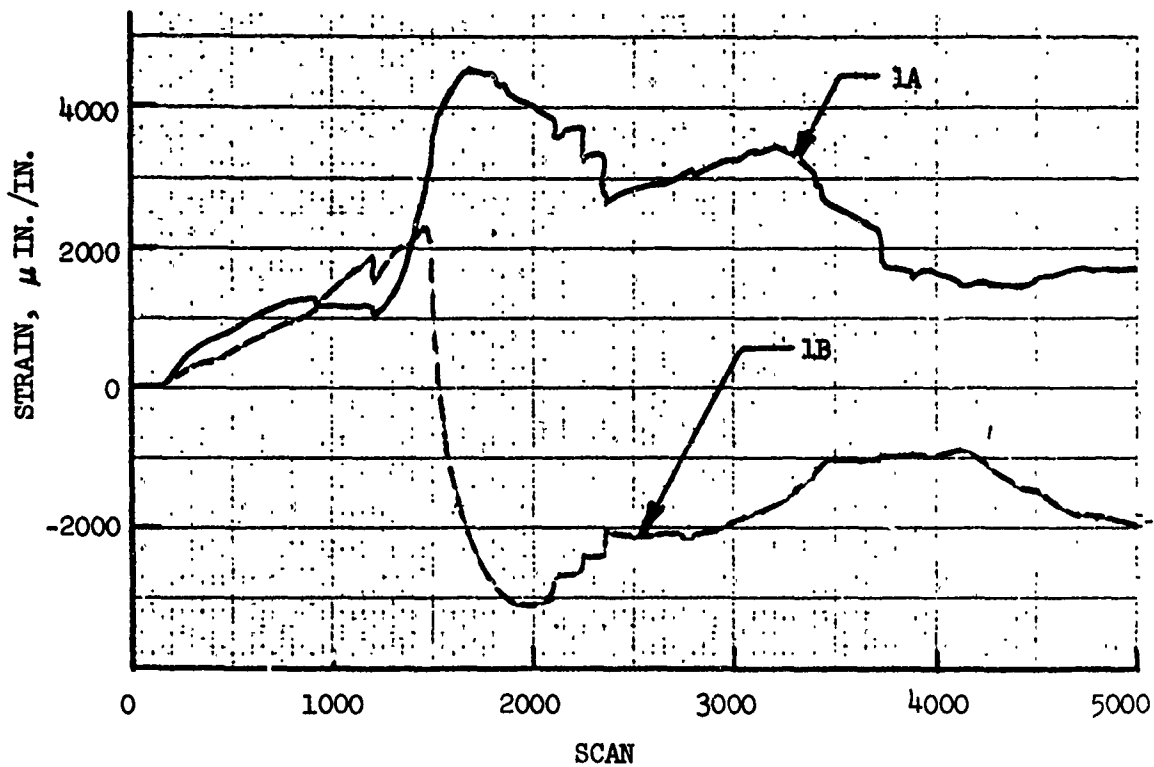


Figure 77. Strain Gages 1A and 1B Versus Scan, Test Specimen 11.

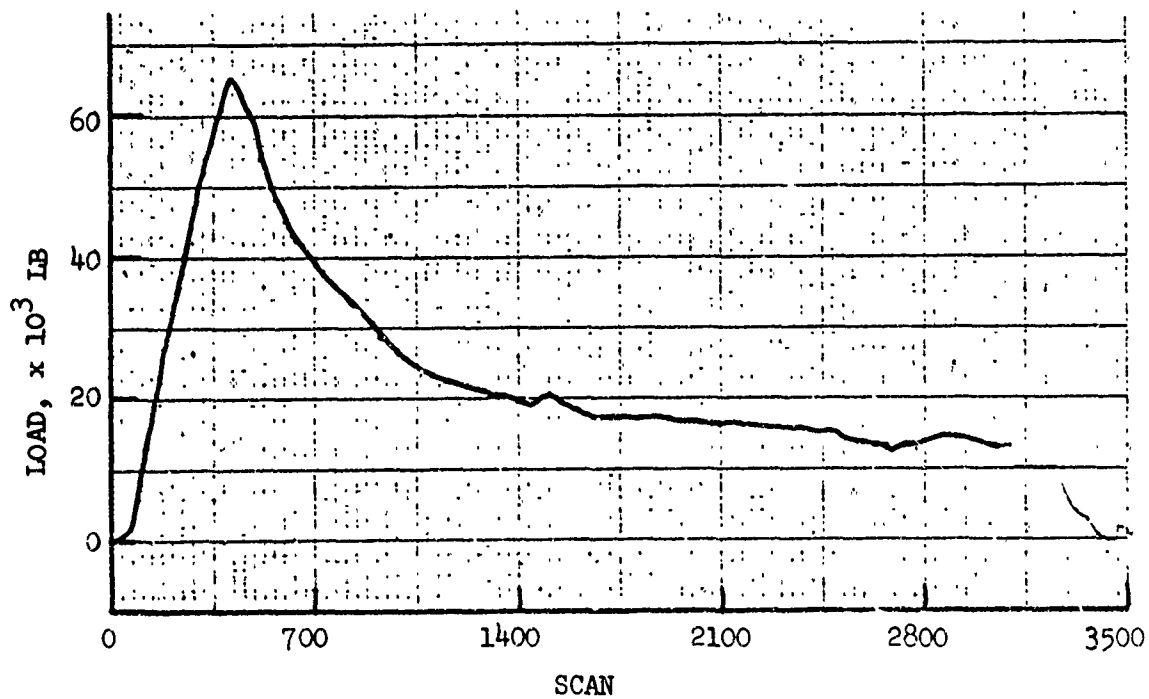


Figure 78. Load Versus Scan, Test Specimen 12.

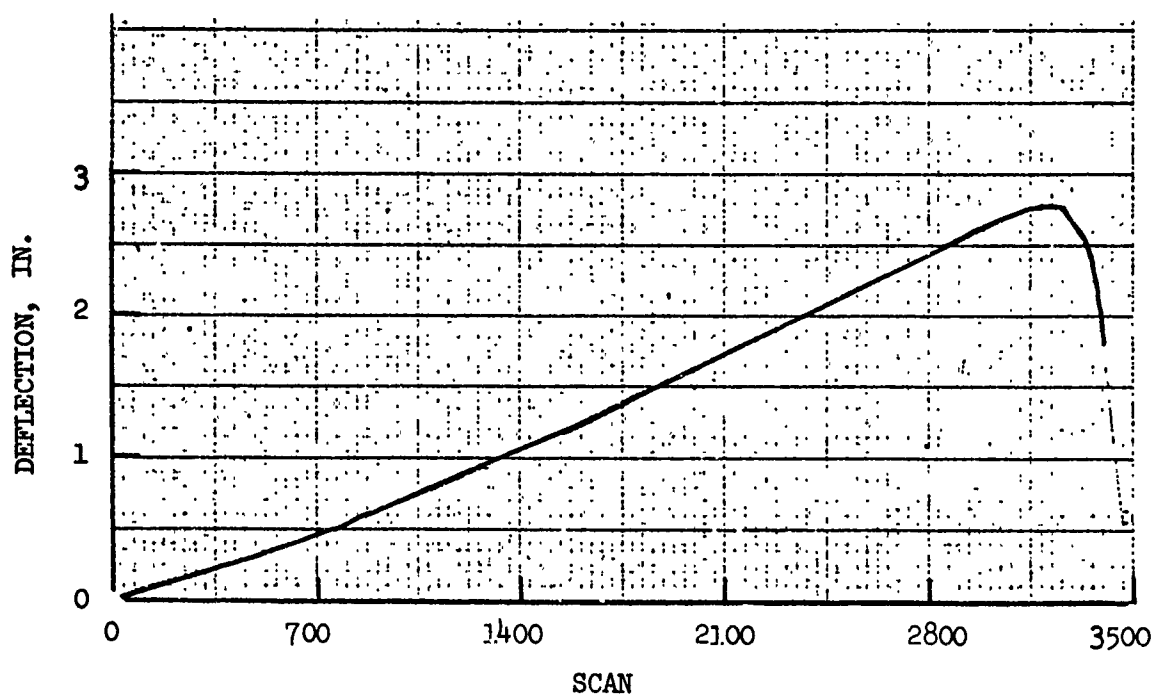


Figure 79. Deflection Versus Scan, Test Specimen 12.

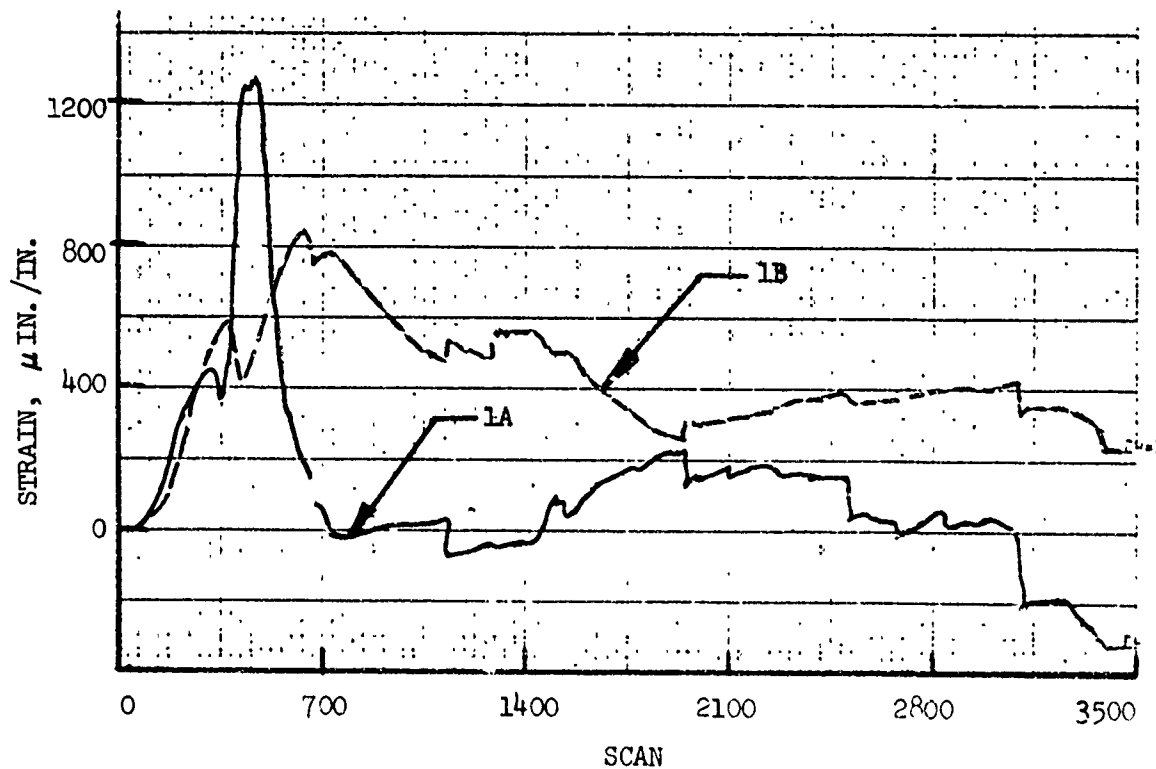


Figure 80. Strain Gages 1A and 1B Versus Scan, Test Specimen 12.

PROGRAM "KRASH" REFINEMENT

GENERAL

Program KRASH was developed as an analytical tool to be used during preliminary phases of design. In Reference 1 the results of a parametric study were integrated into an iterative design procedure by which a tradeoff between potential incremental cost and/or weight versus incremental improvements in crashworthiness capability could be accomplished. The results of that study and a subsequent study for the U.S. Army (Reference 106) showed the potential benefits that could be achieved for designers using a comprehensive but unsophisticated analytical approach. However, as is the situation with rapidly developed advances in the state-of-the-art analytical techniques, there are generally areas which can be simplified for users not thoroughly familiar with the approach.

In an effort to facilitate a designer's usage of program KRASH and to incorporate the results of the literature survey, load sensitivity study, substructure analysis, and tests, the program was revised. In particular, the input data format was changed for ease of data input and subsequent data changes, as would be required during parameter tradeoff studies. In addition, flexibility was added by the manner in which the Stiffness Reduction factors (KR's) are input. The revised program was run for the following conditions:

- Correlation case 31-52 from Reference 1 (to demonstrate that the revised program format was compatible with previous results: 23-ft/sec vertical velocity combined with 18.5-ft/sec lateral velocity impact).
- Three-dimensional velocity (~ 40 ft/sec combined velocity impact - 27.5 ft/sec longitudinal, 23 ft/sec vertical, 18.5 ft/sec lateral).
- Upper mass penetration into a specified occupiable volume.
- Simplified blade contact.
- Utilization of load-deflection data obtained from the 12 substructure tests, performed during the study, and related to actual fuselage structure size.

PROGRAM "KRASH" INPUT FORMAT REVISIONS

The input format changes are divided into three categories:

- Reordered data.
- Standardization of certain inputs.
- Allowance for more general KR curves.

First, the input data is rearranged so that all mass associated data (subscript i) is in one block, followed by all external spring data (subscript ik) and then all internal beam data (subscript ij). Next, standard values are assigned to certain input data items unless otherwise specified. Thus, much repetitive data input is eliminated. The following quantities are given standard values unless otherwise specified:

Quantity	Symbol	Standard Value
Angular Momenta	He_i	0
Euler Angles	$\phi''_i, \theta''_i, \psi''_i$	0
Aerodynamic Lift	lc_i	0
Stiffness Reduction Factors	KR_{ij}	1
Failure Deflection	$v_{max_{ijl}}$	100

Thus, the angular momenta of the masses (He_i) and the aerodynamic lift (lc_i) are normally zero, as are the Euler angles ϕ'' , θ'' , ψ''_i , relating mass body-fixed axes to airplane c.g. axes. For linear internal beam elements, $KR = 1$ for the entire run, so only nonlinear KR data need be input. Similarly, v_{max} 's need only be input for those elements where rupture is expected to occur; for elements not specified, a failure deflection of 100 inches and rotation of 100 radians are assumed. Thus, the nonspecified beams will not rupture during the run.

The input has also been revised to allow more general KR (stiffness reduction factor) curves. Previously, each KR curve could have only six data points, equally spaced. Now each curve can have up to 15 data points, with any desired spacing. This greatly facilitates modeling more complex load-deflection relationships. However, the load-sensitivity study results indicate that for many structural elements a very simple KR representation is adequate. The KR representation for the various load-deflection categories is discussed in Volume I (Design Procedures).

The relationship between the number of masses, internal beam elements and KR tables is governed by the following expression:

$$\text{Total number of bytes} = 235,136 + 1244 \times \text{number of lumped masses (n)} + 1372 \times \text{Internal Beam Elements (IBE)} + 198 \times \text{number of KR tables}$$

$$\begin{aligned} \text{e.g. } N &= 80 \\ \text{IBE} &= 100 \\ \text{KR} &= 120 \end{aligned}$$

$$\text{Bytes} = 235136 + 260,480 = 495616 = 484K$$

$$\text{where } 1K = 1024 \text{ bytes.}$$

Thus, program KRASH has some flexibility with regard to the size of problems to which it can be applied. Depending on the type of vehicle that is being analyzed the program can be redimensionalized to provide the most practical and economical solution.

The three-dimensional impact mass penetration, simplified rotor blade contact and updated load-deflection data cases are described in Volume I under the section entitled Program KRASH Refinement.

ENERGY BALANCE EQUATIONS

The primary objective of a crash analysis in the preliminary design phase is to determine how to absorb the initial vehicle kinetic energy while maintaining a livable environment for the occupants. This task is greatly facilitated if information regarding the spatial distribution of the energy flow through the vehicle is available. With the objective in mind, energy balance equations are developed. These equations do not alter the existing computational procedures or results; they merely provide additional information to assist in understanding how the initial kinetic energy is absorbed.

The total system energy at any time is given by the following expression:

$$E_{TOT} = KE + PE + SE + DE + CE \quad (1)$$

where

E_{TOT} = Total system energy

KE = Total kinetic energy

PE = Total potential energy

SE = Total strain energy absorbed

DE = Total damping energy dissipated

CE = Total crash spring (external spring) energy absorbed.

The total system energy E_{TOT} remains constant during the analysis. The total kinetic and potential energies for each mass over the number of masses (N) are shown in Equation (2).

$$KE = \sum_{i=1}^N KE_i \quad PE = \sum_{i=1}^N PE_i \quad (2)$$

The total strain and damping energies are obtained by summing the strain and damping energy for each interval beam element (ij pair) over the M ij pairs.

$$SE = \sum_{ij=1}^M SE_{ij} \quad DE = \sum_{ij=1}^M DE_{ij} \quad (3)$$

SE_{ij} results from the elastic-plastic behavior of the beam, and DE_{ij} results from its damping properties. The total crash spring energy results from summing the energies for all the individual crash springs (ik pairs) over all the P ik pairs.

$$CE = \sum_{ik=1}^P CE_{ik} \quad (4)$$

Referring to Equation (1), at time zero all energies are zero except KE, PE and E_{TOT} . The potential energy is referenced to the ground plane.

After impact with the ground, the kinetic and potential energies decrease; damping, strain and crash spring energies all increase to keep E_{TOT} constant.

In Equation (3), the summations over the ij pairs exclude those ij pairs that are identified in the input as DRI elements, which are described in Reference 1, page 73. This is done because these ij beams and their masses are isolated from the rest of the system; the forces in DRI beam ij drive mass j but not mass i. Also, the summations in Equation (2) for the kinetic and potential energies exclude mass j in a DRI ij pair, since this mass is isolated from the system. Thus, if ij pairs 6-9 and 11-15 are defined as DRI elements, the summations for SE and DE will exclude these ij pairs and the summations for KE and PE will exclude masses 9 and 15.

The kinetic energy for each mass, including translational and rotational components, is simply

$$KE_i = \frac{1}{2} \{vel_i\}^T [M_i] \{vel_i\} \quad (5)$$

where

$$[M_i] = \begin{bmatrix} M_i & & & & \\ & M_i & & & \\ & & M_i & & \\ & & & I_{xi} & I_{xyi} & I_{xzi} \\ & & & I_{xyi} & I_{yi} & I_{yzi} \\ & & & I_{xzi} & I_{yzi} & I_{zi} \end{bmatrix}$$

and

$$\begin{Bmatrix} \text{vel}_i \end{Bmatrix} = \begin{Bmatrix} u_i \\ v_i \\ w_i \\ p_i \\ q_i \\ r_i \end{Bmatrix}$$

$[M_i]$ is the 6 x 6 inertia matrix for mass i , and $\begin{Bmatrix} \text{vel}_i \end{Bmatrix}$ is a six-element vector of the linear and angular velocity components of mass i in body-fixed axes.

The i^{th} mass potential energy, referenced to the ground plane, is given by

$$PE_i = w_i z_i \quad (6)$$

Note that z_i is positive downward, measured from the ground plane.

The strain energy in internal beam ij is computed as a continuous summation of the incremental energy contributions from each integration interval.

$$\left(SE_{ij} \right)_{\text{current}} = \left(SE_{ij} \right)_{\text{previous}} + \sum_{l=1}^6 F_{ijl} \Delta v_{bijl} \quad (7)$$

$F_{ijl} \Delta v_{bijl}$ is the internal beam force (or moment) in the l^{th} direction, multiplied by the incremental beam deflection (or rotation) in the l^{th} direction; this is the incremental strain energy for the integration interval being considered. This straightforward formulation automatically accounts for the complexities of nonlinear, coupled deflections and unloading-reloading behavior, since these are considered in the calculation of F_{ij} .

The damping energy dissipated in internal beam ij is computed in a similar manner:

$$\left(DE_{ij} \right)_{\text{current}} = \left(DE_{ij} \right)_{\text{previous}} + \sum_{l=1}^6 FD_{ijl} \Delta v_{bijl} \quad (8)$$

FD_{ijl} is the internal beam damping force (or moment) in the l^{th} direction.

The crash spring energy is also computed as a summation of incremental energy changes. The crash spring energy resulting from all the ik springs attached to mass i is

$$(CE_i)_{\text{current}} = (CE_i)_{\text{previous}} - [X_{Ci} \ Y_{Ci} \dots \ N_{Ci}] \begin{pmatrix} \Delta x'_i \\ \Delta y'_i \\ \Delta z'_i \\ \Delta \text{inp}_i \\ \Delta \text{inq}_i \\ \Delta \text{inr}_i \end{pmatrix} \quad (9)$$

$X_{Ci}, Y_{Ci}, \dots, N_{Ci}$ are the six forces and moments acting on mass i in body-fixed axes, resulting from all ik external springs attached to mass i . These are given by Equations (66) in Reference 1, page 61. The vector $\{\Delta x'_i, \Delta y'_i, \Delta z'_i, \Delta \text{inp}_i, \Delta \text{inr}_i\}$ is made up of the six incremental deflections and rotations of mass m_i , in the same body-fixed axes. The first three terms of this vector are given by a simple rotation transformation of the incremental deflections in ground axes, which are obtained directly from numerical integration of the equations of motion.

$$\begin{pmatrix} \Delta x'_i \\ \Delta y'_i \\ \Delta z'_i \end{pmatrix} = [A_i]^T \begin{pmatrix} \Delta x_i \\ \Delta y_i \\ \Delta z_i \end{pmatrix} \quad (10)$$

The last three terms of the incremental displacement vector in Equation (9) are the incremental changes in the integrals of the angular velocities of mass m_i, p_i, q_i, r_i . These are the incremental rotations of mass m_i in body-fixed axes for the integration interval being considered.

The negative sign in Equation (9) results from the fact that the forces acting on mass m_i rather than the forces within the ik springs are being considered. A positive deflection of spring ik results in a negative force on mass m_i . The energy calculated in Equation (9) includes the energy dissipated by the sliding of spring ik on the ground with a friction coefficient; hence the forces in that equation include the ground drag loads due to friction.

The crash spring energies in Equation (9) are not yet in the form desired; the crash spring energy due to each spring ik must be separated out of the total crash spring energy associated with mass m_i, CE_i . This is done

simply by substituting Equations (66) of Reference 1 for the

$\{X_{Ci}, Y_{Ci}, \dots, N_{Ci}\}$ terms in Equation (9). This reformulates Equation (9) into a function of the individual crash spring forces FSP_{ijk} , where i and k refer to the ik spring and j refers to the direction of the forces on the ik spring. These forces are shown in Figure 10 on page 62 of Reference 1. The final equation for the crash spring energy associated with each spring ik is the following:

$$(CE_{ik})_{\text{current}} = (CE_{ik})_{\text{previous}} - \sum_{j=1}^3 FSP_{ijk} \Delta v_{ij} + TERM_{ik} \quad (11)$$

where

$$\{\Delta v_i\} = \begin{pmatrix} \Delta x_i \\ \Delta y_i \\ \Delta z_i \end{pmatrix}$$

and

$$TERM_{ik} = FSP_{i31} l_{i1} \Delta i_{q_i} - FSP_{i21} l_{i1} \Delta i_{nr_i} \quad (k = 1)$$

$$TERM_{ik} = -FSP_{i32} l_{i2} \Delta i_{np_i} + FSP_{i12} l_{i2} \Delta i_{nr_i} \quad (k = 2)$$

$$TERM_{ik} = FSP_{i23} l_{i3} \Delta i_{np_i} - FSP_{i13} l_{i3} \Delta i_{nq_i} \quad (k = 3)$$

This is the crash spring energy to be used in Equation (4) to obtain the total crash spring energy, CE , for the entire vehicle. Note that CE could have been determined more directly by summing the CE_i from Equation (9) over all the masses m_i . However, the CE_{ik} associated with each spring ik is calculated so that it can be determined how much each external spring is contributing to CE .

ENERGY BALANCE DATA

The energy balance technique described in the preceding section and the information presented in the following paragraphs describe a type of format which can be of benefit to a designer during the preliminary stage of design.

The output format for the energy balance data is shown in Figures 81, 82 and 83. For a condition of 42-ft/sec vertical velocity impact and using program KRASH with the existing UH-1H helicopter math model consisting of 31 masses and 38 internal beam elements, Figure 81 shows that at time = 0.0 second only contributions to the energy term are kinetic and potential energies. The kinetic energy represents the $1/2 MV^2$ terms while the potential energy is a function of the height the respective masses are above the ground at the initiation of impact. The internal beam energy contributions due to strain and damping are zero at this time, as is the external spring crushing energy. Figure 82 shows the energy data output during the run at time = .042 second after impact. At this time kinetic energy has decreased from 85.85% of the total energy to 58.6% of the total energy. Similarly, the potential energy has decreased from 14.15% to 8.96% of the total energy. At this time crushing energy, strain energy and damping energy as a percent of the total energy have increased to 9.8%, 7.5% and 15.2%, respectively. Furthermore, Figure 82 shows the following:

- distribution of kinetic energy and potential energy by mass item
- distribution of strain energy and damping energy by beam element
- distribution of crushing energy by external spring element

The negative terms in the damping energy output for beam elements 2 and 4 are of the order of less than .0003 percent of the total energy. The values are obviously insignificant and are considered to reflect damping inherent in the integration routine.

Figure 83 shows a summary of the energy data which is presented at the end of the computer run. The summary shows for each time increment the total energy and the amount and percentage contribution from kinetic, potential, strain damping and crushing. The data in the summary shown in Figure 83 indicates that for the illustrated case the vehicle rebounds at approximately .064 second after impact. The kinetic energy reduces from impact until .064 second after impact, then starts to increase again. The summary data presented in Figure 83 also shows that crushing energy reaches a maximum of 51.73% of the total energy at .062 second after impact and that strain energy reaches a maximum of 21.93% of the total energy at .076 second after impact.

Figure 82. Energy Balance Data Output at $t = 0.042$ Second.

TIME	TOTAL ENERGY	PERCENT	KINETIC ENERGY	PERCENT	POTENTIAL ENERGY	PERCENT	STRAIN ENERGY	PERCENT	DAMPING ENERGY	PERCENT	CRUSHING ENERGY	PERCENT
0	3.256D6	100.	2.796E6	85.85	4.608E5	14.15	0.0	0.0	0.0	0.0	0.0	0.0
.01	3.259D6	100.04	2.662E6	81.71	4.186E5	12.85	4.883E4	1.5	1.030E5	3.16	2.563E4	0.79
.02	3.269D6	100.35	2.458E6	75.21	2.781E5	11.57	1.145E5	3.6	2.603E5	8.57	3.741E4	1.14
.03	3.296D6	101.15	2.234E6	67.80	3.395E5	10.30	2.221E5	6.74	4.602E5	13.99	3.919E4	1.19
.04	3.303D6	101.39	2.038E6	61.66	3.028E5	9.17	2.432E5	7.38	5.011E5	15.17	2.189E5	6.63
.05	3.303D6	101.41	1.68E6	35.38	2.708E5	8.2	3.081E5	9.33	5.153E5	15.40	1.041E5	31.5
.06	3.303D6	101.4	2.622E5	7.94	2.533E5	7.67	5.231E5	16.84	5.584E5	16.90	1.706E6	51.65
.062	3.303D6	101.4	2.080E5	6.30	2.522E5	7.64	5.698E5	17.08	5.698E5	17.25	1.706E6	51.73
.064	3.303D6	101.47	2.017E5	6.11	2.520E5	7.63	5.815E5	18.24	5.815E5	17.61	1.665E6	50.42
.070	3.303D6	101.46	1.016E5	12.16	2.560E5	7.75	6.112E5	21.03	6.112E5	18.50	1.340E6	40.57
.076	3.305D6	101.46	8.053E5	24.36	2.659E5	8.04	6.298E5	21.93	6.298E5	19.06	8.795E6	26.61
.080	3.307D6	101.53	1.137E5	34.38	2.748E5	8.31	6.467E5	19.53	6.467E5	19.55	6.028E5	18.23

*J denotes double precision

Computer Printout is Every .002 Second. For Illustrative Purpose an Abbreviated Computer Printout is Shown

Figure 83. Energy Data Output Summary.

REVISED USER'S GUIDE

This section describes all the input necessary to run the program, and the output available from the program.

Input

The input data format is described in detail in this section and is shown in Figure 84. Unless otherwise specified, all quantities are input in inch, pound, second, and radian units. For cards 0102 and following the input format is 6E12.0 unless otherwise noted. Each card has 6 fields; each field is 12 columns wide. As an example of this format, the number 126.08 can be input in the following ways:

			1	2	6	.	0	8			
			1	2	6	.		8			

			1	.	2	6	0	8			E 2
			1	2	6	0	8	.			E - 2

Blank columns are treated as zeros. When the E format is used, the exponent must be right justified in the field. Integer formats with field widths of 1, 2 or 3 are also used. The numbers in these locations must be right justified integers. Sequence numbers in columns 77 through 80 should be used corresponding to those shown in the input format to facilitate deck assembly and changes. All tabular input is linearly interpolated between input values and extrapolated beyond the two end values, if necessary.

Card 0100 - This card contains the title for the case being analyzed. All text material on card 0100 is reproduced at the top of every page of the output and on every plot.

Card 0101 - N is the total number of lumped masses. The maximum allowable number of masses is 80. $\Delta \text{Print}/\Delta t$ is the integer multiple of Δt at which output is printed. Δt is the numerical integration time interval. c_{\max} is the time span being analyzed.

Cards 0102 through 0104 - These cards contain the overall vehicle initial conditions. \dot{x}_G , \dot{y}_G , and \dot{z}_G are the ground axes components of the initial c.g. velocity. p' , q' , and r' are the c.g. coordinate system components of the initial angular velocity of the vehicle. p' is the roll velocity, q' the pitch velocity, and r' the yaw velocity. ϕ' , θ' , and ψ' are the Euler angles relating the initial position of the vehicle to ground coordinates. ϕ' is the roll angle, θ' the pitch angle, and ψ' the yaw angle. z_G is the negative of the initial vehicle c.g. height above ground. If this input is zero (blank), the initial condition subroutine computes a value of z_G so that the lowest extremity of the vehicle is .1 inch above the ground.

Cards 0201 through 02XX - N of these cards are used to input the weights (not masses) of the N lumped masses.

Cards 0301 through 03XX - N of these cards are required. Each card inputs the six moments and products of inertia for the i th mass, $i = 1, 2, \dots, N$.

Cards 0401 through 04XX - N of these cards are used to input the coordinates of the N lumped masses. x_i'' is the Fuselage Station (increasing aft), y_i'' is the Butt Line (positive left), and z_i'' is the Water Line (increasing upward).

Card 0500 - NI is the number of masses having nonzero He_{xi} , He_{yi} , He_{zi} (angular momenta) or ϕ_i'' , θ_i'' , ψ_i'' , which are the Euler angles relating the c.g. axes to the i th mass body fixed axes. The above quantities are normally zero, so only nonzero values are input on cards 0501 through 05NI. i_1, i_2, \dots, i_{NI} are the actual i 's or mass numbers that have nonzero input data for any of the above quantities. NI and i through i_{NI} are input in integer format I2. If NI equals zero, cards 0501 through 05NI are not input; however, card 0500 is always required. If NI equals zero, a blank card for card 0500 is input.

Cards 0501 through 05NI - As noted above, these cards are used to input any nonzero values of He_{xi} , He_{yi} or ϕ_i'' , θ_i'' , ψ_i'' . One card is used for each mass i having nonzero input. The masses are ordered according to the sequence specified on card 0500. He_{xi} , He_{yi} , and He_{zi} are the body axes components of the angular momenta of masses m_i , due to rotation of internal masses within m_i . These are normally zero. ϕ_i'' , θ_i'' , ψ_i'' are the Euler angles from the c.g. axes to the i th mass axes. If the i th mass body-fixed axes are parallel to the vehicle c.g. coordinate axes, which is usually the case, these are all zero. Note that if any nonzero values are input, then θ_{ij} and ψ_{ij} on cards 0901 through 090M must be input.

Cards 0600 through 06NI - NI is the number of masses having nonzero l_{ci} , which are the aerodynamic lift constants used in Equation (5) on page 30 of Reference 1. i_1, i_2, \dots, i_{NI} are the actual mass numbers having nonzero l_{ci} . If all l_{ci} are zero, card 0600 is input as a blank card and cards 0601 through 06NI are not input. One card is used for each nonzero l_{ci} input, with the ordering defined on card 0600.

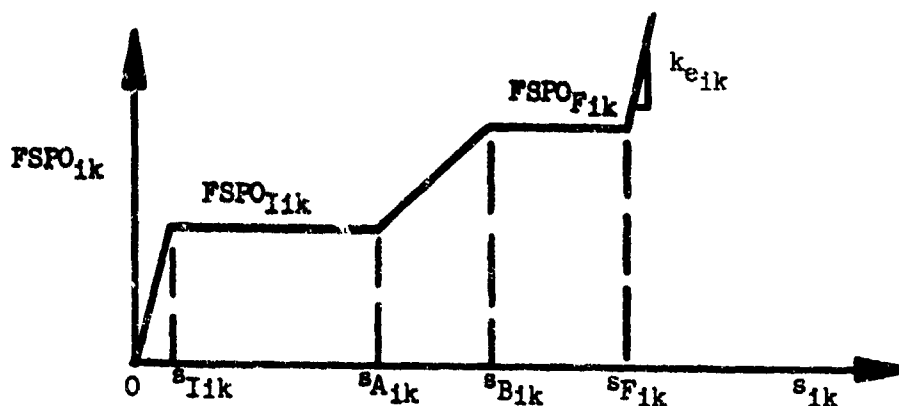
Cards 0701 through 070p - As many cards as necessary are used to input data for the external springs. The maximum allowable number of external springs is 50. Each spring is identified by an ik pair, where i defines the mass to which the spring is attached and k defines the direction of the spring. $k = 1, 2$ and 3 correspond to springs in the x, y and z directions, in mass i body-fixed axes. l_{ik} is the free length of the external spring ik . l_{ik} is positive if it radiates out from mass m_i in

the positive direction of the i th mass body axes; l_{ik} is negative if it radiates in the opposite direction. Springs in both directions are not allowed. μ_{ik} is the friction coefficient between the ground and the end of the ik spring. k_{ik} is the linear unloading stiffness and also the bottoming stiffness for the ik spring.

If no external springs are used, card 0701 is input blank and the next card input is 0901.

Card 07XX - A blank card in this location is required to terminate the above ik external spring list.

Cards 0801 through 080p - External spring load-deflection curve parameters are input on these cards, one card per ik spring, ordered as in the ik list on cards 0701 through 070p. The program is written so that a table of the following form is input:



This table is defined by six parameters s_{Iik} , s_{Aik} , s_{Bik} , s_{Fik} , $FSPO_{Iik}$, and $FSPO_{Fik}$, which are input on cards 0801 through 080p.

Cards 0901 through 090M - These cards contain the Euler angles ϕ_{ij} , θ_{ij} , and ψ_{ij} for all internal beams ij . The beam interconnections are defined by the i 's and j 's input. i must be less than j , but there is no other requirement on the ordering of the ij pairs. ϕ_{ij} is always input; θ_{ij} and ψ_{ij} need not be input if ϕ_i , θ_i and ψ_i on cards 0501 through 050M are all zero. In the latter case, θ_{ij} and ψ_{ij} are computed in initial conditions. ϕ_{ij} will normally be zero. The maximum allowable number of internal beams is 100.

Card 09XX - A blank card is required here to terminate the ij pair internal beam list.

Cards 1001 through 1006 - These six cards are used to input the 6×6 linear stiffness matrix $[K_{ij}]$ for the first ij beam, listed on card 0901. Each card inputs one row of the matrix.

Cards 1007 through 1XXX - As many cards as necessary are used to input all the remaining 6×6 $[K_{ij}]$ matrices. These matrices must be ordered the same as the ij pairs are ordered on cards 0901 through 090M.

Cards 2001 through 200M - These cards are used to input the \bar{C}_{ij} for all the ij pairs defined on cards 0901 through 090M. \bar{C}_{ij} is the damping ratio (damping/critical damping) for the isolated system consisting of masses m_i and m_j connected by beam ij . Values of \bar{C}_{ij} between .01 and .05 are generally representative of the structural damping.

Cards 2101 through 210q - These cards are used to specify which beam elements (ij pairs) and directions (l) have nonlinear stiffness properties. l varies from 1 to 6 corresponding to the $x, y, z, \phi, \theta, \psi$, directions in beam axes. Only beam elements and directions having nonlinear properties ($KR \neq 1$) are input. For each ijl combination listed, a table of KR (stiffness reduction factor) versus deflection is input on the following cards. The NP 's are the number of points in the following tables. For all ijl combinations not listed, $KR = 1$ for the entire run. The maximum allowable number of ijl combinations (nonlinear KR tables) is 80. The maximum allowable number of points per table is 15. For a completely linear analysis, card 2101 is blank and the next card input is 3001.

Card 21XX - A blank card is required here to terminate the ijl list of nonlinear beam/direction combinations.

Cards 2201 through 22NP₁ - These NP_1 cards are used to input the deflection (vb_{ijl}) versus KR_{ijl} table for the ijl combination listed on card 2101. KR_{ijl} normally starts at 1.0 zero deflection (or rotation), corresponding to linear behavior, and varies in any desired manner thereafter to define a general load-deflection curve. Note that KR is not the actual load, but rather the derivative of load with respect to deflection, i.e., the slope of the load-deflection curve.

Cards 22X1 through 2XXX - As many cards as necessary are used to input the KR_{ijl} vs. vb_{ijl} tables for the remaining ijl combinations listed on cards 2101 through 210q above. Note that each table can have a different number of data points NP_1 .

Cards 3001 through 300r - These cards are used to identify which internal beam elements ij have nonstandard maximum deflections for rupture (v_{maxijl}). The standard deflections built into the program are 100 inches for deflections and 100 radians for rotations. These numbers were deliberately chosen to be very high so that rupture would not occur unless reasonable v_{max} 's were specified. If no nonstandard v_{max} 's are to be input, card 3001 is input blank, and the next card input is 5001.

Card 30XX - This blank card is required to terminate the preceding ij list for nonstandard v_{maxijl} .

Cards 3101 through 310r - These cards are used to input the nonstandard v_{maxijl} 's, one card for each ij beam specified on cards 3001 through 300r. Each card inputs the six v_{max} 's for one ij beam, ordered $x, y, z, \phi, \theta, \psi$ in beam axes. v_{maxijl} defines the maximum allowable deflection before rupture occurs. After rupture, all forces in beam ij go to zero, regardless of the direction in which the rupture occurred.

Cards 5001 through 5025 - These cards are used to specify the time history output plots desired. The only input for these cards is either a 1 or a blank. A 1 results in the output of a time history plot for the response quantity indicated; a blank results in no plot for that item. For example, a 1 in the 13th column of card 5003 specifies that a time history plot of z_{13} is to be generated. For cards 5001 through 5012, the column number in which the 1 is input indicates which mass i is desired. For cards 5013 through 5024, the column number in which the 1 is input indicates which ij pair is desired, where the ij pairs are ordered as on cards 0200 through 02XX.

Plots are available for the displacements, velocities and accelerations of all the lumped masses, all the external spring compressions S_{ik} , and all the beam ij total deflections/rotations (vb_{ij}) and forces/moments (F_{ij}). The latter two items are in beam ij axes. Also available are plots of the DRI (Dynamic Response Index). These are identified by the DRI ij element, as described on card 7000 below. The plot variables are labelled automatically, and the plot scales are chosen automatically. The user merely has to specify which plots are desired. Up to 150 plots can be requested per run. Thirty thousand data points are stored for plotting, with the plot time interval depending on run time and number of plots requested. Thus, if the maximum of 150 plots are requested, 200 points will be saved to generate each plot. If the maximum run time is .2 second, this will give one data point every .001 second. For the types of problems analyzed, this appears to be a marginally acceptable resolution. Requesting fewer plots automatically increases the number of points per plot and, hence, the resolution.

Card 6000 - P is the mass to be used to locate the mass penetration control volume. The format is I2. If no mass penetration calculations are required, this card is input blank and card 6001 is not required.

Card 6001 - x_n , x_p , y_n , y_p , z_n , and z_p are the six distances (all positive), measured from the control mass P to the six sides of the control volume. These are measured along the positive (p) and negative (n) body-fixed axes for mass mp.

Card 7000 - This card is used to specify which internal beam ij elements, if any, are to be treated as DRI (Dynamic Response Index) elements. For example, if the 15th ij pair listed on cards 0901 through 090M is to be a DRI element, then a 1 is input in column 15 of card 7000. For DRI beam elements, the forces in beam ij drive mass j but not mass i. To understand the physical significance of this, refer to Reference 1, page 73, which explains the concept of a DRI.

Cards 8000 and 8001 - Cards 8000 and 8001 are used to specify il (mass-direction) pairs for which it is desired to input a time history table of acceleration. Thus, any mass can be driven by a specified acceleration in any direction. The 6 directions are ordered x, y, z, ϕ , θ , ψ in mass i body-fixed axes. The format is 24 (I2I1), where the 1 is right justified in the field of 3 and the i is right justified in the left two columns of the field of 3. A maximum of 48 input tables are allowed. For example, driving mass 3 in the y direction would be specified as |032|.

When any acceleration table is input, that acceleration replaces the normally computed acceleration, so it becomes a driving force to the system. If less than 24 il pairs are specified, only card 8000 is required (omit 8001). If 24 or more il pairs are input, cards 8000 and 8001 are both required (if exactly 24 il pairs are input, card 8001 will be blank, but must be input).

Card 8010 - N_1 is the number of data points in the following acceleration time history table, applicable to the first il pair on card 8000. The maximum allowable value of N_1 is 50.

Cards 8011 through 801 N_1 - N_1 cards are used to input the time history table of acceleration for the first il pair. Each card inputs one time and the corresponding acceleration.

Cards 8XX0 through 8XX N_q - The remaining acceleration time history tables are input on these cards, in the format described above. The tables are ordered according to the sequence of il pairs defined on cards 8000 and 8001.

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[illegible]

Figure 4. Input Data Format.

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PAGE	OF
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GENERAL PURPOSE DATA SHEET

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TITLE		PREPARED BY		DATE		CHECKED BY		DATE		JOB NO.		GROUP		PAGE	OF				
										WO		EWA		3	6				
1	5	7	10	15	20	25	30	35	40	45	50	55	60	65	70	73	76	77	80
<p>(INPUT REMAINING 6x6 K MATRICES FOR EACH (i,j) BEAM, 6 CARDS PER MATRIX. ORDER MATRICES SAME AS (i,j) ELEMENTS ARE ORDERED ON CARDS 0901-0906.)</p>																			
1	5	7	10	15	20	25	30	35	40	45	50	55	60	65	70	73	76	77	80
<p>2001 2002 2003</p>																			
1	5	7	10	15	20	25	30	35	40	45	50	55	60	65	70	73	76	77	80
<p>2101 2102 2103 2104 2105 2106</p>																			
1	5	7	10	15	20	25	30	35	40	45	50	55	60	65	70	73	76	77	80
<p>2201 2202 2203 2204 2205 2206</p>																			
1	5	7	10	15	20	25	30	35	40	45	50	55	60	65	70	73	76	77	80
<p>2301 2302 2303 2304 2305 2306</p>																			

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Figure 3. (Continued)

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F. . . (c n . n . i)

Print Output

First, all the input data is printed out, with self-explanatory identifying titles. Next, at each print time ($= \Delta \text{Print} / \Delta t \times \Delta t$), the data shown in Figure 85 is printed out. At the top of each page, the case identification data is printed out (from input card 0100). Then the current value of time t is printed.

Next, for each of the N masses, five lines of data are output. $x_i, y_i,$ and z_i are the ground coordinates of m_i . $\dot{x}_i, \dot{y}_i,$ and \dot{z}_i are the ground axes components of the velocity of m_i . $\dot{u}_i, \dot{v}_i, \dot{w}_i$ are the time derivatives of $u_i, v_i,$ and w_i . (These are not equal to the acceleration of m_i , as can be seen from Equation (81), Reference 1.) XACCEL, YACCEL, ZACCEL are the body axes components of the acceleration of m_i .

$\phi, \theta,$ and ψ are the Euler angles defining the attitude of body m_i , in roll, pitch and yaw. $\dot{\phi}_i, \dot{\theta}_i,$ and $\dot{\psi}_i$ are the time derivatives of $\phi_i, \theta_i,$ and ψ_i . $\dot{p}_i, \dot{q}_i,$ and \dot{r}_i are the body axes components of the angular velocity of m_i ; they are the velocities in roll, pitch, and yaw, respectively. $p_i, q_i,$ and r_i are the body axes components of the angular acceleration of m_i .

Following that output, the running time sums of the internal forces $\{F_{ij}\}$ (Equation (27), Reference 1) are printed out; the six forces and moments for each ij pair are printed on a line preceded by the identifying i and j . Next, the running time sums of the beam deflections $\{v_{bij}\}$ (Equation (26), Reference 1) are printed out in the same format.

Finally, the external spring compressions s_{ik} (Equation (47), Reference 1) are printed out. Each line starts with the i , followed by the s_{ik} for $k=1, 2, 3$. Only values of i for which a spring is input are shown.

During the course of the run, if any ruptures or control volume mass penetrations occur, the appropriate information is printed out. When a rupture occurs, the word RUPTURE is printed, followed by four items:

1. The ij pair that ruptured, where the numbering corresponds to the ordering of the ij pairs as input on cards 0200 through 02XX.
2. The l (from 1 to 6) indicating in which direction (in beam axes) the rupture occurred. These are ordered $x, y, z, \phi, \theta,$ and ψ . See Figure 24 (Reference 1) for the directions of the beam axes.
3. The v_{bijl} at the time of rupture. This is the total deflection, in beam axes, in the direction of the rupture.
4. The $v_{max,ijl}$ which defines the maximum allowable beam deflection in the l^{th} direction. This is an input constant.

If a control volume mass penetration occurs, the mass which penetrates and the time are printed out. At the end of the run, the ruptures and mass penetrations that occurred during the run are summarized in tables for ready reference.

Sample Cases

Two sample cases are presented. The first sample case represents a combined vertical and lateral impact condition. The vertical impact velocity is 23 ft/sec and the lateral impact velocity is 18.6 ft/sec. The mathematical model is the 31 mass representation shown in Volume I, Figure 6 and Table VI. The sample output for one time period is shown in Figure 85. The input data, which follows the User's Guide Input format (Figure 84), is presented in Figure 86.

Sample output plots are provided for the engine mass vertical velocity, engine mass vertical acceleration and engine mount support vertical deflection in Figures 87, 88 and 89, respectively. Figures 90, 91 and 92 provide similar sample plots for the lateral velocity, lateral acceleration and mount lateral displacement, respectively. Figures 93 and 94 show the forward and aft DRI plots, respectively, for the combined vertical and lateral impact velocity case. The engine vertical responses shown in Figures 87 (velocity), 88 (acceleration) and 89 (displacement) can be compared to Figures 17, 18 and 19 in Volume II, Reference 1 to show that the revised program KRASH performs in the same manner as the final submitted program KRASH from the previous study (Reference 1).

The second sample case provided in this section is the three-dimensional velocity impact condition. The initial c.g. velocity for this case is a combined 40 ft/sec velocity in which the velocity vector components are:

$$V_x = 27 \text{ ft/sec} \quad V_y = 18.6 \text{ ft/sec} \quad V_z = 23 \text{ ft/sec}$$

The initial vehicle attitude is flat (zero roll, pitch and yaw angles), and the initial angular velocities are zero. Figure 101 (page 140) in Volume I shows the mathematical model representation for this condition. The model consists of 32 lumped masses. It is the same model used in the previous sample problem except for the extra mass location which represents the rotor blade during a potential mass penetration (of the upper cabin) condition. The input data for this sample problem is provided in Figure 95. A sample output for one time period is shown in Figure 96. Figures 97 through 99 show the engine mass vertical, lateral and longitudinal velocities, respectively. Figures 100 through 102 present the engine mass vertical, lateral and longitudinal accelerations, respectively. Figures 103 through 105 show the engine support mount vertical, lateral and longitudinal deflections, respectively. The forward and aft DRI's are presented in Figures 106 and 107.

Program Listing

A complete listing of the computer program KRASH, including subroutines, is presented in Figure 108.

TIME = 0.13500

	X	Y	Z	PHI	THETA	PSI
	COORD	COORD	COORD	COORD	COORD	COORD
	U	V	W	P	Q	R
	ACCEL	ACCEL	ACCEL	ACCEL	ACCEL	ACCEL
MASS 1	-3.250560 02	-4.317090 01	-1.106250 02	-8.361200-02	9.347450-02	-1.601300-02
	-1.930260 02	-1.715540 02	1.702990 02	-1.276540 00	4.516750 00	1.657800 00
	-2.053190 02	-1.073710 02	1.357490 02	-1.431280 00	4.352800 00	2.043800 00
	-3.506280 02	1.497320 03	-5.605120 03	-5.452120 01	1.417760 02	5.834700 01
	1.614580 00	3.295260 00	-1.111090 01			
MASS 2	-2.661260 02	-3.491790 01	-5.452410 01	-2.275900-02	5.227000-02	3.797920-02
	-1.446150 01	-7.145320 01	-2.025450 01	-3.938050-01	6.004730-01	-3.878030-01
	-1.608150 01	-7.035360 01	-2.273050 01	-3.735440-01	6.051310-01	-3.735087
	2.307860 03	3.825040 03	-7.145730 03	-5.934810 01	-3.194530 01	-4.2067
	5.875000 00	9.403130 00	-1.441080 01			
MASS 3	-1.023130 02	-3.267740 01	-3.726080 01	-1.929990-02	3.215990-02	3.293480-02
	-2.123310 00	-1.085600 02	-1.011980 02	-6.164260-01	3.824940-01	-1.064790-02
	-2.439980 00	-1.064550 02	-1.034030 02	-6.167490-01	3.824310-01	-3.284740-03
	9.054170 02	-6.119310 02	-6.050930 02	-2.245050 01	-3.372250 01	-7.738650 00
	2.242230 00	-1.420070 00	-2.253310 00			
MASS 4	-7.246160 01	-3.155440 01	-2.908360 01	-1.934330-02	2.628370-02	3.047130-02
	-1.497750 00	-1.130470 02	-1.102820 02	6.995640-01	3.579740-01	1.059750-01
	9.975740-01	-1.106520 02	-1.142090 02	6.967810-01	3.558600-01	1.127420-01
	5.765360 02	-5.274720 02	-5.828620 00	-1.785700 01	-1.984840 01	2.544640 00
	1.427710 00	-3.232720 00	-2.161230-01			
MASS 5	-2.387740 01	-3.011490 01	-3.033350 01	-1.938630-02	2.512670-02	2.876610-02
	9.210640-01	-1.062580 02	-2.297590 02	7.008600-01	3.765770-01	1.790510-01
	1.131230 00	-1.037050 02	-1.318070 02	6.963610-01	3.730370-01	1.862610-01
	5.397130 02	-3.576680 02	8.804820 02	-1.781500 01	-1.638310 01	3.840880 00
	1.320060 00	-6.887680-01	2.092860 00			
MASS 6	-1.774100 00	-2.452470 01	-3.083570 01	-1.947340-02	2.079660-02	2.537870-02
	5.301910-01	-1.012070 02	-1.353660 02	7.049810-01	4.611280-01	2.365620-01
	8.495500-01	-9.845330 01	-1.413430 02	7.000620-01	4.564350-01	2.454450-01
	3.334740 02	-1.215000 02	1.068360 03	-1.406800 01	-4.855560-01	1.672730 01
	7.568040-01	-9.379550-02	2.588210 00			
MASS 7	-4.891240 01	-3.181350 01	-7.003260 01	-2.166540-02	2.590640-02	2.931840-02
	-1.326770 01	-6.455370 01	-1.214680 02	6.222380-01	3.239300-01	1.580260-01
	-1.244560 01	-8.115820 01	-1.287730 02	6.181440-01	3.204510-01	1.849530-01
	1.722740 03	1.513070 02	9.586610 02	3.156940 01	-2.349160 01	8.535260 00
	7.390640 00	5.496710-01	2.332540 00			
MASS 8	-2.420110 00	3.075260 01	-8.175580 01	4.3000-02	1.066320-02	6.315720-02
	-3.862600 01	-3.027150 01	-4.460640 02	5.3010-02	6.852860-01	4.513780 00
	-3.905440 01	-2.155480 01	-1.475330 02	5.261700-01	4.736310-01	4.537640 00
	-2.394060 03	-1.079560 02	1.010520 03	5.796460 01	5.832480 00	-1.574990 02
	-6.130110 00	-5.374440-01	2.636430 00			

Figure 85. Output Data Format.

TIME = 0.14000

	A XDOT U	Y V V	Z W W	PHI PHIDOT P	THETA THETADOT Q	PSI PSIDOT R
	UACCEL	VACCEL	WACCEL	PDOT	QDOT	RDOT
MASS 9	-2.246780 00	-3.042200 01	-5.943700 01	-2.916510-02	6.843460-03	2.781540-02
	-1.508220 01	-7.036700 01	-1.400950 02	1.371520 00	6.234240-01	5.797420-01
	-1.607430 01	-6.580190 01	-1.621880 02	1.371250 00	5.613360-01	6.408510-01
	-7.337330 02	-4.533550 02	1.204420 03	2.233270 00	5.829820 01	-6.552810-01
	-1.998330 01	-9.940330-01	2.907300 00			
MASS 10	-2.340240 01	-2.971910 01	-1.406110 01	-1.872480-02	1.204580-02	2.420870-02
	7.861440 00	-1.182090 02	-1.048550 02	5.719250-01	2.512330 00	5.508460-01
	6.273050 00	-1.453640 02	-1.090130 02	5.676990-01	2.505340 00	5.978060-01
	6.508990 07	2.924800 02	-1.324930 03	-3.101970 01	-4.414430 00	7.504090 00
	5.786350-01	7.255220-01	-3.641340 00			
MASS 11	-1.408210 00	-2.918540 01	-1.438820 01	-2.069040-02	1.628410-02	2.342370-02
	6.955140 00	-1.126740 02	-1.394260 02	7.104940-01	3.943200-01	9.124290-02
	6.624670 00	-1.099000 02	-1.516400 02	7.090080-01	3.022670-01	9.750740-02
	3.493110 02	-1.223560 02	1.040640 03	1.837500 01	6.261370 00	1.954090 01
	8.217710-01	-5.514560-02	2.488920 00			
MASS 12	6.487660 01	1.739160 01	-1.750720 01	-7.567180-02	5.052170-02	1.602980-02
	-3.061280 00	-1.150860 02	-2.251460 02	-3.959520 00	4.635010-01	1.345040-01
	6.420020 00	-9.762990 01	-7.331790 02	-3.966370 00	4.519850-01	1.690790-01
	-1.971370 03	1.142580 03	4.902580 02	2.468370 02	-4.516910 01	-5.599440 01
	-5.237570 00	5.668720-01	2.265710 00			
MASS 13	6.87 040 01	-7.254030 01	-1.368730 01	-6.458330-02	4.210670-02	2.134310-02
	1.071970 01	-1.016340 02	4.693570 01	-6.345460-01	4.714490-01	-1.821620-03
	6.345120 00	-1.046720 02	4.057470 01	-6.344710-01	4.705840-01	2.661040-02
	-6.120580 02	1.276240 03	-5.114460 03	-7.643390 01	-5.813970 00	-4.479860 01
	-2.046550 00	3.344520 00	-1.303140 01			
MASS 14	-2.474030 01	3.352010 01	-5.710330 00	-2.394670-02	-2.083170-02	-4.393490-03
	-1.334610 00	-2.924960 01	-9.792630 01	9.730470 06	-3.449400-02	-3.882300-01
	-3.114330 00	-5.684390 01	-9.928550 01	9.722380 00	-2.319020-02	-3.888400-01
	2.766370 01	-9.757430 02	9.990280 02	3.215280-03	1.239040 00	3.760740-17
	2.083030-02	-2.283920-02	9.994960-01			
MASS 15	-2.310070 01	-1.129530 02	-1.527630 01	-3.879380-01	-3.485620-02	-2.827820-02
	1.246400 01	-5.047250 02	-1.378360 02	-7.200280 00	-1.355450-01	-4.214320-02
	2.317670 01	-3.953040 02	-3.114460 02	-7.201830 00	-1.095410-02	-9.025910-02
	1.578470 01	2.096300 03	-2.492380 03	3.240300-03	-2.130350-01	-1.480300-16
	3.684760-02	-3.780250-01	9.250620-01			
MASS 16	6.774830 01	-2.757770 01	-1.551980 01	-3.044230-07	4.940170-02	1.927090-02
	6.841040 00	-1.062290 02	-3.608440 01	-3.043130 00	1.241520-01	-6.180880-02
	6.575970 00	-1.052020 02	-3.902240 01	-3.080070 00	1.279720-01	-5.786450-02
	1.288980 02	1.239540 03	-5.010420 03	6.789490 01	-1.702950 02	1.945110 01
	3.022250-01	2.898880 00	-1.214310 01			

Figure 85. (Continued)

RUN 31-32 DROP TEST CORRELATION 206/186 ENGINE													100
TIME = 0.16000													
	X XDOT U	Y YDOT V	Z ZDOT W	PHI PHIDOT P	THETA THETADOT Q	PSI PSIDOT R							
	ACCEL	YACCEL	ZACCEL	PDOT	QDOT	RDOT							
MASS 17	6.68480 01 -6.71180 01 6.22130 00 8.47470 01 -2.11410 02	3.77580 01 -5.32640 01 7.23890 01 6.81450 02 -7.36330 01	-1.28180 01 -1.58440 02 -1.39470 02 7.73370 02 6.74610 01	-8.290660-01 -6.94330 00 -6.53960 00 1.257660-01 3.233570-01	5.116340-02 6.70480 01 5.78020 01 1.509920 00 3.849920-02	1.501400-02 -1.468590-01 3.787400-01 -2.721190-16 -7.829300-03							
MASS 18	6.87150 01 2.25660 01 2.60210 01 4.42080 01 -3.84950 02	-6.782210 01 -2.034400 02 -2.078540 02 2.89280 01 3.17520 01	-2.048720 00 -4.859340 01 1.94320 01 -7.147630 02 9.476510-01	3.233570-01 -5.316740 01 -5.314540 00 -1.976630-01 -3.997890-02	3.849920-02 9.987470-01 9.28690 00 -1.130930 00 4.329920-02	-7.829300-03 -5.781200-02 -3.716090-01 2.366440-15 1.961530-02							
MASS 19	1.16300 02 5.05820 00 5.88650 00 1.25630 02 1.24300 01	-2.66340 01 -1.099180 02 -1.071560 02 2.267400 03 5.733780 00	-1.773340 01 -6.906340 01 -7.321440 02 7.979270 02 3.720740 00	-3.997890-02 -6.012650 00 -6.009920 00 1.805978 02 -3.997890-02	4.329920-02 9.497700-01 9.714240 01 -1.410960 02 4.329920-02	1.961530-02 -6.318640-02 -2.427250-02 1.397400 01 1.961530-02							
MASS 20	-2.08180 01 2.93870 01 2.07830 01 -5.82080 03 -1.48420 01	1.425480 01 -6.132680 01 -7.984570 01 -1.069600 03 -2.274440 00	-6.804040 01 -9.920900 01 -1.021570 02 7.690350 02 1.887240 00	-1.843080-02 6.511850-01 6.757110-01 -1.407040 00 -3.586840 01	-7.418530-02 -6.393630-01 -6.453340-01 1.234950 02 1.045040 02	1.436410-02 3.309120-01 3.181420-01 3.750260 01 2.822420 01							
MASS 21	-1.95390 01 4.85970 01 5.78160 01 -3.78300 03 -8.75220 00	-7.572410 01 -8.752200 01 -8.574800 01 -1.564400 03 -3.893030 00	-6.708920 01 -1.469180 02 -1.524910 02 2.345230 03 6.110300 00	-1.488400-02 3.757020-01 6.148150-01 -3.586840 01 6.110300 00	-6.250030-02 -8.457930-01 -8.458440-01 1.045040 02 -6.250030-02	1.448300-02 6.242200-01 6.088950-01 2.822420 01 1.448300-02							
MASS 22	-2.46520 01 -1.19510 01 -2.12310 01 -1.23980 02 1.67310 02	1.52340 01 -1.16490 02 -1.144040 02 -2.41410 02 -5.23190 01	-1.49310 01 -9.599500 01 -9.52150 01 4.408600 02 9.440280-01	-2.149180-02 4.788720-01 5.098700-01 -1.277250 01 9.440280-01	-6.730850-02 -8.454700-01 -8.568700-01 8.357080 01 -6.730850-02	2.406040-02 4.478400-01 4.284980-01 4.297910 01 2.406040-02							
MASS 23	-2.27390 01 1.90670 01 8.89630 00 2.92020 02 1.45570 00	-7.476860 01 -1.164020 02 -1.142240 02 -3.246010 02 -6.233640-01	-1.318470 01 -1.390930 02 -1.418930 02 1.270380 03 3.140380 00	-1.774360-02 5.712470-01 5.816570-01 -2.208490 01 3.140380 00	-5.798190-02 -8.246600-01 -8.276940-01 7.397640 01 -5.798190-02	1.809590-02 1.796240-01 1.628900-01 -7.801970 00 1.809590-02							
MASS 24	6.42480 01 -2.88450 01 -2.20530 01 -4.84050 02 -1.25400 00	1.397830 01 -1.82340 02 -1.89790 02 2.904510 03 8.74150 00	-1.134440 01 -2.21790 02 -2.21980 02 9.43210 02 1.58140 00	-5.191340-02 2.025410 00 2.026110 01 -3.031750 02 1.58140 00	-9.987400-02 5.086340-01 5.086510-01 -2.214770 01 -9.987400-02	2.372300-02 -1.399400-02 1.292190-02 7.904970 00 2.372300-02							

Figure 85. (Continued)

TIME = 0.16000

JUNE 8 9 18 0000													
MASS 25	X U	XROT	Y V	YROT	Z W	ZROT	PHI		THETA		PSI		
							PHIDROT P	PHI Q	THETADROT Q	THETA R	PSIDROT R	PSI S	
MASS 26	UROT	XACCEL	VROT	YACCEL	WROT	ZACCEL	PROT	PACCEL	QROT	QACCEL	RROT	RACCEL	
		6.660340	01	-7.591430	01	-6.753460	01	-5.895890	-02	4.171910	-02	2.490420	-02
		-1.466410	01	-1.660940	02	5.144460	01	-1.986540	-01	5.186540	-01	-2.285290	-02
		-2.094560	01	-1.683720	02	4.078470	01	-1.985560	00	5.190980	-01	7.748230	-03
		-7.347310	02	1.592620	03	-5.298070	03	1.579070	02	1.296380	00	-3.559680	00
		-1.845230	00	4.335330	00	-1.203130	01						
		-2.370340	01	-2.990470	01	-2.385880	01	-2.139070	-02	8.891090	-03	2.200410	-02
		-1.863460	01	-1.135900	02	-1.250880	02	5.212220	-01	3.139160	00	1.042850	-01
		-2.001630	01	-1.104570	02	-1.276630	02	4.907930	01	3.136210	00	1.714010	-01
		6.087720	02	-6.628750	02	8.266570	02			6.118920	01	7.141520	00
		5.889270	-01	-1.553800	00	2.155090	00						
		6.724040	01	-2.786650	01	-2.548030	01	-3.227090	-02	5.655880	-02	2.072320	-02
		-2.734080	00	-1.356300	02	-3.521800	01	-2.603510	00	1.208470	00	-1.576990	-01
		1.915300	00	-1.345220	02	-3.954850	01	-2.594600	00	1.212930	00	-1.183730	-01
		2.109680	03	1.798760	03	-5.253490	03	3.688530	01	-1.917910	02	-1.098730	01
		5.299980	00	4.393560	00	-1.271230	01						
		-2.373880	01	-3.007900	01	-3.171600	01	-2.237010	-02	6.561220	-03	2.190770	-02
		-4.413620	01	-1.115760	02	-1.844580	02	3.467860	-01	3.344450	00	2.548860	-02
		-4.535850	01	-1.064220	02	-1.871870	02	3.466190	-01	3.343050	00	1.002910	-01
		-1.010960	02	-5.931900	02	-5.922580	03	8.122920	01	7.816030	01	3.572040	00
		-1.855440	00	-1.280460	00	-1.504620	01						
		6.671000	01	-2.818710	01	-3.501200	01	-3.293750	-02	5.832960	-02	2.212800	-02
		-1.586140	01	-1.621940	02	-1.094040	02	-2.296150	00	1.689350	00	-4.244020	-02
		-1.303530	01	-1.580810	02	-1.158200	02	-2.292510	00	1.690480	00	-4.472400	-03
		3.227900	03	1.969660	03	1.347160	03	4.705270	00	-1.728290	02	-1.732330	01
		9.146090	00	4.416280	00	4.480010	00						
		-2.373770	01	-5.004410	01	-3.018080	01	-2.257820	-02	4.992240	-03	2.159860	-02
		-3.951640	01	-1.115250	02	-1.576750	02	3.535020	-01	3.378030	00	1.529320	-02
		-4.080700	01	-1.070510	02	-1.664220	02	3.534040	-01	3.378870	00	8.995300	-02
		1.505480	02	-6.018650	02	-8.253480	03	8.524400	01	6.534030	01	4.482140	00
		-9.885700	-01	-1.939960	00	-2.112310	01						
		6.677720	01	-2.815140	01	-3.370860	01	-3.259830	-02	5.875250	-02	2.212480	-02
		-1.452480	01	-1.590060	02	-1.262080	02	-2.293700	00	1.695710	00	-4.603280	-02
		-1.060050	01	-1.544230	02	-1.321530	02	-2.290990	00	1.696300	00	9.338400	-03
		3.574080	03	1.761180	03	-2.192790	03	-6.867590	00	-1.721900	02	-1.645390	01
		8.682260	00	3.778030	00	-4.717680	00						
1G(I,J),JG(I,J),SUMGE(1,1),SUMDE(1,1),SUMDE(2,1),SUMDE(3,1),SUMDE(4,1),SUMDE(5,1),SUMDE(6,1),SUMDE(7,1),SUMDE(8,1),SUMDE(9,1),SUMDE(10,1),SUMDE(11,1),SUMDE(12,1),SUMDE(13,1),SUMDE(14,1),SUMDE(15,1),SUMDE(16,1),SUMDE(17,1),SUMDE(18,1),SUMDE(19,1),SUMDE(20,1),SUMDE(21,1),SUMDE(22,1)													
1	2	-1.316500	03	9.422560	01	-4.297280	02	0.0		-6.998340	04	-1.735290	04
2	3	1.620810	02	1.420760	03	-3.063890	03	6.910400	03	-6.800300	05	-2.262900	05
3	4	-3.205860	02	1.535530	03	-2.662980	03	-2.643780	04	-8.042650	05	-2.436190	05
4	5	2.354140	02	5.672020	01	-6.737560	02	6.353610	04	-1.028930	04	-2.932140	05
5	6	-5.214160	02	-1.305070	03	2.121940	03	0.0		4.264330	04	-2.202220	04
6	7	4.484850	03	-9.189030	03	1.667720	04	-1.281430	05	-7.678130	05	-1.514530	05
7	8	3.529850	03	-1.799270	03	-1.790770	03	0.0		-2.614870	04	-4.482160	03
8	9	-3.248680	03	0.0		1.068330	03	0.0		3.321110	04	0.0	
9	10	-7.616340	02	0.0		-7.613280	02	0.0		-3.085030	04	0.0	
10	11	-4.725690	02	0.0		-7.630700	03	0.0		-1.770430	05	0.0	

Figure 85. (Continued)

5	23	-1.20350-03	0.0	-1.16150-03	-1.13310-04	-2.39270-05	0.0	-2.08240-05
6	9	3.06170-03	-3.64780-03	9.16150-03	-1.63520-05	-1.63520-05	-2.08240-05	-2.08240-05
7	11	1.74910-04	4.49170-03	9.32330-03	-1.60530-05	-2.68390-05	0.0	6.69190-04
8	9	0.0	0.0	0.0	-1.71360-04	0.0	0.0	0.0
10	11	3.85130-03	1.97020-03	6.62150-03	-8.41780-04	1.93630-05	-1.73440-05	-1.73440-05
10	14	-1.34930-03	4.15180-03	-2.78040-03	-7.66440-03	2.95480-02	-6.13930-04	-6.13930-04
10	15	-2.27210-03	-1.611850-02	-9.45980-02	7.00950-03	1.88080-05	2.47090-03	2.47090-03
10	22	5.72970-02	5.40460-02	1.23490-03	1.26770-03	1.78630-05	1.521150-03	1.521150-03
10	23	5.83700-03	3.49160-02	2.00290-03	0.0	1.06260-05	1.41830-03	1.41830-03
11	16	7.86690-02	-3.54270-03	5.61780-03	0.0	1.39180-05	8.81970-03	8.81970-03
12	16	-2.57680-02	1.07060-03	6.65220-03	-7.20870-03	3.09420-05	-1.05950-05	-1.05950-05
12	24	5.43720-02	4.78420-02	6.19260-02	-2.31570-03	-3.07660-04	7.04330-04	7.04330-04
13	16	-1.47750-03	3.46930-02	6.44130-03	7.87020-03	3.04360-03	1.80540-04	1.80540-04
13	25	-6.33070-02	5.40920-01	5.50740-02	-1.05690-03	2.22610-04	-2.46970-04	-2.46970-04
14	17	-1.07480-04	4.54360-01	1.51790-03	-4.07870-02	-1.57570-02	5.741200-03	5.741200-03
16	18	-9.14510-02	-3.35610-02	7.45130-02	-2.40510-03	6.71190-04	-6.03920-00	-6.03920-00
16	19	1.39660-01	1.38930-03	3.98850-03	-1.02430-04	9.17940-04	-6.42470-02	-6.42470-02
20	21	2.18350-02	4.08630-00	1.25760-03	0.0	-1.74910-04	4.96370-03	4.96370-03
20	22	2.40150-03	4.68840-02	7.59140-02	0.0	5.75370-03	5.60840-03	5.60840-03
21	23	5.67720-02	1.65070-03	8.75010-02	0.0	-5.44870-03	5.84430-04	5.84430-04
24	25	1.96120-01	3.91650-01	3.56020-02	5.55770-02	-1.75690-04	-4.69480-02	-4.69480-02
24	26	-1.55470-03	1.00180-02	9.35480-02	2.40580-03	-2.97070-04	-2.59880-04	-2.59880-04
26	27	-6.21570-02	6.01730-02	-1.21860-03	-1.52270-03	6.79120-04	-1.97080-04	-1.97080-04
26	28	-2.63110-03	-2.38660-02	2.84210-03	1.56740-02	-2.01320-04	-9.38070-03	-9.38070-03
27	29	-5.51710-02	1.94510-03	4.72900-02	-1.42640-03	1.76340-04	-9.36490-03	-9.36490-03
27	30	-2.88970-03	1.70840-01	1.70360-03	4.51500-02	-1.69240-04	-1.61930-04	-1.61930-04
27	31	-1.37890-03	1.08050-03	-7.36030-01	-1.43400-03	2.10000-04	-5.13260-03	-5.13260-03
16(11J), JG(11J), VEE(11J), VEE(11J), VEE(11J), VEE(11J), VEE(11J), VEE(11J), VEE(11J)								
1	2	-1.10490-02	-5.13910-01	1.54470-00	8.01660-02	-4.26890-02	-1.39360-02	-1.39360-02
2	3	2.64510-03	3.10150-01	1.31890-00	2.80480-02	-2.00480-02	-5.44190-03	-5.44190-03
3	4	-9.93140-04	-3.69890-02	8.86230-02	-7.24520-04	-5.92920-03	-2.46990-03	-2.46990-03
4	5	1.12040-04	4.10370-02	2.66070-02	-7.75490-07	-1.12360-03	-1.72620-03	-1.72620-03
4	7	-9.22840-03	-1.96470-01	2.74610-02	-2.14640-04	-3.60130-03	-2.55350-03	-2.55350-03
5	6	1.03070-02	-4.36960-02	4.99120-02	-1.43590-05	-4.26340-03	-3.66970-03	-3.66970-03
5	7	-5.83320-03	-1.56760-01	-3.43040-02	6.79420-04	7.77030-04	-2.24820-03	-2.24820-03
5	20	-2.03020-02	-5.32170-00	2.20490-02	-5.68160-02	-1.04430-05	-7.58840-02	-7.58840-02
5	21	-4.76020-03	3.96850-00	4.51460-02	7.60690-02	4.52250-02	4.52250-02	4.52250-02
5	22	-2.69110-03	-1.65050-01	-1.94830-01	-0.84770-02	2.61230-03	-4.06640-02	-4.06640-02
5	23	-6.85350-02	-5.56470-01	1.65400-01	7.36310-02	9.49960-04	-9.79970-03	-9.79970-03
6	11	1.00710-03	-1.51150-01	2.30280-01	-2.20820-03	-1.36940-02	1.17620-03	1.17620-03
6	12	3.29470-03	1.09790-02	3.94830-02	-2.08970-03	-4.46960-03	9.92640-05	9.92640-05
7	9	3.83310-01	-5.27430-02	-3.25840-02	-7.82500-03	-1.88060-02	-9.26460-05	-9.26460-05
8	9	-5.75000-01	-1.39600-01	4.85580-01	-3.73130-02	-1.49290-03	-6.87160-03	-6.87160-03
10	11	2.96630-03	-2.76490-04	-6.20640-02	-2.01370-03	4.24090-03	-8.00450-04	-8.00450-04
10	14	-2.81410-03	-1.27480-01	-1.51030-01	1.23170-04	3.88190-01	-2.73890-03	-2.73890-03
10	15	-4.73870-03	-7.32540-01	-4.51030-01	2.25560-05	3.83810-01	-2.16140-02	-2.16140-02
10	22	-6.45560-02	-5.39150-02	-4.84120-02	-8.13490-02	2.84590-03	-1.70350-03	-1.70350-03
10	23	6.56110-02	-2.43310-01	2.97060-02	6.99190-02	5.75590-04	-7.52770-03	-7.52770-03
11	16	4.43170-03	-1.61260-02	-2.10420-03	-9.600120-03	3.35870-02	-3.91360-04	-3.91360-04
12	16	-3.44860-03	4.00350-02	1.42240-00	9.72930-04	4.52620-02	2.153120-03	2.153120-03
12	24	3.67130-03	7.24730-01	2.90160-02	-7.76840-03	-7.637030-04	2.356680-02	2.356680-02
13	16	-8.31130-04	-1.49310-02	1.07340-00	7.376010-03	-3.42210-02	-7.494780-04	-7.494780-04
13	25	-4.27460-03	1.57950-01	3.01270-02	-3.545740-03	-8.39210-04	5.47980-03	5.47980-03
16	17	-4.46090-02	1.26410-01	-1.51070-01	-1.83490-05	3.58280-01	-3.85020-03	-3.85020-03
16	18	-3.81520-02	-9.92820-01	-1.51020-01	8.139040-05	3.81210-01	-3.88070-02	-3.88070-02
20	21	6.75250-03	6.71350-04	1.97330-01	-5.55790-03	-6.33070-03	2.604070-05	2.604070-05
20	22	4.81210-03	1.16140-01	8.85390-02	-1.16350-02	-2.704150-04	2.548640-03	2.548640-03
20	23	3.24220-02	5.77680-02	-1.84890-01	9.767500-03	4.68020-03	2.367670-03	2.367670-03
21	23	7.64530-03	-1.10940-02	-1.88770-01	1.48780-03	4.49960-03	-1.21250-03	-1.21250-03
24	25	2.211600-05	3.15670-02	-9.55950-01	8.436290-03	-7.108260-03	4.825490-04	4.825490-04
24	26	-1.99970-01	-1.61310-03	1.245010-02	-2.40580-03	-3.09520-03	-2.415020-03	-2.415020-03
16	27	-2.17190-02	4.08730-03	-1.91690-02	-1.52270-03	6.963220-03	-1.930020-03	-1.930020-03
26	28	-2.14090-00	-3.358260-03	3.07430-02	1.567440-04	-2.31350-03	-9.71630-04	-9.71630-04
27	29	-4.489120-01	1.87020-02	2.99560-03	-1.42640-03	1.73340-03	-7.494690-04	-7.494690-04
28	30	-3.67670-00	-1.344230-05	1.89120-02	4.585500-04	-1.862040-03	-1.17590-03	-1.17590-03

Figure 85. (Continued)

27 31 -1.75400 00 1.03958D-02 -2.86475D-03 2.12871D-03 -4.07982D-04
 1.5C(11),5C(11+2),5C(11+3)

10	0.0	0.0	0.0	2.93540D 00		
11	0.0	0.0	0.0	2.40482D 00		
12	0.0	0.0	0.0	4.21148D-03		
13	0.0	0.0	0.0	2.27189D 00		
14	0.0	0.0	0.0	2.10820D-03		
15	0.0	0.0	0.0	2.94423D-03		
16	0.0	0.0	0.0	1.45891D 00		
17	0.0	0.0	0.0	6.59394D-04		
18	0.0	0.0	0.0	8.75920D-02		
22	0.0	0.0	0.0	2.00759D 00		
23	0.0	0.0	0.0	3.79107D 00		

MASS DRI
 30 2.40809D 01
 31 1.05213D 01
 RUPTURE SUMMARY

I	J	TIME
10	15	0.04784
16	17	0.08624
10	14	0.11572
16	18	0.14320

Figure 85. (Continued)

23	4.650000 01	7.000000 01	7.000000 01	0.0	0.0	0.0
24	1.358000 02	2.688000 02	1.565000 02	0.0	0.0	0.0
25	1.358000 02	2.688000 02	1.565000 02	0.0	0.0	0.0
26	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
27	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
28	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
29	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
30	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
31	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0

1.800000 02

1	4.587500 02	0.0	1.322500 02	0.0	0.0	0.0
2	4.042500 02	0.0	7.200000 01	0.0	0.0	0.0
3	2.412200 02	0.0	4.156000 01	0.0	0.0	0.0
4	2.116000 02	0.0	3.845000 01	0.0	0.0	0.0
5	1.630000 02	0.0	3.845000 01	0.0	0.0	0.0
6	1.410000 02	0.0	3.845000 01	0.0	0.0	0.0
7	1.870200 02	0.0	7.200000 01	0.0	0.0	0.0
8	1.365000 02	0.0	9.000000 01	0.0	0.0	0.0
9	1.410000 02	0.0	6.706000 01	0.0	0.0	0.0
10	1.630000 02	0.0	2.200000 01	0.0	0.0	0.0
11	1.410000 02	0.0	2.200000 01	0.0	0.0	0.0
12	7.163000 01	-4.500000 01	2.200000 01	0.0	0.0	0.0
13	7.163000 01	4.500000 01	2.200000 01	0.0	0.0	0.0
14	1.630000 02	-5.080000 01	-7.000000 00	0.0	0.0	0.0
15	1.630000 02	5.080000 01	-7.000000 00	0.0	0.0	0.0
16	7.162000 01	0.0	2.200000 01	0.0	0.0	0.0
17	7.162000 01	-5.080000 01	-7.000000 00	0.0	0.0	0.0
18	7.162000 01	5.080000 01	-7.000000 00	0.0	0.0	0.0
19	2.300000 01	0.0	2.200000 01	0.0	0.0	0.0
20	1.630000 02	-4.500000 01	7.600000 01	0.0	0.0	0.0
21	1.630000 02	4.500000 01	7.600000 01	0.0	0.0	0.0
22	1.630000 02	-4.500000 01	2.200000 01	0.0	0.0	0.0
23	1.630000 02	4.500000 01	2.200000 01	0.0	0.0	0.0
24	7.162000 01	-4.500000 01	7.600000 01	0.0	0.0	0.0
25	7.162000 01	4.500000 01	7.600000 01	0.0	0.0	0.0
26	1.630000 02	0.0	3.200000 01	0.0	0.0	0.0
27	7.162000 01	0.0	3.200000 01	0.0	0.0	0.0
28	1.630000 02	0.0	4.200000 01	0.0	0.0	0.0
29	7.162000 01	0.0	4.200000 01	0.0	0.0	0.0
30	1.630000 02	0.0	4.200000 01	0.0	0.0	0.0
31	7.162000 01	0.0	4.200000 01	0.0	0.0	0.0

THERE ARE 0 1'S HAVING NON-ZERO ME OR PHI'S, THETA'S, PSI'S

THERE ARE 0 1'S HAVING NON-ZERO LC'S

10	3	1.700000 01	3.000000 01	3.300000 04		
11	3	1.700000 01	3.000000 01	1.100000 04		
12	3	1.700000 01	3.000000 01	1.100000 04		
13	3	1.700000 01	3.000000 01	1.100000 04		
14	3	2.250000 00	3.000000 01	2.200000 04		
15	3	2.250000 00	3.000000 01	2.200000 04		
16	3	1.700000 01	3.000000 01	3.300000 04		
17	3	2.250000 00	3.000000 01	2.200000 04		
18	3	2.250000 00	3.000000 01	2.200000 04		
19	3	1.700000 01	3.000000 01	1.100000 04		
20	3	1.700000 01	3.000000 01	1.100000 04		

10	3	1.000000 03	1.000000 00	2.000000 00	1.650000 04	1.650000 04
11	3	1.000000 03	1.000000 00	2.000000 00	5.500000 03	5.500000 03
12	3	1.000000 03	1.000000 00	2.000000 00	5.500000 03	5.500000 03
13	3	1.000000 03	1.000000 00	2.000000 00	5.500000 03	5.500000 03
14	3	2.000000 01	4.000000 01	6.000000 01	1.200000 00	4.400000 03

Figure 86. (Continued).

TJ, J, PHI(I, J), THETA(I, J), PSI(I, J) (INTERNAL BEAMS)									
15	3	2.500000	-01	4.000000	-01	4.000000	-01	4.000000	-01
16	3	1.000000 <td>-03</td> <td>1.000000</td> <td>00</td> <td>2.000000</td> <td>00</td> <td>3.500000</td> <td>00</td>	-03	1.000000	00	2.000000	00	3.500000	00
17	3	2.000000 <td>-01</td> <td>4.000000<td>-01</td><td>4.000000<td>-01</td><td>1.200000</td><td>00</td></td></td>	-01	4.000000 <td>-01</td> <td>4.000000<td>-01</td><td>1.200000</td><td>00</td></td>	-01	4.000000 <td>-01</td> <td>1.200000</td> <td>00</td>	-01	1.200000	00
18	3	2.000000 <td>-01</td> <td>4.000000<td>-01</td><td>4.000000<td>-01</td><td>1.200000</td><td>00</td></td></td>	-01	4.000000 <td>-01</td> <td>4.000000<td>-01</td><td>1.200000</td><td>00</td></td>	-01	4.000000 <td>-01</td> <td>1.200000</td> <td>00</td>	-01	1.200000	00
22	3	1.000000 <td>-03</td> <td>1.000000</td> <td>00</td> <td>2.000000</td> <td>00</td> <td>3.500000</td> <td>00</td>	-03	1.000000	00	2.000000	00	3.500000	00
23	3	1.000000 <td>-03</td> <td>1.000000</td> <td>00</td> <td>2.000000</td> <td>00</td> <td>3.500000</td> <td>00</td>	-03	1.000000	00	2.000000	00	3.500000	00
1									
1	1	2	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	3	4	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	4	5	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	4	7	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	5	6	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	5	7	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	5	20	0.0	0.0	0.0	0.0	0.0	0.0	0.0
9	5	21	0.0	0.0	0.0	0.0	0.0	0.0	0.0
10	5	22	0.0	0.0	0.0	0.0	0.0	0.0	0.0
11	5	23	0.0	0.0	0.0	0.0	0.0	0.0	0.0
12	6	9	0.0	0.0	0.0	0.0	0.0	0.0	0.0
13	6	11	0.0	0.0	0.0	0.0	0.0	0.0	0.0
14	7	9	0.0	0.0	0.0	0.0	0.0	0.0	0.0
15	8	9	0.0	0.0	0.0	0.0	0.0	0.0	0.0
16	10	11	0.0	0.0	0.0	0.0	0.0	0.0	0.0
17	10	14	0.0	0.0	0.0	0.0	0.0	0.0	0.0
18	10	15	0.0	0.0	0.0	0.0	0.0	0.0	0.0
19	10	22	0.0	0.0	0.0	0.0	0.0	0.0	0.0
20	10	23	0.0	0.0	0.0	0.0	0.0	0.0	0.0
21	11	16	0.0	0.0	0.0	0.0	0.0	0.0	0.0
22	12	16	0.0	0.0	0.0	0.0	0.0	0.0	0.0
23	12	24	0.0	0.0	0.0	0.0	0.0	0.0	0.0
24	13	16	0.0	0.0	0.0	0.0	0.0	0.0	0.0
25	13	25	0.0	0.0	0.0	0.0	0.0	0.0	0.0
26	16	17	0.0	0.0	0.0	0.0	0.0	0.0	0.0
27	16	18	0.0	0.0	0.0	0.0	0.0	0.0	0.0
28	16	19	0.0	0.0	0.0	0.0	0.0	0.0	0.0
29	20	21	0.0	0.0	0.0	0.0	0.0	0.0	0.0
30	20	22	0.0	0.0	0.0	0.0	0.0	0.0	0.0
31	21	23	0.0	0.0	0.0	0.0	0.0	0.0	0.0
32	24	24	0.0	0.0	0.0	0.0	0.0	0.0	0.0
33	10	26	0.0	0.0	0.0	0.0	0.0	0.0	0.0
34	18	21	0.0	0.0	0.0	0.0	0.0	0.0	0.0
35	26	28	0.0	0.0	0.0	0.0	0.0	0.0	0.0
36	27	29	0.0	0.0	0.0	0.0	0.0	0.0	0.0
37	26	30	0.0	0.0	0.0	0.0	0.0	0.0	0.0
38	27	31	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1									
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2	3	0.0	0.0	0.0	0.0	0.0	0.0	0.0
1	2								

Figure 86. (Continued).

[illegible]

Figure 86. (Continued).

0.0	0.0	0.0	2.123000 04	0.0	0.0	4.774000 05	0.0
0.0	0.0	0.0	0.0	1.047000 06	0.0	1.433000 07	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	8.604000 07
12 24							
1.481000 05	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	5.411000 03	0.0	0.0	0.0	0.0	0.0	-1.461000 05
0.0	0.0	7.377000 04	0.0	0.0	1.992000 06	0.0	0.0
0.0	0.0	0.0	2.981000 05	0.0	0.0	0.0	0.0
0.0	0.0	1.992000 06	0.0	0.0	7.170000 07	0.0	0.0
0.0	-1.461000 05	0.0	0.0	0.0	0.0	0.0	5.259000 06
13 16							
1.778000 06	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.275000 05	0.0	0.0	0.0	0.0	0.0	-2.868000 06
0.0	0.0	2.123000 04	0.0	0.0	4.774000 05	0.0	0.0
0.0	0.0	0.0	1.067000 06	0.0	0.0	0.0	0.0
0.0	0.0	4.774000 05	0.0	0.0	1.433000 07	0.0	0.0
0.0	-2.868000 06	0.0	0.0	0.0	0.0	0.0	8.604000 07
13 25							
1.481000 05	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	5.411000 03	0.0	0.0	0.0	0.0	0.0	-1.461000 05
0.0	0.0	7.377000 04	0.0	0.0	1.992000 06	0.0	0.0
0.0	0.0	0.0	2.981000 05	0.0	0.0	0.0	0.0
0.0	0.0	1.992000 06	0.0	0.0	7.170000 07	0.0	0.0
0.0	-1.461000 05	0.0	0.0	0.0	0.0	0.0	5.259000 06
16 17							
2.397000 05	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.429000 03	0.0	0.0	0.0	0.0	0.0	-3.516000 04
0.0	0.0	1.429000 03	0.0	0.0	3.516000 04	0.0	0.0
0.0	0.0	0.0	2.944000 07	0.0	0.0	0.0	0.0
0.0	0.0	3.516000 04	0.0	0.0	1.154000 06	0.0	0.0
0.0	-3.516000 04	0.0	0.0	0.0	0.0	0.0	1.154000 06
16 18							
2.397000 05	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.429000 03	0.0	0.0	0.0	0.0	0.0	-3.516000 04
0.0	0.0	1.429000 03	0.0	0.0	3.516000 04	0.0	0.0
0.0	0.0	0.0	2.944000 07	0.0	0.0	0.0	0.0
0.0	0.0	3.516000 04	0.0	0.0	1.154000 06	0.0	0.0
0.0	-3.516000 04	0.0	0.0	0.0	0.0	0.0	1.154000 06
16 19							
2.057000 05	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	8.933000 06	0.0	0.0	0.0	0.0	0.0	-2.172000 08
0.0	0.0	9.188000 04	0.0	0.0	2.234000 06	0.0	0.0
0.0	0.0	0.0	1.072000 06	0.0	0.0	0.0	0.0
0.0	0.0	2.234000 06	0.0	0.0	7.240000 07	0.0	0.0
0.0	-2.172000 08	0.0	0.0	0.0	0.0	0.0	7.039000 09
20 21							
4.445000 04	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	2.800000 03	0.0	0.0	0.0	0.0	0.0	-1.260000 05
0.0	0.0	1.447000 04	0.0	0.0	7.410000 05	0.0	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	7.410000 05	0.0	0.0	4.445000 07	0.0	0.0
0.0	-1.260000 05	0.0	0.0	0.0	0.0	0.0	7.550000 06
20 22							
7.407000 04	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	7.621000 04	0.0	0.0	0.0	0.0	0.0	-2.058000 06
0.0	0.0	1.296000 04	0.0	0.0	3.498000 05	0.0	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	3.498000 05	0.0	0.0	1.259000 07	0.0	0.0
0.0	-2.058000 06	0.0	0.0	0.0	0.0	0.0	7.407000 07
21 23							
7.407000 04	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	7.621000 04	0.0	0.0	0.0	0.0	0.0	-2.058000 06
0.0	0.0	1.296000 04	0.0	0.0	3.498000 05	0.0	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	3.498000 05	0.0	0.0	1.259000 07	0.0	0.0
0.0	-2.058000 06	0.0	0.0	0.0	0.0	0.0	1.259000 07

Figure 86. (Continued).

24 25	0.0	-7.058000 06	0.0	0.0	0.0	7.407000 07
0.0	8.890000 03	0.0	0.0	0.0	0.0	0.0
0.0	3.940000 03	0.0	0.0	0.0	0.0	-1.792000 05
0.0	0.0	3.045000 02	0.0	0.0	1.370000 04	0.0
0.0	0.0	0.0	0.0	6.550000 04	0.0	0.0
0.0	0.0	1.370000 04	0.0	0.0	8.220000 05	0.0
10 26	0.0	-1.792000 05	0.0	0.0	0.0	1.075000 07
0.0	1.000000 04	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05
0.0	0.0	1.000000 05	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0
16 27	0.0	1.000000 05	0.0	0.0	1.000000 07	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07
0.0	0.0	0.0	0.0	0.0	0.0	0.0
3.000000 04	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05
0.0	0.0	1.000000 05	0.0	1.000000 06	0.0	0.0
0.0	0.0	1.000000 05	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07
26 28	0.0	0.0	0.0	0.0	0.0	0.0
1.225000 03	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05
0.0	0.0	1.000000 05	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0
0.0	0.0	1.000000 05	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07
27 29	0.0	0.0	0.0	0.0	0.0	0.0
1.225000 03	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05
0.0	0.0	1.000000 05	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0
0.0	0.0	1.000000 05	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07
26 30	0.0	0.0	0.0	0.0	0.0	0.0
7.840000 02	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05
0.0	0.0	1.000000 05	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0
0.0	0.0	1.000000 05	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07
27 31	0.0	0.0	0.0	0.0	0.0	0.0
7.840000 02	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05
0.0	0.0	1.000000 05	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0
0.0	0.0	1.000000 05	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07
1J,1J,CBAR(1,1)	0.0	0.0	0.0	0.0	0.0	0.0
1 1 2	5.000000-02	0.0	0.0	0.0	0.0	0.0
2 2 3	5.000000-02	0.0	0.0	0.0	0.0	0.0
3 3 4	5.000000-02	0.0	0.0	0.0	0.0	0.0
4 4 5	5.000000-02	0.0	0.0	0.0	0.0	0.0
5 4 7	5.000000-02	0.0	0.0	0.0	0.0	0.0
6 5 6	5.000000-02	0.0	0.0	0.0	0.0	0.0
7 5 7	5.000000-02	0.0	0.0	0.0	0.0	0.0
8 5 20	5.000000-02	0.0	0.0	0.0	0.0	0.0
9 5 21	5.000000-02	0.0	0.0	0.0	0.0	0.0
10 5 22	1.000000-01	0.0	0.0	0.0	0.0	0.0
11 5 23	1.000000-01	0.0	0.0	0.0	0.0	0.0
12 6 9	5.000000-02	0.0	0.0	0.0	0.0	0.0
13 6 11	1.000000-01	0.0	0.0	0.0	0.0	0.0
14 7 9	5.000000-02	0.0	0.0	0.0	0.0	0.0

Figure 86. (Continued).

15	8	9	5.00000D-02
16	10	11	1.00000D-01
17	10	14	1.00000D-01
18	16	15	1.00000D-01
19	10	22	1.00000D-01
20	15	23	1.50000D-01
21	11	16	1.00000D-01
22	12	16	1.00000D-01
23	12	25	5.60000D-02
24	13	16	1.00000D-01
25	13	25	5.00000D-02
26	16	17	1.00000D-01
27	16	18	1.00000D-01
28	16	19	1.00000D-01
29	20	21	5.00000D-02
30	20	22	5.00000D-02
31	21	23	5.00000D-02
32	24	25	5.00000D-02
33	10	26	5.00000D-02
34	16	27	5.00000D-02
35	24	28	2.47000D-01
36	27	29	2.34000D-01
37	26	30	1.81800D-01
38	27	31	1.75200D-01

NR TABLE SPECS. 1-J-L-M-P			
2	3	2	4
2	3	3	4
2	3	5	4
2	3	4	4
4	7	7	4
4	7	2	4
4	7	3	4
4	7	4	4
4	7	5	4
4	7	6	4
5	7	1	4
5	7	2	4
5	7	3	4
5	7	4	4
5	7	5	4
5	7	6	4
8	9	1	4
8	9	2	4
8	9	3	4
8	9	4	4
8	9	5	4
8	9	6	4
10	14	1	4
10	14	2	4
10	14	3	4
10	14	4	4
10	14	5	4
10	14	6	4
10	15	1	4
10	15	2	4
10	15	3	4
10	15	4	4
10	15	5	4
10	15	6	4
16	17	1	4
16	17	2	4
16	17	3	4
16	17	4	4
16	17	5	4
16	17	6	4

Figure 86. (Continued).

14	18	1	4
14	18	2	4
14	18	3	4
14	18	4	4
14	18	5	4
14	18	6	4

NR TABLE FOR I,J,L = 2 3 2			
1	0.0	1.00000E 00	
2	1.00000E 00	1.00000E 00	
3	4.00000E 00	-1.00000E 00	
4	6.00000E 00	-1.00000E 00	
5	8.00000E 00	-1.00000E 00	
6	1.00000E 01	-1.00000E 00	

NR TABLE FOR I,J,L = 2 3 3			
1	0.0	1.00000E 00	
2	2.00000E 00	1.00000E 00	
3	4.00000E 00	-1.00000E 00	
4	6.00000E 00	-1.00000E 00	
5	8.00000E 00	-1.00000E 00	
6	1.00000E 01	-1.00000E 00	

NR TABLE FOR I,J,L = 2 3 5			
1	0.0	1.00000E 00	
2	1.00000E 00	1.00000E 00	
3	2.00000E 00	0.0	
4	3.00000E 00	-1.00000E 00	
5	4.00000E 00	-1.00000E 00	
6	5.00000E 00	-1.00000E 00	

NR TABLE FOR I,J,L = 2 3 6			
1	0.0	1.00000E 00	
2	1.00000E 00	1.00000E 00	
3	2.00000E 00	0.0	
4	3.00000E 00	-1.00000E 00	
5	4.00000E 00	-1.00000E 00	
6	5.00000E 00	-1.00000E 00	

NR TABLE FOR I,J,L = 4 7 1			
1	0.0	1.00000E 00	
2	1.00000E -01	1.00000E 00	
3	2.00000E -01	-7.00000E -01	
4	3.00000E -01	-7.00000E -01	
5	4.00000E -01	0.0	
6	5.00000E -01	0.0	

NR TABLE FOR I,J,L = 4 7 2			
1	0.0	1.00000E 00	
2	6.67000E -02	1.00000E 00	
3	1.33400E -01	1.00000E 00	
4	2.00100E -01	1.00000E 00	
5	2.66800E -01	0.0	
6	3.33500E -01	0.0	

NR TABLE FOR I,J,L = 4 7 3			
1	0.0	1.00000E 00	
2	6.67000E -02	1.00000E 00	
3	1.33400E -01	1.00000E 00	
4	2.00100E -01	1.00000E 00	
5	2.66800E -01	0.0	
6	3.33500E -01	0.0	

NR TABLE FOR I,J,L = 4 7 4			
1	0.0	1.00000E 00	
2	1.00000E -01	1.00000E 00	

Figure 86. (Continued).

3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00
5	4.00000E-01	0.0
6	5.00000E-01	0.0

KR TABLE FOR I,J,L = 4 7 5		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00
5	4.00000E-01	0.0
6	5.00000E-01	0.0

KR TABLE FOR I,J,L = 4 7 6		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00
5	4.00000E-01	0.0
6	5.00000E-01	0.0

KR TABLE FOR I,J,L = 5 7 1		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	-7.00000E-01
4	3.00000E-01	-7.00000E-01
5	4.00000E-01	0.0
6	5.00000E-01	0.0

KR TABLE FOR I,J,L = 5 7 2		
1	0.0	1.00000E 00
2	6.67000E-02	1.00000E 00
3	1.33400E-01	1.00000E 00
4	2.00100E-01	1.00000E 00
5	2.66800E-01	0.0
6	3.33500E-01	0.0

KR TABLE FOR I,J,L = 5 7 3		
1	0.0	1.00000E 00
2	6.67000E-02	1.00000E 00
3	1.33400E-01	1.00000E 00
4	2.00100E-01	1.00000E 00
5	2.66800E-01	0.0
6	3.33500E-01	0.0

KR TABLE FOR I,J,L = 5 7 4		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00
5	4.00000E-01	0.0
6	5.00000E-01	0.0

KR TABLE FOR I,J,L = 5 7 5		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00
5	4.00000E-01	0.0
6	5.00000E-01	0.0

KR TABLE FOR I,J,L = 5 7 6		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00

Figure 86. (Continued).

5	4.00000E-01	0.0
6	5.00000E-01	0.0
KR TABLE FOR I,J,L = 8 9 1		
1	0.0	7.85000E-03
2	4.00000E-01	7.85000E-03
3	9.60000E-01	1.00000E 00
4	1.44000E 00	1.00000E 00
5	1.92000E 00	0.0
6	2.40000E 00	0.0
KR TABLE FOR I,J,L = 8 9 2		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	0.0
6	1.00000E 00	0.0
KR TABLE FOR I,J,L = 8 9 3		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	0.0
6	1.00000E 00	0.0
KR TABLE FOR I,J,L = 8 9 4		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00
5	4.00000E-01	0.0
6	5.00000E-01	0.0
KR TABLE FOR I,J,L = 8 9 5		
1	0.0	1.00000E 00
2	5.00000E-03	1.00000E 00
3	1.00000E-02	1.00000E 00
4	1.50000E-02	1.00000E 00
5	2.00000E-02	0.0
6	2.50000E-02	0.0
KR TABLE FOR I,J,L = 8 9 6		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	2.00000E-01	1.00000E 00
4	3.00000E-01	1.00000E 00
5	4.00000E-01	0.0
6	5.00000E-01	0.0
KR TABLE FOR I,J,L = 10 14 1		
1	0.0	1.00000E 00
2	1.00000E-01	1.00000E 00
3	1.00000E 00	1.00000E 00
4	1.50000E 00	1.00000E 00
5	2.00000E 00	-3.33000E 00
6	2.50000E 00	-3.33000E 00
KR TABLE FOR I,J,L = 10 14 2		
1	0.0	1.00000E 00
2	4.00000E 00	1.00000E 00
3	8.00000E 00	1.00000E 00
4	1.20000E 01	1.00000E 00
5	1.60000E 01	-3.33000E 00
6	2.00000E 01	-3.33000E 00

Figure 86. (Continued).

KR TABLE FOR I,J,L = 10 14 3					
1	0.0	1.00000E 00			
2	4.00000E 00	1.00000E 00			
3	7.00000E 00	1.00000E 00			
4	1.20000E 01	1.00000E 00			
5	1.40000E 01	-3.33000E 00			
6	2.00000E 01	-3.33000E 00			
KR TABLE FOR I,J,L = 10 14 4					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	4.00000E -01	1.00000E 00			
4	6.00000E -01	1.00000E 00			
5	8.00000E -01	-3.33000E 00			
6	1.00000E 00	-3.33000E 00			
KR TABLE FOR I,J,L = 10 14 5					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	4.00000E -01	1.00000E 00			
4	6.00000E -01	1.00000E 00			
5	8.00000E -01	-3.33000E 00			
6	1.00000E 00	-3.33000E 00			
KR TABLE FOR I,J,L = 10 14 6					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	4.00000E -01	1.00000E 00			
4	6.00000E -01	1.00000E 00			
5	8.00000E -01	-3.33000E 00			
6	1.00000E 00	-3.33000E 00			
KR TABLE FOR I,J,L = 10 15 1					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	1.00000E 00	1.00000E 00			
4	1.50000E 00	1.00000E 00			
5	2.00000E 00	-3.33000E 00			
6	2.50000E 00	-3.33000E 00			
KR TABLE FOR I,J,L = 10 15 2					
1	0.0	1.00000E 00			
2	4.00000E 00	1.00000E 00			
3	8.00000E 00	1.00000E 00			
4	1.20000E 01	1.00000E 00			
5	1.40000E 01	-3.33000E 00			
6	2.00000E 01	-3.33000E 00			
KR TABLE FOR I,J,L = 10 15 3					
1	0.0	1.00000E 00			
2	4.00000E 00	1.00000E 00			
3	8.00000E 00	1.00000E 00			
4	1.20000E 01	1.00000E 00			
5	1.40000E 01	-3.33000E 00			
6	2.00000E 01	-3.33000E 00			
KR TABLE FOR I,J,L = 10 15 4					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	4.00000E -01	1.00000E 00			
4	6.00000E -01	1.00000E 00			
5	8.00000E -01	-3.33000E 00			
6	1.00000E 00	-3.33000E 00			
KR TABLE FOR I,J,L = 10 15 5					

Figure 86. (Continued).

1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 10 15 6		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 16 17 1		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 16 17 2		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 16 17 3		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 16 17 4		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 16 17 5		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 16 17 6		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00
NR TABLE FOR I,J,L = 16 18 1		
1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00

Figure 86. (Continued).

3	1.00000E 00	1.00000E 00			
4	1.30000E 00	1.00000E 00			
5	2.00000E 00	-3.33000E 00			
6	2.50000E 00	-3.33000E 00			

KR TABLE FOR I,J,L = 16 18 2					
1	0.0	1.00000E 00			
2	4.00000E 00	1.00000E 00			
3	8.00000E 00	1.00000E 00			
4	1.20000E 01	1.00000E 00			
5	1.60000E 01	-3.33000E 00			
6	2.00000E 01	-3.33000E 00			

KP TABLE FOR I,J,L = 16 18 3					
1	0.0	1.00000E 00			
2	4.00000E 00	1.00000E 00			
3	8.00000E 00	1.00000E 00			
4	1.20000E 01	1.00000E 00			
5	1.60000E 01	-3.33000E 00			
6	2.00000E 01	-3.33000E 00			

KR TABLE FOR I,J,L = 16 18 4					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	4.00000E -01	1.00000E 00			
4	8.00000E -01	1.00000E 00			
5	8.00000E -01	-3.33000E 00			
6	1.00000E 00	-3.33000E 00			

KR TABLE FOR I,J,L = 16 18 5					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	4.00000E -01	1.00000E 00			
4	6.00000E -01	1.00000E 00			
5	8.00000E -01	-3.33000E 00			
6	1.00000E 00	-3.33000E 00			

KR TABLE FOR I,J,L = 16 18 6					
1	0.0	1.00000E 00			
2	2.00000E -01	1.00000E 00			
3	4.00000E -01	1.00000E 00			
4	6.00000E -01	1.00000E 00			
5	8.00000E -01	-3.33000E 00			
6	1.00000E 00	-3.33000E 00			

I,J,L,JVMAM I,J,L-1					
1	2	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
2	3	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
3	4	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
4	5	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
5	6	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
6	7	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
7	8	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
8	9	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
9	10	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
10	11	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
11	12	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
12	13	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
13	14	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
14	15	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
15	16	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
16	17	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
17	18	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
18	19	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
19	20	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
20	21	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01

Figure 86. (Continued).

	1	2	3	4	5	6	7	8
21 11 16	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
22 12 16	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
23 12 24	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
24 13 16	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
25 13 25	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
26 16 17	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
27 16 18	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
28 16 19	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
29 20 21	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
30 20 22	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
31 21 23	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
32 24 25	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
33 10 24	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
34 16 27	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
35 24 28	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
36 27 29	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
37 24 30	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01
38 27 31	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01	1.000000 01

PLOT CARDS WHICH ARE NOT ALL BLANK

	1	2	3	4	5	6	7	8
12345678901234567890123456789012345678901234567890								
5	11	11	11	11	11	11	11	11
6	11	11	11	11	11	11	11	11
7	11	11	11	11	11	11	11	11
8	11	11	11	11	11	11	11	11
9	11	11	11	11	11	11	11	11
10	11	11	11	11	11	11	11	11
11	11	11	11	11	11	11	11	11
12	11	11	11	11	11	11	11	11
13	11	11	11	11	11	11	11	11
14	11	11	11	11	11	11	11	11
15	11	11	11	11	11	11	11	11
16	11	11	11	11	11	11	11	11
17	11	11	11	11	11	11	11	11
18	11	11	11	11	11	11	11	11
19	11	11	11	11	11	11	11	11
20	11	11	11	11	11	11	11	11
21	11	11	11	11	11	11	11	11
22	11	11	11	11	11	11	11	11

INOP = 0

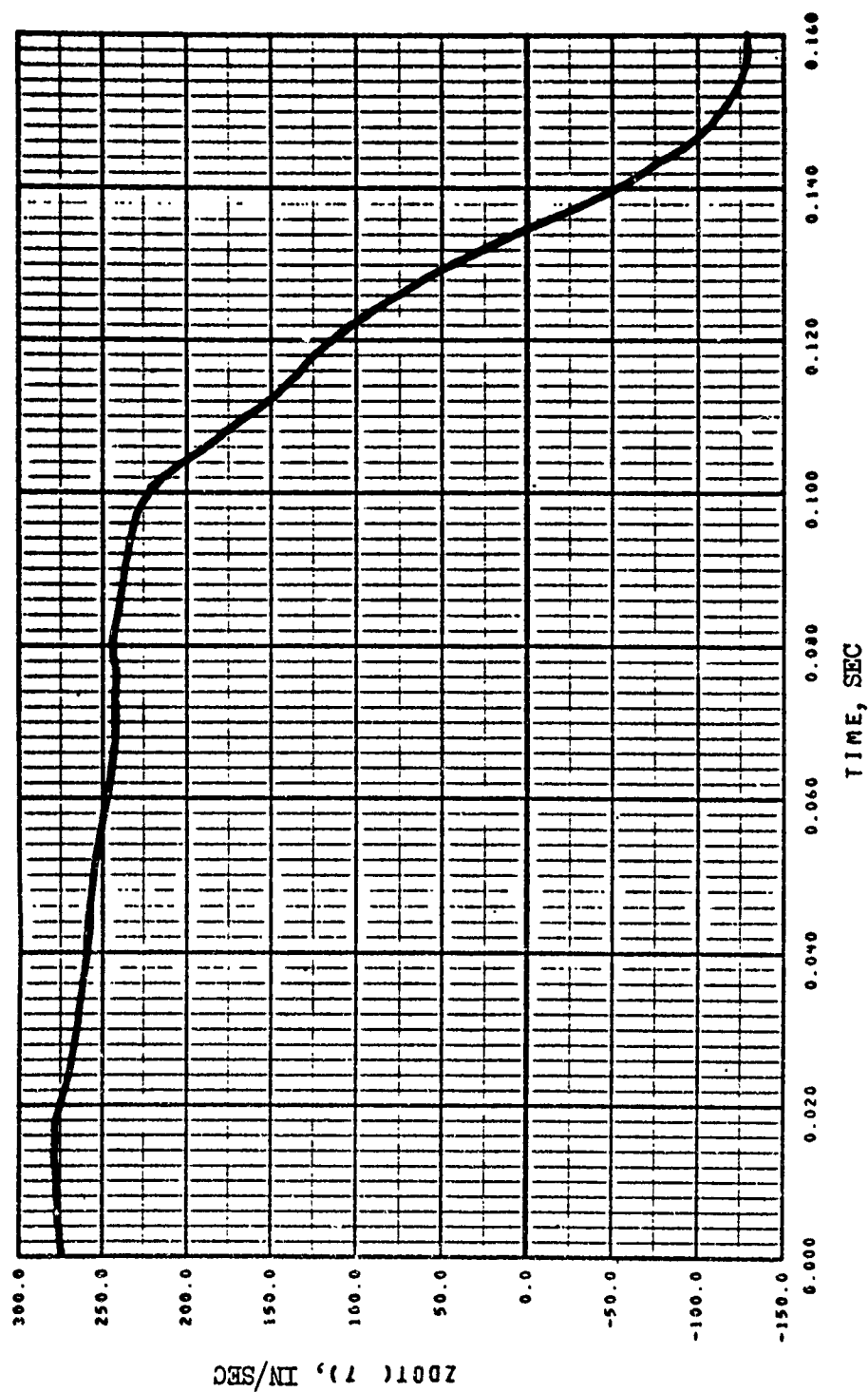
UNIT(I) INDICATORS

	1	2	3	4	5	6	7	8
12345678901234567890123456789012345678901234567890								
1	11	11	11	11	11	11	11	11
2	11	11	11	11	11	11	11	11
3	11	11	11	11	11	11	11	11
4	11	11	11	11	11	11	11	11
5	11	11	11	11	11	11	11	11
6	11	11	11	11	11	11	11	11
7	11	11	11	11	11	11	11	11
8	11	11	11	11	11	11	11	11
9	11	11	11	11	11	11	11	11
10	11	11	11	11	11	11	11	11
11	11	11	11	11	11	11	11	11
12	11	11	11	11	11	11	11	11
13	11	11	11	11	11	11	11	11
14	11	11	11	11	11	11	11	11
15	11	11	11	11	11	11	11	11
16	11	11	11	11	11	11	11	11
17	11	11	11	11	11	11	11	11
18	11	11	11	11	11	11	11	11
19	11	11	11	11	11	11	11	11
20	11	11	11	11	11	11	11	11
21	11	11	11	11	11	11	11	11
22	11	11	11	11	11	11	11	11

Figure 86. (Continued).

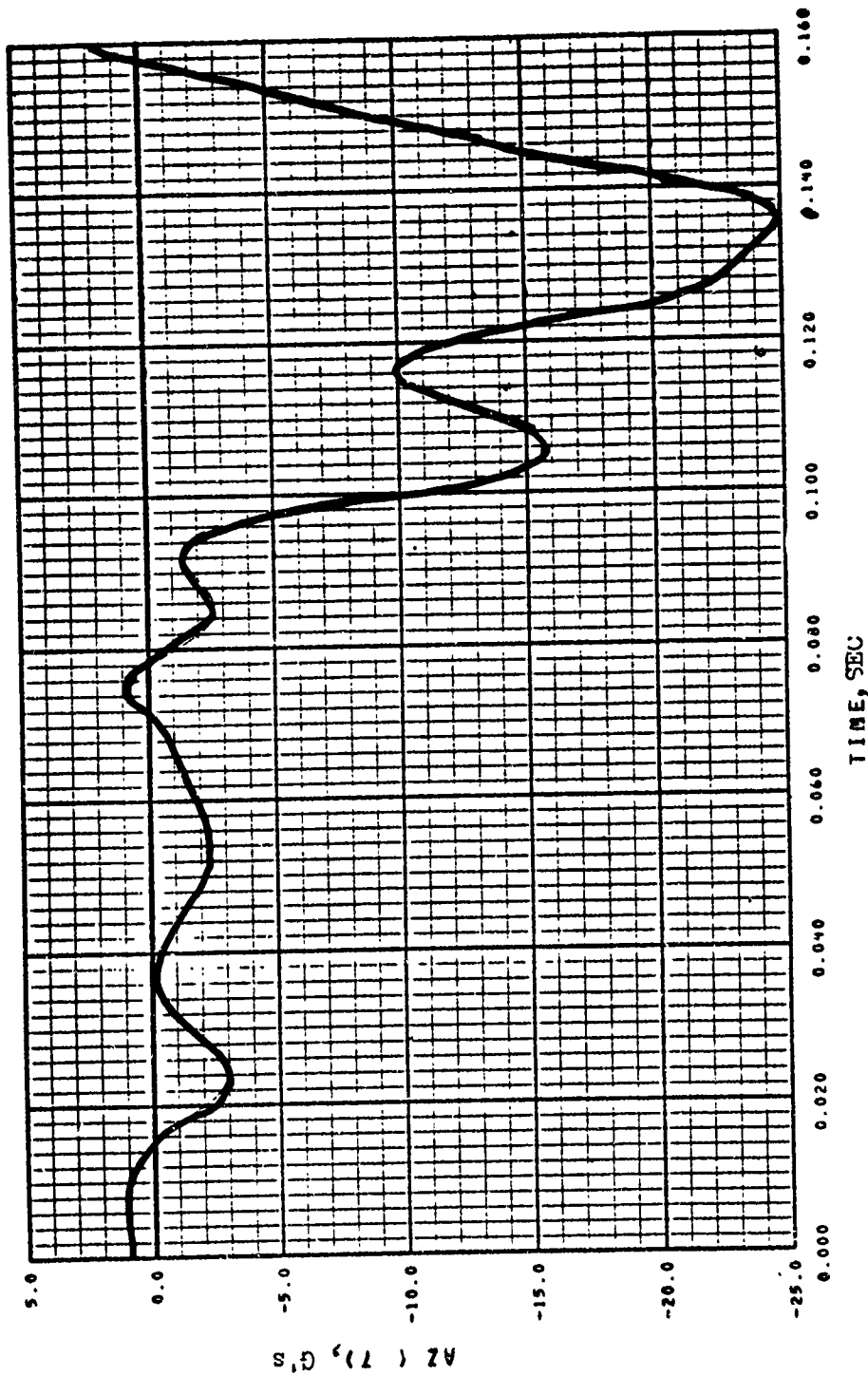
25	1.570610 00	0.0	
26	0.0	1.570570 00	
27	1.570610 00	0.0	
28	-5.187220-01	1.570800 00	
29	-5.187220-01	-1.570800 00	
30	0.0	0.0	
31	-1.570800 00	0.0	
32	0.0	-1.570800 00	
33	1.570800 00	0.0	
34	1.570800 00	0.0	
35	1.570800 00	0.0	
36	1.570800 00	0.0	
37	1.570800 00	0.0	
38	1.570800 00	0.0	

Figure 86. (Continued).



RUN 31-52 DROP TEST CORRELATION 20G/18G ENGINE

Figure 87. Engine Mass Vertical Velocity Time History Plot, Combined Vertical and Lateral Impact Sample Case.



RUN 31-52 DROP TEST CORRELATION 206/186 ENGINE

Figure 88. Engine Mass Vertical Acceleration Time History Plot, Combined Vertical and Lateral Impact Sample Case.

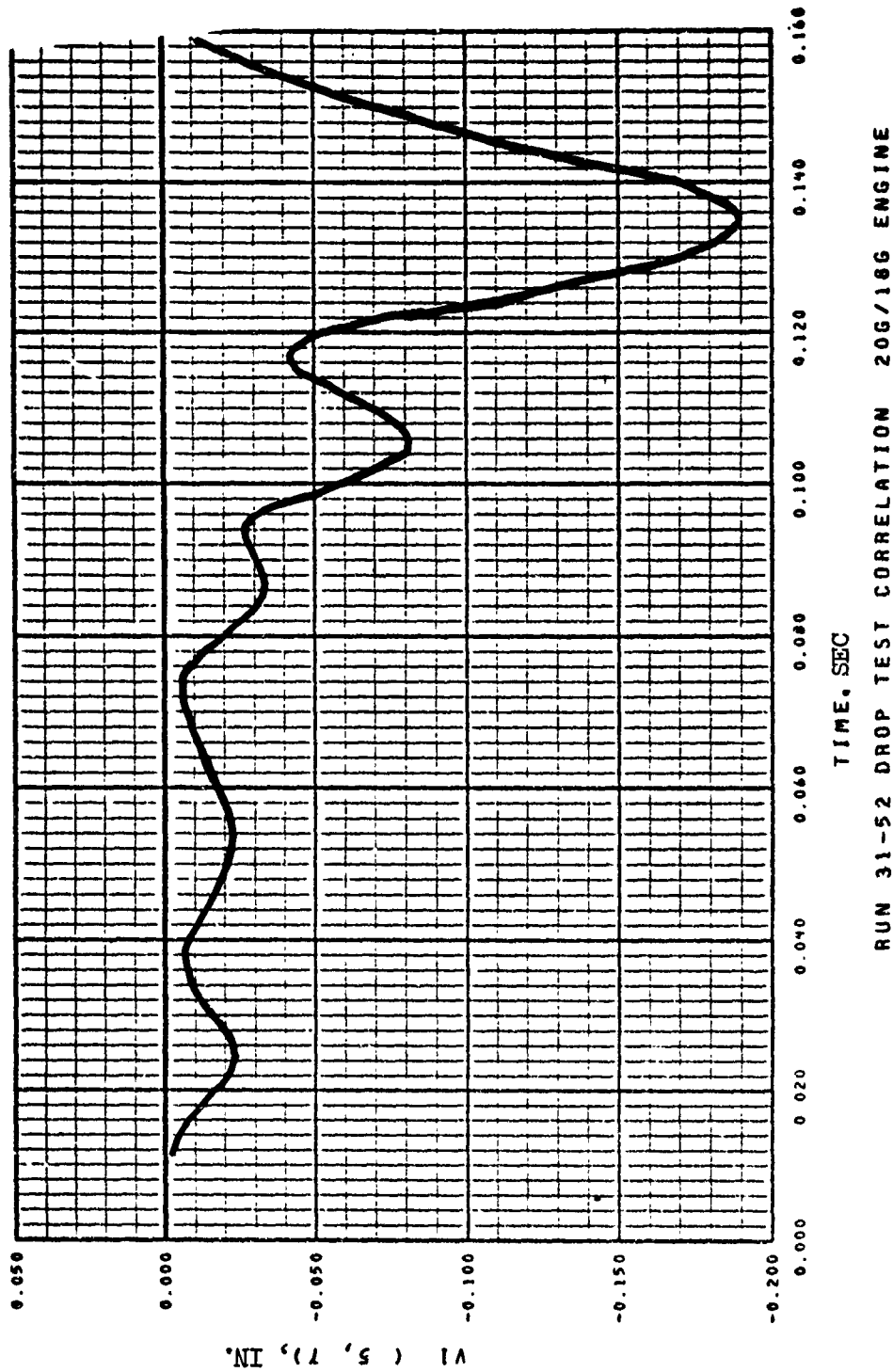
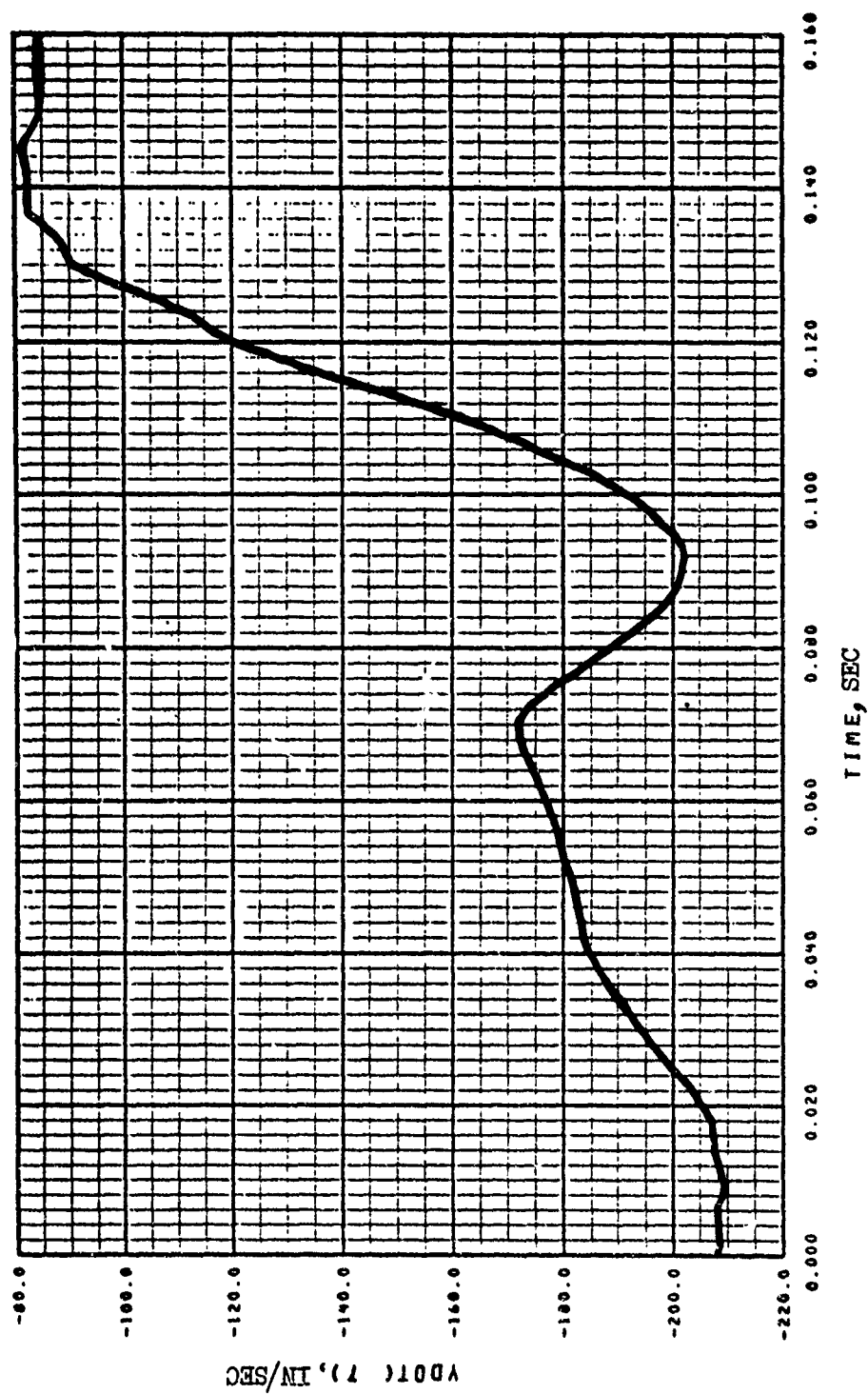


Figure 89. Engine Mount Vertical Deflection Time History Plot,
Combined Vertical and Lateral Impact Sample Case.



RUN 31-52 DROP TEST CORRELATION 206/186 ENGINE

Figure 90. Engine Mass Lateral Velocity Time History Plot,
Combined Vertical and Lateral Impact Sample Case.

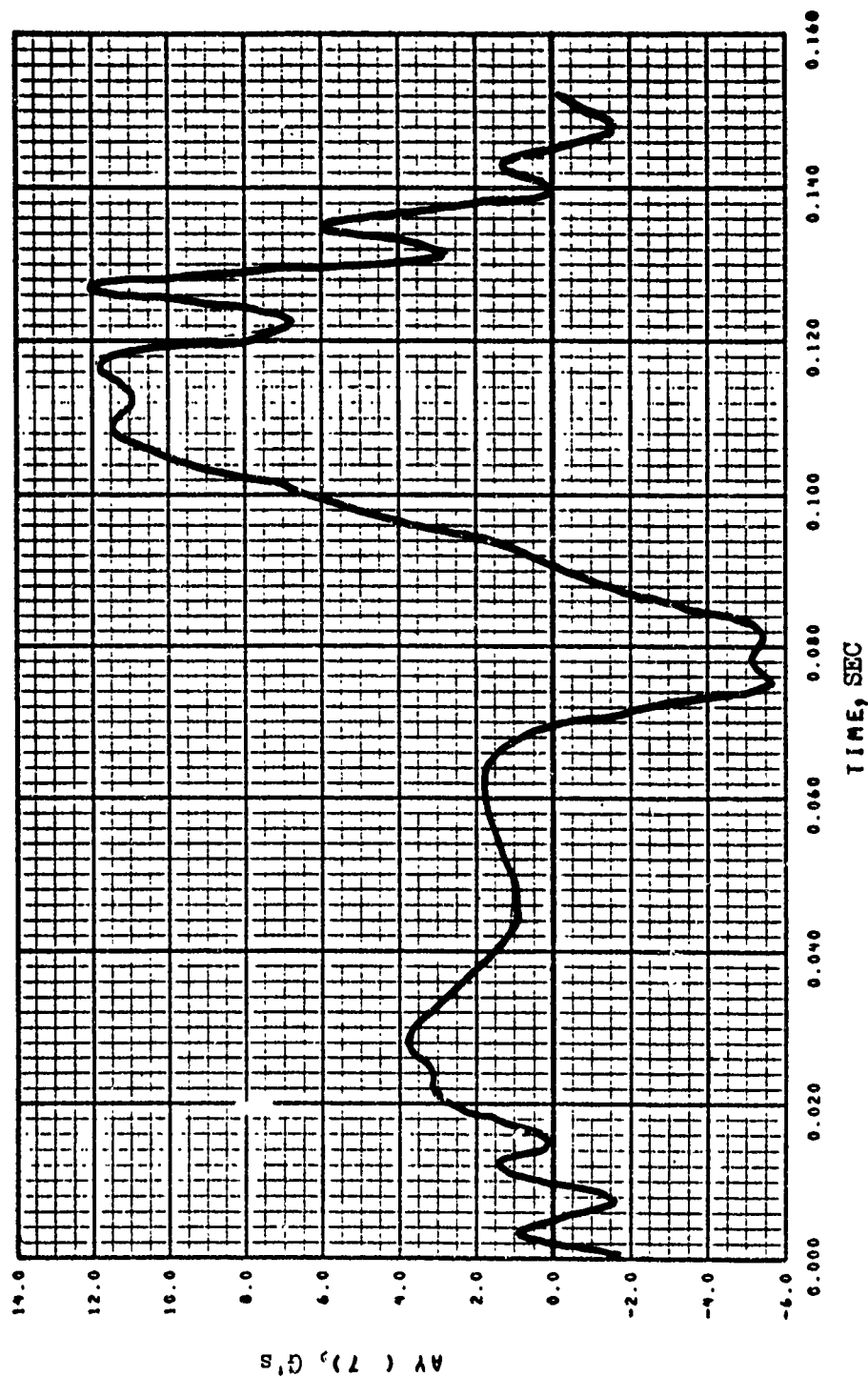


Figure 91. Engine Mass Lateral Acceleration Time History Plot, Combined Vertical and Lateral Impact Sample Case.

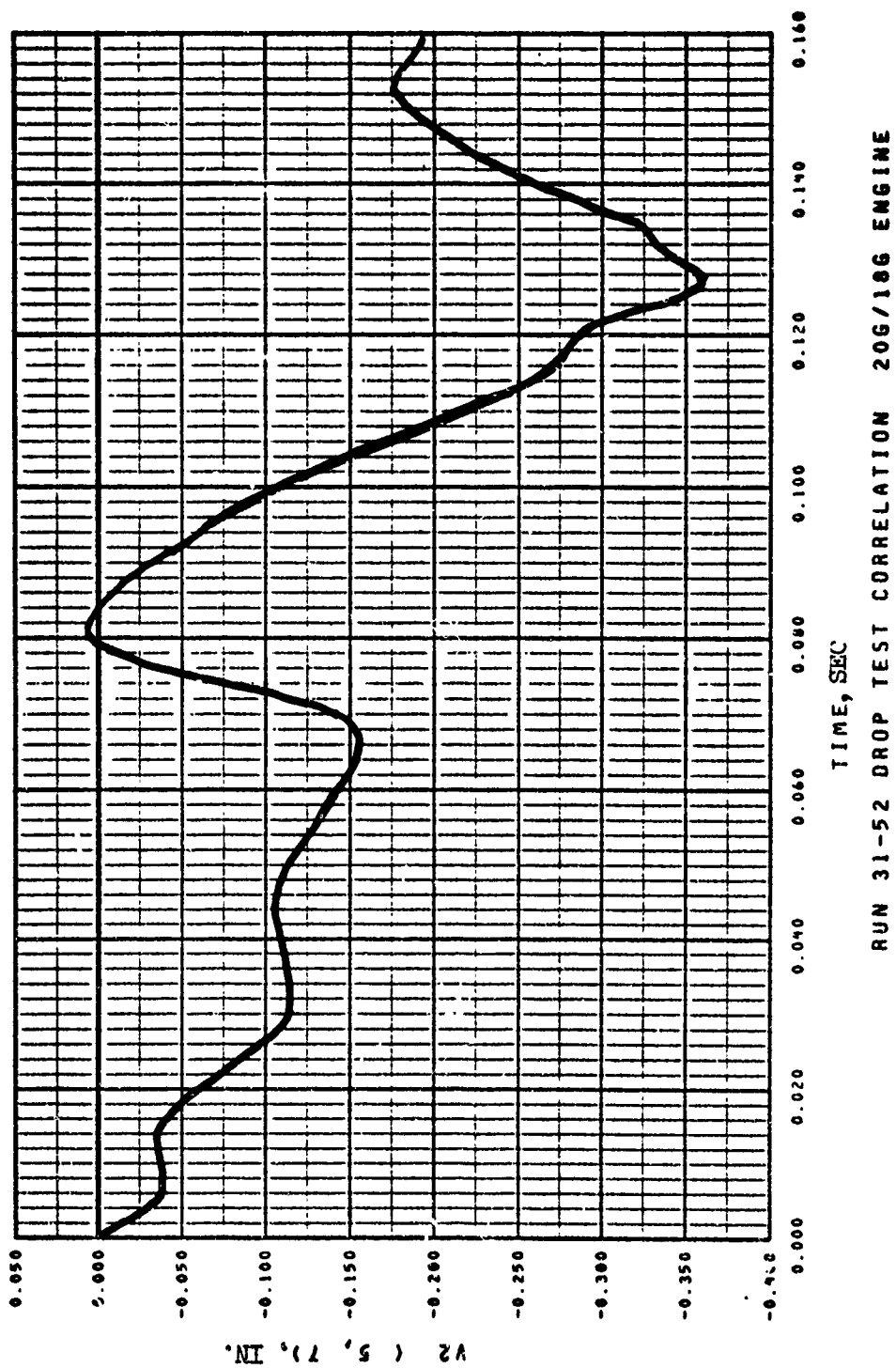
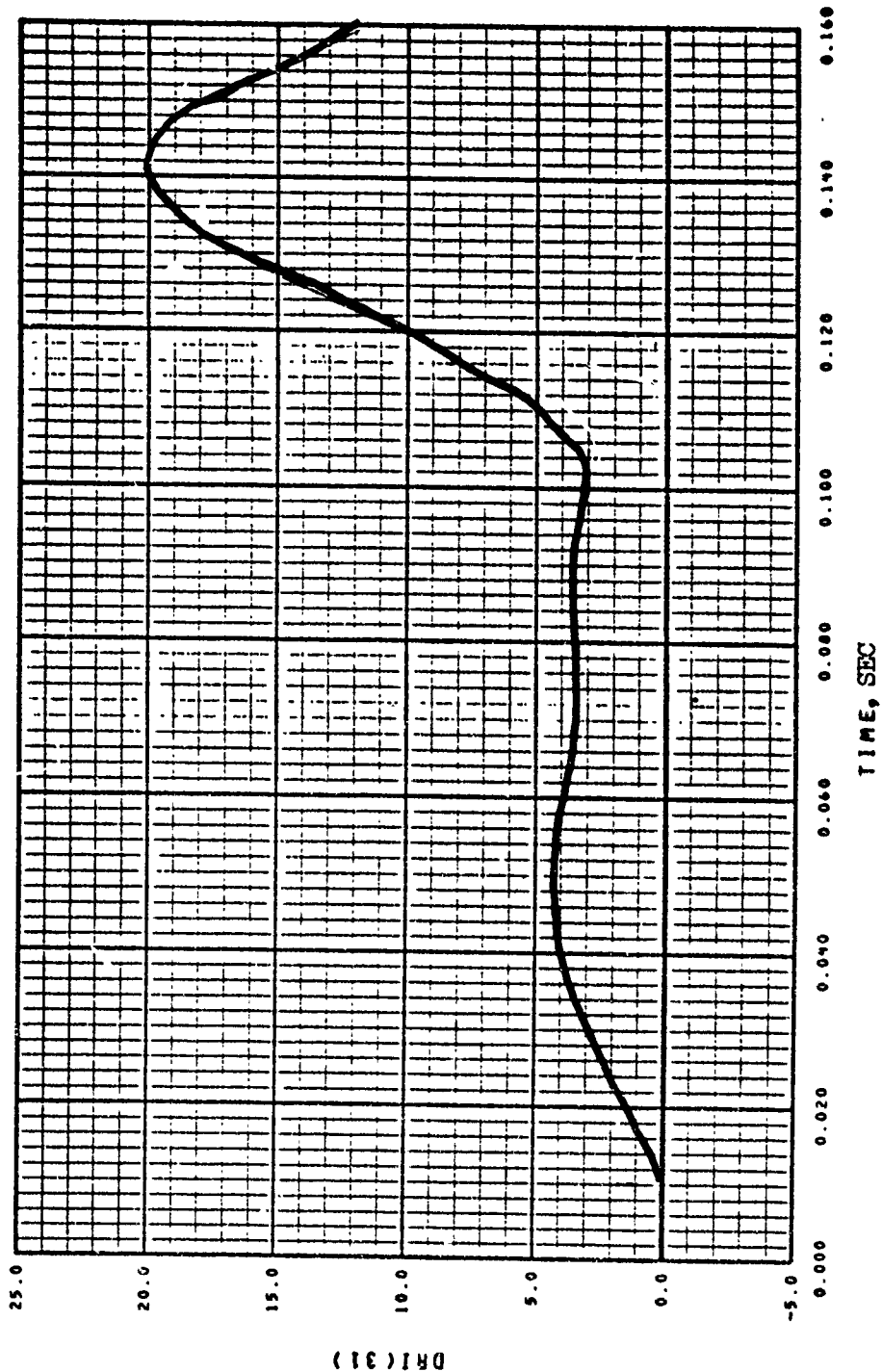
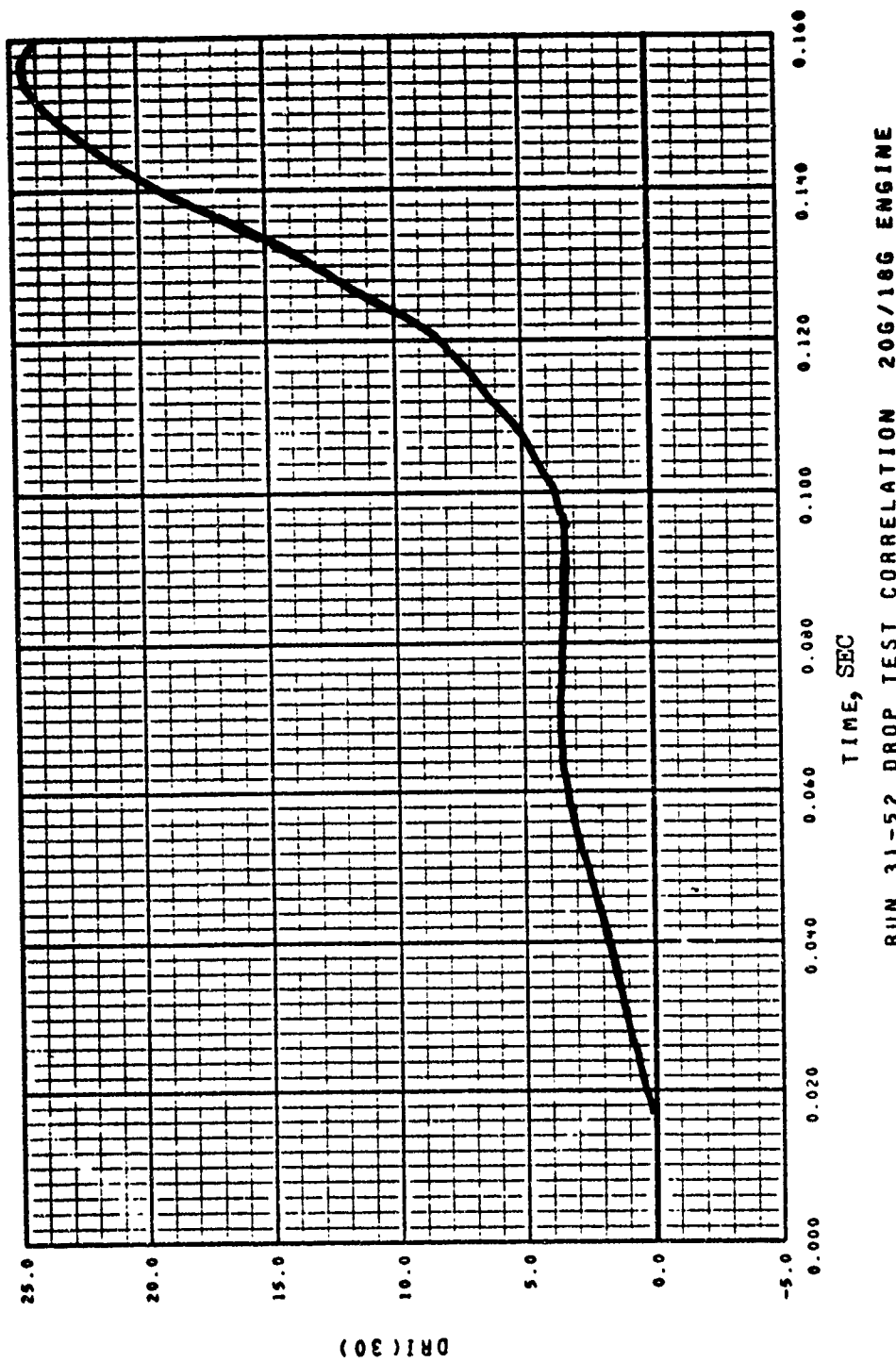


Figure 92. Engine Mount Lateral Deflection Time History Plot, Combined Vertical and Lateral Impact Sample Case.



RUN 31-52 DROP TEST CORRELATION 206/186 ENGINE

Figure 93. Forward DRI Time History Plot, Combined Vertical and Lateral Impact Sample Case.



RUN 31-52 DROP TEST CORRELATION 206/186 ENGINE

Figure 94. Aft DRI Time History Plot, Combined Vertical and Lateral Impact Sample Case.

MASSES = 32 DP/DT = 100 DT = 0.00002 TMAX = 0.100
 YDOT = 3.240000 02 YDOT = -2.235000 02 I5DOT = 2.760000 02
 P1 = 0.0 TMEAT = 0.0 R1 = 0.0
 PHI = 0.0 TMEAT = 0.0 PSI = 0.0 ZC = 0.0

WEIGHTS

1	1.390000 02
2	8.500000 01
3	4.200000 01
4	1.720000 02
5	2.780000 02
6	9.320000 02
7	7.310000 02
8	1.215000 03
9	9.800000 01
10	2.570000 02
11	1.823000 03
12	1.750000 02
13	1.750000 02
14	2.500000 01
15	2.400000 01
16	9.500000 02
17	2.400000 01
18	2.400000 01
19	3.070000 02
20	3.100000 01
21	3.100000 01
22	1.660000 02
23	1.660000 02
24	2.850000 01
25	2.850000 01
26	5.200000 01
27	7.000000 01
28	1.200000 02
29	1.200000 02
30	1.200000 02
31	1.200000 02
32	2.000000 02

1.390000 02 1.750000 02 1.750000 02 1.750000 02

1	1.390000 02	1.750000 02	1.750000 02	1.750000 02	0.0	0.0	0.0
2	1.000000 01	1.000000 02	1.000000 02	1.000000 02	0.0	0.0	0.0
3	1.500000 01	1.470000 02	1.470000 02	1.470000 02	0.0	0.0	0.0
4	2.500000 01	2.400000 01	2.400000 01	2.400000 01	0.0	0.0	0.0
5	2.500000 02	2.500000 02	2.500000 02	2.500000 02	0.0	0.0	0.0
6	1.000000 03	1.000000 03	1.000000 03	1.000000 03	0.0	0.0	0.0
7	3.000000 02	3.000000 02	3.000000 02	3.000000 02	0.0	0.0	0.0
8	3.500000 03	3.500000 03	3.500000 03	3.500000 03	0.0	0.0	0.0
9	3.700000 01	4.700000 01	4.700000 01	4.700000 01	0.0	0.0	0.0
10	2.500000 02	2.500000 02	2.500000 02	2.500000 02	0.0	0.0	0.0
11	5.000000 03	5.000000 03	5.000000 03	5.000000 03	0.0	0.0	0.0
12	4.300000 01	2.500000 02	2.500000 02	2.500000 02	0.0	0.0	0.0
13	4.300000 01	2.500000 02	2.500000 02	2.500000 02	0.0	0.0	0.0
14	3.570000 01	3.570000 01	3.570000 01	3.570000 01	0.0	0.0	0.0
15	3.570000 01	3.570000 01	3.570000 01	3.570000 01	0.0	0.0	0.0
16	1.000000 03	1.000000 03	1.000000 03	1.000000 03	0.0	0.0	0.0
17	5.620000 01	5.620000 01	5.620000 01	5.620000 01	0.0	0.0	0.0
18	5.620000 01	5.620000 01	5.620000 01	5.620000 01	0.0	0.0	0.0
19	1.716000 02	2.476000 02	2.476000 02	2.476000 02	0.0	0.0	0.0
20	1.284000 02	1.284000 02	1.284000 02	1.284000 02	0.0	0.0	0.0
21	1.284000 02	1.284000 02	1.284000 02	1.284000 02	0.0	0.0	0.0
22	4.650000 01	7.000000 01	7.000000 01	7.000000 01	0.0	0.0	0.0

Figure 95. Input Data, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

23	4.650000 01	7.000000 01	7.000000 01	0.0	0.0	0.0
24	1.358000 02	2.688000 02	1.565000 02	0.0	0.0	0.0
25	1.358000 02	2.688000 02	1.565000 02	0.0	0.0	0.0
26	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
27	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
28	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
29	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
30	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
31	1.800000 02	1.800000 02	1.800000 02	0.0	0.0	0.0
32	2.000000 01	2.000000 02	2.000000 02	0.0	0.0	0.0

1	4.587500 02	0.0	1.325200 02	0.0	0.0	0.0
2	4.042300 02	0.0	7.200000 01	0.0	0.0	0.0
3	2.417200 02	0.0	4.750000 01	0.0	0.0	0.0
4	2.116000 02	0.0	3.843000 01	0.0	0.0	0.0
5	1.630000 02	0.0	3.843000 01	0.0	0.0	0.0
6	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
7	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
8	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
9	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
10	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
11	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
12	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
13	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
14	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
15	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
16	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
17	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
18	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
19	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
20	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
21	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
22	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
23	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
24	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
25	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
26	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
27	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
28	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
29	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
30	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
31	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0
32	1.410000 02	0.0	3.843000 01	0.0	0.0	0.0

THERE ARE 0 I'S HAVING NON-ZERO HE OR PH I'S, THE TA'S, PSI'S

THERE ARE 0 I'S HAVING NON-ZERO LC'S

10	3	1.700000 01	3.000000-01	3.300000 04
11	3	1.700000 01	3.000000-01	1.100000 04
12	3	1.700000 01	3.000000-01	1.100000 04
13	3	1.700000 01	3.000000-01	1.100000 04
14	3	2.250000 00	3.000000-01	2.200000 04
15	3	2.250000 00	3.000000-01	2.200000 04
16	3	1.700000 01	3.000000-01	3.100000 04
17	3	2.250000 00	3.000000-01	2.200000 04
18	3	2.250000 00	3.000000-01	2.200000 04
19	3	1.700000 01	3.000000-01	1.100000 04
20	3	1.700000 01	3.000000-01	1.100000 04

10	3	1.000000-03	1.000000 00	3.500000 04	1.650000 04
11	3	1.000000-03	1.000000 00	3.500000 00	5.500000 03
12	3	1.000000-03	1.000000 00	2.000000 00	5.500000 03

Figure 95. (Continued).

[illegible]

Figure 95. (Continued).

0.0	0.0	0.0	1.296000 04	0.0	0.0	3.498000 05	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	3.498000 05	0.0	0.0	1.259000 07	0.0
24 25	0.0	-2.058000 06	0.0	0.0	0.0	0.0	7.407000 07
8.840000 05	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	3.980000 03	0.0	0.0	0.0	0.0	0.0	-1.792000 05
0.0	0.0	3.045000 02	0.0	0.0	1.370000 04	0.0	0.0
0.0	0.0	0.0	0.0	6.550000 04	0.0	0.0	0.0
0.0	0.0	1.370000 04	0.0	0.0	8.220000 05	0.0	0.0
0.0	-1.792000 05	0.0	0.0	0.0	0.0	1.075000 07	0.0
10 26	1.000000 04	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	0.0	-1.000000 05
0.0	0.0	1.000000 05	0.0	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07	0.0
16 27	3.000000 04	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05	0.0
0.0	0.0	1.000000 05	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07	0.0
26 28	1.229000 03	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.050000 05	0.0	0.0	0.0	0.0	-1.000000 05	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07	0.0
27 29	1.229000 03	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 06	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 05	0.0	1.000000 07	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07	0.0
26 30	7.840000 02	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07	0.0
27 31	7.840000 02	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.000000 05	0.0	0.0	0.0	0.0	-1.000000 05	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0	0.0
0.0	0.0	1.000000 05	0.0	0.0	1.000000 07	0.0	0.0
0.0	-1.000000 05	0.0	0.0	0.0	0.0	1.000000 07	0.0
8 32	1.000000 05	0.0	0.0	0.0	0.0	0.0	0.0
0.0	8.000000 03	0.0	0.0	0.0	0.0	-1.000000 05	0.0
0.0	0.0	8.000000 03	0.0	0.0	4.000000 05	0.0	0.0
0.0	0.0	0.0	0.0	1.000000 06	0.0	0.0	0.0
0.0	0.0	4.000000 05	0.0	0.0	2.447000 07	0.0	0.0
0.0	-4.000000 05	0.0	0.0	0.0	0.0	2.447000 07	0.0
1J,1J,CSART(1,1)							
1 1 2	5.000000-02						
2 2 3	5.000000-02						
3 3 4	5.000000-02						
4 4 5	5.000000-02						

Figure 95. (Continued).

2	4	7	3.00000-02
4	5	6	5.00000-02
7	5	7	5.00000-02
8	5	20	5.00000-02
9	5	21	5.00000-02
10	5	22	1.00000-01
11	6	23	1.00000-01
12	6	9	5.00000-02
13	6	11	1.00000-01
14	7	9	5.00000-02
15	8	9	5.00000-02
16	10	11	1.00000-01
17	10	14	1.00000-01
18	10	15	1.00000-01
19	10	22	1.00000-01
20	10	23	1.00000-01
21	11	16	1.00000-01
22	12	16	1.00000-01
23	12	24	5.00000-02
24	13	16	1.00000-01
25	13	25	5.00000-02
26	14	17	1.00000-01
27	16	18	1.00000-01
28	16	19	1.00000-01
29	20	21	5.00000-02
30	20	22	5.00000-02
31	21	23	5.00000-02
32	24	25	5.00000-02
33	10	26	5.00000-02
34	16	27	5.00000-02
35	16	28	2.00000-01
36	27	29	2.50000-01
37	26	30	1.51800-01
38	27	31	1.75200-01
39	8	32	5.00000-02

RR TABLE SPECS: I,J,L,MP

2	3	2	6
2	3	3	6
2	3	5	6
2	3	6	6
4	7	1	6
4	7	2	6
4	7	3	6
4	7	4	6
4	7	5	6
4	7	6	6
5	7	1	6
5	7	2	6
5	7	3	6
5	7	4	6
5	7	5	6
5	7	6	6
8	9	1	6
8	9	2	6
8	9	3	6
8	9	4	6
8	9	5	6
8	9	6	6
10	14	1	6
10	14	2	6
10	14	3	6
10	14	4	6
10	14	5	6
10	14	6	6
10	15	1	6

Figure 95. (Continued).

10	15	2	6
10	15	3	6
10	15	4	6
10	15	5	6
10	15	6	6
16	17	1	6
16	17	2	6
16	17	3	6
16	17	4	6
16	17	5	6
16	17	6	6
16	18	1	6
16	18	2	6
16	18	3	6
16	18	4	6
16	18	5	6
16	18	6	6

KR TABLE FOR I,J,L =			
1	2	3	2
1 0.0	1.0000E 00		
2 2.0000E 00	1.0000E 00		
3 4.0000E 00	-1.0000E 00		
4 6.0000E 00	-1.0000E 00		
5 8.0000E 00	-1.0000E 00		
6 1.0000E 01	-1.0000E 00		

KR TABLE FOR I,J,L =			
1	2	3	3
1 0.0	1.0000E 00		
2 2.0000E 00	1.0000E 00		
3 4.0000E 00	-1.0000E 00		
4 6.0000E 00	-1.0000E 00		
5 8.0000E 00	-1.0000E 00		
6 1.0000E 01	-1.0000E 00		

KR TABLE FOR I,J,L =				
1	2	3	5	
1 0.0	1.0000E 00			
2 1.0000E 00	1.0000E 00			
3 2.0000E 00	0.0			
4 3.0000E 00	-1.0000E 00			
5 4.0000E 00	-1.0000E 00			
6 5.0000E 00	-1.0000E 00			

KR TABLE FOR I,J,L =					
1	2	3	6		
1 0.0	1.0000E 00				
2 1.0000E 00	1.0000E 00				
3 2.0000E 00	0.0				
4 3.0000E 00	-1.0000E 00				
5 4.0000E 00	-1.0000E 00				
6 5.0000E 00	-1.0000E 00				

KR TABLE FOR I,J,L =						
1	2	3	4	7	1	
1 0.0	1.0000E 00					
2 1.0000E-01	1.0000E 00					
3 2.0000E-01	-7.0000E-01					
4 3.0000E-01	-7.0000E-01					
5 4.0000E-01	0.0					
6 5.0000E-01	0.0					

KR TABLE FOR I,J,L =						
1	2	3	4	7	2	
1 0.0	1.0000E 00					
2 6.6700E-02	1.0000E 00					
3 1.3340E-01	1.0000E 00					
4 2.0010E-01	1.0000E 00					
5 2.6680E-01	0.0					
6 3.3350E-01	0.0					

Figure 95. (Continued).

2	1.00000E-01	1.00000E 00			
3	2.00000E-01	1.00000E 00			
4	3.00000E-01	1.00000E 00			
5	4.00000E-01	0.0			
6	5.00000E-01	0.0			
NR TABLE FOR I,J,L = 5 7 6					
1	0.0	1.00000E 00			
2	1.00000E-01	1.00000E 00			
3	2.00000E-01	1.00000E 00			
4	3.00000E-01	1.00000E 00			
5	4.00000E-01	0.0			
6	5.00000E-01	0.0			
NR TABLE FOR I,J,L = 8 9 1					
1	0.0	7.85000E-03			
2	4.80000E-01	7.85000E-03			
3	9.60000E-01	1.00000E 00			
4	1.44000E 00	1.00000E 00			
5	1.92000E 00	0.0			
6	2.40000E 00	0.0			
NR TABLE FOR I,J,L = 8 9 2					
1	0.0	1.00000E 00			
2	2.00000E-01	1.00000E 00			
3	4.00000E-01	1.00000E 00			
4	6.00000E-01	1.00000E 00			
5	8.00000E-01	0.0			
6	1.00000E 00	0.0			
NR TABLE FOR I,J,L = 8 9 3					
1	0.0	1.00000E 00			
2	5.00000E-02	1.00000E 00			
3	1.00000E-01	1.00000E 00			
4	1.50000E-01	1.00000E 00			
5	2.00000E-01	0.0			
6	2.50000E-01	0.0			
NR TABLE FOR I,J,L = 8 9 4					
1	0.0	1.00000E 00			
2	1.00000E-01	1.00000E 00			
3	2.00000E-01	1.00000E 00			
4	3.00000E-01	1.00000E 00			
5	4.00000E-01	0.0			
6	5.00000E-01	0.0			
NR TABLE FOR I,J,L = 8 9 5					
1	0.0	1.00000E 00			
2	1.00000E-03	1.00000E 00			
3	2.00000E-03	1.00000E 00			
4	3.00000E-03	1.00000E 00			
5	4.00000E-03	0.0			
6	5.00000E-03	0.0			
NR TABLE FOR I,J,L = 8 9 6					
1	0.0	1.00000E 00			
2	1.00000E-01	1.00000E 00			
3	2.00000E-01	1.00000E 00			
4	3.00000E-01	1.00000E 00			
5	4.00000E-01	0.0			
6	5.00000E-01	0.0			
NR TABLE FOR I,J,L = 10 14 1					
1	0.0	1.00000E 00			
2	5.00000E-01	1.00000E 00			
3	1.00000E 00	1.00000E 00			

Figure 95. (Continued).

4	1.50000E 00	1.00000E 00	
5	2.00000E 00	-3.33000E 00	
6	2.50000E 00	-3.33000E 00	

KR TABLE FOR I,J,L = 10 14 2			
1	0.0	1.00000E 00	
2	4.00000E 00	1.00000E 00	
3	8.00000E 00	1.00000E 00	
4	1.20000E 01	1.00000E 00	
5	1.60000E 01	-3.33000E 00	
6	2.00000E 01	-3.33000E 00	

KR TABLE FOR I,J,L = 10 14 3			
1	0.0	1.00000E 00	
2	4.00000E 00	1.00000E 00	
3	8.00000E 00	1.00000E 00	
4	1.20000E 01	1.00000E 00	
5	1.60000E 01	-3.33000E 00	
6	2.00000E 01	-3.33000E 00	

KR TABLE FOR I,J,L = 10 14 4			
1	0.0	1.00000E 00	
2	2.00000E -01	1.00000E 00	
3	4.00000E -01	1.00000E 00	
4	6.00000E -01	1.00000E 00	
5	8.00000E -01	-3.33000E 00	
6	1.00000E 00	-3.33000E 00	

KR TABLE FOR I,J,L = 10 14 5			
1	0.0	1.00000E 00	
2	2.00000E -01	1.00000E 00	
3	4.00000E -01	1.00000E 00	
4	6.00000E -01	1.00000E 00	
5	8.00000E -01	-3.33000E 00	
6	1.00000E 00	-3.33000E 00	

KR TABLE FOR I,J,L = 10 14 6			
1	0.0	1.00000E 00	
2	2.00000E -01	1.00000E 00	
3	4.00000E -01	1.00000E 00	
4	6.00000E -01	1.00000E 00	
5	8.00000E -01	-3.33000E 00	
6	1.00000E 00	-3.33000E 00	

KR TABLE FOR I,J,L = 10 15 1			
1	0.0	1.00000E 00	
2	5.00000E -01	1.00000E 00	
3	1.00000E 00	1.00000E 00	
4	1.50000E 00	1.00000E 00	
5	2.00000E 00	-3.33000E 00	
6	2.50000E 00	-3.33000E 00	

KR TABLE FOR I,J,L = 10 15 2			
1	0.0	1.00000E 00	
2	4.00000E 00	1.00000E 00	
3	8.00000E 00	1.00000E 00	
4	1.20000E 01	1.00000E 00	
5	1.60000E 01	-3.33000E 00	
6	2.00000E 01	-3.33000E 00	

KR TABLE FOR I,J,L = 10 15 3			
1	0.0	1.00000E 00	
2	4.00000E 00	1.00000E 00	
3	8.00000E 00	1.00000E 00	
4	1.20000E 01	1.00000E 00	
5	1.60000E 01	-3.33000E 00	
6	2.00000E 01	-3.33000E 00	

Figure 95. (Continued).

6	2.0000E 01	-3.3300E 00			
KR TABLE FOR I,J,L = 10 15 4					
1	0.0	1.0000E 00			
2	2.0000E-01	1.0000E 00			
3	4.0000E-01	1.0000E 00			
4	6.0000E-01	1.0000E 00			
5	8.0000E-01	3.3300E 00			
6	1.0000E 00	-3.3300E 00			
KR TABLE FOR I,J,L = 10 15 5					
1	0.0	1.0000E 00			
2	2.0000E-01	1.0000E 00			
3	4.0000E-01	1.0000E 00			
4	6.0000E-01	1.0000E 00			
5	8.0000E-01	-3.3300E 00			
6	1.0000E 00	-3.3300E 00			
KR TABLE FOR I,J,L = 10 15 6					
1	0.0	1.0000E 00			
2	2.0000E-01	1.0000E 00			
3	4.0000E-01	1.0000E 00			
4	6.0000E-01	1.0000E 00			
5	8.0000E-01	-3.3300E 00			
6	1.0000E 00	-3.3300E 00			
KR TABLE FOR I,J,L = 16 17 1					
1	0.0	1.0000E 00			
2	5.0000E-01	1.0000E 00			
3	1.0000E 00	1.0000E 00			
4	1.5000E 00	1.0000E 00			
5	2.0000E 00	-3.3300E 00			
6	2.5000E 00	-3.3300E 00			
KR TABLE FOR I,J,L = 16 17 2					
1	0.0	1.0000E 00			
2	4.0000E 00	1.0000E 00			
3	8.0000E 00	1.0000E 00			
4	1.2000E 01	1.0000E 00			
5	1.6000E 01	-3.3300E 00			
6	2.0000E 01	-3.3300E 00			
KR TABLE FOR I,J,L = 16 17 3					
1	0.0	1.0000E 00			
2	4.0000E 00	1.0000E 00			
3	8.0000E 00	1.0000E 00			
4	1.2000E 01	1.0000E 00			
5	1.6000E 01	-3.3300E 00			
6	2.0000E 01	-3.3300E 00			
KR TABLE FOR I,J,L = 16 17 4					
1	0.0	1.0000E 00			
2	2.0000E-01	1.0000E 00			
3	4.0000E-01	1.0000E 00			
4	6.0000E-01	1.0000E 00			
5	8.0000E-01	-3.3300E 00			
6	1.0000E 00	-3.3300E 00			
KR TABLE FOR I,J,L = 16 17 5					
1	0.0	1.0000E 00			
2	2.0000E-01	1.0000E 00			
3	4.0000E-01	1.0000E 00			
4	6.0000E-01	1.0000E 00			
5	8.0000E-01	-3.3300E 00			
6	1.0000E 00	-3.3300E 00			

Figure 95. (Continued).

NR TABLE FOR I,J,L = 16 17 6

1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00

NR TABLE FOR I,J,L = 16 18 1

1	0.0	1.00000E 00
2	5.00000E-01	1.00000E 00
3	1.00000E 00	1.00000E 00
4	1.50000E 00	1.00000E 00
5	2.00000E 00	-3.33000E 00
6	2.50000E 00	-3.33000E 00

NR TABLE FOR I,J,L = 16 18 2

1	0.0	1.00000E 00
2	4.00000E 00	1.00000E 00
3	8.00000E 00	1.00000E 00
4	1.20000E 01	1.00000E 00
5	1.60000E 01	-3.33000E 00
6	2.00000E 01	-3.33000E 00

NR TABLE FOR I,J,L = 16 18 3

1	0.0	1.00000E 00
2	4.00000E 00	1.00000E 00
3	8.00000E 00	1.00000E 00
4	1.20000E 01	1.00000E 00
5	1.60000E 01	-3.33000E 00
6	2.00000E 01	-3.33000E 00

NR TABLE FOR I,J,L = 16 18 4

1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00

NR TABLE FOR I,J,L = 16 18 5

1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00

NR TABLE FOR I,J,L = 16 18 6

1	0.0	1.00000E 00
2	2.00000E-01	1.00000E 00
3	4.00000E-01	1.00000E 00
4	6.00000E-01	1.00000E 00
5	8.00000E-01	-3.33000E 00
6	1.00000E 00	-3.33000E 00

J,I,J,NMAX(I,J,L=6)

1	1	2	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
2	2	3	1.00000E 01	6.00000E 00	6.00000E 00	1.00000E 01	1.00000E 01	1.00000E 01
3	3	4	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
4	4	5	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
5	4	7	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
6	2	6	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
7	5	7	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
8	5	20	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01
9	5	21	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01	1.00000E 01

Figure 95. (Continued).

10	-3.50450-01	1.570800 00
11	-3.50450-01	-1.570800 00
12	1.570800 00	0.0
13	1.570800 00	0.0
14	-2.497790-01	0.0
15	-1.742282 00	0.0
16	0.0	0.0
17	-5.187220-01	1.570800 00
18	-5.187220-01	-1.570800 00
19	0.0	1.570800 00
20	0.0	-1.570800 00
21	0.0	0.0
22	0.0	-1.570375 00
23	1.570610 00	0.0
24	0.0	1.570570 00
25	1.570610 00	0.0
26	-5.187220-01	1.570800 00
27	-5.187220-01	-1.570800 00
28	0.0	0.0
29	0.0	-1.570800 00
30	-1.570800 00	0.0
31	-1.570800 00	0.0
32	0.0	-1.570800 00
33	1.570800 00	0.0
34	1.570800 00	0.0
35	1.570800 00	0.0
36	1.570800 00	0.0
37	1.570800 00	0.0
38	1.570800 00	0.0
39	-1.971710-01	0.0

Figure 95. (Continued).

RUM SE-31-100 COMBINED VEL. IMPACT

TIME = 0.10000

0100

	X	Y	Z	PHI	THETA	PSI
	U	V	W	Q	R	S
	UACCEL	VACCEL	WACCEL	QACCEL	RACCEL	SACCEL
MASS 1	-2.937840 01	-2.316490 01	-1.174970 02	-5.311000-03	2.004480-02	5.725200-03
	1.468840 02	-2.569870 02	1.451010 02	-4.027280-01	-6.040580-01	4.400620-01
	1.901610 02	-2.569160 02	1.476760 02	-4.118820-01	-6.064740-01	4.366310-01
	-0.978250 03	-1.062200 03	-6.921680 03	-1.076730 01	6.801120 01	1.321820 01
	-1.801980 01	-2.379130 00	-1.735890 01			
MASS 2	-2.375200 02	-2.239830 01	-5.816320 01	-1.791600-02	2.471600-02	6.511870-03
	2.211050 02	-2.058480 02	1.165370 02	-4.662400-01	1.418340 00	4.396890-01
	2.168120 02	-2.094340 02	1.181990 02	-4.771150-01	1.410240 00	4.644930-01
	-2.440930 03	2.973480 03	-1.087260 04	-8.732060 01	3.664120 00	-7.684490 00
	-5.639570 00	8.110500 00	-2.870070 01			
MASS 3	-7.416440 01	-2.100460 01	-3.654310 01	-2.525740-02	5.743380-03	4.108720-03
	2.348780 02	-1.480040 02	-4.076820 01	-6.702950-01	1.015370-01	-3.348400-03
	2.364990 02	-1.479350 02	-4.315590 01	-6.702660-01	1.015890-01	-7.830010-04
	-1.405020 03	3.800020 03	-1.059310 04	-1.312800 01	-7.182070 00	-1.192670 01
	-8.632960 00	9.745130 00	-2.724860 01			
MASS 4	-4.653600 01	-2.066360 01	-2.758970 01	-2.579440-02	-3.043300-04	3.444990-03
	2.349190 02	-1.441310 02	1.824570 01	-7.020800-01	-3.108780-01	-1.560390-01
	7.364070 02	-1.439030 02	4.204110 01	-7.020800-01	-3.067510-01	-1.640020-01
	-3.526750 03	3.508930 03	-1.042150 04	-7.968380 00	-1.148460 01	-1.617430 01
	-1.459350 00	8.914430 00	-2.655080 01			
MASS 5	4.057450 00	-2.050660 01	-2.755320 01	-2.578290-02	-1.469680-03	2.715100-03
	2.347350 02	-1.532330 02	-2.180100 01	-7.017320-01	-3.671850-01	-2.138810-01
	2.342860 02	-1.532480 02	-2.610480 01	-7.020870-01	-3.615490-01	-2.232740-01
	-3.618240 03	2.723020 03	-9.788530 03	-7.960110 00	-1.382020 01	-1.455940 01
	-6.437880 00	6.871470 00	-2.486070 01			
MASS 6	2.503240 01	-2.047380 01	-2.745040 01	-2.578280-02	-7.089530-03	1.065000-03
	2.331710 02	-1.583460 02	-1.259640 01	-7.018310-01	-4.645500-01	-2.337330-01
	2.329170 02	-1.581750 02	-1.833170 01	-7.034880-01	-4.583700-01	-2.456250-01
	-3.512770 03	2.361010 03	-9.664160 03	-7.9160 00	-3.526050 00	-1.775990 01
	-9.179320 00	6.012700 00	-2.447180 01			
MASS 7	-1.987720 01	-2.163480 01	-6.737370 01	-2.666260-02	-1.847090-03	3.754710-03
	2.470380 02	-1.741180 02	3.397900 01	-6.001530-01	-2.578090-01	-3.296300-01
	2.463190 02	-1.740720 02	3.905360 01	-6.007710-01	-2.590000-01	-3.360710-01
	-1.826840 03	2.234770 03	-1.161130 04	-2.732810 01	-5.717350 01	-2.960240 01
	-4.860130 00	5.514320 00	-2.965760 01			
MASS 8	3.326430 01	-2.201850 01	-8.290960 01	-2.316350-02	-5.404550-02	-7.325020-03
	3.149710 02	-2.046640 02	4.280530 01	-9.434440-01	-1.925680 00	-4.059980-01
	3.162310 02	-2.020000 02	-1.750400 01	-5.853760-01	-1.915780 00	-4.998980-01
	3.534950 00	1.516210 03	-9.348230 03	-2.448300 01	-7.801950 01	-1.367240 01
	-1.394060-01	3.514740 00	-2.213300 01			

Figure 96. Typical Output, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

RUN SC-31-100 COMBINED VEL. IMPACT

TIME = 0.10000

0100

	X	Y	Z	PHI	THETA	PSI
	ADOT	VDOT	ZDOT	PHIDOT	THETADOT	PSIDOT
	U	V	W	P	Q	R
	ACCEL	YACCEL	ZACCEL	PDOT	QDOT	RDOT
MASS 9	2.631900 01	-2.124670 01	-5.603770 01	-2.777930 02	-1.305530 02	2.154960 04
	2.502520 02	-1.800170 02	-1.232710 01	-6.101590 01	-6.704870 01	-2.482480 01
	2.500310 02	-1.795690 02	-2.058830 01	-6.136410 01	-6.627780 01	-2.867450 01
	-3.075220 03	2.109720 03	-9.740600 03	-1.310430 01	-1.755510 01	-2.178370 01
	-8.075290 00	5.236470 00	-2.447920 01			
MASS 10	3.837220 00	-2.002530 01	-1.179530 01	-2.427930 02	-1.937590 02	-9.419330 03
	2.228440 02	-1.419830 02	-3.304940 01	-5.489590 01	-1.854330 00	-3.819740 01
	2.223940 02	-1.393770 02	-4.972640 01	-5.716240 01	-1.844730 00	-4.52130 01
	-3.485440 03	2.649330 03	-8.041380 03	1.516740 01	8.103670 01	1.102760 01
	-8.972130 00	6.544350 00	-1.956340 01			
MASS 11	2.589350 01	-2.008660 01	-1.100750 01	-2.427840 02	-1.054170 02	1.702580 03
	2.235340 02	-1.470470 02	-1.257940 01	-6.881470 01	-6.745840 01	-1.750560 01
	2.231360 02	-1.470220 02	-1.850720 01	-6.897930 01	-6.701360 01	-1.913650 01
	-3.333560 03	2.656740 03	-9.707520 03	-2.505020 01	-2.912140 01	-1.556550 01
	-8.676930 00	6.739170 00	-2.449880 01			
MASS 12	4.527820 01	2.494960 01	-1.400310 01	-2.891720 02	-2.280640 02	-6.181590 04
	2.266500 02	-1.603340 02	-9.271710 00	1.229310 00	-3.258560 01	-9.854370 02
	2.264680 02	-1.597100 02	-1.906570 01	1.271130 00	-3.279580 01	-1.049020 01
	-5.006810 03	1.540520 03	-9.092420 03	-3.282150 01	-3.329300 01	7.507000 01
	-1.299850 01	3.990040 00	-2.387370 01			
MASS 13	9.234950 01	-6.498400 01	-1.258250 01	-9.716330 03	-2.825570 02	4.051550 04
	2.006330 02	-1.610610 02	-4.142740 01	7.657660 01	-8.744670 01	-7.259600 01
	1.993370 02	-1.608220 02	-4.815930 01	7.452540 01	-8.493730 01	-7.315270 01
	-4.178590 03	1.441305 03	-1.270400 04	1.131230 02	-2.257860 01	-3.346760 01
	-1.257610 01	3.375970 00	-3.277330 01			
MASS 14	1.045400 01	4.233390 01	-4.441950 00	-7.448000 03	-1.389490 01	1.404430 01
	3.073410 02	-7.589720 01	-8.247340 01	9.888910 00	-1.753570 00	-2.459680 01
	2.795330 02	-1.170650 02	-1.231320 02	9.834740 00	-1.751780 00	-2.566970 01
	-1.323350 02	-1.144560 03	1.046240 03	1.473700 01	8.290800 01	0.0
	1.381070 01	-7.396310 03	9.903900 01			
MASS 15	4.440220 01	-8.796720 01	-7.974130 00	8.565930 02	-1.243250 01	-1.344580 01
	2.842070 02	-4.055270 02	-1.549410 02	-9.958410 00	-1.707130 00	-1.114420 00
	3.214200 02	-3.723540 02	-1.650590 02	-7.076610 00	-1.795480 00	-9.557120 01
	1.074070 02	1.511310 03	-2.838220 03	5.623730 01	-2.222770 00	0.0
	1.240050 01	8.489430 02	9.886430 01			
MASS 16	9.529940 01	-1.998520 01	-1.281440 01	-1.999330 02	2.629910 03	1.804230 03
	2.203560 02	-1.409680 02	-3.347960 01	-2.106050 01	-1.036510 00	-1.375360 01
	2.201530 02	-1.406760 02	-3.612020 01	-2.102430 01	-1.033590 00	-1.582320 01
	-3.070180 03	1.488960 03	-9.603640 03	8.186020 01	-2.020510 01	-1.990700 01
	-7.922990 00	3.747540 00	-2.420290 01			

Figure 96. (Continued).

RUN SE-31-100 COMBINED VEL. IMPACT
TIME = 0.10000

0100

	X			Y			Z			PHI			THETA			PSI		
	XDOT	YDOT	ZDOT	XDOT	YDOT	ZDOT	XDOT	YDOT	ZDOT	PHIDOT	PHIDOT	PHIDOT	THETADOT	THETADOT	THETADOT	PSIDOT	PSIDOT	PSIDOT
	U	V	W	U	V	W	U	V	W	P	P	P	Q	Q	Q	R	R	R
	ACCEL	YACCEL	ZACCEL	ACCEL	YACCEL	ZACCEL	ACCEL	YACCEL	ZACCEL	POOT	POOT	POOT	GOOT	GOOT	GOOT	RDOT	RDOT	RDOT
MASS 17	9.0734 01	4.2192 01	-3.9255 00	-7.0792 01	-7.7518 01	-1.2678 02	-3.9255 00	-7.0792 01	-7.7518 01	-3.4247 01	-3.4247 01	-3.4247 01	-3.7245 02	-3.7245 02	-3.7245 02	1.7954 01	1.7954 01	1.7954 01
	2.7583 02	-7.0792 01	-1.2678 02	-7.0792 01	-7.7518 01	-1.2678 02	-3.9255 00	-7.0792 01	-7.7518 01	-3.4247 01	-3.4247 01	-3.4247 01	-3.7245 02	-3.7245 02	-3.7245 02	1.7954 01	1.7954 01	1.7954 01
	2.5389 02	-8.1047 01	-1.2678 02	-7.0792 01	-7.7518 01	-1.2678 02	-3.9255 00	-7.0792 01	-7.7518 01	-3.4247 01	-3.4247 01	-3.4247 01	-3.7245 02	-3.7245 02	-3.7245 02	1.7954 01	1.7954 01	1.7954 01
	-1.9771 02	-6.6548 02	2.6549 02	-7.0792 01	-7.7518 01	-1.2678 02	-3.9255 00	-7.0792 01	-7.7518 01	-3.4247 01	-3.4247 01	-3.4247 01	-3.7245 02	-3.7245 02	-3.7245 02	1.7954 01	1.7954 01	1.7954 01
	5.7234 02	-3.3526 02	9.4038 02	-7.0792 01	-7.7518 01	-1.2678 02	-3.9255 00	-7.0792 01	-7.7518 01	-3.4247 01	-3.4247 01	-3.4247 01	-3.7245 02	-3.7245 02	-3.7245 02	1.7954 01	1.7954 01	1.7954 01
MASS 18	9.2012 01	-8.4217 01	-4.3035 00	-3.3952 02	-3.1048 02	-1.4080 01	-4.3035 00	-3.3952 02	-3.1048 02	-4.0122 01	-4.0122 01	-4.0122 01	-4.7617 02	-4.7617 02	-4.7617 02	-1.4848 01	-1.4848 01	-1.4848 01
	2.8116 02	-3.3952 02	-1.4080 01	-3.3952 02	-3.1048 02	-1.4080 01	-4.3035 00	-3.3952 02	-3.1048 02	-4.0122 01	-4.0122 01	-4.0122 01	-4.7617 02	-4.7617 02	-4.7617 02	-1.4848 01	-1.4848 01	-1.4848 01
	2.4079 02	-3.1048 02	-1.4080 01	-3.3952 02	-3.1048 02	-1.4080 01	-4.3035 00	-3.3952 02	-3.1048 02	-4.0122 01	-4.0122 01	-4.0122 01	-4.7617 02	-4.7617 02	-4.7617 02	-1.4848 01	-1.4848 01	-1.4848 01
	4.7213 02	-3.4199 02	-4.7711 01	-3.3952 02	-3.1048 02	-1.4080 01	-4.3035 00	-3.3952 02	-3.1048 02	-4.0122 01	-4.0122 01	-4.0122 01	-4.7617 02	-4.7617 02	-4.7617 02	-1.4848 01	-1.4848 01	-1.4848 01
	4.7599 02	3.5479 01	9.3573 01	-3.3952 02	-3.1048 02	-1.4080 01	-4.3035 00	-3.3952 02	-3.1048 02	-4.0122 01	-4.0122 01	-4.0122 01	-4.7617 02	-4.7617 02	-4.7617 02	-1.4848 01	-1.4848 01	-1.4848 01
MASS 19	1.4393 02	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.8925 01	-4.4296 05	-4.4296 05	-4.4296 05	1.9447 03	1.9447 03	1.9447 03
	2.1941 02	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.8925 01	-4.4296 05	-4.4296 05	-4.4296 05	1.9447 03	1.9447 03	1.9447 03
	2.1909 02	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.8925 01	-4.4296 05	-4.4296 05	-4.4296 05	1.9447 03	1.9447 03	1.9447 03
	-2.9908 03	9.4461 02	-7.5142 03	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.8925 01	-4.4296 05	-4.4296 05	-4.4296 05	1.9447 03	1.9447 03	1.9447 03
	-7.6187 00	2.3750 00	-1.6782 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.2660 01	-1.8925 01	-1.8925 01	-1.8925 01	-4.4296 05	-4.4296 05	-4.4296 05	1.9447 03	1.9447 03	1.9447 03
MASS 20	8.5721 00	2.3917 01	-6.6356 01	-1.7841 02	-1.8346 02	-1.1744 04	-6.6356 01	-1.7841 02	-1.8346 02	-1.7841 02	-1.7841 02	-1.7841 02	-9.9073 02	-9.9073 02	-9.9073 02	1.3333 02	1.3333 02	1.3333 02
	5.0905 02	-1.7841 02	-1.8346 02	-1.7841 02	-1.8346 02	-1.1744 04	-6.6356 01	-1.7841 02	-1.8346 02	-1.7841 02	-1.7841 02	-1.7841 02	-9.9073 02	-9.9073 02	-9.9073 02	1.3333 02	1.3333 02	1.3333 02
	4.5983 02	-1.8346 02	-1.1744 04	-1.7841 02	-1.8346 02	-1.1744 04	-6.6356 01	-1.7841 02	-1.8346 02	-1.7841 02	-1.7841 02	-1.7841 02	-9.9073 02	-9.9073 02	-9.9073 02	1.3333 02	1.3333 02	1.3333 02
	7.0775 03	2.3135 03	-2.2257 01	-1.7841 02	-1.8346 02	-1.1744 04	-6.6356 01	-1.7841 02	-1.8346 02	-1.7841 02	-1.7841 02	-1.7841 02	-9.9073 02	-9.9073 02	-9.9073 02	1.3333 02	1.3333 02	1.3333 02
	2.0700 01	6.9286 00	-2.2257 01	-1.7841 02	-1.8346 02	-1.1744 04	-6.6356 01	-1.7841 02	-1.8346 02	-1.7841 02	-1.7841 02	-1.7841 02	-9.9073 02	-9.9073 02	-9.9073 02	1.3333 02	1.3333 02	1.3333 02
MASS 21	9.3460 00	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.6322 01	-1.3986 01	-1.3986 01	-1.3986 01	1.0599 02	1.0599 02	1.0599 02
	5.6631 02	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.6322 01	-1.3986 01	-1.3986 01	-1.3986 01	1.0599 02	1.0599 02	1.0599 02
	5.6284 02	-1.8961 02	-5.4255 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.6322 01	-1.3986 01	-1.3986 01	-1.3986 01	1.0599 02	1.0599 02	1.0599 02
	1.0015 04	7.9930 02	-1.3026 00	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.6322 01	-1.3986 01	-1.3986 01	-1.3986 01	1.0599 02	1.0599 02	1.0599 02
	2.7450 01	3.4194 00	-3.0799 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.4147 01	-6.6322 01	-6.6322 01	-6.6322 01	-1.3986 01	-1.3986 01	-1.3986 01	1.0599 02	1.0599 02	1.0599 02
MASS 22	3.1430 00	2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-2.4784 01	-9.4490 02	-9.4490 02	-9.4490 02	1.6778 02	1.6778 02	1.6778 02
	2.3236 02	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-2.4784 01	-9.4490 02	-9.4490 02	-9.4490 02	1.6778 02	1.6778 02	1.6778 02
	2.2315 02	-1.4379 02	-8.6077 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-2.4784 01	-9.4490 02	-9.4490 02	-9.4490 02	1.6778 02	1.6778 02	1.6778 02
	-2.0598 03	2.5863 00	-2.2709 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-2.4784 01	-9.4490 02	-9.4490 02	-9.4490 02	1.6778 02	1.6778 02	1.6778 02
	-4.7671 00	6.5816 00	-2.1034 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-1.2128 01	-2.4784 01	-2.4784 01	-2.4784 01	-9.4490 02	-9.4490 02	-9.4490 02	1.6778 02	1.6778 02	1.6778 02
MASS 23	2.4104 00	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-6.5175 01	-1.3143 01	-1.3143 01	-1.3143 01	-4.0841 02	-4.0841 02	-4.0841 02
	1.5342 02	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-6.5175 01	-1.3143 01	-1.3143 01	-1.3143 01	-4.0841 02	-4.0841 02	-4.0841 02
	1.5611 02	-1.3946 02	-3.4057 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-6.5175 01	-1.3143 01	-1.3143 01	-1.3143 01	-4.0841 02	-4.0841 02	-4.0841 02
	-2.2055 03	2.3826 00	-1.0335 04	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-6.5175 01	-1.3143 01	-1.3143 01	-1.3143 01	-4.0841 02	-4.0841 02	-4.0841 02
	-5.6381 00	5.2319 00	-2.3739 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-1.0736 01	-6.5175 01	-6.5175 01	-6.5175 01	-1.3143 01	-1.3143 01	-1.3143 01	-4.0841 02	-4.0841 02	-4.0841 02
MASS 24	9.6512 01	2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-2.3724 01	-2.3281 02	-2.3281 02	-2.3281 02	3.3842 03	3.3842 03	3.3842 03
	2.4113 02	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-2.3724 01	-2.3281 02	-2.3281 02	-2.3281 02	3.3842 03	3.3842 03	3.3842 03
	2.6030 02	-1.4849 02	-1.7572 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-2.3724 01	-2.3281 02	-2.3281 02	-2.3281 02	3.3842 03	3.3842 03	3.3842 03
	-3.3870 03	8.7413 02	-1.8459 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-2.3724 01	-2.3281 02	-2.3281 02	-2.3281 02	3.3842 03	3.3842 03	3.3842 03
	-8.7582 00	2.2811 01	-2.4484 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-6.7872 01	-2.3724 01	-2.3724 01	-2.3724 01	-2.3281 02	-2.3281 02	-2.3281 02	3.3842 03	3.3842 03	3.3842 03

Figure 96. (Continued).

RUN SE-31-100 COMBINED VEL. IMPACT

0100

TIME = 0.10000

	X	Y	Z	PHI	THETA	PSI
	XDOT	YDOT	ZDOT	PHIDOT	THETADOT	PSIDOT
	U	V	W	P	Q	R
	UACCEL	VACCEL	WACCEL	POOT	QOOT	ROOT
MASS 25	9.681900 01	-6.626370 01	-6.654630 01	-3.087490 -02	-2.612220 -02	3.785710 -03
	2.442640 02	-1.697780 02	-4.104110 01	-1.992380 -01	-8.140850 -01	-9.844010 -04
	2.424600 02	-1.691570 02	-5.263720 01	-1.992380 -01	-8.140850 -01	-2.611740 -02
	-2.484000 03	8.522940 03	-1.295300 04	5.036990 01	-6.400890 01	8.198300 -01
	-4.335470 00	2.203650 01	-3.295740 01			
MASS 26	4.422980 00	-2.027120 01	-2.137960 01	-2.198670 -02	-4.197930 -02	1.317840 -03
	2.436950 02	-1.470280 02	-1.911640 01	-7.237950 -01	-4.114640 00	-3.344610 -01
	2.416500 02	-1.465620 02	-7.393200 01	-7.445750 -01	-4.108480 00	-4.264230 -01
	-3.714180 03	3.269430 03	-9.192680 03	-4.427370 01	1.251380 02	-2.335090 01
	-4.364080 00	8.130830 00	-2.097390 01			
MASS 27	9.527400 01	-2.019180 01	-2.264940 01	-2.013100 -02	6.714790 -03	3.903830 -03
	2.298580 02	-1.440320 02	-3.307460 01	-5.582300 -01	-1.112500 00	1.514070 -04
	2.294330 02	-1.442600 02	-3.464830 01	-5.582310 -01	-1.112500 00	-2.214310 -02
	-2.432110 03	2.245180 03	-1.079610 04	1.832070 01	-1.050300 02	-8.534890 00
	-4.209850 00	5.752920 00	-2.707220 01			
MASS 28	4.902730 00	-2.045210 01	-2.295470 01	-2.034260 -02	-5.864440 -02	4.642210 -03
	2.769930 02	-1.510370 02	4.104260 01	-5.880990 -01	-5.395370 00	-1.372740 -01
	2.732800 02	-1.527900 02	2.168560 01	-5.961750 -01	-5.391460 00	-2.467540 -01
	-4.174260 03	2.883440 03	-1.055930 04	-1.301050 02	5.794400 01	-3.004610 01
	-1.121640 01	7.585330 00	-2.324620 01			
MASS 29	9.521080 01	-2.036510 01	-3.119200 01	-1.935730 -02	9.122070 -03	4.526040 -03
	2.346530 02	-1.672510 02	6.472440 01	-6.388680 -01	-1.079190 00	5.304500 -02
	2.372400 02	-1.695930 02	6.362100 01	-6.393320 -01	-1.080020 00	3.216390 -02
	-1.359100 03	2.297520 03	-9.408310 03	-3.057460 01	-1.344620 02	-6.610210 00
	-3.684870 00	6.082450 00	-2.342900 01			
MASS 30	4.873610 00	-2.043920 01	-2.866850 01	-2.015460 -02	-5.943500 -02	5.014220 -03
	2.719780 02	-1.501370 02	6.469630 01	-6.070120 -01	-5.639580 00	-1.140920 -01
	2.745670 02	-1.524440 02	4.539300 01	-6.137920 -01	-5.635140 00	-2.275030 -01
	-4.221650 03	3.122630 03	-9.383910 03	-1.340270 02	5.747200 01	-3.075480 01
	-1.165940 01	8.000050 00	-2.005960 01			
MASS 31	9.521560 01	-2.035880 01	-3.081090 01	-1.906940 -02	9.174380 -03	4.633700 -03
	2.387210 02	-1.668240 02	7.434220 01	-4.007170 -01	-1.100970 00	5.700310 -02
	2.372350 02	-1.693950 02	7.532400 01	-4.012400 -01	-1.101050 00	3.599440 -02
	-1.395150 03	2.312030 03	-7.052760 03	-4.011210 01	-1.347730 02	-6.716970 00
	-3.813600 00	6.140850 00	-1.730450 01			
MASS 32	1.178450 02	-2.213580 01	-6.128670 01	-2.197020 -02	-5.296760 -02	-5.291830 -03
	2.737740 02	-1.776100 02	1.672820 02	-4.781160 -01	-2.037890 00	-3.372820 -01
	2.835600 02	-1.960600 02	1.476630 02	-4.959730 -01	-2.030000 00	-3.814970 -01
	-5.228550 02	2.018220 02	-5.893110 03	-3.576140 01	2.891950 01	-2.578610 01
	-2.348160 00	6.237730 -01	-1.321010 01			
[CIT]J1,JC(I,J),SUMDF(I,I,J),SUMDF(I2,I,J),SUMDF(I3,I,J),SUMDF(I4,I,J),SUMDF(I5,I,J),SUMDF(I6,I,J)						
1 2	-3.508070 03	-4.867180 02	2.902240 02	0.0	1.658580 04	2.919290 04
2 3	-3.643970 03	6.428450 01	-3.601350 03	-1.877770 04	-6.997490 05	-3.526330 04
3 4	-5.381350 03	6.952450 02	-5.735750 03	-2.697240 04	-8.784010 05	-7.184110 04
4 5	-1.321090 04	4.246730 03	-2.792140 04	8.899380 04	-1.693270 06	-2.770870 05
4 7	-6.711760 03	-1.169140 03	1.275940 04	0.0	2.687940 05	1.846640 04

Figure 96. (Continued).

5	6	-1.172360 04	-1.488650 04	4.659290 04	-6.408060 04	-7.312060 05	4.354310 04
4	7	-6.334200 03	-2.028840 02	-9.782660 03	0.0	-2.388770 05	-1.813860 04
5	20	1.111600 04	0.0	-6.267770 02	0.0	6.358850 03	0.0
5	21	6.835670 03	0.0	-6.449450 03	0.0	-1.405360 05	0.0
5	22	-1.167400 04	0.0	-2.658140 04	0.0	-7.241340 05	0.0
5	23	-3.377850 04	0.0	-3.470700 04	0.0	-8.435860 05	0.0
6	9	-3.325750 04	-2.810000 03	-1.682090 02	2.578300 04	-1.431080 05	-1.381070 04
6	11	-6.929000 03	-1.786060 04	1.104960 04	-1.768190 05	2.566050 03	0.0
7	9	0.0	0.0	0.0	-4.928820 03	0.0	0.0
8	9	-3.022290 04	2.564540 03	4.888310 03	2.082070 04	1.231230 05	-1.793510 04
10	11	2.872950 04	5.418070 03	2.864780 03	7.769400 02	3.855230 05	1.723390 04
10	15	1.538540 03	2.838690 03	-2.213690 03	3.657800 03	1.466110 05	7.481030 03
10	16	-3.190440 03	-2.561170 03	-6.393380 02	8.024240 03	1.916040 05	4.459730 03
10	22	1.380300 03	3.151910 03	3.263720 04	0.0	7.853880 05	1.280240 04
10	23	1.532740 04	-5.977430 03	2.174440 04	0.0	6.245060 05	-9.541440 02
11	16	2.154990 03	1.325020 04	-1.197000 04	3.220260 03	-3.847760 05	-1.683020 05
12	16	7.478300 02	-3.603310 02	1.897490 03	-2.696270 04	7.508100 04	7.409350 04
12	24	-3.764800 02	2.813720 02	4.243070 01	-1.192030 03	5.539040 03	3.822210 03
13	16	-1.546600 03	6.458610 02	7.636390 03	3.293080 04	2.238240 05	2.089470 04
13	25	-1.311980 03	1.450160 03	7.537790 02	-1.019090 03	1.675750 04	7.531660 03
16	17	1.879440 03	1.900590 03	-5.781720 02	-0.151950 03	9.727850 04	-8.951820 03
16	18	-2.147740 03	-1.608720 03	-5.624490 02	5.990040 03	9.786600 04	9.517850 03
16	19	2.564150 03	7.243780 03	1.710040 03	1.279850 03	-5.218640 05	-4.239170 03
20	21	-5.814820 03	-8.140610 01	-7.523790 02	0.0	3.028160 04	-3.621860 03
20	22	-8.766200 03	1.027100 03	-2.646910 02	0.0	-5.796670 04	-3.115780 04
21	23	-9.764240 03	3.278190 03	-4.453820 03	0.0	-9.136250 04	-1.003570 05
24	25	-9.113640 02	7.096900 01	-2.671580 02	2.500860 02	-1.418970 04	-2.257000 03
10	26	-3.956490 03	-1.181300 03	1.310930 03	-2.578920 03	-2.999900 04	2.921590 04
16	27	-4.883720 03	-9.800920 02	4.094110 02	-2.178620 03	4.041480 04	-4.620880 02
26	28	-2.232060 03	-3.423850 02	-2.427600 03	-3.344900 03	2.772880 04	1.874530 04
27	29	-1.765600 03	-1.819060 03	-4.109400 02	-6.482070 02	2.329870 04	7.798610 03
28	30	-1.961540 03	-5.709920 03	-1.167240 03	-3.719300 03	2.309440 04	2.110870 04
27	31	-1.357570 03	-3.492840 02	-1.704960 02	-7.745710 02	2.405530 04	1.080740 04
8	32	9.845050 02	1.605780 02	-1.785410 02	1.686810 03	-2.053020 03	3.644190 03
IG(IJJ),G(IJJ),VEE2(11,IJJ),VEE2(12,IJJ),VEE2(13,IJJ),VEE2(14,IJJ),VEE2(15,IJJ),VEE2(16,IJJ)							
1	2	-2.827530 02	1.160180 01	-2.715120 02	-7.802220 03	3.817840 03	8.441540 03
2	3	-5.973560 02	-1.516530 01	-1.711440 00	-7.808100 03	-1.872350 02	-1.977330 03
3	4	-1.667080 02	-9.176360 03	8.790920 02	-7.391730 04	-6.033500 03	9.293870 04
4	5	-8.287920 03	-1.110700 02	2.136940 02	1.086220 06	-1.143840 03	-7.596570 04
4	7	-1.256550 01	-2.080610 02	1.503340 01	-0.814190 04	-1.945100 03	-4.398270 04
5	6	-2.669430 02	-2.868610 02	6.975280 02	-1.807170 06	-5.575440 03	-1.794430 03
5	7	-1.765880 01	-2.794580 04	-7.790500 02	-3.357330 04	-4.026050 04	-1.108730 03
5	20	6.953730 02	-4.577220 00	-8.794910 02	-0.024330 02	2.211160 03	-5.855780 02
5	21	4.272420 02	5.110360 00	-2.521630 01	1.036350 01	8.859310 03	9.206760 02
5	22	-6.761940 02	3.691200 01	-4.619740 01	-3.932150 02	1.859170 03	4.592400 02
5	23	-1.924740 01	-1.743100 00	-7.683210 01	1.034590 01	8.226210 03	-5.778940 02
6	9	-1.094000 02	-3.590960 02	8.285270 02	1.007159 03	-5.940640 03	-2.005150 03
6	11	-1.424840 03	-1.665130 02	3.174780 02	-2.508290 04	-3.547370 03	-1.210820 03
7	9	1.608160 01	-9.259530 02	-1.738530 01	-2.250600 03	-1.109830 02	-3.392970 03
8	9	-5.940810 01	9.050850 02	2.059550 01	9.229030 03	4.078800 02	2.763310 03
10	11	5.266630 02	1.128720 01	-5.189210 01	3.281220 04	6.862640 02	9.251010 03
10	14	3.208640 03	4.546780 00	-1.510060 01	6.790780 05	3.693720 01	1.419700 01
10	15	-6.653680 03	-3.595580 00	-1.510380 01	1.431340 04	3.898030 01	-1.074740 01
10	22	1.555912 02	6.391780 01	1.260820 01	-3.391540 02	2.299440 03	2.215460 02
10	23	1.724200 01	-1.073290 00	2.816420 02	7.152560 02	6.086790 03	-3.584190 02
11	16	1.269940 02	2.488820 02	-2.538710 00	4.288540 03	1.316460 02	4.696990 04
12	16	4.206070 04	6.612700 02	-1.138620 01	-2.526960 02	9.034290 03	3.065390 03
12	24	-2.541990 03	2.867590 01	-5.897680 03	-3.898970 03	2.397100 04	8.693230 03
13	16	-6.495390 04	4.834950 02	7.127420 02	3.086290 02	1.324380 02	1.854500 03
13	25	-8.853730 03	-9.177080 01	-6.617010 02	-3.518630 03	2.072080 02	-2.506270 02
16	17	7.840820 03	4.550260 00	-1.510230 01	-2.091460 04	3.860040 01	1.308440 01
16	18	-8.960120 03	-3.680420 00	-1.510100 01	2.029000 04	3.866410 01	-1.038380 01
16	19	1.246550 02	3.188340 03	8.153920 02	1.193850 03	-2.588080 03	9.777930 05
20	21	-1.308120 01	-2.037390 01	-3.055910 01	4.070020 02	5.775580 03	-3.779840 03
20	22	-1.183750 01	8.476900 03	-3.197450 01	3.548620 03	4.279490 03	-1.851240 04
21	23	-1.318240 01	2.574080 02	-5.909600 01	-3.082220 02	9.162470 03	-6.366930 04

Figure 96. (Continued).

24	25	-1.025160-03	3.358790-02	-4.025800-01	3.818110-03	-1.059270-02	3.499490-04
10	26	-3.950490-01	-8.981210-03	1.564810-02	-7.578920-03	-2.759590-03	2.831780-03
16	27	-1.627910-01	-9.946460-03	1.275940-05	-2.178620-03	4.041350-03	-1.454730-04
26	28	-1.816140-00	-1.544970-03	-2.732210-02	-3.344940-03	3.044100-03	1.858880-03
27	29	-1.435400-00	-1.049720-03	-6.904010-03	-6.682070-04	2.394610-03	7.699640-04
26	30	-2.495460-00	-3.435450-03	-1.612300-02	-3.719300-03	2.450470-03	2.074320-03
27	31	-1.824800-00	-2.414540-03	-4.152020-03	-7.945710-04	2.447050-03	1.054360-03
28	32	9.945090-03	1.077770-01	-7.365710-02	1.686810-03	1.000740-03	1.753090-03
1.55(1.21)56(1.21)56(1.21)							
10	0.0	0.0	0.0	3.180140-00			
11	0.0	0.0	0.0	5.988450-00			
12	0.0	0.0	0.0	2.917370-00			
13	0.0	0.0	0.0	4.712120-00			
14	0.0	0.0	0.0	1.057320-03			
15	0.0	0.0	0.0	1.674310-03			
16	0.0	0.0	0.0	4.162980-00			
17	0.0	0.0	0.0	1.653420-04			
18	0.0	0.0	0.0	1.043750-03			
22	0.0	0.0	0.0	4.208850-00			
23	0.0	0.0	0.0	6.168710-00			
MASS							
30	DA1	1.634440-01					
31		1.202970-01					

Figure 96. (Continued).

RUPTURE SUMMARY		
I	J	TIME
10	15	0.03376
10	14	0.04142
16	18	0.04284
16	17	0.04510

Figure 96. (Continued).

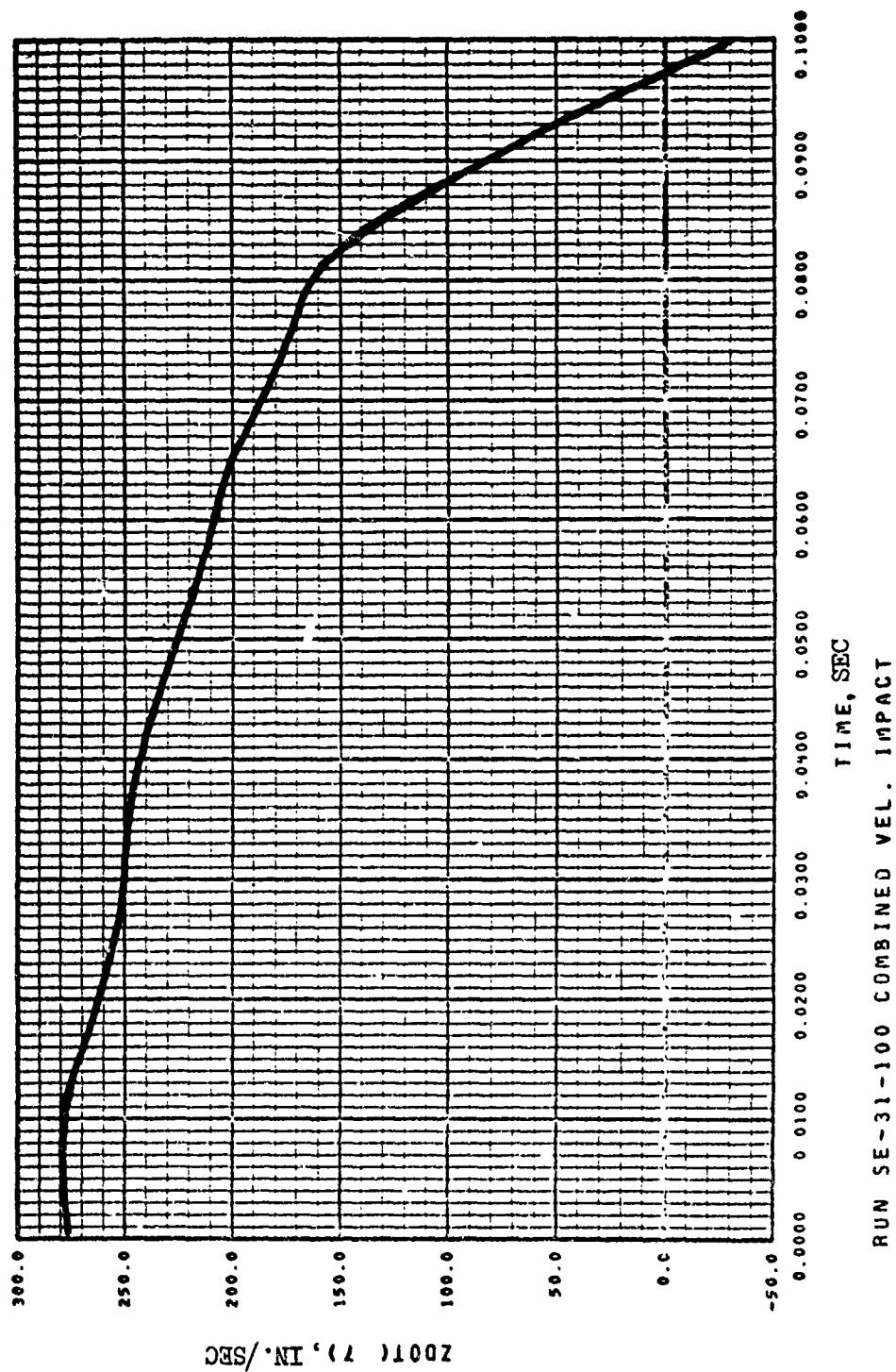


Figure 97. Engine Mass Vertical Velocity, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

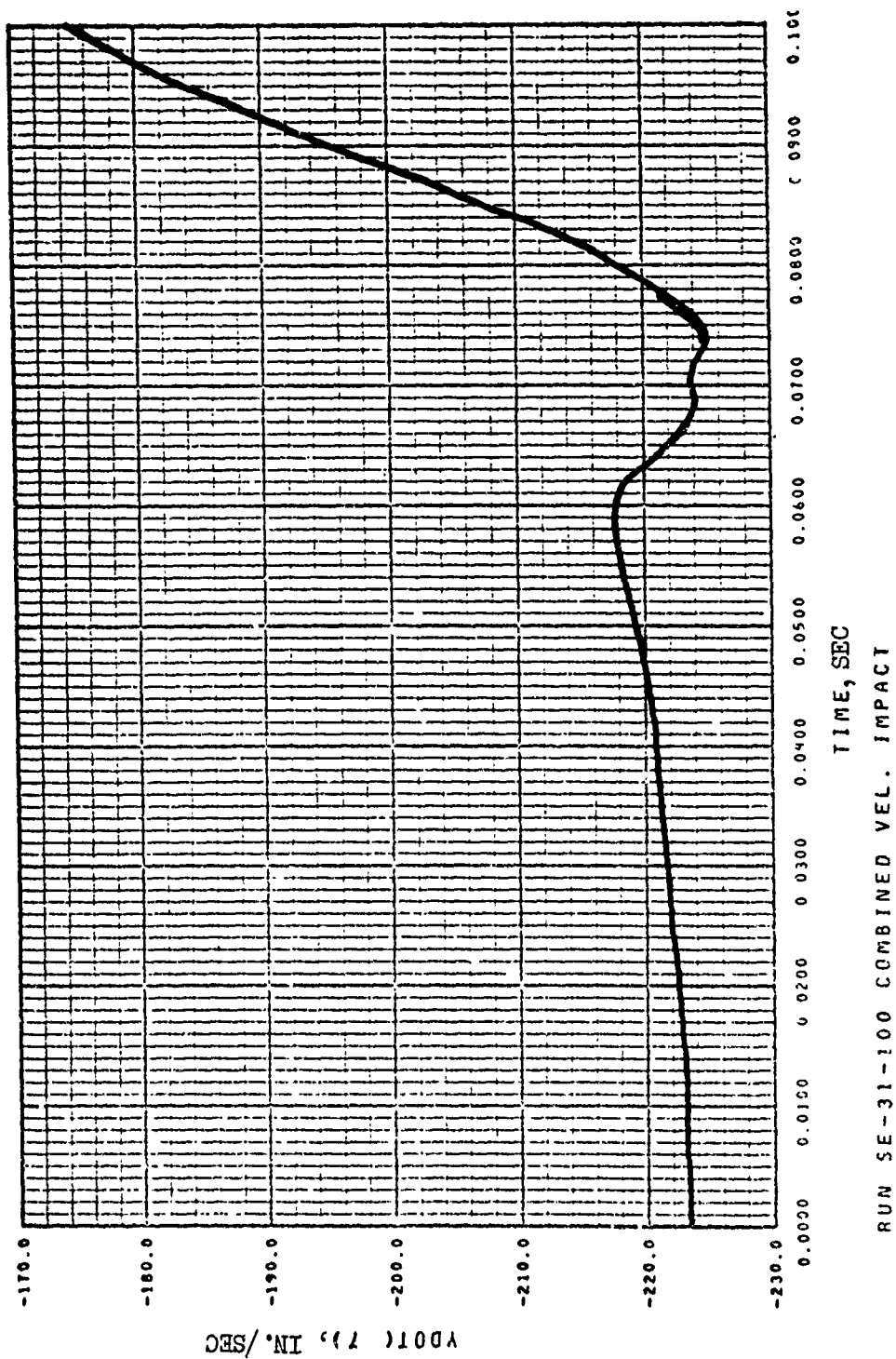


Figure 98. Engine Mass Lateral Velocity, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

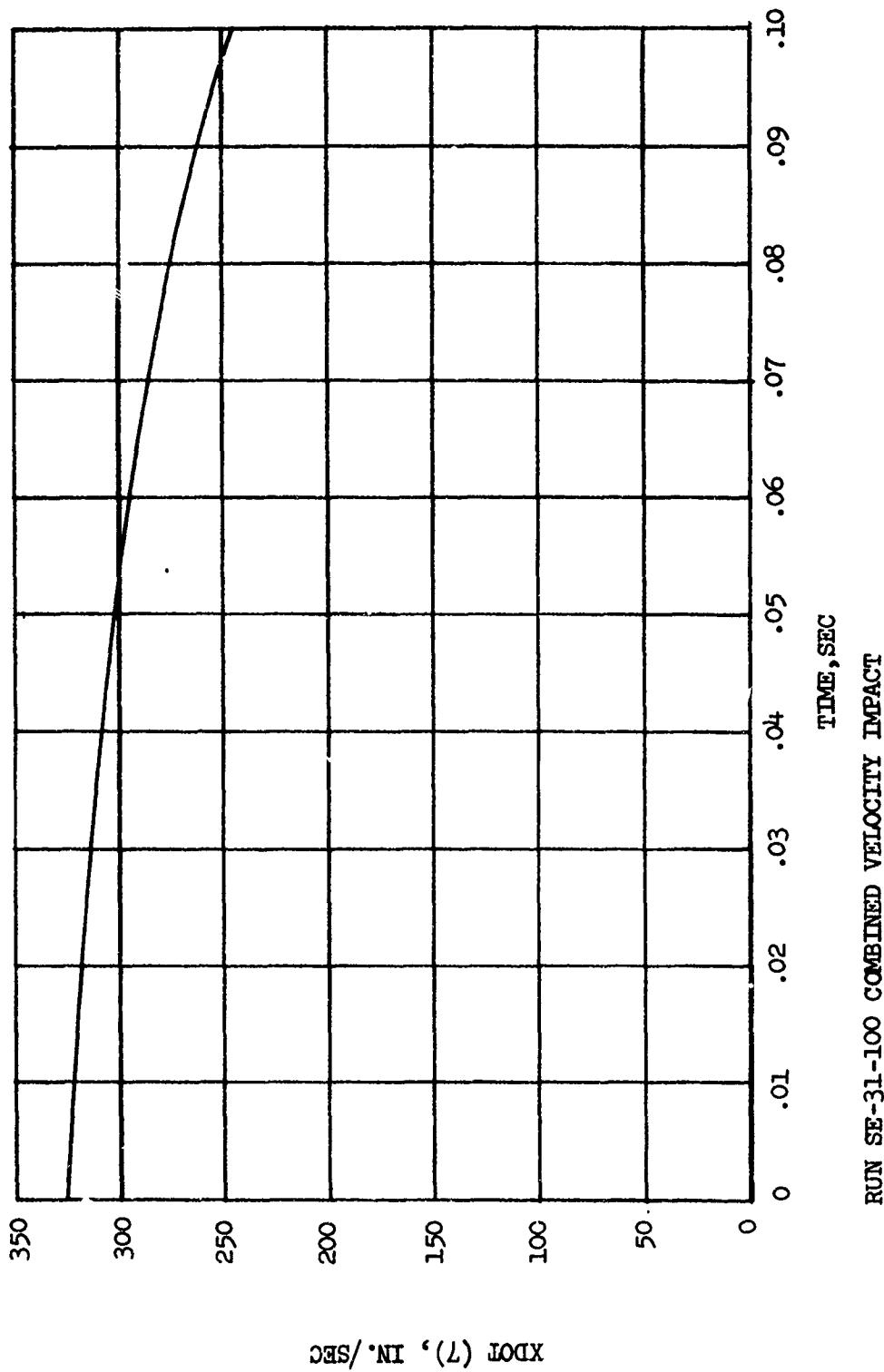


Figure 99. Engine Mass Longitudinal Velocity, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

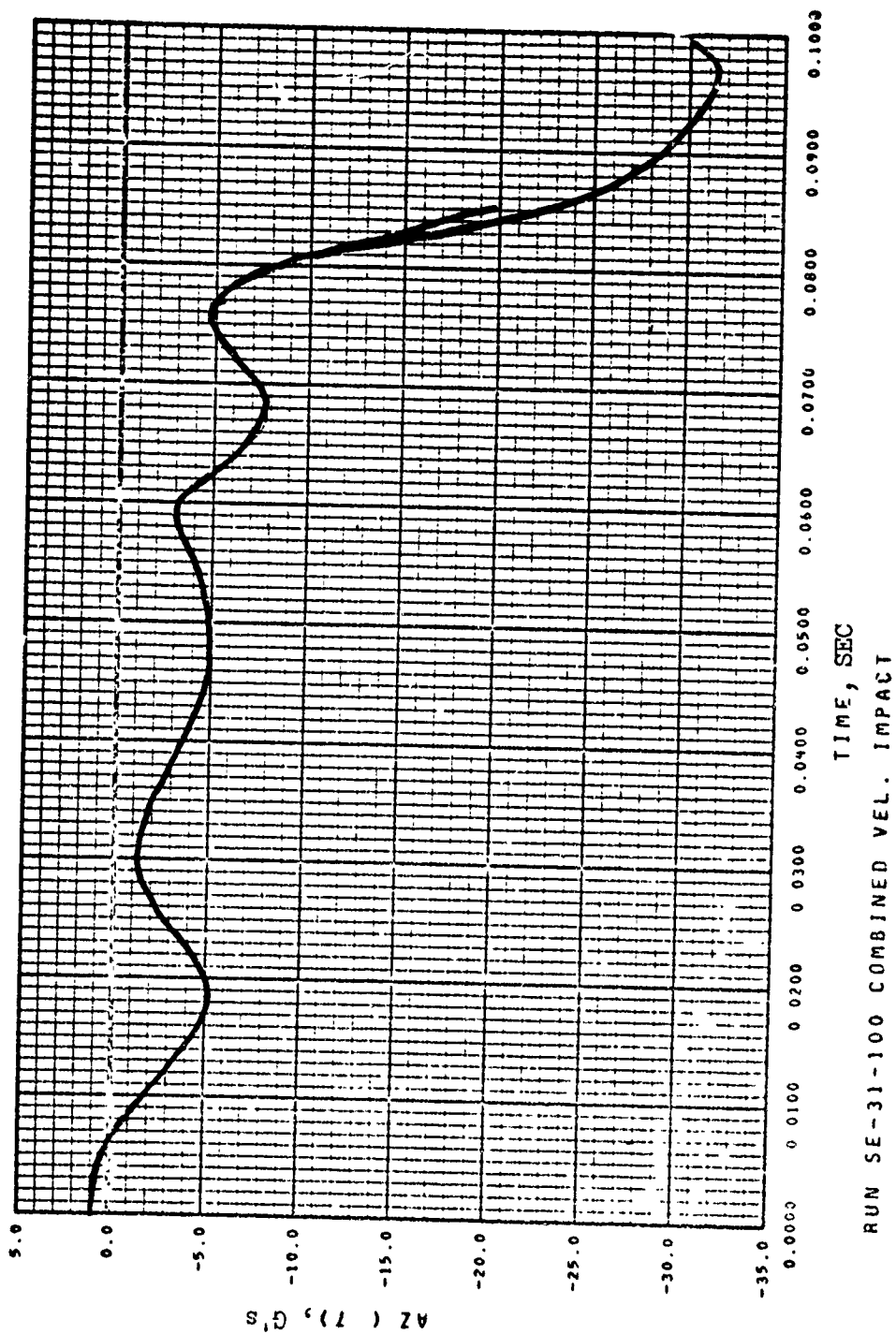


Figure 100. Engine Mass Vertical Acceleration, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

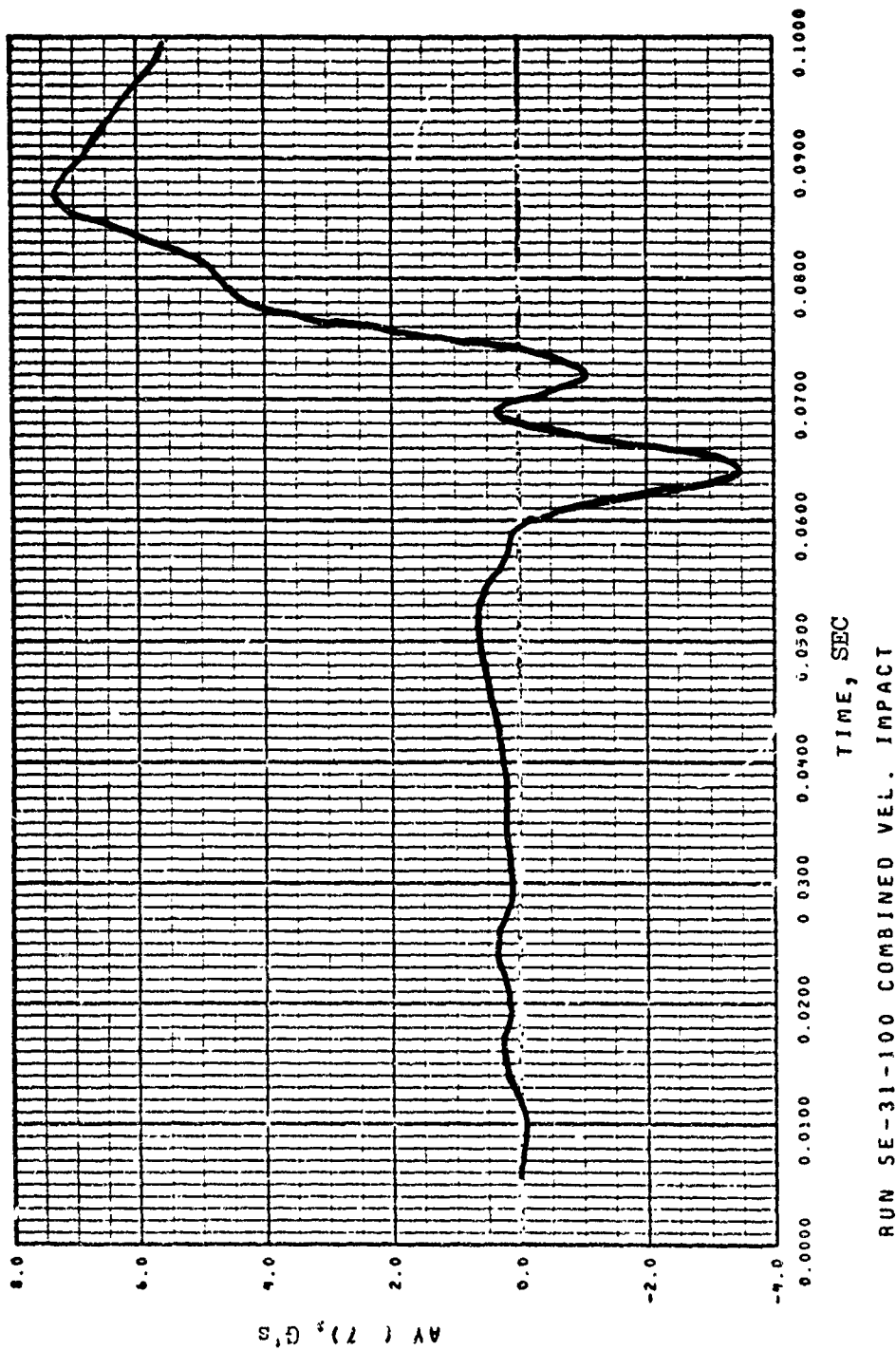


Figure 101. Engine Mass Lateral Acceleration, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

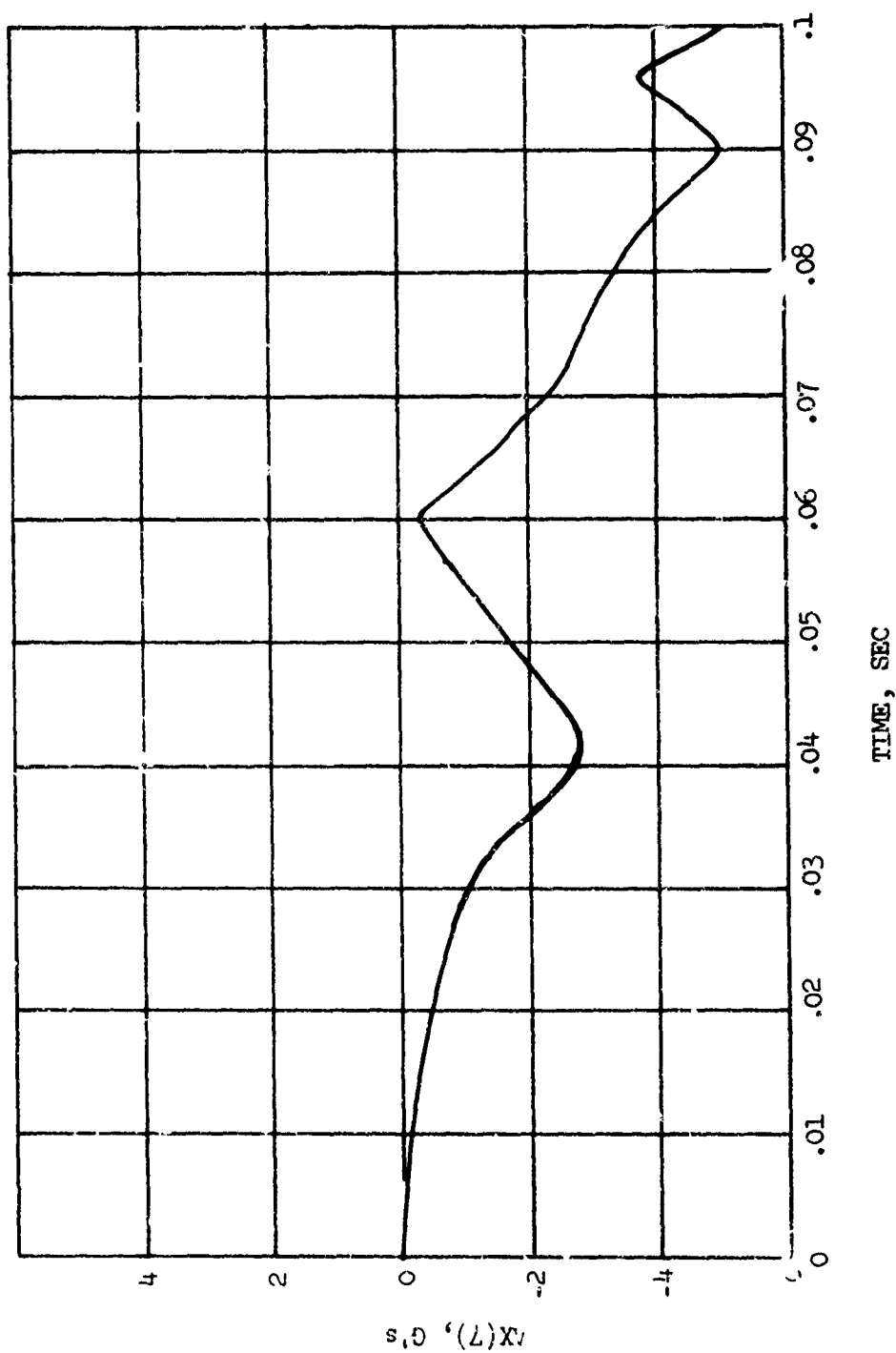


Figure 102. Engine Mass Longitudinal Acceleration, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

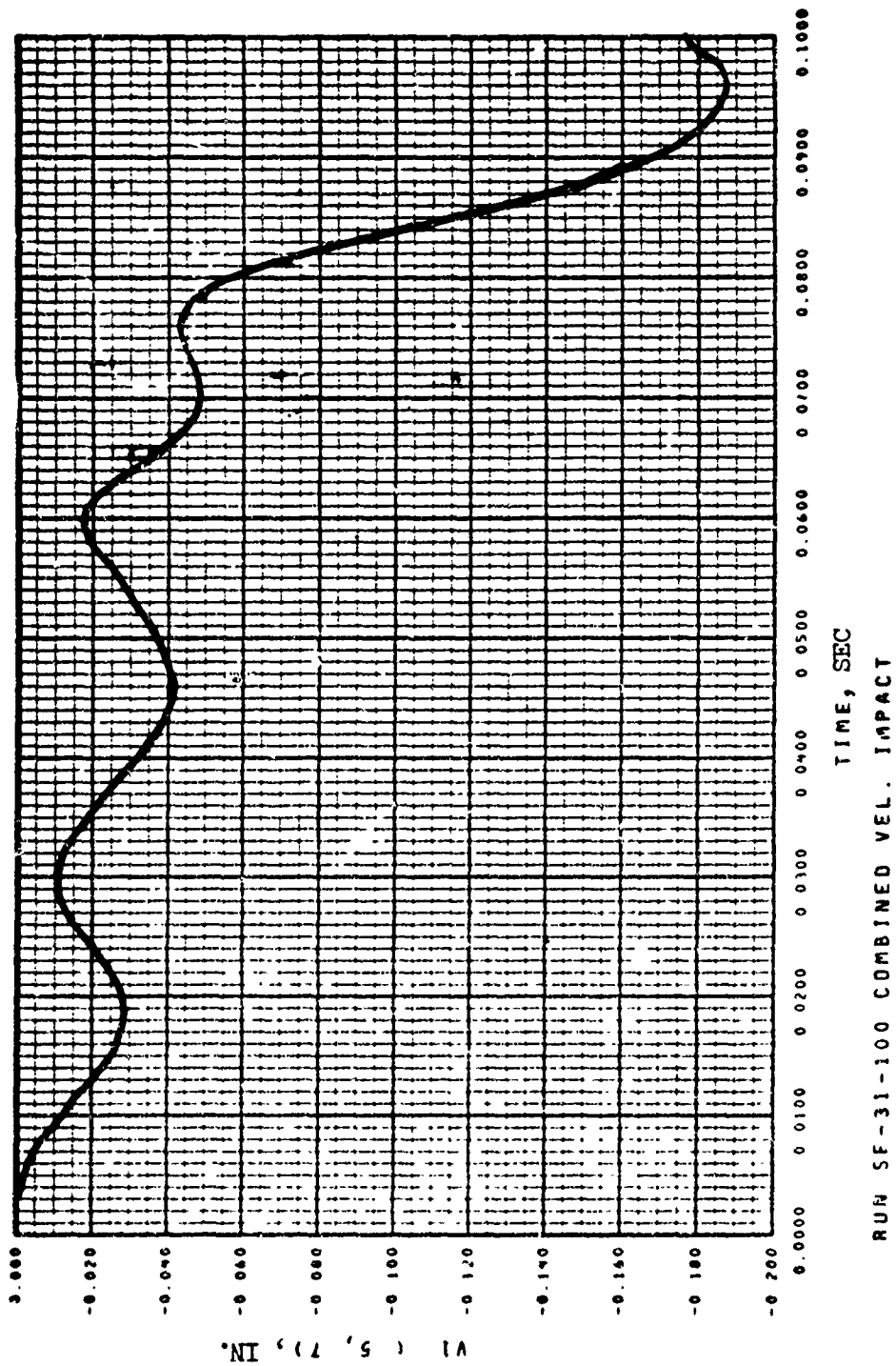


Figure 103. Engine Mount Vertical Reflection, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

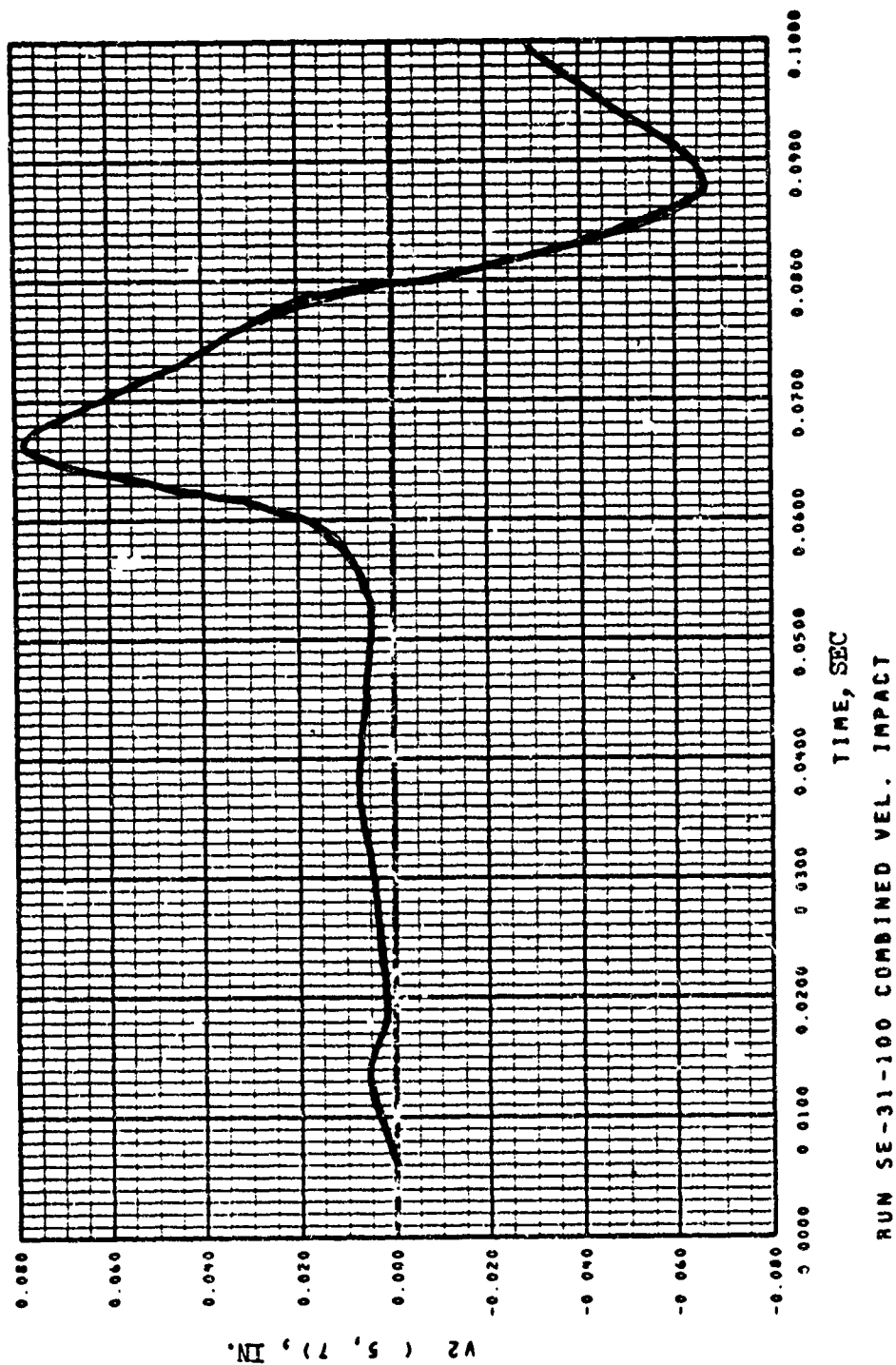


Figure 104. Engine Mount Lateral Deflection, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

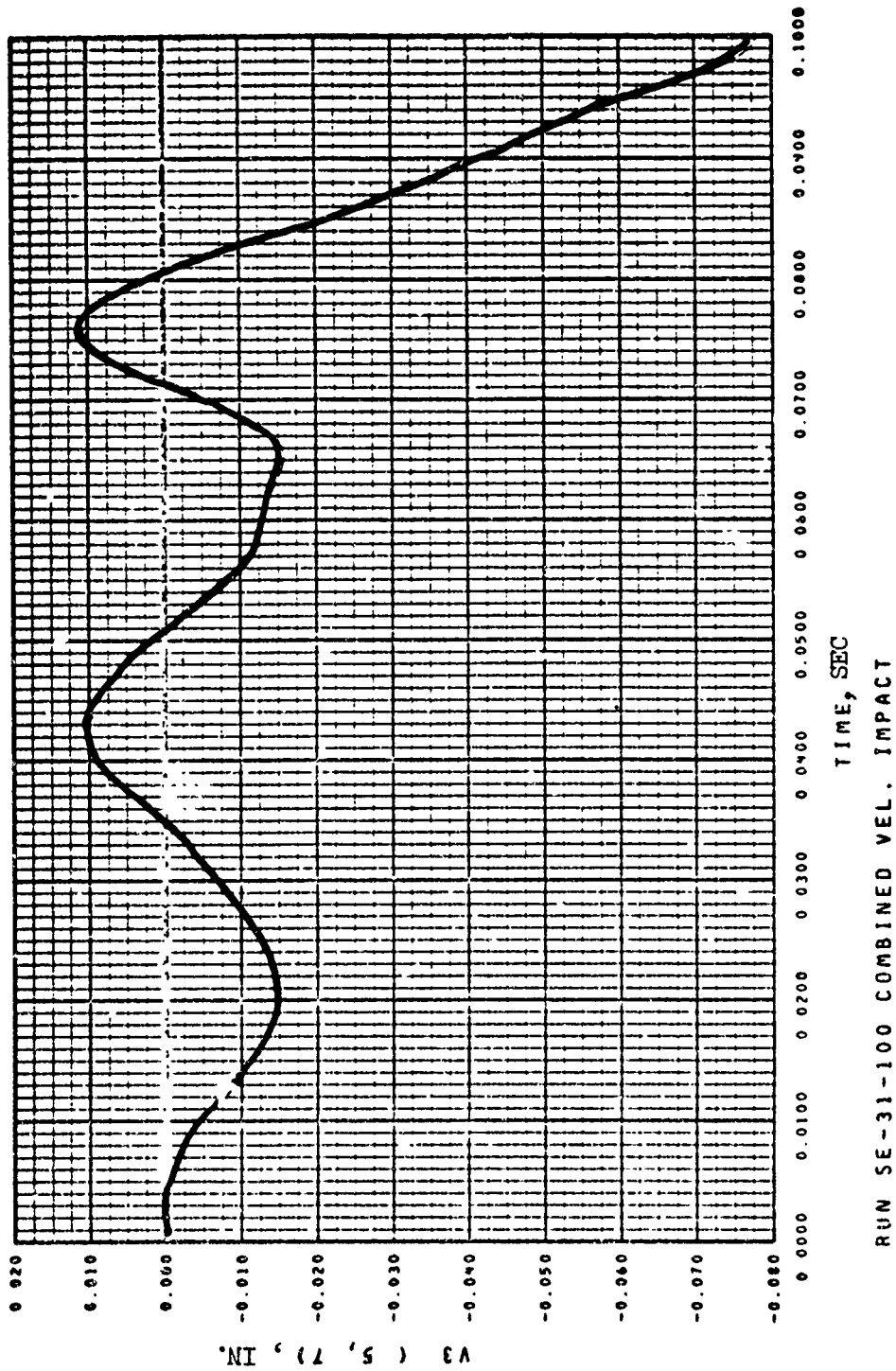


Figure 105. Engine Mount Longitudinal Deflection, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

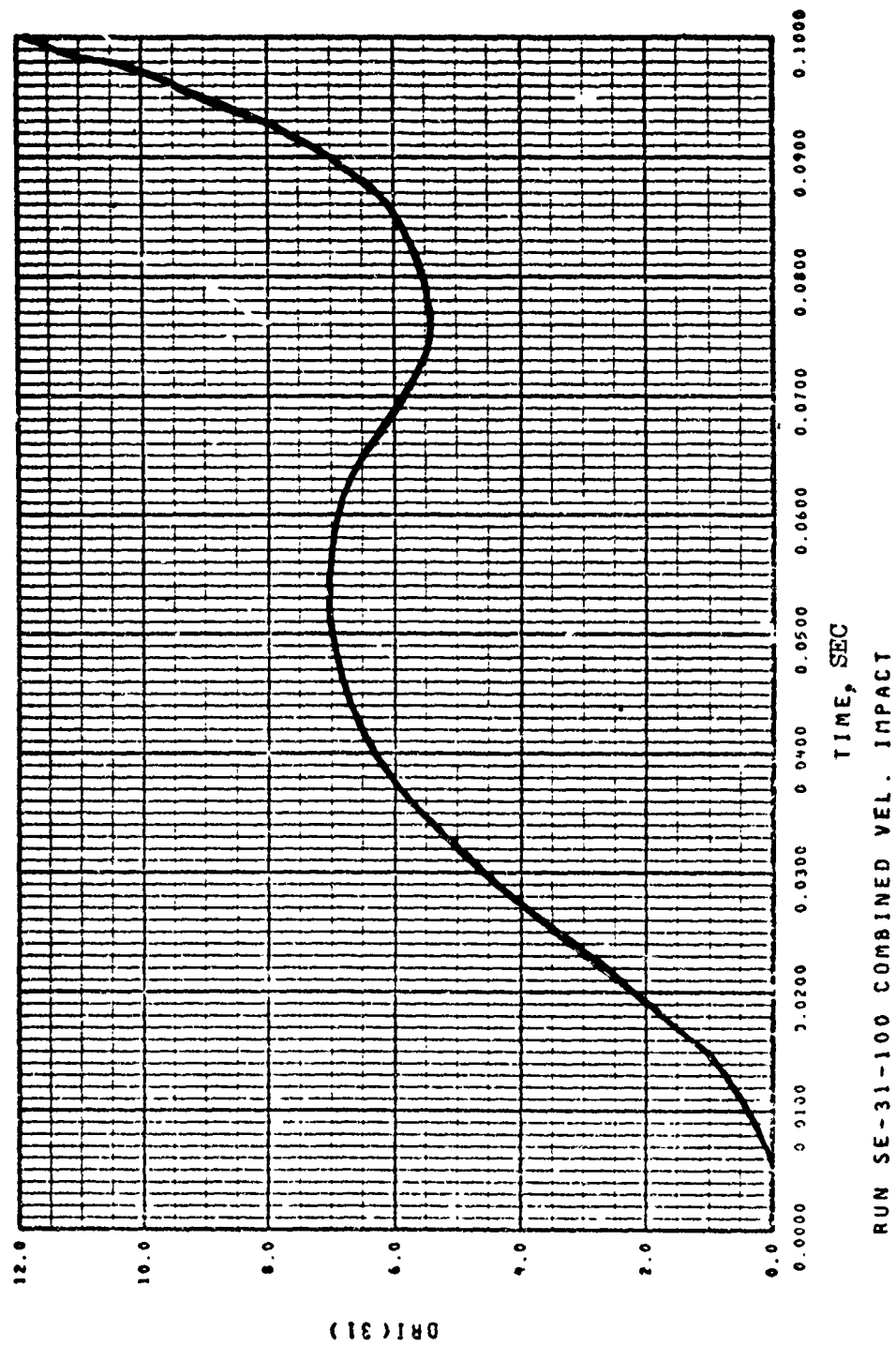


Figure 106. Forward DRI, Combined Vertical, Lateral and Longitudinal Impact Sample Case.

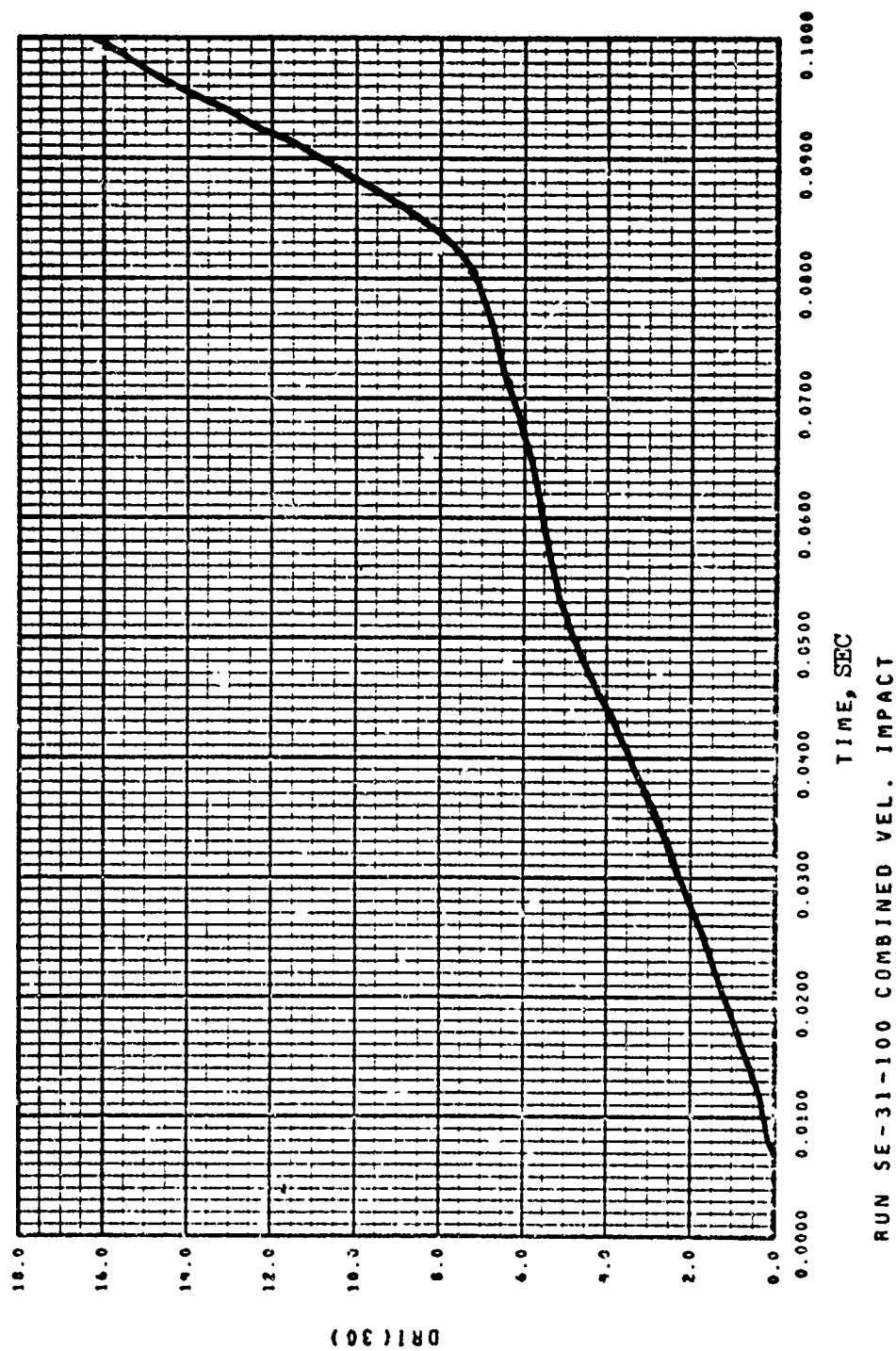


Figure 107. Aft DRI, Combined Vertical, Lateral and Longitudinal Impact Sample Case.



REV 20.1 MAY 71)

(7-0-0-0) 907114 (11) JAMF
SUNO, CUBU, NULJST, MULEX, LUGU, MAP, NUTUY, ID, KNTT

ISM 0018 C (26) DU 130 I = 1.0M

Figure 108. Program Listing.

174

1SM 0072	ANK(I,J) = 0.0	CRS00940
1SM 0073	ANK(I,J) = 0.0	CRS00950
1SM 0074	DN(I,J) = 0.0	CRS00960
1SM 0075	DN(I,J) = 0.0	CRS00970
1SM 0076	DN(I,J) = 0.0	CRS00980
1SM 0077	SE(I,J) = 0.0	
1SM 0078	DE(I,J) = 0.0	
1SM 0079	LU 160 L = 1.6	CRS00990
1SM 0080	SUMU(L,I,J) = 0.0	CRS01000
1SM 0081	N(I,J,L) = 0	CRS01010
1SM 0082	VE(I,L,I,J) = 0.0	CRS01020
1SM 0083	UG 160 K = 1.6	CRS01030
1SM 0084	FR(I,K,I,J) = 0.0	CRS01040
1SM 0085	160 FHEA(I,L,K,I,J) = 0.0	CRS01050
1SM 0086	DU 160 J = 1.3	CRS01060
1SM 0087	UG 160 K = 1.3	CRS01070
1SM 0088	165 VELDOT(J,K) = 0.0	CRS01080
1SM 0089	C DO INITIAL CONDITIONS	CRS01090
1SM 0090	CALL JL	CRS01100
1SM 0091	C DO ALL THE LA(I,J) INTO DJJ	CRS01110
1SM 0092	CALL DUATJ	CRS01120
1SM 0093	CALL LFRIV	CRS01130
1SM 0094	CALL PRINT	CRS01140
1SM 0095	IPL = 0	CRS01150
1SM 0096	IF (TEMPU-NE-G) CALL SAVE	CRS01160
1SM 0097	DU 200 I = 1-MM	CRS01170
1SM 0098	C PRESII GAD VALUES	CRS01180
1SM 0099	PI(I,J) = 0.0	CRS01190
1SM 0100	Q(I,J) = 0.0	CRS01200
1SM 0101	M(I,J) = 0.0	CRS01210
1SM 0102	Y(I,J) = 0.0	CRS01220
1SM 0103	Z(I,J) = 0.0	CRS01230
1SM 0104	PH(I,J) = 0.0	CRS01240
1SM 0105	THELOC(I) = THELOC(I)	CRS01250
1SM 0106	PSIGLOC(I) = PSIGLOC(I)	CRS01260
1SM 0107	P(I) = P(I)	CRS01270
1SM 0108	Q(I) = Q(I)	CRS01280
1SM 0109	R(I) = R(I)	CRS01290
1SM 0110	U(I) = U(I)	CRS01300
1SM 0111	V(I) = V(I)	CRS01310
1SM 0112	W(I) = W(I)	CRS01320
1SM 0113	M(I) = M(I)	CRS01330
1SM 0114	C DO 1ST STEP CYCLE	CRS01340
1SM 0115	UP(I) = DELTAP(I)	CRS01350
1SM 0116	DN(I) = DELTAD(I)	CRS01360
1SM 0117	PI(I) = DELTAP(I)	CRS01370
1SM 0118	Q(I) = DELTAP(I)	CRS01380
1SM 0119	M(I) = DELTAP(I)	CRS01390
1SM 0120	Y(I) = DELTAP(I)	CRS01400
1SM 0121	Z(I) = DELTAP(I)	CRS01410
1SM 0122	PH(I) = DELTAP(I)	CRS01420
1SM 0123	THELOC(I) = THELOC(I)	CRS01430
1SM 0124	PSIGLOC(I) = PSIGLOC(I)	CRS01440
1SM 0125	P(I) = P(I)	CRS01450

Figure 108. (Continued)

Figure 108. (Continued)


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1SM 0176      IMCLOUT1 = IPETAT1)      CR502000
1SM 0179      IMETAT1 = 1              CR502010
1SM 0180      DTMETAT1 = IMETAT1 - IMCLOUT1)      CR502020
1SM 0181      T = PSICLOUT1 + DT2 * PSIDOT1)      CR502030
1SM 0182      PSICLOUT1 = PST1)      CR502040
1SM 0183      PST1 = 1                CR502050
1SM 0184      UPS1111 = PST1111 - PSICLOUT1)      CR502060
1SM 0185      T = POLCLOUT1 + DT2 * PUOT1)      CR502070
1SM 0186      POLCLOUT1 = P11)      CR502080
1SM 0187      P11 = 1                CR502090
1SM 0188      T = CLOD11 + DT2 * QUOT1)      CR502100
1SM 0189      CLOD11 = Q11)      CR502110
1SM 0190      QUOT1 = 1              CR502120
1SM 0191      T = FOLD11 + DT2 * RDOT1)      CR502130
1SM 0192      RDOT1 = R11)      CR502140
1SM 0193      R11 = 1                CR502150
1SM 0194      T = WULD11 + DT2 * UDOT1)      CR502160
1SM 0195      WULD11 = U11)      CR502170
1SM 0196      UDOT1 = 1              CR502180
1SM 0197      T = VULD11 + DT2 * VDOT1)      CR502190
1SM 0198      VULD11 = V11)      CR502200
1SM 0199      V11 = 1                CR502210
1SM 0200      T = WULD11 + DT2 * WDOT1)      CR502220
1SM 0201      WULD11 = W11)      CR502230
1SM 0202      W11 = 1                CR502240
1SM 0203      30C CONTINUE          CR502250
1SM 0204      IF (TIME - TIME1) .LE. 1.0, 1.0, 500      CR502260
1SM 0205      50C IF (TIME1) .5500, 5500, 5000      CR502270
1SM 0206      5000 PRINT 2000      CR502280
1SM 0207      2000 FORMAT(1M1, 7X, 15H RUPTURE SUMMARY, /)      CR502290
1SM 0208      PRINT 3000          CR502300
1SM 0209      3000 FORMAT(1M, 9X, 1M1, 7X, 1M3, 4TIME, /)      CR502310
1SM 0210      PRINT 4000, ((RUPT1(KR1)), RUPT1(KAT1), RUPT1(KRT1), KRT=1, KRUPT1)      CR502320
1SM 0211      4000 FORMAT(1M, 2110, F10.5)      CR502330
1SM 0212      5500 IF (RPM1, LE. 0) GO TO 6000      CR502340
1SM 0213      PRINT 5501          CR502350
1SM 0214      5501 FORMAT(1M1, 7X, 15H CONTROL VOLUME PENETRATIONS, /)      CR502360
1SM 0215      PRINT 5502          CR502370
1SM 0216      5502 FORMAT(1M, 13X, 4TIME, 5, 10X, 5PASS, /)      CR502380
1SM 0217      PRINT 5503, ((PEN1(K1), IPEN1(K1), IPEN1(K1), K1=1, KPEN1)      CR502390
1SM 0218      5503 FORMAT(1M, 10X, F10.5, 110)      CR502400
1SM 0219      6000 IF (IMPLUT, NE. 0) CALL TOLP      CR502410
1SM 0220      GO TO 1              CR502420
1SM 0221      1000 IF (IPLSM, NE. 0) CALL EXITG(ZAR)      CR502430
1SM 0222      STOP                CR502440
1SM 0223      END                  CR502450

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Figure 108. (Continued)

		PAGE 002
ISM 0021	70 ISUPSMIJJ = 0	CRS02250
ISM 0042	DO 72 I=1,NM	CRS02260
ISM 0023	7, IPENSMIJJ=0	CRS02270
ISM 0024	C DO ALL THE (AII)(AJJJ)	CP50280
ISM 0025	60 DO 10 I = 1,NM	CRS02890
ISM 0026	ARG = PHIIJ	CRS02900
ISM 0027	S1 = SIN(ARG)	CRS02910
ISM 0028	C1 = CUS(ARG)	CRS02920
ISM 0029	ARG = THETA(1)	CRS02930
ISM 0030	S2 = SIN(ARG)	CRS02940
ISM 0031	C2 = CUS(ARG)	CRS02950
ISM 0032	ARC = PSI(1)	CRS02960
ISM 0033	S3 = SIN(ARG)	CRS02970
ISM 0034	C3 = CUS(ARG)	CRS02980
ISM 0035	DO 40 J = 1,6	CRS02990
ISM 0036	1 = SINGOS(IJ)	CRS03000
ISM 0037	IF(1) 45,40,50	CRS03010
ISM 0038	45 I = -1	CRS03020
ISM 0039	50 IF(1-1,1-10) 55,40,40	CRS03030
ISM 0040	55 SINGOS(IJ) = 0.0	CRS03040
ISM 0041	40 CONTINUE	CRS03050
ISM 0042	C	CRS03060
ISM 0043	J = 4*(I-1)	CRS03070
ISM 0044	C MOVE AI 5 TO OLD AI 5	CRS03080
ISM 0045	DO 4 JJ = 1,4	CRS03090
ISM 0046	4 OAI(JJJ) = BIJ(JJJ)	CRS03100
ISM 0047	S12 = S1+S2	CRS03110
ISM 0048	C12 = C1+C2	CRS03120
ISM 0049	AI(1) = C2+C3	CRS03130
ISM 0050	BIJ(J+1) = AI(1)	CRS03140
ISM 0051	AI(2) = C2+S3	CRS03150
ISM 0052	BIJ(J+2) = AI(2)	CRS03160
ISM 0053	AI(3) = -S2	CRS03170
ISM 0054	BIJ(J+3) = AI(3)	CRS03180
ISM 0055	AI(4) = -C1+S3+S12+C3	CRS03190
ISM 0056	BIJ(J+4) = AI(4)	CRS03200
ISM 0057	AI(5) = C1+C3+S12+S3	CRS03210
ISM 0058	BIJ(J+5) = AI(5)	CRS03220
ISM 0059	AI(6) = S1+C4	CRS03230
ISM 0060	BIJ(J+6) = AI(6)	CRS03240
ISM 0061	AI(7) = S1+S3+C12+C3	CRS03250
ISM 0062	BIJ(J+7) = AI(7)	CRS03260
ISM 0063	AI(8) = -S1+C3+C12+S3	CRS03270
ISM 0064	BIJ(J+8) = AI(8)	CRS03280
ISM 0065	AI(9) = C1+C2	CRS03290
ISM 0066	BIJ(J+9) = AI(9)	CRS03300
ISM 0067	C (27)	CRS03310
ISM 0068	PP = P(1)	CRS03320
ISM 0069	QQ = Q(1)	CRS03330
ISM 0070	RR = R(1)	CRS03340
ISM 0071	UU = U(1)	CRS03350
ISM 0072	VV = V(1)	CRS03360
ISM 0073	WW = W(1)	CRS03370
ISM 0074	XX = X(1)	CRS03380
ISM 0075	YY = Y(1)	CRS03390
ISM 0076	ZZ = Z(1)	CRS03400
ISM 0077	AA = A(1)	CRS03410
ISM 0078	BB = B(1)	CRS03420
ISM 0079	CC = C(1)	CRS03430
ISM 0080	DD = D(1)	CRS03440
ISM 0081	EE = E(1)	CRS03450
ISM 0082	FF = F(1)	CRS03460
ISM 0083	GG = G(1)	CRS03470
ISM 0084	HH = H(1)	CRS03480
ISM 0085	II = I(1)	CRS03490
ISM 0086	JJ = J(1)	CRS03500
ISM 0087	KK = K(1)	CRS03510
ISM 0088	LL = L(1)	CRS03520
ISM 0089	MM = M(1)	CRS03530
ISM 0090	NN = N(1)	CRS03540
ISM 0091	OO = O(1)	CRS03550
ISM 0092	PP = P(1)	CRS03560
ISM 0093	QQ = Q(1)	CRS03570
ISM 0094	RR = R(1)	CRS03580
ISM 0095	SS = S(1)	CRS03590
ISM 0096	TT = T(1)	CRS03600
ISM 0097	UU = U(1)	CRS03610
ISM 0098	VV = V(1)	CRS03620
ISM 0099	WW = W(1)	CRS03630
ISM 0100	XX = X(1)	CRS03640
ISM 0101	YY = Y(1)	CRS03650
ISM 0102	ZZ = Z(1)	CRS03660
ISM 0103	AA = A(1)	CRS03670
ISM 0104	BB = B(1)	CRS03680
ISM 0105	CC = C(1)	CRS03690
ISM 0106	DD = D(1)	CRS03700
ISM 0107	EE = E(1)	CRS03710
ISM 0108	FF = F(1)	CRS03720
ISM 0109	GG = G(1)	CRS03730
ISM 0110	HH = H(1)	CRS03740
ISM 0111	II = I(1)	CRS03750
ISM 0112	JJ = J(1)	CRS03760
ISM 0113	KK = K(1)	CRS03770
ISM 0114	LL = L(1)	CRS03780
ISM 0115	MM = M(1)	CRS03790
ISM 0116	NN = N(1)	CRS03800
ISM 0117	OO = O(1)	CRS03810
ISM 0118	PP = P(1)	CRS03820
ISM 0119	QQ = Q(1)	CRS03830
ISM 0120	RR = R(1)	CRS03840
ISM 0121	SS = S(1)	CRS03850
ISM 0122	TT = T(1)	CRS03860
ISM 0123	UU = U(1)	CRS03870
ISM 0124	VV = V(1)	CRS03880
ISM 0125	WW = W(1)	CRS03890
ISM 0126	XX = X(1)	CRS03900
ISM 0127	YY = Y(1)	CRS03910
ISM 0128	ZZ = Z(1)	CRS03920
ISM 0129	AA = A(1)	CRS03930
ISM 0130	BB = B(1)	CRS03940
ISM 0131	CC = C(1)	CRS03950
ISM 0132	DD = D(1)	CRS03960
ISM 0133	EE = E(1)	CRS03970
ISM 0134	FF = F(1)	CRS03980
ISM 0135	GG = G(1)	CRS03990
ISM 0136	HH = H(1)	CRS04000
ISM 0137	II = I(1)	CRS04010
ISM 0138	JJ = J(1)	CRS04020
ISM 0139	KK = K(1)	CRS04030
ISM 0140	LL = L(1)	CRS04040
ISM 0141	MM = M(1)	CRS04050
ISM 0142	NN = N(1)	CRS04060
ISM 0143	OO = O(1)	CRS04070
ISM 0144	PP = P(1)	CRS04080
ISM 0145	QQ = Q(1)	CRS04090
ISM 0146	RR = R(1)	CRS04100
ISM 0147	SS = S(1)	CRS04110
ISM 0148	TT = T(1)	CRS04120
ISM 0149	UU = U(1)	CRS04130
ISM 0150	VV = V(1)	CRS04140
ISM 0151	WW = W(1)	CRS04150
ISM 0152	XX = X(1)	CRS04160
ISM 0153	YY = Y(1)	CRS04170
ISM 0154	ZZ = Z(1)	CRS04180
ISM 0155	AA = A(1)	CRS04190
ISM 0156	BB = B(1)	CRS04200
ISM 0157	CC = C(1)	CRS04210
ISM 0158	DD = D(1)	CRS04220
ISM 0159	EE = E(1)	CRS04230
ISM 0160	FF = F(1)	CRS04240
ISM 0161	GG = G(1)	CRS04250
ISM 0162	HH = H(1)	CRS04260
ISM 0163	II = I(1)	CRS04270
ISM 0164	JJ = J(1)	CRS04280
ISM 0165	KK = K(1)	CRS04290
ISM 0166	LL = L(1)	CRS04300
ISM 0167	MM = M(1)	CRS04310
ISM 0168	NN = N(1)	CRS04320
ISM 0169	OO = O(1)	CRS04330
ISM 0170	PP = P(1)	CRS04340
ISM 0171	QQ = Q(1)	CRS04350
ISM 0172	RR = R(1)	CRS04360
ISM 0173	SS = S(1)	CRS04370
ISM 0174	TT = T(1)	CRS04380
ISM 0175	UU = U(1)	CRS04390
ISM 0176	VV = V(1)	CRS04400
ISM 0177	WW = W(1)	CRS04410
ISM 0178	XX = X(1)	CRS04420
ISM 0179	YY = Y(1)	CRS04430
ISM 0180	ZZ = Z(1)	CRS04440
ISM 0181	AA = A(1)	CRS04450
ISM 0182	BB = B(1)	CRS04460
ISM 0183	CC = C(1)	CRS04470
ISM 0184	DD = D(1)	CRS04480
ISM 0185	EE = E(1)	CRS04490
ISM 0186	FF = F(1)	CRS04500
ISM 0187	GG = G(1)	CRS04510
ISM 0188	HH = H(1)	CRS04520
ISM 0189	II = I(1)	CRS04530
ISM 0190	JJ = J(1)	CRS04540
ISM 0191	KK = K(1)	CRS04550
ISM 0192	LL = L(1)	CRS04560
ISM 0193	MM = M(1)	CRS04570
ISM 0194	NN = N(1)	CRS04580
ISM 0195	OO = O(1)	CRS04590
ISM 0196	PP = P(1)	CRS04600
ISM 0197	QQ = Q(1)	CRS04610
ISM 0198	RR = R(1)	CRS04620
ISM 0199	SS = S(1)	CRS04630
ISM 0200	TT = T(1)	CRS04640
ISM 0201	UU = U(1)	CRS04650
ISM 0202	VV = V(1)	CRS04660
ISM 0203	WW = W(1)	CRS04670
ISM 0204	XX = X(1)	CRS04680
ISM 0205	YY = Y(1)	CRS04690
ISM 0206	ZZ = Z(1)	CRS04700
ISM 0207	AA = A(1)	CRS04710
ISM 0208	BB = B(1)	CRS04720
ISM 0209	CC = C(1)	CRS04730
ISM 0210	DD = D(1)	CRS04740
ISM 0211	EE = E(1)	CRS04750
ISM 0212	FF = F(1)	CRS04760
ISM 0213	GG = G(1)	CRS04770
ISM 0214	HH = H(1)	CRS04780
ISM 0215	II = I(1)	CRS04790
ISM 0216	JJ = J(1)	CRS04800
ISM 0217	KK = K(1)	CRS04810
ISM 0218	LL = L(1)	CRS04820
ISM 0219	MM = M(1)	CRS04830
ISM 0220	NN = N(1)	CRS04840
ISM 0221	OO = O(1)	CRS04850
ISM 0222	PP = P(1)	CRS04860
ISM 0223	QQ = Q(1)	CRS04870
ISM 0224	RR = R(1)	CRS04880
ISM 0225	SS = S(1)	CRS04890
ISM 0226	TT = T(1)	CRS04900
ISM 0227	UU = U(1)	CRS04910
ISM 0228	VV = V(1)	CRS04920
ISM 0229	WW = W(1)	CRS04930
ISM 0230	XX = X(1)	CRS04940
ISM 0231	YY = Y(1)	CRS04950
ISM 0232	ZZ = Z(1)	CRS04960
ISM 0233	AA = A(1)	CRS04970
ISM 0234	BB = B(1)	CRS04980
ISM 0235	CC = C(1)	CRS04990
ISM 0236	DD = D(1)	CRS05000
ISM 0237	EE = E(1)	CRS05010
ISM 0238	FF = F(1)	CRS05020
ISM 0239	GG = G(1)	CRS05030
ISM 0240	HH = H(1)	CRS05040
ISM 0241	II = I(1)	CRS05050
ISM 0242	JJ = J(1)	CRS05060
ISM 0243	KK = K(1)	CRS05070
ISM 0244	LL = L(1)	CRS05080
ISM 0245	MM = M(1)	CRS05090
ISM 0246	NN = N(1)	CRS05100
ISM 0247	OO = O(1)	CRS05110
ISM 0248	PP = P(1)	CRS05120
ISM 0249	QQ = Q(1)	CRS05130
ISM 0250	RR = R(1)	CRS05140
ISM 0251	SS = S(1)	CRS05150
ISM 0252	TT = T(1)	CRS05160
ISM 0253	UU = U(1)	CRS05170
ISM 0254	VV = V(1)	CRS05180
ISM 0255	WW = W(1)	CRS05190
ISM 0256	XX = X(1)	CRS05200
ISM 0257	YY = Y(1)	CRS05210
ISM 0258	ZZ = Z(1)	CRS05220
ISM 0259	AA = A(1)	CRS05230
ISM 0260	BB = B(1)	CRS05240
ISM 0261	CC = C(1)	CRS05250
ISM 0262	DD = D(1)	CRS05260
ISM 0263	EE = E(1)	CRS05270
ISM 0264	FF = F(1)	CRS05280
ISM 0265	GG = G(1)	CRS05290
ISM 0266	HH = H(1)	CRS05300
ISM 0267	II = I(1)	CRS05310
ISM 0268	JJ = J(1)	CRS05320
ISM 0269	KK = K(1)	CRS05330
ISM 0270	LL = L(1)	CRS05340
ISM 0271	MM = M(1)	CRS05350
ISM 0272	NN = N(1)	CRS05360
ISM 0273	OO = O(1)	CRS05370
ISM 0274	PP = P(1)	CRS05380
ISM 0275	QQ = Q(1)	CRS05390
ISM 0276	RR = R(1)	CRS05400
ISM 0277	SS = S(1)	CRS05410
ISM 0278	TT = T(1)	CRS05420
ISM 0279	UU = U(1)	CRS05430
ISM 0280	VV = V(1)	CRS05440
ISM 0281	WW = W(1)	CRS05450
ISM 0282	XX = X(1)	CRS05460
ISM 0283	YY = Y(1)	CRS05470
ISM 0284	ZZ = Z(1)	CRS05480
ISM 0285	AA = A(1)	CRS05490
ISM 0286	BB = B(1)	CRS05500
ISM 0287	CC = C(1)	CRS05510
ISM 0288	DD = D(1)	CRS05520
ISM 0289	EE = E(1)	CRS05530
ISM 0290	FF = F(1)	CRS05540
ISM 0291	GG = G(1)	CRS05550
ISM 0292	HH = H(1)	CRS05560
ISM 0293	II = I(1)	CRS05570
ISM 0294	JJ = J(1)	CRS05580
ISM 0295	KK = K(1)	CRS05590
ISM 0296	LL = L(1)	CRS05600
ISM 0297	MM = M(1)	CRS05610
ISM 0298	NN = N(1)	CRS05620
ISM 0299	OO = O(1)	CRS05630
ISM 0300	PP = P(1)	CRS05640
ISM 0301	QQ = Q(1)	CRS05650
ISM 0302	RR = R(1)	CRS05660
ISM 0303	SS = S(1)	CRS05670
ISM 0304	TT = T(1)	CRS05680
ISM 0305	UU = U(1)	CRS05690
ISM 0306	VV = V(1)	CRS05700
ISM 0307	WW = W(1)	CRS05710
ISM 0308	XX = X(1)	CRS05720
ISM 0309	YY = Y(1)	CRS05730
ISM 0310	ZZ = Z(1)	CRS05740
ISM 0311	AA = A(1)	CRS05750
ISM 0312	BB = B(1)	CRS05760
ISM 0313	CC = C(1)	CRS05770
ISM 0314	DD = D(1)	CRS05780
ISM 0315	EE = E(1)	CRS05790
ISM 0316	FF = F(1)	CRS05800
ISM 0317	GG = G(1)	CRS05810
ISM 0318	HH = H(1)	CRS05820
ISM 0319	II = I(1)	CRS05830
ISM 0320	JJ = J(1)	CRS05840
ISM 0321	KK = K(1)	CRS05850
ISM 0322	LL = L(1)	CRS05860
ISM 0323	MM = M(1)	CRS05870
ISM 0324	NN = N(1)	CRS05880
ISM 0325	OO = O(1)	CRS05890
ISM 0326	PP = P(1)	CRS05900
ISM 0327	QQ = Q(1)	CRS05910
ISM 0328	RR = R(1)	CRS05920
ISM 0329	SS = S(1)	CRS05930
ISM 0330	TT = T(1)	CRS05940
ISM 0331	UU = U(1)	CRS05950
ISM 0332	V	

1SM 0071	260111) = 21111000+AL1610VV+AL1710W	CR503500
1SM 0072	70011) = 7001111	CR503400
1SM 0073	700111) = AL161000+AL1510VV+AL1610W	CR503410
1SM 0074	70011) = 7101111	CR503420
1SM 0075	700111) = AL161000+AL1610VV+AL1910W	CR503430
	C (24)1291	CR503440
1SM 0076	CC = 3170	CR503450
1SM 0077	CC = 3170	CR503460
1SM 0078	PH0011) = PH00111	CR503470
1SM 0079	PH00111) = PH00111+PH00111+PH00111	CR503480
1SM 0080	PH0011) = PH00111	CR503490
1SM 0081	PH0011) = PH00111	CR503500
1SM 0082	PH0011) = PH00111	CR503510
1SM 0083	PH0011) = PH00111	CR503520
	C LO AICU1 NUM	CR503530
1SM 0084	T = PH00111+CC	CR503540
1SM 0085	T1 = PH00111+CC	CR503550
1SM 0086	T2 = PH00111+CC	CR503560
1SM 0087	T3 = PH00111+CC	CR503570
1SM 0088	CL1J111) = -CL1J111+CL1J111	CR503580
1SM 0089	CL1J111) = -CL1J111+CL1J111	CR503590
1SM 0090	CL1J111) = -CL1J111+CL1J111	CR503600
1SM 0091	CL1J111) = -CL1J111+CL1J111	CR503610
1SM 0092	CL1J111) = -CL1J111+CL1J111	CR503620
1SM 0093	CL1J111) = -CL1J111+CL1J111	CR503630
1SM 0094	CL1J111) = -CL1J111+CL1J111	CR503640
1SM 0095	CL1J111) = -CL1J111+CL1J111	CR503650
1SM 0096	CL1J111) = -CL1J111+CL1J111	CR503660
1SM 0097	CL1J111) = -CL1J111+CL1J111	CR503670
	C CUMULAT 2, Y, Z, PH00111, T, P51	CR503680
1SM 0098	Y111) = Y111+Y111+Y111+Y111+Y111+Y111	CR503690
1SM 0099	Y111) = Y111+Y111+Y111+Y111+Y111+Y111	CR503700
1SM 0100	Y111) = Y111+Y111+Y111+Y111+Y111+Y111	CR503710
1SM 0101	Y111) = Y111+Y111+Y111+Y111+Y111+Y111	CR503720
1SM 0102	Y111) = Y111+Y111+Y111+Y111+Y111+Y111	CR503730
1SM 0103	Y111) = Y111+Y111+Y111+Y111+Y111+Y111	CR503740
	C CLEAR THE DAMPING TERMS.	CR503750
1SM 0104	DP111) = 0.0	CR503760
1SM 0105	DP111) = 0.0	CR503770
1SM 0106	DP111) = 0.0	CR503780
1SM 0107	DP111) = 0.0	CR503790
1SM 0108	DP111) = 0.0	CR503800
1SM 0109	DP111) = 0.0	CR503810
1SM 0110	DP111) = 0.0	CR503820
	C DO 1000, IS MAIN DO LOOP TO GET TOTAL INTERNAL FORCES AND MOMENTS	CR503830
1SM 0111	CL1J11) = 0	CR503840
1SM 0112	CL1J11) = 0	CR503850
1SM 0113	CL1J11) = 0	CR503860
1SM 0114	DO 1000, IS = 1165	CR503870
1SM 0115	DO 1000, IS = 1165	CR503880
1SM 0116	DO 1000, IS = 1165	CR503890
1SM 0117	DO 1000, IS = 1165	CR503900
1SM 0118	DO 1000, IS = 1165	CR503910
1SM 0119	DO 1000, IS = 1165	CR503920

Figure 108. (Continued)

15M 0121	1 = 16(113)	CR503910
15M 0122	J = 36(113)	CR503940
15M 0123	C. IF WE GET TO A NEW I WE MUST MOVE (AI) INTO AI AND (AIDOT) INTO AIDOT	CR503950
15M 0124	IF (I-ELAST) 26,30,20	CR503960
15M 0125	20 ELAST = 1	CR503970
15M 0126	15 = 96(11-1)	CR503980
15M 0127	60 3-0 KS = 1,5	CR503990
15M 0128	15 = 15*1	CR504000
15M 0129	AI(OUT(KS)) = C1(115)	CR504010
15M 0130	DAI(KS) = OAI(115)-O1(115)	CR504020
15M 0131	320 AI(KS) = 61(115)	CR504030
15M 0132	30 X13 = X1(1)-X1(1)	CR504040
15M 0133	Y13 = Y1(1)-Y1(1)	CR504050
15M 0134	Z14 = Z1(1)-Z1(1)	CR504060
15M 0135	X140 = O2(113)	CR504070
15M 0136	Y140 = O2(113)	CR504080
15M 0137	Z140 = O2(113)	CR504090
15M 0138	O2(113) = X13	CR504100
15M 0139	O2(113) = Y13	CR504110
15M 0140	O2(113) = Z13	CR504120
15M 0141	15 = 96(11-1)	CR504130
15M 0142	135 = 96(11-1)	CR504140
15M 0143	60 310 KS = 1,5	CR504150
15M 0144	15 = 15*1	CR504160
15M 0145	135 = 135*1	CR504170
15M 0146	AI(KS) = O1(115)	CR504180
15M 0147	310 AI(KS) = 61(115)	CR504190
15M 0148	11 = O2(1)-O2(1)	CR504200
15M 0149	12 = O2(1)-O2(1)	CR504210
15M 0150	13 = O2(1)-O2(1)	CR504220
15M 0151	14 = 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Figure 108. (Continued)

Figure 108. (Continued)

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C AKL SAPI TO EE SU TIGHT.
C-----CHUG 14MM
15N 0224 110 PTH = P1R-1 CRS05070
15N 0225 IF (AKL-1-2RR(PTR)) GO TO 110 CRS05080
15N 0226 130 CHUG(11N) = P1R CRS05090
15N 0227 GO TO 14C CRS05100
15N 0228 CRS05110
15N 0229 CRS05120
15N 0230 CRS05130
15N 0231 120 PTH = P1R+1 CRS05140
15N 0232 IF (AKL-1-2RR(PTR)) GO TO 120 CRS05150
15N 0233 GO TO 130 CRS05160
15N 0234 220 PTH(14N) = PTH(14N)+UELEF CRS05170
15N 0235 0PTR = 0PTR+UELEF CRS05180
15N 0236 150 CONTINUE CRS05190
15N 0237 GO 631 K = 1+6 CRS05200
15N 0238 630 SUM(FK,IJ) = SUM(FK,IJ)+FKI CRS05210
15N 0239 C-----COMPUTE THE ENERGY HERE BUT ADD IT AFTER 230 (NO RUPTURE)
15N 0240 C-----CHUG USE AN IJ IF IT'S A DRI ELEMENT
15N 0241 IF (IJK(IJ,NL-1) BLANK) GO TO 632
15N 0242 DO 631 K = 1+6
15N 0243 SUM(I) = SUM(I)+SUM(FK,IJ)+FKI
15N 0244 IF (IJK(IJ,NL-1) BLANK) GO TO 632
15N 0245 PRINT 55, (IJK(IJ,NL-1),K=1,6)
15N 0246 999 FORMAT(1X,1P12(11.3))
15N 0247 632 IJL = IJL+6
15N 0248 DO 630 I = 1+6
15N 0249 IJL = IJL+1
15N 0250 I = VEE(IJL)+5011
15N 0251 VEE(IJL) = 7
15N 0252 C MOVE ON TO D FLP (13) ETC.
15N 0253 IJL = 0F(1)
15N 0254 IF (11) = 0F(1)
15N 0255 240 T = -1
15N 0256 250 IF (1-VMAX(IJL)) 230,240,260
15N 0257 260 TRUPT(IJL) = 1
15N 0258 X(IJ) = X(IJ)-XK(IJ)
15N 0259 Y(IJ) = Y(IJ)-YK(IJ)
15N 0260 Z(IJ) = Z(IJ)-ZK(IJ)
15N 0261 X(IJ) = X(IJ)-XK(IJ)
15N 0262 X(IJ) = X(IJ)-XK(IJ)
15N 0263 Y(IJ) = Y(IJ)-YK(IJ)
15N 0264 Y(IJ) = Y(IJ)-YK(IJ)
15N 0265 Z(IJ) = Z(IJ)-ZK(IJ)
15N 0266 Z(IJ) = Z(IJ)-ZK(IJ)
15N 0267 X(IJ) = X(IJ)-XK(IJ)
15N 0268 X(IJ) = X(IJ)-XK(IJ)
15N 0269 Y(IJ) = Y(IJ)-YK(IJ)
15N 0270 Y(IJ) = Y(IJ)-YK(IJ)
15N 0271 Z(IJ) = Z(IJ)-ZK(IJ)
15N 0272 TRUPT(IJL) = TIME
15N 0273 PRINT 1040, IJL, VEE(IJL), VMAX(IJL)

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Figure 108. (Continued)

ISM 0274	1040	FORMATTED	TRUPLOKE	215	JP215.61	CRS05500
ISM 0275	GO TO 1600					CRS05510
ISM 0276	230	CONTINUE				CRS05520
ISM 0277	SEI(11) = 51111111	SUMSE				CRS05530
ISM 0278	U(11) = 01111111	SUMU				CRS05540
ISM 0279	IF (11) .EQ. 1	NE-BLANK	U(11) = -6.550000000000000E-1			CRS05550
ISM 0280	ALAI(1) = AT(1)ALAI(1)ALAI(2)ALAI(3)					CRS05560
ISM 0281	ALAI(2) = AT(2)ALAI(2)ALAI(3)ALAI(4)					CRS05570
ISM 0282	ALAI(3) = AT(3)ALAI(3)ALAI(4)ALAI(5)					CRS05580
ISM 0283	ALAI(4) = AT(4)ALAI(4)ALAI(5)ALAI(6)					CRS05590
ISM 0284	ALAI(5) = AT(5)ALAI(5)ALAI(6)ALAI(7)					CRS05600
ISM 0285	ALAI(6) = AT(6)ALAI(6)ALAI(7)ALAI(8)					CRS05610
ISM 0286	ALAI(7) = AT(7)ALAI(7)ALAI(8)ALAI(9)					CRS05620
ISM 0287	ALAI(8) = AT(8)ALAI(8)ALAI(9)ALAI(10)					CRS05630
ISM 0288	ALAI(9) = AT(9)ALAI(9)ALAI(10)ALAI(11)					CRS05640
ISM 0289	ALAI(10) = AT(10)ALAI(10)ALAI(11)ALAI(12)					CRS05650
ISM 0290	11 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05660
ISM 0291	12 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05670
ISM 0292	13 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05680
ISM 0293	14 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05690
ISM 0294	15 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05700
ISM 0295	16 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05710
ISM 0296	17 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05720
ISM 0297	18 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05730
ISM 0298	19 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05740
ISM 0299	20 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05750
ISM 0300	21 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05760
ISM 0301	22 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05770
ISM 0302	23 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05780
ISM 0303	24 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05790
ISM 0304	25 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05800
ISM 0305	26 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05810
ISM 0306	27 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05820
ISM 0307	28 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05830
ISM 0308	29 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05840
ISM 0309	30 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05850
ISM 0310	31 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05860
ISM 0311	32 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05870
ISM 0312	33 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05880
ISM 0313	34 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05890
ISM 0314	35 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05900
ISM 0315	36 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05910
ISM 0316	37 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05920
ISM 0317	38 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05930
ISM 0318	39 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05940
ISM 0319	40 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05950
ISM 0320	41 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05960
ISM 0321	42 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05970
ISM 0322	43 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05980
ISM 0323	44 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS05990
ISM 0324	45 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS06000
ISM 0325	46 = ALAI(1)ALAI(2)ALAI(3)ALAI(4)ALAI(5)ALAI(6)ALAI(7)ALAI(8)ALAI(9)ALAI(10)ALAI(11)ALAI(12)					CRS06010

Figure 108. (Continued)

1SM 0325	C (16A)	UXV = A1(1)U(1)A1(2)U(2)A1(3)U(3)	CR506020
1SM 0326		UXV = A1(4)U(4)A1(5)U(5)A1(6)U(6)	CR506030
1SM 0327		UXV = A1(7)U(7)A1(8)U(8)A1(9)U(9)	CR506040
1SM 0328		UXV = A1(10)U(10)A1(11)U(11)A1(12)U(12)	CR506050
1SM 0329		UXV = A1(13)U(13)A1(14)U(14)A1(15)U(15)	CR506060
1SM 0330		UXV = A1(16)U(16)A1(17)U(17)A1(18)U(18)	CR506070
1SM 0331		UXV = A1(19)U(19)A1(20)U(20)A1(21)U(21)	CR506080
1SM 0332		UXV = A1(22)U(22)A1(23)U(23)A1(24)U(24)	CR506090
1SM 0333		UXV = A1(25)U(25)A1(26)U(26)A1(27)U(27)	CR506100
1SM 0334		UXV = A1(28)U(28)A1(29)U(29)A1(30)U(30)	CR506110
1SM 0335		UXV = A1(31)U(31)A1(32)U(32)A1(33)U(33)	CR506120
1SM 0336		UXV = A1(34)U(34)A1(35)U(35)A1(36)U(36)	CR506130
1SM 0337		UXV = A1(37)U(37)A1(38)U(38)A1(39)U(39)	CR506140
1SM 0338		UXV = A1(40)U(40)A1(41)U(41)A1(42)U(42)	CR506150
1SM 0339		UXV = A1(43)U(43)A1(44)U(44)A1(45)U(45)	CR506160
1SM 0340		UXV = A1(46)U(46)A1(47)U(47)A1(48)U(48)	CR506170
1SM 0341		UXV = A1(49)U(49)A1(50)U(50)A1(51)U(51)	CR506180
1SM 0342		UXV = A1(52)U(52)A1(53)U(53)A1(54)U(54)	CR506190
1SM 0343		UXV = A1(55)U(55)A1(56)U(56)A1(57)U(57)	CR506200
1SM 0344		UXV = A1(58)U(58)A1(59)U(59)A1(60)U(60)	CR506210
1SM 0345		UXV = A1(61)U(61)A1(62)U(62)A1(63)U(63)	CR506220
1SM 0346		UXV = A1(64)U(64)A1(65)U(65)A1(66)U(66)	CR506230
1SM 0347		UXV = A1(67)U(67)A1(68)U(68)A1(69)U(69)	CR506240
1SM 0348		UXV = A1(70)U(70)A1(71)U(71)A1(72)U(72)	CR506250
1SM 0349		UXV = A1(73)U(73)A1(74)U(74)A1(75)U(75)	CR506260
1SM 0350		UXV = A1(76)U(76)A1(77)U(77)A1(78)U(78)	CR506270
1SM 0351		UXV = A1(79)U(79)A1(80)U(80)A1(81)U(81)	CR506280
1SM 0352		UXV = A1(82)U(82)A1(83)U(83)A1(84)U(84)	CR506290
1SM 0353		UXV = A1(85)U(85)A1(86)U(86)A1(87)U(87)	CR506300
1SM 0354		UXV = A1(88)U(88)A1(89)U(89)A1(90)U(90)	CR506310
1SM 0355		UXV = A1(91)U(91)A1(92)U(92)A1(93)U(93)	CR506320
1SM 0356		UXV = A1(94)U(94)A1(95)U(95)A1(96)U(96)	CR506330
1SM 0357		UXV = A1(97)U(97)A1(98)U(98)A1(99)U(99)	CR506340
1SM 0358		UXV = A1(100)U(100)A1(101)U(101)A1(102)U(102)	CR506350
1SM 0359		UXV = A1(103)U(103)A1(104)U(104)A1(105)U(105)	CR506360
1SM 0360		UXV = A1(106)U(106)A1(107)U(107)A1(108)U(108)	CR506370
1SM 0361		UXV = A1(109)U(109)A1(110)U(110)A1(111)U(111)	CR506380
1SM 0362		UXV = A1(112)U(112)A1(113)U(113)A1(114)U(114)	CR506390
1SM 0363		UXV = A1(115)U(115)A1(116)U(116)A1(117)U(117)	CR506400
1SM 0364		UXV = A1(118)U(118)A1(119)U(119)A1(120)U(120)	CR506410
1SM 0365		UXV = A1(121)U(121)A1(122)U(122)A1(123)U(123)	CR506420
1SM 0366		UXV = A1(124)U(124)A1(125)U(125)A1(126)U(126)	CR506430
1SM 0367		UXV = A1(127)U(127)A1(128)U(128)A1(129)U(129)	CR506440
1SM 0368		UXV = A1(130)U(130)A1(131)U(131)A1(132)U(132)	CR506450
1SM 0369		UXV = A1(133)U(133)A1(134)U(134)A1(135)U(135)	CR506460
1SM 0370		UXV = A1(136)U(136)A1(137)U(137)A1(138)U(138)	CR506470
1SM 0371		UXV = A1(139)U(139)A1(140)U(140)A1(141)U(141)	CR506480
1SM 0372		UXV = A1(142)U(142)A1(143)U(143)A1(144)U(144)	CR506490
1SM 0373		UXV = A1(145)U(145)A1(146)U(146)A1(147)U(147)	CR506500
1SM 0374		UXV = A1(148)U(148)A1(149)U(149)A1(150)U(150)	CR506510
1SM 0375		UXV = A1(151)U(151)A1(152)U(152)A1(153)U(153)	CR506520
1SM 0376		UXV = A1(154)U(154)A1(155)U(155)A1(156)U(156)	CR506530
1SM 0377		UXV = A1(157)U(157)A1(158)U(158)A1(159)U(159)	CR506540
1SM 0378		UXV = A1(160)U(160)A1(161)U(161)A1(162)U(162)	CR506550
1SM 0379		UXV = A1(163)U(163)A1(164)U(164)A1(165)U(165)	CR506560
1SM 0380		UXV = A1(166)U(166)A1(167)U(167)A1(168)U(168)	CR506570
1SM 0381		UXV = A1(169)U(169)A1(170)U(170)A1(171)U(171)	CR506580
1SM 0382		UXV = A1(172)U(172)A1(173)U(173)A1(174)U(174)	CR506590
1SM 0383		UXV = A1(175)U(175)A1(176)U(176)A1(177)U(177)	CR506600
1SM 0384		UXV = A1(178)U(178)A1(179)U(179)A1(180)U(180)	CR506610
1SM 0385		UXV = A1(181)U(181)A1(182)U(182)A1(183)U(183)	CR506620
1SM 0386		UXV = A1(184)U(184)A1(185)U(185)A1(186)U(186)	CR506630
1SM 0387		UXV = A1(187)U(187)A1(188)U(188)A1(189)U(189)	CR506640
1SM 0388		UXV = A1(190)U(190)A1(191)U(191)A1(192)U(192)	CR506650
1SM 0389		UXV = A1(193)U(193)A1(194)U(194)A1(195)U(195)	CR506660
1SM 0390		UXV = A1(196)U(196)A1(197)U(197)A1(198)U(198)	CR506670
1SM 0391		UXV = A1(199)U(199)A1(200)U(200)A1(201)U(201)	CR506680
1SM 0392		UXV = A1(202)U(202)A1(203)U(203)A1(204)U(204)	CR506690
1SM 0393		UXV = A1(205)U(205)A1(206)U(206)A1(207)U(207)	CR506700
1SM 0394		UXV = A1(208)U(208)A1(209)U(209)A1(210)U(210)	CR506710
1SM 0395		UXV = A1(211)U(211)A1(212)U(212)A1(213)U(213)	CR506720
1SM 0396		UXV = A1(214)U(214)A1(215)U(215)A1(216)U(216)	CR506730
1SM 0397		UXV = A1(217)U(217)A1(218)U(218)A1(219)U(219)	CR506740
1SM 0398		UXV = A1(220)U(220)A1(221)U(221)A1(222)U(222)	CR506750
1SM 0399		UXV = A1(223)U(223)A1(224)U(224)A1(225)U(225)	CR506760
1SM 0400		UXV = A1(226)U(226)A1(227)U(227)A1(228)U(228)	CR506770
1SM 0401		UXV = A1(229)U(229)A1(230)U(230)A1(231)U(231)	CR506780
1SM 0402		UXV = A1(232)U(232)A1(233)U(233)A1(234)U(234)	CR506790
1SM 0403		UXV = A1(235)U(235)A1(236)U(236)A1(237)U(237)	CR506800
1SM 0404		UXV = A1(238)U(238)A1(239)U(239)A1(240)U(240)	CR506810
1SM 0405		UXV = A1(241)U(241)A1(242)U(242)A1(243)U(243)	CR506820
1SM 0406		UXV = A1(244)U(244)A1(245)U(245)A1(246)U(246)	CR506830
1SM 0407		UXV = A1(247)U(247)A1(248)U(248)A1(249)U(249)	CR506840
1SM 0408		UXV = A1(250)U(250)A1(251)U(251)A1(252)U(252)	CR506850
1SM 0409		UXV = A1(253)U(253)A1(254)U(254)A1(255)U(255)	CR506860
1SM 0410		UXV = A1(256)U(256)A1(257)U(257)A1(258)U(258)	CR506870
1SM 0411		UXV = A1(259)U(259)A1(260)U(260)A1(261)U(261)	CR506880
1SM 0412		UXV = A1(262)U(262)A1(263)U(263)A1(264)U(264)	CR506890
1SM 0413		UXV = A1(265)U(265)A1(266)U(266)A1(267)U(267)	CR506900
1SM 0414		UXV = A1(268)U(268)A1(269)U(269)A1(270)U(270)	CR506910
1SM 0415		UXV = A1(271)U(271)A1(272)U(272)A1(273)U(273)	CR506920
1SM 0416		UXV = A1(274)U(274)A1(275)U(275)A1(276)U(276)	CR506930
1SM 0417		UXV = A1(277)U(277)A1(278)U(278)A1(279)U(279)	CR506940
1SM 0418		UXV = A1(280)U(280)A1(281)U(281)A1(282)U(282)	CR506950
1SM 0419		UXV = A1(283)U(283)A1(284)U(284)A1(285)U(285)	CR506960
1SM 0420		UXV = A1(286)U(286)A1(287)U(287)A1(288)U(288)	CR506970
1SM 0421		UXV = A1(289)U(289)A1(290)U(290)A1(291)U(291)	CR506980
1SM 0422		UXV = A1(292)U(292)A1(293)U(293)A1(294)U(294)	CR506990
1SM 0423		UXV = A1(295)U(295)A1(296)U(296)A1(297)U(297)	CR507000
1SM 0424		UXV = A1(298)U(298)A1(299)U(299)A1(300)U(300)	CR507010
1SM 0425		UXV = A1(301)U(301)A1(302)U(302)A1(303)U(303)	CR507020
1SM 0426		UXV = A1(304)U(304)A1(305)U(305)A1(306)U(306)	CR507030
1SM 0427		UXV = A1(307)U(307)A1(308)U(308)A1(309)U(309)	CR507040
1SM 0428		UXV = A1(310)U(310)A1(311)U(311)A1(312)U(312)	CR507050
1SM 0429		UXV = A1(313)U(313)A1(314)U(314)A1(315)U(315)	CR507060
1SM 0430		UXV = A1(316)U(316)A1(317)U(317)A1(318)U(318)	CR507070
1SM 0431		UXV = A1(319)U(319)A1(320)U(320)A1(321)U(321)	CR507080
1SM 0432		UXV = A1(322)U(322)A1(323)U(323)A1(324)U(324)	CR507090
1SM 0433		UXV = A1(325)U(325)A1(326)U(326)A1(327)U(327)	CR507100
1SM 0434		UXV = A1(328)U(328)A1(329)U(329)A1(330)U(330)	CR507110
1SM 0435		UXV = A1(331)U(331)A1(332)U(332)A1(333)U(333)	CR507120
1SM 0436		UXV = A1(334)U(334)A1(335)U(335)A1(336)U(336)	CR507130
1SM 0437		UXV = A1(337)U(337)A1(338)U(338)A1(339)U(339)	CR507140
1SM 0438		UXV = A1(340)U(340)A1(341)U(341)A1(342)U(342)	CR507150
1SM 0439		UXV = A1(343)U(343)A1(344)U(344)A1(345)U(345)	CR507160
1SM 0440		UXV = A1(346)U(346)A1(347)U(347)A1(348)U(348)	CR507170
1SM 0441		UXV = A1(349)U(349)A1(350)U(350)A1(351)U(351)	CR507180
1SM 0442		UXV = A1(352)U(352)A1(353)U(353)A1(354)U(354)	CR507190
1SM 0443		UXV = A1(355)U(355)A1(356)U(356)A1(357)U(357)	CR507200
1SM 0444		UXV = A1(358)U(358)A1(359)U(359)A1(360)U(360)	CR507210
1SM 0445		UXV = A1(361)U(361)A1(362)U(362)A1(363)U(363)	CR507220
1SM 0446		UXV = A1(364)U(364)A1(365)U(365)A1(366)U(366)	CR507230
1SM 0447		UXV = A1(367)U(367)A1(368)U(368)A1(369)U(369)	CR507240
1SM 0448		UXV = A1(370)U(370)A1(371)U(371)A1(372)U(372)	CR507250
1SM 0449		UXV = A1(373)U(373)A1(374)U(374)A1(375)U(375)	CR507260
1SM 0450		UXV = A1(376)U(376)A1(377)U(377)A1(378)U(378)	CR507270
1SM 0451		UXV = A1(379)U(379)A1(380)U(380)A1(381)U(381)	CR507280
1SM 0452		UXV = A1(382)U(382)A1(383)U(383)A1(384)U(384)	CR507290
1SM 0453		UXV = A1(385)U(385)A1(386)U(386)A1(387)U(387)	CR507300
1SM 0454		UXV = A1(388)U(388)A1(389)U(389)A1(390)U(390)	CR507310
1SM 0455		UXV = A1(391)U(391)A1(392)U(392)A1(393)U(393)	CR507320
1SM 0456		UXV = A1(394)U(394)A1(395)U(395)A1(396)U(396)	CR507330
1SM 0457		UXV = A1(397)U(397)A1(398)U(398)A1(399)U(399)	CR507340
1SM 0458		UXV = A1(400)U(400)A1(401)U(401)A1(402)U(402)	CR507350
1SM 0459		UXV = A1(403)U(403)A1(404)U(404)A1(405)U(405)	CR507360
1SM 0460		UXV = A1(406)U(406)A1(407)U(407)A1(408)U(408)	CR507370
1SM 0461		UXV = A1(409)U(409)A1(410)U(410)A1(411)U(411)	CR507380
1SM 0462		UXV = A1(412)U(412)A1(413)U(413)A1(414)U(414)	CR507390
1SM 0463		UXV = A1(415)U(415)A1(416)U(416)A1(417)U(417)	CR507400
1SM 0464		UXV = A1(418)U(418)A1(419)U(419)A1(420)U(420)	CR507410
1SM 0465		UXV = A1(421)U(421)A1(422)U(422)A1(423)U(423)	CR507420
1SM 0466		UXV = A1(424)U(424)A1(425)U(425)A1(426)U(426)	CR507430
1SM 0467		UXV = A1(427)U(427)A1(428)U(428)A1(429)U(429)	CR507440
1SM 0468		UXV = A1(430)U(430)A1(431)U(431)A1(432)U(432)	CR507450
1SM 0469		UXV = A1(433)U(433)A1(434)U(434)A1(435)U(435)	CR507460
1SM 0470		UXV = A1(436)U(436)A1(437)U(437)A1(438)U(438)	CR507470
1SM 0471		UXV = A1(439)U(439)A1(440)U(440)A1(441)U(441)	CR507480
1SM 0472		UXV = A1(442)U(442)A1(443)U(443)A1(444)U(444)	CR507490
1SM 0473		UXV = A1(445)U(445)A1(446)U(446)A1(447)U(447)	CR507500
1SM 0474		UXV = A1(448)U(448)A1(449)U(449)A1(450)U(450)	CR507510
1SM 0475		UXV = A1(451)U(451)A1(452)U(452)A1(453)U(453)	CR507520
1SM 0476		UXV = A1(454)U(454)A1(455)U(455)A1(456)U(456)	CR507530
1SM 0477		UXV = A1(457)U(457)A1(458)U(458)A1(459)U(459)	CR507540
1SM 0478		UXV = A1(460)U(460)A1(461)U(461)A1(462)U(462)	CR507550
1SM 0479		UXV = A1(463)U(463)A1(464)U(464)A1(465)U(465)	CR507560
1SM 0480		UXV = A1(466)U(466)A1(467)U(467)A1(468)U(468)	CR507570
1SM 0481		UXV = A1(469)U(469)A1(4	

15M 0373	C (BA)	XX = A111*0011)*A112)*0012)*A113)*0013)	CRS06570
15M 0374		XX = A114)*0011)*A115)*0012)*A116)*0013)	CRS06570
15M 0375		XX = A117)*0011)*A118)*0012)*A119)*0013)	CRS06570
15M 0376		DPX11) = DPX11)*DX	CRS06570
15M 0377		UPV11) = UPV11)*DX	CRS06570
15M 0378		OPZ11) = OPZ11)*DX	CRS06570
15M 0379	C (BB)	DXL = A111)*0011)*A112)*0013)*A114)*0016)	CRS06570
15M 0380		DXM = A114)*0011)*A115)*0013)*A116)*0016)	CRS06570
15M 0381		DXN = A117)*0011)*A118)*0013)*A119)*0016)	CRS06570
15M 0382		DPL11) = DPL11)*DX	CRS06570
15M 0383		DPN11) = DPN11)*DX	CRS06570
15M 0384		UPN11) = UPN11)*DX	CRS06570
15M 0385		1000 CUNTINUT	CRS06570
15M 0386		15*99(1NUP-1)	CRS06570
15M 0387		DO 1010 KS=1.9	CRS06570
15M 0388		15=15*1	CRS06570
15M 0389		1010 APIRS=BJTIS	CRS06570
15M 0390	C FINISH COMPUTING DERIVATIVES	DO 2000 I=1,MM	CRS06570
15M 0391		IS = 99(1-1)	CRS06570
15M 0392		DO 300 KS = 1.9	CRS06570
15M 0393		IS = 15*	CRS06570
15M 0394		ADUTTRS) = C1J(15)	CRS06570
15M 0395		330 ATRS) = 01J(15)	CRS06570
15M 0396	C DO CRASH FORCIS	DO 340 K = 1.6	CRS06570
15M 0397		440 XCIN) = 0.0	CRS06570
15M 0398		IFISPI(11)=EG.1 .OR.1SP(11,2).EQ.1.OR.1SP(11,3).EQ.1) CALL CFORCE	CRS06570
15M 0400	C (201,123)12*	XA = MGT(11)-ALIF(11)	CRS06570
15M 0401		SA = XN(11)*XCAI(11)*XC(11)*UPX(11)	CRS06570
15M 0402		SV = XN(11)*XCAI(11)*XC(11)*OPV(11)	CRS06570
15M 0403		SZ = XN(11)*XCAI(11)*XC(11)*DPZ(11)	CRS06570
15M 0404		SL = XN(11)*XC(11)*DPL(11)	CRS06570
15M 0405		SM = XN(11)*XC(11)*DPN(11)	CRS06570
15M 0406		SN = XN(11)*XC(11)*DPN(11)	CRS06570
15M 0407	C GET P1,Q,K,U,V,M	PP = P(1)	CRS06570
15M 0408		QQ = Q(1)	CRS06570
15M 0409		RR = R(1)	CRS06570
15M 0410		UU = U(1)	CRS06570
15M 0411		VV = V(1)	CRS06570
15M 0412		MM = M(1)	CRS06570
15M 0413	C MASS	MGT1 = 1.0/MGT(11)	CRS06570
15M 0414		ZM = 386.0/MGT1	CRS06570
15M 0415		XAC(11) = 289/MGT1	CRS06570
15M 0416		YACC(11) = 289/MGT1	CRS06570
15M 0417		ZACC(11) = 52/MGT1	CRS06570
15M 0418	C (25)	UOUT1 = UOUT(11)	CRS06570
15M 0419		UOUT(11) = 52*ZM-40*MM*RR*VV	CRS06570

Figure 108. (Continued)

Figure 108. (Continued)

DATE 73-298/10-02-52

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100

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COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=56,SIZE=0000K,
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[illegible]

Figure 108. (Continued)

15M 0030	1070 1*-1	CR501010
15M 0031	1075 1*(1-1-1-10) 1060,1085,1085	CR508020
15M 0032	1080 51MCS(J)=0.	CR508030
15M 0033	1085 CONTINUE	CR508040
15M 0034	J=9*(1-1)	CR508050
	C MOVE A1'S TO ULC A1'S	CR508060
15M 0035	DO 1000 JJ = 1,9	CR508070
15M 0036	1090 0A1(J+1)=P1(J+1,J)	CR508080
15M 0037	51S=51A52	CR508090
15M 0038	61S=61A52	CR508100
15M 0039	81J(J+1)=C*9C3	CR508110
15M 0040	81J(J+1)=C*9C3	CR508120
15M 0041	81J(J+1)=S2	CR508130
15M 0042	81J(J+1)=C1*9S+S1S2*9C3	CR508140
15M 0043	81J(J+1)=C1*9C3+S1S2*53	CP508150
15M 0044	81J(J+1)=S1*9C2	CR508160
15M 0045	81J(J+1)=S1*9S+S1S2*9C3	CR508170
15M 0046	81J(J+1)=S1*9C3+C1S2*53	CR508180
15M 0047	81J(J+1)=C1*9C	CR508190
15M 0048	1100 CONTINUE	CR508200
	C	CR508210
15M 0049	DO 1010 IJ = 1,165	CR508220
15M 0050	51 = 51*(PM1(IJ,IJ))	CR508230
15M 0051	C1 = C05(PM1(IJ,IJ))	CR508240
15M 0052	S2 = 51M(THE1(IJ,IJ))	CR508250
15M 0053	C2 = C05(THE1(IJ,IJ))	CR508260
15M 0054	S3 = 51M(P51(IJ,IJ))	CR508270
15M 0055	C3 = C05(P51(IJ,IJ))	CR508280
15M 0056	DO 1040 J = 1,6	CR508290
15M 0057	I = 51MCS(IJ)	CR508300
15M 0058	IF (I) 1045,1040,1050	CR508310
15M 0059	1045 I = I - 1	CR508320
15M 0060	1050 11(1-1-1-10) 1155,1040,1040	CP508330
15M 0061	1055 51MCS(IJ) = 0.4	CR508340
15M 0062	1040 CONTINUE	CR508350
15M 0063	AIJ(I) = C2*9C	CR508360
15M 0064	AIJ(I) = C2*53	CR508370
15M 0065	AIJ(I) = -S2	CR508380
15M 0066	AIJ(I) = -C1*9S+S1S2*9C3	CR508390
15M 0067	AIJ(I) = C1*9C3+S1S2*53	CR508400
15M 0068	AIJ(I) = S1*9C2	CP508410
15M 0069	AIJ(I) = S1*9S+S1S2*9C3	CR508420
15M 0070	AIJ(I) = -S1*9C3+C1S2*53	CR508430
15M 0071	AIJ(I) = C1*9C2	CR508440
15M 0072	I = 9*(IJ-1)	CR508450
15M 0073	DO 1015 J = 1,9	CR508460
15M 0074	1015 11J(I+1) = AIJ(I)	CR508470
15M 0075	CB1J = CBAN(IJ)	CR508480
15M 0076	I = 16(IJ)	CR508490
15M 0077	J = J(IJ)	CR508500
15M 0078	IF (11UD-NE-01-AMU-11-64,10LD) GO TO 1120	CR508510
15M 0080	15=9*(1-1)	CR508520
15M 0081	DO 1110 JJ = 1,9	CR508530
15M 0082	15=15+1	CR508540

Figure 108. (Continued)

15M 0063	1110 A1(JJ)=B1(JJ)	CR508550
15M 0064	1120 10LD=1	CR508560
15M 0065	15=4*(J-1)	CR508570
15M 0066	00 1125 JJ = 1,4	CR508580
15M 0067	15=15*1	CR508590
15M 0068	1125 A1(JJ) = B1(JJ)	CR508600
15M 0069	A1TAJ(1)=A1(1)*A1(1)+A1(2)*A1(2)+A1(3)*A1(3)	CR508610
15M 0070	A1TAJ(2)=A1(1)*A1(2)+A1(2)*A1(2)+A1(3)*A1(3)	CR508620
15M 0071	A1TAJ(3)=A1(1)*A1(3)+A1(2)*A1(2)+A1(3)*A1(3)	CR508630
15M 0072	A1TAJ(4)=A1(1)*A1(4)+A1(2)*A1(2)+A1(3)*A1(3)	CR508640
15M 0073	A1TAJ(5)=A1(1)*A1(5)+A1(2)*A1(2)+A1(3)*A1(3)	CR508650
15M 0074	A1TAJ(6)=A1(1)*A1(6)+A1(2)*A1(2)+A1(3)*A1(3)	CR508660
15M 0075	A1TAJ(7)=A1(1)*A1(7)+A1(2)*A1(2)+A1(3)*A1(3)	CR508670
15M 0076	A1TAJ(8)=A1(1)*A1(8)+A1(2)*A1(2)+A1(3)*A1(3)	CR508680
15M 0077	A1TAJ(9)=A1(1)*A1(9)+A1(2)*A1(2)+A1(3)*A1(3)	CR508690
15M 0078	A1TAJ(10)=A1(1)*A1(10)+A1(2)*A1(2)+A1(3)*A1(3)	CR508700
15M 0079	A1TAJ(11)=A1(1)*A1(11)+A1(2)*A1(2)+A1(3)*A1(3)	CR508710
15M 0080	A1TAJ(12)=A1(1)*A1(12)+A1(2)*A1(2)+A1(3)*A1(3)	CR508720
15M 0081	A1TAJ(13)=A1(1)*A1(13)+A1(2)*A1(2)+A1(3)*A1(3)	CR508730
15M 0082	A1TAJ(14)=A1(1)*A1(14)+A1(2)*A1(2)+A1(3)*A1(3)	CR508740
15M 0083	A1TAJ(15)=A1(1)*A1(15)+A1(2)*A1(2)+A1(3)*A1(3)	CR508750
15M 0084	A1TAJ(16)=A1(1)*A1(16)+A1(2)*A1(2)+A1(3)*A1(3)	CR508760
15M 0085	A1TAJ(17)=A1(1)*A1(17)+A1(2)*A1(2)+A1(3)*A1(3)	CR508770
15M 0086	A1TAJ(18)=A1(1)*A1(18)+A1(2)*A1(2)+A1(3)*A1(3)	CR508780
15M 0087	A1TAJ(19)=A1(1)*A1(19)+A1(2)*A1(2)+A1(3)*A1(3)	CR508790
15M 0088	A1TAJ(20)=A1(1)*A1(20)+A1(2)*A1(2)+A1(3)*A1(3)	CR508800
15M 0089	A1TAJ(21)=A1(1)*A1(21)+A1(2)*A1(2)+A1(3)*A1(3)	CR508810
15M 0090	A1TAJ(22)=A1(1)*A1(22)+A1(2)*A1(2)+A1(3)*A1(3)	CR508820
15M 0091	A1TAJ(23)=A1(1)*A1(23)+A1(2)*A1(2)+A1(3)*A1(3)	CR508830
15M 0092	A1TAJ(24)=A1(1)*A1(24)+A1(2)*A1(2)+A1(3)*A1(3)	CR508840
15M 0093	A1TAJ(25)=A1(1)*A1(25)+A1(2)*A1(2)+A1(3)*A1(3)	CR508850
15M 0094	A1TAJ(26)=A1(1)*A1(26)+A1(2)*A1(2)+A1(3)*A1(3)	CR508860
15M 0095	A1TAJ(27)=A1(1)*A1(27)+A1(2)*A1(2)+A1(3)*A1(3)	CR508870
15M 0096	A1TAJ(28)=A1(1)*A1(28)+A1(2)*A1(2)+A1(3)*A1(3)	CR508880
15M 0097	A1TAJ(29)=A1(1)*A1(29)+A1(2)*A1(2)+A1(3)*A1(3)	CR508890
15M 0098	A1TAJ(30)=A1(1)*A1(30)+A1(2)*A1(2)+A1(3)*A1(3)	CR508900
15M 0099	A1TAJ(31)=A1(1)*A1(31)+A1(2)*A1(2)+A1(3)*A1(3)	CR508910
15M 0100	A1TAJ(32)=A1(1)*A1(32)+A1(2)*A1(2)+A1(3)*A1(3)	CR508920
15M 0101	A1TAJ(33)=A1(1)*A1(33)+A1(2)*A1(2)+A1(3)*A1(3)	CR508930
15M 0102	A1TAJ(34)=A1(1)*A1(34)+A1(2)*A1(2)+A1(3)*A1(3)	CR508940
15M 0103	A1TAJ(35)=A1(1)*A1(35)+A1(2)*A1(2)+A1(3)*A1(3)	CR508950
15M 0104	A1TAJ(36)=A1(1)*A1(36)+A1(2)*A1(2)+A1(3)*A1(3)	CR508960
15M 0105	A1TAJ(37)=A1(1)*A1(37)+A1(2)*A1(2)+A1(3)*A1(3)	CR508970
15M 0106	A1TAJ(38)=A1(1)*A1(38)+A1(2)*A1(2)+A1(3)*A1(3)	CR508980
15M 0107	A1TAJ(39)=A1(1)*A1(39)+A1(2)*A1(2)+A1(3)*A1(3)	CR508990
15M 0108	A1TAJ(40)=A1(1)*A1(40)+A1(2)*A1(2)+A1(3)*A1(3)	CR509000
15M 0109	A1TAJ(41)=A1(1)*A1(41)+A1(2)*A1(2)+A1(3)*A1(3)	CR509010
15M 0110	A1TAJ(42)=A1(1)*A1(42)+A1(2)*A1(2)+A1(3)*A1(3)	CR509020
15M 0111	A1TAJ(43)=A1(1)*A1(43)+A1(2)*A1(2)+A1(3)*A1(3)	CR509030
15M 0112	A1TAJ(44)=A1(1)*A1(44)+A1(2)*A1(2)+A1(3)*A1(3)	CR509040
15M 0113	A1TAJ(45)=A1(1)*A1(45)+A1(2)*A1(2)+A1(3)*A1(3)	CR509050
15M 0114	A1TAJ(46)=A1(1)*A1(46)+A1(2)*A1(2)+A1(3)*A1(3)	CR509060
15M 0115	A1TAJ(47)=A1(1)*A1(47)+A1(2)*A1(2)+A1(3)*A1(3)	CR509070
15M 0116	A1TAJ(48)=A1(1)*A1(48)+A1(2)*A1(2)+A1(3)*A1(3)	CR509080
15M 0117	A1TAJ(49)=A1(1)*A1(49)+A1(2)*A1(2)+A1(3)*A1(3)	CR509090
15M 0118	A1TAJ(50)=A1(1)*A1(50)+A1(2)*A1(2)+A1(3)*A1(3)	CR509100
15M 0119	A1TAJ(51)=A1(1)*A1(51)+A1(2)*A1(2)+A1(3)*A1(3)	CR509110
15M 0120	A1TAJ(52)=A1(1)*A1(52)+A1(2)*A1(2)+A1(3)*A1(3)	CR509120
15M 0121	A1TAJ(53)=A1(1)*A1(53)+A1(2)*A1(2)+A1(3)*A1(3)	CR509130
15M 0122	A1TAJ(54)=A1(1)*A1(54)+A1(2)*A1(2)+A1(3)*A1(3)	CR509140
15M 0123	A1TAJ(55)=A1(1)*A1(55)+A1(2)*A1(2)+A1(3)*A1(3)	CR509150
15M 0124	A1TAJ(56)=A1(1)*A1(56)+A1(2)*A1(2)+A1(3)*A1(3)	CR509160
15M 0125	A1TAJ(57)=A1(1)*A1(57)+A1(2)*A1(2)+A1(3)*A1(3)	CR509170
15M 0126	A1TAJ(58)=A1(1)*A1(58)+A1(2)*A1(2)+A1(3)*A1(3)	CR509180
15M 0127	A1TAJ(59)=A1(1)*A1(59)+A1(2)*A1(2)+A1(3)*A1(3)	CR509190
15M 0128	A1TAJ(60)=A1(1)*A1(60)+A1(2)*A1(2)+A1(3)*A1(3)	CR509200
15M 0129	A1TAJ(61)=A1(1)*A1(61)+A1(2)*A1(2)+A1(3)*A1(3)	CR509210
15M 0130	A1TAJ(62)=A1(1)*A1(62)+A1(2)*A1(2)+A1(3)*A1(3)	CR509220
15M 0131	A1TAJ(63)=A1(1)*A1(63)+A1(2)*A1(2)+A1(3)*A1(3)	CR509230
15M 0132	A1TAJ(64)=A1(1)*A1(64)+A1(2)*A1(2)+A1(3)*A1(3)	CR509240
15M 0133	A1TAJ(65)=A1(1)*A1(65)+A1(2)*A1(2)+A1(3)*A1(3)	CR509250

Figure 108. (Continued)

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15M 0134 PROD(4)=XVZIJ1(1)+AIJ(4)+XVZIJ1(1)+AIJ(5)+XVZIJ1(7)+AIJ(6) CRS09090
15M 0135 PROD(5)=XVZIJ1(2)+AIJ(4)+XVZIJ1(5)+AIJ(5)+XVZIJ1(8)+AIJ(6) CRS09100
15M 0136 PROD(6)=XVZIJ1(3)+AIJ(4)+XVZIJ1(6)+AIJ(5)+XVZIJ1(5)+AIJ(6) CRS09110
15M 0137 PROD(7)=XVZIJ1(1)+AIJ(7)+XVZIJ1(6)+AIJ(6)+XVZIJ1(7)+AIJ(9) CRS09120
15M 0138 PROD(8)=XVZIJ1(2)+AIJ(7)+XVZIJ1(5)+AIJ(6)+XVZIJ1(8)+AIJ(9) CRS09130
15M 0139 PROD(9)=XVZIJ1(3)+AIJ(7)+XVZIJ1(6)+AIJ(8)+XVZIJ1(9)+AIJ(9) CRS09140
15M 0140 X11J(1)=AIJ(1)+PRUD(1)+AIJ(2)+PRUD(2)+AIJ(3)+PRUD(3) CRS09150
15M 0141 X11J(2)=AIJ(4)+PRUD(4)+AIJ(5)+PRUD(5)+AIJ(6)+PRUD(6) CRS09160
15M 0142 X11J(3)=AIJ(7)+PRUD(7)+AIJ(8)+PRUD(8)+AIJ(9)+PRUD(9) CRS09170
15M 0143 DU 1020 K = 1.3 CRS09180
15M 0144 1020 L(6,1,1) = .101600*CB1*SQRT(XK3(K,K,1,1)+WGT(1,1)+WGT(1,1)) CRS09190
15M 0145 DU 1030 K = 1.6 CRS09200
15M 0146 1030 C(K,1,1)=C(61)*SQRT(XK3(K,K,1,1)+WGT(1,1)+WGT(1,1)) CRS09210
15M 0147 1010 CONTINUE CRS09220
15M 0148 ***** CRS09230
15M 0149 ***** CRS09240
15M 0149 ***** CRS09250
15M 0149 ***** CRS09260
15M 0149 ***** CRS09270
15M 0149 ***** CRS09280
15M 0150 ***** CRS09290
15M 0151 ***** CRS09300
15M 0152 ***** CRS09310
15M 0153 ***** CRS09320
15M 0154 ***** CRS09330
15M 0154 ***** CRS09340
15M 0154 ***** CRS09350
15M 0155 ***** CRS09360
15M 0155 ***** CRS09370
15M 0155 ***** CRS09380
15M 0156 ***** CRS09390
15M 0157 ***** CRS09400
15M 0158 ***** CRS09410
15M 0159 ***** CRS09420
15M 0160 ***** CRS09430
15M 0161 ***** CRS09440
15M 0162 ***** CRS09450
15M 0163 ***** CRS09460
15M 0164 ***** CRS09470
15M 0165 ***** CRS09480
15M 0166 ***** CRS09490
15M 0167 ***** CRS09500
15M 0168 ***** CRS09510
15M 0169 ***** CRS09520
15M 0170 ***** CRS09530
15M 0171 ***** CRS09540
15M 0172 ***** CRS09550
15M 0173 ***** CRS09560
15M 0174 ***** CRS09570
15M 0175 ***** CRS09580
15M 0176 ***** CRS09590
15M 0177 ***** CRS09600
15M 0178 ***** CRS09610
15M 0179 ***** CRS09620

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Figure 108. (Continued)

Figure 108. (Continued)


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15N 0226 5500 FUMAT(1,2,0) CRS10160
15N 0227 PRINT 5514 CRS10170
15N 0228 5514 FUMAT(1,01,LC(1)) CRS10180
15N 0229 PRINT 5506, (FUMAT(1,1),ALFF(1,MUF(1,1),1,1,M1)) CRS10190
C-----CLAS EXTERNAL SPRING ELMS (AND THE ASSOCIATED DATA ALTHO THIS
C SHOULD NOT BE NECESSARY BECAUSE WE ONLY USE IT IF THE FLAG IS 1. CRS10200
C HOWEVER, THEY MUST BE CLEARED FOR THE SEARCH WHICH PRINTS THE INPUT.) CRS10210
5534 DO 5150 K = 1,3 CRS10220
5535 DO 5150 I = 1,M1 CRS10230
15N 0230 I = 1,3 CRS10240
15N 0231 I = 1,3 CRS10250
15N 0232 I = 1,3 CRS10260
15N 0233 I = 1,3 CRS10270
15N 0234 I = 1,3 CRS10280
15N 0235 I = 1,3 CRS10290
15N 0236 I = 1,3 CRS10300
15N 0237 I = 1,3 CRS10310
15N 0238 I = 1,3 CRS10320
15N 0239 I = 1,3 CRS10330
15N 0240 I = 1,3 CRS10340
15N 0241 I = 1,3 CRS10350
15N 0242 I = 1,3 CRS10360
15N 0243 I = 1,3 CRS10370
15N 0244 I = 1,3 CRS10380
15N 0245 I = 1,3 CRS10390
15N 0246 I = 1,3 CRS10400
15N 0247 I = 1,3 CRS10410
15N 0248 I = 1,3 CRS10420
15N 0249 I = 1,3 CRS10430
15N 0250 I = 1,3 CRS10440
15N 0251 I = 1,3 CRS10450
15N 0252 I = 1,3 CRS10460
15N 0253 I = 1,3 CRS10470
15N 0254 I = 1,3 CRS10480
15N 0255 I = 1,3 CRS10490
15N 0256 I = 1,3 CRS10500
15N 0257 I = 1,3 CRS10510
15N 0258 I = 1,3 CRS10520
15N 0259 I = 1,3 CRS10530
15N 0260 I = 1,3 CRS10540
15N 0261 I = 1,3 CRS10550
15N 0262 I = 1,3 CRS10560
15N 0263 I = 1,3 CRS10570
C-----DO 5160 J = 1,IRCT CRS10580
15N 0264 I = 1,3 CRS10590
15N 0265 I = 1,3 CRS10600
15N 0266 I = 1,3 CRS10610
15N 0267 I = 1,3 CRS10620
15N 0268 I = 1,3 CRS10630
15N 0269 I = 1,3 CRS10640
15N 0270 I = 1,3 CRS10650
15N 0271 I = 1,3 CRS10660
15N 0272 I = 1,3 CRS10670
15N 0273 I = 1,3 CRS10680
5015 READ=500, 1,3,PHIIN,THEIN,PSIIN CRS10690

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Figure 108. (Continued)

15N 0274	5400 FORMATTED(13,5,12,0)	CRS10700
15N 0275	1611 5051,5051,5020	CRS10710
15N 0276	5020 1611 5051,5051,5020	CRS10720
15N 0277	1611 5051,5051,5020	CRS10730
15N 0278	1611 5051,5051,5020	CRS10740
15N 0279	1611 5051,5051,5020	CRS10750
15N 0280	1611 5051,5051,5020	CRS10760
15N 0281	1611 5051,5051,5020	CRS10770
15N 0282	1611 5051,5051,5020	CRS10780
15N 0283	1611 5051,5051,5020	CRS10790
15N 0284	1611 5051,5051,5020	CRS10800
15N 0285	1611 5051,5051,5020	CRS10810
15N 0286	1611 5051,5051,5020	CRS10820
15N 0287	1611 5051,5051,5020	CRS10830
15N 0288	1611 5051,5051,5020	CRS10840
15N 0289	1611 5051,5051,5020	CRS10850
15N 0290	1611 5051,5051,5020	CRS10860
15N 0291	1611 5051,5051,5020	CRS10870
15N 0292	1611 5051,5051,5020	CRS10880
15N 0293	1611 5051,5051,5020	CRS10890
15N 0294	1611 5051,5051,5020	CRS10900
15N 0295	1611 5051,5051,5020	CRS10910
15N 0296	1611 5051,5051,5020	CRS10920
15N 0297	1611 5051,5051,5020	CRS10930
15N 0298	1611 5051,5051,5020	CRS10940
15N 0299	1611 5051,5051,5020	CRS10950
15N 0300	1611 5051,5051,5020	CRS10960
15N 0301	1611 5051,5051,5020	CRS10970
15N 0302	1611 5051,5051,5020	CRS10980
15N 0303	1611 5051,5051,5020	CRS10990
15N 0304	1611 5051,5051,5020	CRS11000
15N 0305	1611 5051,5051,5020	CRS11010
15N 0306	1611 5051,5051,5020	CRS11020

Figure 108. (Continued)

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ISM 0307      IF(IJ11).EQ.1C(J1,NO,JQ11)2C(J1,J1) GO TO 5040
ISM 0308      5030 CONTINUE
ISM 0309      C***** SUCH I,J PAIR, ABORT
ISM 0310      PRINT5410, I011,J011
ISM 0311      STOP
ISM 0312      5910 FORMAT(11,'NON-EXISTENT I,J PAIR IN KR TABLE SPECS',215)
ISM 0313      C***** FOUND IT
ISM 0314      5040 MSLPGL6=(J1-1)*LO(11) = NO
ISM 0315      C***** TOO MANY KR TABLES, ABORT
ISM 0316      PRINT5420
ISM 0317      STOP
ISM 0318      5020 FORMAT(11,'TOO MANY KR TABLES')
ISM 0319      C***** PRINT KR TABLE SPECS
ISM 0320      5050 IF(INQ.IG.0) GO TO 5536
ISM 0321      PRINT 5930
ISM 0322      5930 FORMAT('KR TABLE SPECS, I-J-L, NP')
ISM 0323      PRINT5430,IG11,JQ11,LO11,NPQ11,I011,N01
ISM 0324      C***** LOOP TO HEAD IN A TABLE
ISM 0325      K = -14
ISM 0326      DO 5070 I = 1,NO
ISM 0327      IF(INP-LE-15) GO TO 5055
ISM 0328      C***** TOO MANY POINTS IN KR TABLE, ABORT
ISM 0329      PRINT 5960, NP+1
ISM 0330      STOP
ISM 0331      5960 FORMAT(11,'5 POINTS IN KR TABLE',13,' (MAX IS 15)')
ISM 0332      C***** SET CHNG TO 1,10,31,....
ISM 0333      5055 K = K+15
ISM 0334      CHNG(1) = K
ISM 0335      ICH = CHNG(1)-1
ISM 0336      READ5450, (XKR(ICH),KR(J1,J1),NP)
ISM 0337      5950 FORMAT(12,0)
ISM 0338      C***** PRINT TABLE
ISM 0339      PRINT5460, KR TABLE FOR I,J,L = 315
ISM 0340      5970 FORMAT(11,'(J,KR(ICH),KR(J1,J1),NP)')
ISM 0341      C***** COMPUTE SLOPES AND INTERCEPTS
ISM 0342      NP1 = NP-1
ISM 0343      DO 5461 J = 1,NP1
ISM 0344      SLOP1 = (KR(J1)-KR(J1))/XKR(ICH-J1-XKR(ICH+J1))
ISM 0345      XKR(ICH+J1) = SLOP1
ISM 0346      5080 XKR(ICH+J1) = KR(J1)-SLOP1*XKR(ICH-J1)
ISM 0347      C***** MOVE ENDPOINTS 'MAY OUT
ISM 0348      XKR(ICH+1) = -1-E35
ISM 0349      XKR(ICH+NP) = 1+E35
ISM 0350      C***** CONTINUE
ISM 0351      5070 CONTINUE
ISM 0352      C***** STANDARD VMAX = 100
ISM 0353      5536 DO 5180 I = 1,460
ISM 0354      5180 VMAX(1) = 100.0
ISM 0355      PRINT 5525
ISM 0356      5525 FORMAT('01J1,I,J,VMAX(1),J1-61')

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Figure 108. (Continued)

[illegible]

Figure 108. (Continued)

Figure 108. (Continued)

15N 0453	915 XVC(JJ,K) = 0.0	CAS12620
15N 0454	910 CONTINUE	CAS12630
	C LOOP C	CAS12640
15N 0455	100 920 K = 1.3	CAS12650
15N 0456	IF (I1SP(K)) 925,920,925	CAS12660
15N 0457	925 ISUB = 39K	CAS12670
15N 0458	DVC = A(I1SUB)*X(LBAR(K))	CAS12680
15N 0459	DVLOT = A(LOT(I1SUB)*XLBAR(K))	CAS12690
15N 0460	WC = VAL(3)*DVC	CAS12700
15N 0461	VLLOT = VADOT(3)*DVC	CAS12710
15N 0462	C4(K) = VC/DVC	CAS12720
15N 0463	C5(K) = DVC/VLOT - VC*DVCLOT/(DVC*DVC)	CAS12730
15N 0464	VC3(K) = VC	CAS12740
15N 0465	920 CONTINUE	CAS12750
	C LOOP H	CAS12760
15N 0466	100 30 K = 1.3	CAS12770
15N 0467	IF (I1SP(K)) 35,30,35	CAS12780
15N 0468	35 IF (VC3(K)) 30,30,40	CAS12790
15N 0469	40 ISUB = 39(K-1)	CAS12800
15N 0470	PARL = X(LBAR(K))	CAS12810
15N 0471	SUM = 0.0	CAS12820
15N 0472	SUM = 0.0	CAS12830
	C LOOP J	CAS12840
15N 0473	00 50 J = 1.3	CAS12850
15N 0474	ISUB = 1501.1	CAS12860
15N 0475	LVC = A(I1SUB)*PARL	CAS12870
15N 0476	VLCLOT = A(LOT(I1SUB)*PARL)	CAS12880
15N 0477	DVP = C4(K)*LVC	CAS12890
15N 0478	DVP = C4(K)*DVCLOT + C5(K)*DVC	CAS12900
15N 0479	SUM = SUM + DVP	CAS12910
15N 0480	SUM = SUM + DVP	CAS12920
15N 0481	50 CONTINUE	CAS12930
15N 0482	SR = SUM/ISUB	CAS12940
15N 0483	SR = SUM/ISUB	CAS12950
15N 0484	SC(1,K) = SR	CAS12960
15N 0485	SC(1,K) = SUM/ISUB	CAS12970
	C GET LENGTH	CAS12980
15N 0486	IF (PARL) 55,60,60	CAS12990
15N 0487	55 T = PARL - SR	CAS13000
15N 0488	IF (T) 65,65,70	CAS13010
15N 0489	70 T = -1	CAS13020
15N 0490	60 T = PARL - SR	CAS13030
15N 0491	65 XLNGTHIF = T	CAS13040
15N 0492	30 CONTINUE	CAS13050
15N 0493	PL(1,1) = -5001(1)	CAS13060
15N 0494	PL(2,2) = -5001(2)	CAS13070
15N 0495	PL(3,3) = -5001(3)	CAS13080
15N 0496	PL(1,1) = PAR(1)*XLNGTHIF(1)	CAS13090
15N 0497	PL(2,2) = PAR(2)*XLNGTHIF(2)	CAS13100
15N 0498	PL(3,3) = PAR(3)*XLNGTHIF(3)	CAS13110
15N 0499	PL(1,2) = -PAR(1)*XLNGTHIF(2)	CAS13120
15N 0500	PL(2,1) = -PAR(2)*XLNGTHIF(1)	CAS13130
15N 0501	PL(3,2) = PAR(1)*XLNGTHIF(2)	CAS13140
15N 0502	PL(1,3) = PAR(2)*XLNGTHIF(3)	CAS13150

Figure 108. (Continued)

ISM 0503	C LOOP K	PL17,3) = -PBART(10,X)ENG(M,3)	CRS13360
ISM 0504	DO 75 JJ = 1,3		CRS13370
ISM 0505	ISUR = JJ-3		CRS13380
ISM 0506	VAD = VADUT(JJ)		CRS13390
	C LOOP L		CRS13400
ISM 0507	DO 100 K = 1,3		CRS13410
ISM 0508	IF(IISPK(K)) RS=RO,RS		CRS13420
ISM 0509	85 IF(VL3(K)) RC=RO,RC		CRS13430
ISM 0510	90 SUM = 0.0		CRS13440
	C LOOP N		CRS13450
ISM 0511	DO 125 L = 1,3		CRS13460
ISM 0512	ISUR = ISUR+3		CRS13470
ISM 0513	95 SUM = SUM+IISUB*PL(L,K)		CRS13480
ISM 0514	VELTOT(JJ,K) = VAD*SUM		CRS13490
ISM 0515	80 CONTINUE		CRS13500
ISM 0516	75 CONTINUE		CRS13510
	C LOOP N		CRS13520
ISM 0517	DO 105 K = 1,3		CRS13530
ISM 0518	IF(IISPK(K)) 110,105,110		CRS13540
ISM 0519	110 IF(VL3(K)) 105,105,115		CRS13550
ISM 0520	115 SK = SIK		CRS13560
ISM 0521	SDIF = SK-SF(1,K)		CRS13570
ISM 0522	IF(SDIF) 120,120,125		CRS13580
ISM 0523	120 IF(IISPK(K)) 130,130,135		CRS13590
ISM 0524	125 ISD(1,K) = 1		CRS13600
ISM 0525	135 FSPU = FSPOT(1,K)*KKE(1,K)*SDIF		CRS13610
ISM 0526	130 FSPU = 0.0		CRS13620
ISM 0527	140 FSPU = 0.0		CRS13630
ISM 0528	60 TO 190		CRS13640
ISM 0529	130 SP = SA-FSPBAR(1,K)		CRS13650
ISM 0530	C FSPBAR = SHAR-FSPBAR/RE		CRS13660
ISM 0531	150 FSHU = 0.0		CRS13670
ISM 0532	60 TO 160		CRS13680
	C COMPUTE FSPU PER NEW EXTERNAL SPRING LOAD-STROKE CURVES 7/25/72		CRS13690
ISM 0533	155 FSPU=FSPU(1,K)		CRS13700
ISM 0534	IF (ISP-GE-SR(1,K)) 60 TO 160		CRS13710
ISM 0535	IF (ISP-GE-SR(1,K)) 60 TO 157		CRS13720
ISM 0536	FSPU=FSPU(1,K)		CRS13730
ISM 0537	IF (ISP-GE-SR(1,K)) 60 TO 160		CRS13740
ISM 0538	FSPU=FSPU(1,K)		CRS13750
ISM 0539	60 TO 160		CRS13760
ISM 0540	FSPU=FSPU(1,K)		CRS13770
ISM 0541	60 TO 160		CRS13780
ISM 0542	157 FSPU=FSPU(1,K)*(SP-SA(1,K))/(FSPU-FSPOT(1,K))/(ISB(1,K)-SA(1,K))		CRS13790
ISM 0543	160 IF(SR(1,K)) 165,165,170		CRS13800
ISM 0544	170 MNT(K) = 0		CRS13810
ISM 0545	60 TO 190		CRS13820
ISM 0546	165 IF(MNT(K)) 190,175,190		CRS13830
ISM 0547	175 MNT(K) = 1		CRS13840
ISM 0548	FSPBAR(K) = SK-FSPU/KKE(1,K)		CRS13850
ISM 0549	C (13A)		CRS13860
ISM 0550	190 VX = AT(3,K)*FSPU		CRS13870
ISM 0551	XVOC(3,K) = VX		CRS13880
ISM 0552	V1 = VFEDOT(1,K)		CRS13890

Figure 108. (Continued)

ISM 0553	V2 = VEEDUT(L,K)	CRS13900
ISM 0554	V66 = SURT(VIOL+2*V2)	CRS13910
ISM 0555	IF(VIOL) 210,210,200	CRS13920
ISM 0556	260 VA = XNU(I,K)*VA/V66	CRS13930
ISM 0557	AVLC(I,K) = VRAVE(LUT(I,K))	CRS13940
ISM 0558	AVUC(I,K) = VRAVE(LUT(I,K))	CRS13950
ISM 0559	210 IS = C	CRS13960
	C LOOP C	CRS13970
ISM 0560	DO 220 J = 1,3	CRS13980
ISM 0561	SUM = 0.0	CRS13990
	C LOOP R	CRS14000
ISM 0562	DO 230 L = 1,3	CRS14010
ISM 0563	IS = IS+1	CRS14020
ISM 0564	230 SUM = SUM-AL(I,S)*AVOL(L,K)	CRS14030
ISM 0565	220 FSP(J,K) = SUM	CRS14040
	C END OF LOOP N	CRS14050
ISM 0566	105 CONTINUE	CRS14060
	C CRASH PLALS	CRS14070
ISM 0567	DO 240 J = 1,3	CRS14080
ISM 0568	SUM = 0.0	CRS14090
ISM 0569	DO 250 K = 1,3	CRS14100
ISM 0570	250 SUM = SUM+FSP(J,K)	CRS14110
ISM 0571	240 XCI(J) = SUM	CRS14120
ISM 0572	UXV(I) = UX(I)	
ISM 0573	UXV(I) = UX(I)	
ISM 0574	UXV(I) = UX(I)	
ISM 0575	UPG(IH(1)) = CLIN(I)	
ISM 0576	UPG(IH(2)) = CLIN(I)	
ISM 0577	UPG(IH(3)) = CLIN(I)	
ISM 0578	UPG(IH(4)) = CLIN(I)	
ISM 0579	IF(XI) = FSP(I)*ALNG(IH(1))	
ISM 0580	IF(XI) = FSP(I)*ALNG(IH(2))	
ISM 0581	IF(XI) = FSP(I)*ALNG(IH(3))	
ISM 0582	IF(XI) = FSP(I)*ALNG(IH(4))	
ISM 0583	IF(XI) = FSP(I)*ALNG(IH(5))	
ISM 0584	IF(XI) = FSP(I)*ALNG(IH(6))	
	C CRASH TERMINIS	CRS14130
ISM 0585	XCI(4) = TERM(5)-TERM(3)	
ISM 0586	XCI(5) = TERM(6)-TERM(4)	
ISM 0587	XCI(6) = TERM(7)-TERM(5)	
ISM 0588	CALL MAI V(LC(I),UXV,UXVZPK+1)	
ISM 0589	DO 460 B = 1,3	
ISM 0590	IF(I)UXV(I) = UXV(I)	
ISM 0591	SUM = 0.0	
ISM 0592	DO 470 J = 1,3	
ISM 0593	270 SUM = SUM-SP(I,K)*UXVZPK(J)	
ISM 0594	UXV(I,K) = CLIN(I,K)*SUM-TERM(IK)*OPORIN(IK+1)	
ISM 0595	260 CONTINUE	
ISM 0596	C	
ISM 0597	RETURN	CRS14170
	*****	CRS14180
ISM 0598	*****	CRS14190
	*****	CRS14200
	*****	CRS14210
	*****	CRS14220

Figure 108. (Continued)

15M 0599	***** LIMENSION XMPR(3),ABAPR(3,3),ANGDPR(3,3),DPR(3,3),ATDP(3,3), 1 ATC(3,2),APR(3,3),VAP(60,3),VAP(3,3),VAP(3,3),VAP(3,3), EQUVALENCE (PHR,XMPR(1)), (OPR,XMPR(2)), (THPR,XMPR(3)), 1 (PHIDPR,ANGDPR(1)), (THEDPR,ANGDPR(2)), (PSIDPR,ANGDPR(3)) WTOT = 0.0	CRS14230 CRS14240 CRS14250 CRS14260 CRS14270 CRS14280
15M 0601	DO 2040 I = 1,NM WTOT = WTOT + MGT(I)	CRS14290 CRS14300 CRS14310 CRS14320 CRS14330 CRS14340
15M 0602	XGUP = 0.0	CRS14350 CRS14360 CRS14370 CRS14380 CRS14390 CRS14400
15M 0603	YGUP = 0.0	CRS14410 CRS14420 CRS14430 CRS14440 CRS14450 CRS14460
15M 0604	ZGUP = 0.0	CRS14470 CRS14480 CRS14490 CRS14500 CRS14510 CRS14520
15M 0605	DO 2020 I = 1,NM XGUP = XGUP + MGT(I)*VAP(I)	CRS14530 CRS14540 CRS14550 CRS14560 CRS14570 CRS14580
15M 0606	ZGUP = ZGUP + MGT(I)*VAP(I)	CRS14590 CRS14600 CRS14610 CRS14620 CRS14630 CRS14640
15M 0607	YGUP = YGUP + MGT(I)*VAP(I)	CRS14650 CRS14660 CRS14670 CRS14680 CRS14690 CRS14700
15M 0608	ZGUP = ZGUP + MGT(I)*VAP(I)	CRS14710 CRS14720 CRS14730 CRS14740 CRS14750 CRS14760
15M 0609	DO 2020 I = 1,NM XGUP = XGUP + MGT(I)*VAP(I)	CRS14770 CRS14780 CRS14790 CRS14800 CRS14810 CRS14820
15M 0610	ZGUP = ZGUP + MGT(I)*VAP(I)	CRS14830 CRS14840 CRS14850 CRS14860 CRS14870 CRS14880
15M 0611	XGUP = XGUP + MGT(I)*VAP(I)	CRS14890 CRS14900 CRS14910 CRS14920 CRS14930 CRS14940
15M 0612	YGUP = YGUP + MGT(I)*VAP(I)	CRS14950 CRS14960 CRS14970 CRS14980 CRS14990 CRS15000
15M 0613	ZGUP = ZGUP + MGT(I)*VAP(I)	CRS15010 CRS15020 CRS15030 CRS15040 CRS15050 CRS15060
15M 0614	CALL LUNKEPR,PHIDPR,THEDPR,PSIDPR	CRS15070 CRS15080 CRS15090 CRS15100 CRS15110 CRS15120
15M 0615	SI = SIN(PHIDPR)	CRS15130 CRS15140 CRS15150 CRS15160 CRS15170 CRS15180
15M 0616	CI = COS(PHIDPR)	CRS15190 CRS15200 CRS15210 CRS15220 CRS15230 CRS15240
15M 0617	S2 = SIN(THEDPR)	CRS15250 CRS15260 CRS15270 CRS15280 CRS15290 CRS15300
15M 0618	C2 = COS(THEDPR)	CRS15310 CRS15320 CRS15330 CRS15340 CRS15350 CRS15360
15M 0619	C NUM ABAPR(1,1) = 1.0	CRS15370 CRS15380 CRS15390 CRS15400 CRS15410 CRS15420
15M 0620	ABAPR(1,2) = 0.0	CRS15430 CRS15440 CRS15450 CRS15460 CRS15470 CRS15480
15M 0621	ABAPR(1,3) = 0.0	CRS15490 CRS15500 CRS15510 CRS15520 CRS15530 CRS15540
15M 0622	ABAPR(1,2) = 51*52/C2	CRS15550 CRS15560 CRS15570 CRS15580 CRS15590 CRS15600
15M 0623	ABAPR(2,2) = 1.1	CRS15610 CRS15620 CRS15630 CRS15640 CRS15650 CRS15660
15M 0624	ABAPR(1,2) = 51/C2	CRS15670 CRS15680 CRS15690 CRS15700 CRS15710 CRS15720
15M 0625	ABAPR(1,3) = C1*52/C2	CRS15730 CRS15740 CRS15750 CRS15760 CRS15770 CRS15780
15M 0626	ABAPR(2,3) = -51	CRS15790 CRS15800 CRS15810 CRS15820 CRS15830 CRS15840
15M 0627	ABAPR(3,2) = C1/C2	CRS15850 CRS15860 CRS15870 CRS15880 CRS15890 CRS15900
15M 0628	C ANGLE DLT PRIMES (6)	CRS15910 CRS15920 CRS15930 CRS15940 CRS15950 CRS15960
15M 0629	CALL MATVEC(ABAPR,XMPR,ANGDPR,0)	CRS15970 CRS15980 CRS15990 CRS16000 CRS16010 CRS16020
15M 0630	C U PRIME (7)	CRS16030 CRS16040 CRS16050 CRS16060 CRS16070 CRS16080
15M 0631	DPK(1,1) = 0.0	CRS16090 CRS16100 CRS16110 CRS16120 CRS16130 CRS16140
15M 0632	DPK(1,2) = THEDPR*SI-PSIDPR*CI*52	CRS16150 CRS16160 CRS16170 CRS16180 CRS16190 CRS16200
15M 0633	DPK(1,3) = THEDPR*CI-PSIDPR*SI*52	CRS16210 CRS16220 CRS16230 CRS16240 CRS16250 CRS16260
15M 0634	DPK(2,1) = -DPK(1,2)	CRS16270 CRS16280 CRS16290 CRS16300 CRS16310 CRS16320
15M 0635	DPK(2,2) = 0.0	CRS16330 CRS16340 CRS16350 CRS16360 CRS16370 CRS16380
15M 0636	DPK(2,3) = -DPK(1,3)	CRS16390 CRS16400 CRS16410 CRS16420 CRS16430 CRS16440
15M 0637	DPK(3,1) = 0.0	CRS16450 CRS16460 CRS16470 CRS16480 CRS16490 CRS16500
15M 0638	DPK(3,2) = 0.0	CRS16510 CRS16520 CRS16530 CRS16540 CRS16550 CRS16560
15M 0639	DPK(3,3) = 0.0	CRS16570 CRS16580 CRS16590 CRS16600 CRS16610 CRS16620
15M 0640	C A DUT PRIME (8)	CRS16630 CRS16640 CRS16650 CRS16660 CRS16670 CRS16680
15M 0641	CALL MATMUL(APR,DPR,ADPR)	CRS16690 CRS16700 CRS16710 CRS16720 CRS16730 CRS16740
15M 0642	ZCRAX = 0.0	CRS16750 CRS16760 CRS16770 CRS16780 CRS16790 CRS16800
15M 0643	C LOOP A	CRS16810 CRS16820 CRS16830 CRS16840 CRS16850 CRS16860
15M 0644	DO 2040 I = 1,NM	CRS16870 CRS16880 CRS16890 CRS16900 CRS16910 CRS16920
15M 0645	WTOT = WTOT + MGT(I)	CRS16930 CRS16940 CRS16950 CRS16960 CRS16970 CRS16980
15M 0646	CALL EULER(ADPR,PHIDPR,THEDPR(1),PSIDPR(1))	CRS16990 CRS17000 CRS17010 CRS17020 CRS17030 CRS17040
15M 0647	C AI (10)	CRS17050 CRS17060 CRS17070 CRS17080 CRS17090 CRS17100

Figure 108. (Continued)

15M 0642	CALL MATMULTAW, AILP, AIC	CRS14770
15M 0643	IPETAT(1) = -ARSTHAI(1,1)	CRS14780
15M 0644	CT = 1.0/COS(THETA(1))	CRS14790
15M 0645	PHI(1) = ARSTHAI(1,2)	CRS14800
15M 0646	PSI(1) = ARSTHAI(1,3)	CRS14810
	C (1,2)	CRS14820
15M 0647	VJP(1,1) = XGDP-XLP(1)	CRS14830
15M 0648	VJP(1,2) = YGDP-YDP(1)	CRS14840
15M 0649	VJP(1,3) = ZGDP-ZDP(1)	CRS14850
	C LOPP U	CRS14860
15M 0650	DO 2050 K = 1,3	CRS14870
15M 0651	IF(1,1) = 1, 2060, 2050, 2060	CRS14880
15M 0652	2060 VC = AIC(1,1)*XGDP(1,1)	CRS14890
15M 0653	DO 2070 L = 1,3	CRS14900
15M 0654	2070 VC = VL*APR(1,1)*VJP(1,1)	CRS14910
15M 0655	IF(VC-ZCMAX) 2080, 2050, 2080	CRS14920
15M 0656	2080 ZCMAX = VC	CRS14930
15M 0657	2050 CONTINUE	CRS14940
	C END OF LOOP A	CRS14950
15M 0658	2040 CONTINUE	CRS14960
15M 0659	2100 IF(1,1) 2220, 2210, 2220	CRS14970
15M 0660	2210 ZC = ZCMAX*(CT/DO)	CRS14980
	C 3PC IF PHI(1) 1, ETC. ARE ALL ZERO	CRS14990
15M 0661	2220 DO 2100 J = 1, MM	CRS15000
15M 0662	IF(1,1) 2150, 2110, 2150	CRS15010
15M 0663	2110 IF(1,1) 2150, 2120, 2150	CRS15020
15M 0664	2120 IF(1,1) 2150, 2100, 2150	CRS15030
15M 0665	2100 CONTINUE	CRS15040
	C IF WE GET HERE WE COMPUTE NEW THETA(1,1) AND PSI(1,1)	CRS15050
15M 0666	P1 = 3.141592653589793240	CRS15060
15M 0667	P1Z = .500871	CRS15070
15M 0668	DO 2160 IJ = 1, JGS	CRS15080
15M 0669	I = 10(IJ)	CRS15090
15M 0670	J = 10(IJ)	CRS15100
15M 0671	XIJP = VJP(1,1)-VJP(1,1)	CRS15110
15M 0672	YIJP = VJP(1,2)-VJP(1,2)	CRS15120
15M 0673	ZIJP = VJP(1,3)-VJP(1,3)	CRS15130
15M 0674	IF(VIJP) 2140, 2130, 2140	CRS15140
15M 0675	2130 IF(1,1) 2180, 2170, 2180	CRS15150
15M 0676	2180 PSI(1,1) = 0.0	CRS15160
15M 0677	THI(1,1) = -ATAN2(ZIJP, XIJP)	CRS15170
15M 0678	CU 10 2260	CRS15180
15M 0679	2170 PSI(1,1) = 0.0	CRS15190
15M 0680	THI(1,1) = -PIZ	CRS15200
15M 0681	IF(1,1) 2140, 2200, 2200	CRS15210
15M 0682	2160 THI(1,1) = P1Z	CRS15220
15M 0683	CU 10 2200	CRS15230
15M 0684	2140 PSI(1,1) = ATAN2(YIJP, XIJP)	CRS15240
15M 0685	THI(1,1) = -ATAN2(ZIJP, YIJP)	CRS15250
15M 0686	2200 CONTINUE	CRS15260
	C LOPPC	CRS15270
15M 0687	2150 DO 2090 I = 1, MM	CRS15280
15M 0688	VIP(1) = VJP(1,1)	CRS15290
15M 0689	VIP(2) = VJP(1,2)	CRS15300

Figure 108. (Continued)

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ISM 0690      VIP(3) = VJP(1,3)
C (114)
ISM 0691      CALL MATVEC(AIPR,VIP,XV,0)
ISM 0692      XV(3) = XV(3)+2G
ISM 0693      X(1) = XV(1)
ISM 0694      Y(1) = XV(2)
ISM 0695      Z(1) = XV(3)
C (115)
ISM 0696      CALL MATVEC(AUPR,VIP,XV,0)
ISM 0697      XV(1) = XV(1)+6G001
ISM 0698      XV(2) = XV(2)+6G001
ISM 0699      XV(3) = XV(3)+6G001
ISM 0700      X001(1) = Z(1)
ISM 0701      X001(1) = XV(2)
ISM 0702      X001(1) = XV(3)
C (116)
ISM 0703      CALL MATVEC(AIC,XV,VIP,1)
ISM 0704      U(1) = VIP(1)
ISM 0705      V(1) = VIP(2)
ISM 0706      W(1) = V*(13)
C (117)
ISM 0707      CALL MATVEC(AIDP,XRPR,VIP,1)
ISM 0708      P(1) = VIP(1)
ISM 0709      Q(1) = VIP(2)
ISM 0710      R(1) = VIP(3)
C AIDAR (118)
ISM 0711      S1 = SIM(PHI(1))
ISM 0712      C1 = COS(PHI(1))
ISM 0713      S2 = SIM(THETA(1))
ISM 0714      C2 = COS(THETA(1))
ISM 0715      ABARPR(1,2) = S1*S2/C2
ISM 0716      ABARPR(2,2) = C1
ISM 0717      ABARPR(3,2) = S1/C2
ISM 0718      ABARPR(1,3) = C1*S2/C2
ISM 0719      ABARPR(2,3) = -S1
ISM 0720      ABARPR(3,3) = C1/C2
C (119)
ISM 0721      CALL MATVEC(ABARPR,VIP,XV,0)
ISM 0722      PHI(011) = XV(1)
ISM 0723      THETA(11) = XV(2)
ISM 0724      PSI(0011) = XV(3)
C END LOOP C
ISM 0725      2090 CONTINUE
ISM 0726      PRINT /301
ISM 0727      2301 FORMAT(IHO,'IJ,THEI(JIJ),PSII(JIJ), )
ISM 0728      PRINT 2300,(IJ,THEI(JIJ),PSII(JIJ),IJ=1,IGS)
ISM 0729      2300 FORMAT (1H ,I5,1P2E15.5)
ISM 0730      RETURN
C *****
ISM 0731      ENJAY PRINT
C *****
ISM 0732      HEAL=4 REI(60),PEI 601
C *****

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Figure 108. (Continued)

Figure 108. (Continued)

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1SM 0712      820 FURNAT(11,17,2X,1P,15.5)
1SM 0713      3040 CONTINUE
1SM 0714      151140
1SM 0715      DO 3070 I = 1,IGS
1SM 0716      IF (11PR(1)-EQ-LLANK) GO TO 3070
1SM 0717      IF (15EL,EO,1) GO TO 3065
1SM 0718      PRINT 221
1SM 0719      821 FURNAT(11 //1X,MASS,7X,DR1*)
1SM 0720      151141
1SM 0721      3065 PRINT 822, JG(1),DR1(1)
1SM 0722      3070 CONTINUE
1SM 0723      822 FURNAT(11,1X,12,3X,1PE15.5)
1SM 0724      SUMCL1 = 0.0
1SM 0725      SUMPL1 = 0.0
1SM 0726      SUMSE1 = 0.0
1SM 0727      SUMCL1 = 0.0
1SM 0728      SUMPL1 = 0.0
1SM 0729      SUMSE1 = 0.0
1SM 0730      SUMCL1 = 0.0
1SM 0731      SUMPL1 = 0.0
1SM 0732      SUMSE1 = 0.0
1SM 0733      DO 3400 I = 1,MM
1SM 0734      C*****DON'T USE AN I IF IT'S = TO A J OF A DRI I-J PAIR
1SM 0735      DO 3400 IJ = 1,IGS
1SM 0736      IF (11PR(1J)-EQ-LLANK) GO TO 3400
1SM 0737      IF (11M,1J) GO TO 3400
1SM 0738      K(11) = 0.0
1SM 0739      K(1J) = 0.0
1SM 0740      GO TO 3400
1SM 0741      3400 CONTINUE
1SM 0742      PL(11) = -M(11)/2(11)
1SM 0743      SUMPL1 = SUMPL1+PL(11)
1SM 0744      K(11) = .5*(M(11)+PL(11)+M(11)+M(11)+M(11)+M(11))/386.0
1SM 0745      1 *PL(11)*PL(11) X(11)*X(11)+X(11)*X(11)+X(11)*X(11)
1SM 0746      2 *PL(11)*X(11)+X(11)*X(11)+X(11)*X(11)+X(11)*X(11)
1SM 0747      3 *K(11)*PL(11)+X(11)*X(11)+X(11)*X(11)+X(11)*X(11)
1SM 0748      SUMCL1 = SUMCL1+K(11)
1SM 0749      3400 CONTINUE
1SM 0750      DO 3400 IJ = 1,IGS
1SM 0751      SUMSE1 = SUMSE1+SE1(11)
1SM 0752      IF (11K(1)-EQ-0) GO TO 3400
1SM 0753      DO 3400 IK = 1,1MCI
1SM 0754      3407 SUMCL1 = SUMCL1+CL1(11)*K(11)
1SM 0755      3409 L107 = SUMPL1+SUMPL1+SUMSE1+SUMSE1+SUMCL1+SUMCL1
1SM 0756      PRINT 2401
1SM 0757      3401 PL*MA(11,0,0X,0.11AL,7X,KINETIC,0X,POTENTIAL,7X,STRAIN,7X,
1SM 0758      1 *DAMPING,7X,CUSHING,7X,ENERGY,510X,ENERGY,1/)
1SM 0759      PRINT 3402, L107, SUMCL1, SUMSE1, SUMSE1, SUMCL1, SUMCL1
1SM 0760      3402 FURNAT(11,12,1X,15.5)
1SM 0761      PCSE = SUMSE1/101
1SM 0762      PCCL = SUMCL1/101
1SM 0763      PCSE = SUMSE1/101
1SM 0764      PCCL = SUMCL1/101
1SM 0765      PRINT 3403, PCSE, PCSE, PCSE, PCSE, PCSE, PCSE
1SM 0766      PRINT 3403, PCSE, PCSE, PCSE, PCSE, PCSE, PCSE
1SM 0767      PRINT 3404, PERCENT OF, 2P5F14.3/3X, TOTAL ENERGY)
1SM 0768      PRINT 3404

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Figure 108. (Continued)

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1SM 0836 3404 FORMAT(1X, 'INTERNAL', 40X, 'EXTERNAL', 753X, 'BEAM', 49X, 'SPRING', 7
1 7A, 'KINETIC', 15X, 'POTENTIAL', 20X, 'STRAIN', 17X, 'DAMPING', 20X,
2 'CRUSHING', 7X, 'MASS ENERGY PER CENT ENERGY PER CENT I
3J I J ENERGY PER CENT ENERGY PER CENT I K ENER
4GY PER CENT I J)
1SM 0837 MAXLN = MAX(INH, IUS, IJCT)
1SM 0838 IF (TIME - LG - 0.0) MAXEN = NM
1SM 0840 DO 3410 IU = 1, MAXEN
1SM 0841 IOPTR = 0
1SM 0842 IF (IU - G1 - NM) GO TO 3411
1SM 0844 IOPTR = IOPTR + 4
1SM 0845 PCSE = KL(IU) / SUMKEI
1SM 0846 PCPE = PE(IU) / SUMPEI
1SM 0847 IF (IHL - LG - 0.0) GO TO 3504
1SM 0849 3411 IF (IU - G1 - IGS) GO TO 3412
1SM 0851 IOPTR = IOPTR + 2
1SM 0852 PCSE = SE(IU) / SUMSEI
1SM 0853 PCDE = DE(IU) / SUMDEI
1SM 0854 3412 IF (IU - G1 - IJCT) GO TO 3413
1SM 0856 IOPTR = IOPTR + 1
1SM 0857 I = JI(IU)
1SM 0858 K = KR(IU)
1SM 0859 CE = CEIN(I, K)
1SM 0860 PCSE = CE / SUMCEI
1SM 0861 3413 GO TO 3501, 3502, 3503, 3504, 3505, 3506, 3507, IOPTR
C-----ONLY CE
1SM 0862 3501 PRINT 3511, I, K, CE, PCCE
1SM 0863 3511 FORMAT(1X, 13, 12, 1PE13.5, 2PF9.3)
1SM 0864 GO TO 3410
C-----SE AND DE
1SM 0865 3502 PRINT 3511, I, G1, IJCT, JG(IU), SE(IU), PCSE, DE(IU), PCDE
1SM 0866 3512 FORMAT(1X, 13, 12, 1PE13.5, 2PF9.3, 1PE14.5, 2PF9.3)
1SM 0867 GO TO 3410
C-----SE, DE, CE
1SM 0868 3503 PRINT 3511, I, G1, IJCT, JG(IU), SE(IU), PCSE, DE(IU), PCDE, I, K, CE,
1 PCCE
1SM 0869 3513 FORMAT(1X, 13, 12, 1PE13.5, 2PF9.3, 1PE14.5, 2PF9.3, 14, 12, 1PE13.5, 2PF9.3)
1SM 0870 GO TO 3410
C-----KE AND PE
1SM 0871 3504 PRINT 3514, I, KE(IU), PCKE, PE(IU), PCPE
1SM 0872 3514 FORMAT(1X, 13, 1PE13.5, 2PF9.3, 1PE14.5, 2PF9.3)
1SM 0873 GO TO 3410
C-----KE, PE, CE
1SM 0874 3505 PRINT 3515, I, KE(IU), PCKE, PE(IU), PCPE, I, K, CE, PCCE
1SM 0875 3515 FORMAT(1X, 13, 1PE13.5, 2PF9.3, 1PE14.5, 2PF9.3, 50X, 212, 1PE13.5, 2PF9.3)
1SM 0876 GO TO 3410
C-----KE, PE, SE, DE
1SM 0877 3506 PRINT 3516, I, KE(IU), PCKE, PE(IU), PCPE, I, G1, IJCT, JG(IU), SE(IU),
1 PCSE, DE(IU), PCDE
1SM 0878 3516 FORMAT(1X, 13, 1PE13.5, 2PF9.3, 1PE14.5, 2PF9.3, 2X, 313, 1PE13.5, 2PF9.3,
1 1PE14.5, 2PF9.3)
1SM 0879 GO TO 3410
C-----KE, PE, SE, DE, CE
1SM 0880 3507 PRINT 3517, I, KE(IU), PCKE, PE(IU), PCPE, I, G1, IJCT, JG(IU), SE(IU),
1 1PE14.5, 2PF9.3)

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Figure 108. (Continued)

		PAGE 020	
ISM 0881	3517	PCSE-DAT(10),PCSE-LK(15),PCCE 1 3PE13.5,2PF9.3,1PE13.5,2PF9.3,1PE13.5,2PF9.3,1PE13.5,2PF9.3	CRS16500 CRS16510 CRS16520 CRS16530 CRS16540 CRS16550 CRS16560 CRS16570 CRS16580 CRS16590 CRS16600 CRS16610 CRS16620 CRS16630 CRS16640 CRS16650 CRS16660 CRS16670 CRS16680 CRS16690 CRS16700 CRS16710 CRS16720 CRS16730 CRS16740 CRS16750 CRS16760 CRS16770 CRS16780 CRS16790 CRS16800 CRS16810 CRS16820 CRS16830 CRS16840 CRS16850 CRS16860 CRS16870 CRS16880 CRS16890 CRS16900 CRS16910 CRS16920 CRS16930 CRS16940 CRS16950 CRS16960 CRS16970 CRS16980 CRS16990
ISM 0882	3518	CONTINUE	
ISM 0883	3519	CONTINUE	
ISM 0884	3520	ENTRY SAVE	
ISM 0885	3521	REAL*4 PLOT,ZAK,IMPL01	
ISM 0886	3522	IPLC = 0	
ISM 0887	3523	JPL01 = JPL01*1	
ISM 0888	3524	IF(JPL01.GT.1P) RETURN	
ISM 0889	3525	IMPL01(JPL01) = TIME	
ISM 0890	3526	DO 35 J = 1,IMPL01	
ISM 0891	3527	IS = JPL01(J)	
ISM 0892	3528	ID = IMPL01(J)	
ISM 0893	3529	GO TO (1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0894	3530	1 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0895	3531	1 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0896	3532	1 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0897	3533	2 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0898	3534	3 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0899	3535	4 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0900	3536	5 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0901	3537	6 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0902	3538	7 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0903	3539	8 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0904	3540	9 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0905	3541	10 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0906	3542	11 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0907	3543	12 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0908	3544	13 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0909	3545	14 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0910	3546	15 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0911	3547	16 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0912	3548	17 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0913	3549	18 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0914	3550	19 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0915	3551	20 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0916	3552	21 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0917	3553	22 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0918	3554	23 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0919	3555	24 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0920	3556	25 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0921	3557	26 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0922	3558	27 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0923	3559	28 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0924	3560	29 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0925	3561	30 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0926	3562	31 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0927	3563	32 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	
ISM 0928	3564	33 2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,21,	

Figure 108. (Continued)

PAGE 021		
15N 0929	16 1 = VEE2(6,15)	CRS17000
15N 0930	GO TO 39	CRS17010
15N 0931	19 1 = SUMOF(1,15)	CRS17020
15N 0932	GO TO 39	CRS17030
15N 0933	20 1 = SUMOF(2,15)	CRS17040
15N 0934	GO TO 39	CRS17050
15N 0935	21 1 = SUMOF(3,15)	CRS17060
15N 0936	GO TO 39	CRS17070
15N 0937	22 1 = SUMOF(4,15)	CRS17080
15N 0938	GO TO 39	CRS17090
15N 0939	23 1 = SUMOF(5,15)	CRS17100
15N 0940	GO TO 39	CRS17110
15N 0941	24 1 = SUMOF(6,15)	CRS17120
15N 0942	GO TO 39	CRS17130
15N 0943	25 1 = SUMOF(7,15)	CRS17140
15N 0944	39 PLOT(15,15)	CRS17150
15N 0945	RETURN	CRS17160
15N 0946	END	CRS17170

Figure 108. (Continued)


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15M 0030 IF(IPLPLOT(I),LE,12) GO TO 4020 CRS17590
15M 0032 NY = 12 CRS17600
15M 0033 IYCHAR(2) = LIT(2) CRS17610
15M 0034 CALL FMSG(ZAR,1,4,0,IGT,PLOT(I),IYCHAR(2)) CRS17620
15M 0035 CALL GTC(ZZ(ZAR,IMULU,3,IMUF)) CRS17630
15M 0036 CALL PUTC(ZZ(ZAR,IMULU,3,IYCHAR(2))) CRS17640
15M 0037 CALL GTC(ZZ(ZAR,IMULU,4,IMUF)) CRS17650
15M 0038 CALL PUTC(ZZ(ZAR,IMULU,4,IYCHAR(2))) CRS17660
15M 0039 CALL GTC(ZZ(ZAR,IMULU,5,IMUF)) CRS17670
15M 0040 CALL PUTC(ZZ(ZAR,IMULU,5,IYCHAR(2))) CRS17680
15M 0041 CALL GTC(ZZ(ZAR,IMULU,6,IMUF)) CRS17690
15M 0042 CALL PUTC(ZZ(ZAR,IMULU,6,IYCHAR(2))) CRS17700
15M 0043 CALL GTC(ZZ(ZAR,IMULU,7,IYCHAR(2))) CRS17710
15M 0044 GO TO 4030 CRS17720
15M 0045 CALL FMSG(ZAR,1,4,0,IGT,PLOT(I),IYCHAR(2)) CRS17730
15M 0046 IYCHAR(2) = LIT(2) CRS17740
15M 0047 NY = 6 CRS17750
15M 0048 GO TO 4025 CRS17760
15M 0049 CALL FMSG(ZAR,1,4,0,IGT,PLOT(I),IYCHAR(2)) CRS17770
15M 0050 CALL GTC(ZZ(ZAR,IMULU,3,IMUF)) CRS17780
15M 0051 CALL PUTC(ZZ(ZAR,IMULU,3,IYCHAR(2))) CRS17790
15M 0052 CALL GTC(ZZ(ZAR,IMULU,4,IMUF)) CRS17800
15M 0053 CALL PUTC(ZZ(ZAR,IMULU,4,IYCHAR(2))) CRS17810
15M 0054 CALL GTC(ZZ(ZAR,IMULU,5,IMUF)) CRS17820
15M 0055 CALL PUTC(ZZ(ZAR,IMULU,5,IYCHAR(2))) CRS17830
15M 0056 VMIN = VMIN CRS17840
15M 0057 VMAX = VMAX CRS17850
15M 0058 DO 4050 J = 2,JPL0T CRS17860
15M 0059 VMIN = AMIN(VMIN,PLOT(I,J)) CRS17870
15M 0060 VMAX = AMAX(VMAX,PLOT(I,J)) CRS17880
15M 0061 IF (ABS(VMAX-VMIN)-GT,1E-7) GO TO 4050 CRS17890
15M 0062 VMAX = 1 CRS17900
15M 0063 VMIN = -1 CRS17910
15M 0064 GO TO 4050 CRS17920
15M 0065 VMAX = VMAX+0.1*VMAX CRS17930
15M 0066 VMIN = VMIN-0.1*VMIN CRS17940
15M 0067 GO TO 4050 CRS17950
15M 0068 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS17960
15M 0069 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS17970
15M 0070 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS17980
15M 0071 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS17990
15M 0072 DO 4060 J = 10,104 CRS18000
15M 0073 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18010
15M 0074 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18020
15M 0075 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18030
15M 0076 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18040
15M 0077 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18050
15M 0078 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18060
15M 0079 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18070
15M 0080 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18080
15M 0081 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18090
15M 0082 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18100
15M 0083 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18110
15M 0084 CALL SUBJ(GZAR,VMIN,VMAX,VMAX) CRS18120

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Figure 108. (Continued)

LEVEL 20.1 (MAY 71)	US/360 FORTRAN M	DATE 73-298/10.03.44
*****		*****
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*		*
*		*
*		*
*****		*****
COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=56,SIZE=0000K, SOURCE=H00,WHIST=NOBCE,LOAD=HAP,WRDIT=10,XREF		
ISM 0002	SUBROUTINE EULERIA,PHI,TM,TAP,PSI	CRS18120
ISM 0003	IMPLICIT REAL*8 (A-H,O-Z)	CRS18130
ISM 0004	COMMON (A19)	CRS18140
ISM 0005	SIN(PI) = DSIN(PI)	CRS18150
ISM 0006	COS(PI) = DCOS(PI)	CRS18160
ISM 0007	S1 = S1RPHI	CRS18170
ISM 0008	C1 = C1SPH(PI)	CRS18180
ISM 0009	S2 = S1RTHETA	CRS18190
ISM 0010	C2 = C1STHETA	CRS18200
ISM 0011	S3 = S1RPSI	CRS18210
ISM 0012	C3 = C1SPSI	CRS18220
ISM 0013	A11 = C2*PC3	CRS18230
ISM 0014	A121 = C2*PC3	CRS18240
ISM 0015	A131 = -S2	CRS18250
ISM 0016	A141 = -C1*PC3+S1*PC3	CRS18260
ISM 0017	A151 = C1*PC3+S1*PC3	CRS18270
ISM 0018	A161 = S1*PC2	CRS18280
ISM 0019	A171 = S1*PC3+C1*PC3	CRS18290
ISM 0020	A181 = -S1*PC3+C1*PC3	CRS18300
ISM 0021	A191 = C1*PC2	CRS18310
ISM 0022	RETURN	CRS18320
ISM 0023	END	CRS18330

Figure 108. (Continued)

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LEVEL 20.1 (MAY 71)
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*****

                                OS/360  FORTRAN M
                                DATE 73.298/10.03.54
*****

                                COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=56,SIZE=0000K,
                                SOURCE=BCU,MKLIST=HUBCK,LOAD=MAP,MODIFY=10,XREF
                                SUBROUTINE MATMUL(A,B,C)
                                IMPLICIT REAL*8 (A-H,O-Z)
                                DIMENSION A(3,3),B(3,3),C(3,3)
                                C=A*B
                                DO 10 I=1,3
                                DO 10 J=1,3
                                SUM=0.0
                                DO 20 K=1,3
                                SUM=SUM+A(I,K)*B(K,J)
                                C(I,J)=SUM
                                RETURN
                                END

                                CS18340
                                CS18350
                                CS18360
                                CS18370
                                CS18380
                                CS18390
                                CS18400
                                CS18410
                                CS18420
                                CS18430
                                CS18440
                                CS18450

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Figure 108. (Continued)

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LEVEL 20.1 (MAY 71)
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*
*****

US/J60 FORTRAN M
DATE 73-298/10-04-05
*****

CUMPIER OPTIONS - NAME= MAIN,OPT=02,LINECT=56,SIZE=000K,
SOURCE=BCD,NOLIST,MODECK,LOAD=MAP,MODIT=10,REF
ISN 0002 SUBROUTINE MATVECTIA,V,P,ISM)
ISN 0003 IMPLICIT REAL*8 (A-H,D-Z)
ISN 0004 DIMENSION AT(3),V(3),P(3)
C A*V TO P IF ISM = 0, ELSE AT*V TO P
DO 10 I = 1,3
SUM = 0.0
DO 20 K = 1,3
IF(I)SM) 40,30,50
30 SUM = SUM+AT(I,K)*V(K)
40 SUM = SUM+AT(K,I)*V(K)
20 CONTINUE
10 P(I) = SUM
RETURN
END
ISN 0005
ISN 0006
ISN 0007
ISN 0008
ISN 0009
ISN 0010
ISN 0011
ISN 0012
ISN 0013
ISN 0014
ISN 0015

CRS10460
CRS10470
CRS10480
CRS10490
CRS10500
CRS10510
CRS10520
CRS10530
CRS10540
CRS10550
CRS10560
CRS10570
CRS10580
CRS10590
CRS10600

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Figure 108. (Continued)

REFERENCES

1. Wittlin, G., Gamon, M. A., EXPERIMENTAL PROGRAM FOR THE DEVELOPMENT OF IMPROVED HELICOPTER STRUCTURAL CRASHWORTHINESS ANALYTICAL AND DESIGN TECHNIQUES, Lockheed-California Company; USAAMRDL Technical Report 72-72A, 72B, Eustis Directorate, U. S. Army Air Mobility Research and Development Laboratory, Ft. Eustis, Virginia, May 1973, AD 764985.
2. Turnbow, J. W., Carroll, D. F., Haley, J. L., Jr., Robertson, S. N., CRASH SURVIVAL DESIGN GUIDE, Dynamic Science; USAAMRDL Technical Report 71-22, Eustis Directorate, U. S. Army Air Mobility and Research Laboratory, Ft. Eustis, Virginia, Oct. 1971, AD 733358.
3. Reed, William, H., Avery, James P., Ph.D., PRINCIPLES FOR IMPROVING STRUCTURAL CRASHWORTHINESS FOR STOL AND CTOL AIRCRAFT, Aviation Safety Engineering and Research; USAAVLABS Technical Report 66-39, U. S. Army Aviation Materiel Laboratories, Ft. Eustis, Virginia, June 1966, AD 637133.
4. Greer D. L., et al, DESIGN STUDY AND MODEL STRUCTURES TEST PROGRAM TO IMPROVE FUSELAGE CRASHWORTHINESS, General Dynamics, Convair, FAA Technical Report DS-67-20, Federal Aviation Administration, Washington, D.C., October 1967, AD 666816.
5. Saczalski, K. J., Park, K. C., TRANSIENT RESPONSE OF INELASTICITY CONSTRAINED RIGID BODY SYSTEMS, to appear in Journal of Engineering for Industry, ASME Transactions, 1974.
6. Massonnet, C. E., and Save, M. A., PLASTIC ANALYSIS AND DESIGN, VOL. I, BEAMS AND FRAMES: Blusdell Publishers, New York, 1965.
7. Roark, R. J., FORMULAS FOR STRESS AND STRAIN, McGraw-Hill Publishers, New York, 1965.
8. Ayre, Robert S., Shock and Vibration Hdbk., Chap. 8, Vol. 1, TRANSIENT RESPONSE TO STEP AND PULSE FUNCTIONS., Hoppman, W. K., Chap. 9, Vol. I, EFFECTS OF IMPACT ON STRUCTURES, McGraw-Hill Publishers, New York, 1961.
9. Timoshenko, S., and Woinowsky-Krieger, S., THEORY OF PLATES AND SHELLS, McGraw-Hill Publishers, New York, 1959.
10. Tulk, F. D., BUCKLING OF CIRCULAR CYLINDRICAL SHELLS UNDER DYNAMICALLY APPLIED AXIAL LOADS, UTIAS Report 160, 1972.
11. Striklin, J. E., et al, LARGE DEFLECTION ELASTIC-PLASTIC DYNAMIC RESPONSE OF STIFFENED SHELLS OF REVOLUTION, TEES-RPT-72-25 and SLA-73-0128, 1972.

12. Becker, H., Gerard, G., HANDBOOKS OF STRUCTURAL STABILITY, PARTS I - V. NACA Technical Notes, TN 3781 - 3785, 1957.
13. Skogh, J., and Stern, P., POSTBUCKLING BEHAVIOR OF A SECTION REPRESENTATIVE OF THE B-1 AFT INTERMEDIATE FUSELAGE, Lockheed Palo Alto Research, AFFDL-TR-73-63, Air Force Flight Dynamics Laboratory, Air Force Systems Command, Wright-Patterson Air Force Base, Ohio, May 1973.
14. Atluri, S., PETROS 3: A FINITE-DIFFERENCE METHOD AND PROGRAM FOR THE CALCULATION OF LARGE ELASTIC-PLASTIC DYNAMICALLY-INDUCED DEFORMATIONS OF MULTILAYER VARIABLE THICKNESS SHELLS, BRL, U. S. Army Aberdeen Research and Development Center, Aberdeen Proving Ground, Maryland, Contract #DAADO5-68-C-0314, Nov. 1971.
15. Haftka, R. T., A KOITER-TYPE METHOD FOR FINITE ELEMENT ANALYSIS OF NON-LINEAR STRUCTURAL BEHAVIOR, AFFDL-TR-70-130, Vol. I, Air Force Flight Dynamics Laboratory, Wright-Patterson Air Force Base, Ohio, Nov. 1970.
16. Stilwell, W. C., and Ball, R. E., A DIGITAL COMPUTER STUDY OF THE BUCKLING OF SHALLOW SPHERICAL CAPS AND TRUNCATED HEMISPHERES, NASA CR 1998, June 1972.
17. Witmer, E. A., LARGE DYNAMIC DEFORMATIONS OF BEAMS, RINGS, PLATES AND SHELLS, AIAA Journal, Vol. I, No. 2, August 1963.
18. Perrone, N., ON A SIMPLIFIED METHOD FOR SOLVING IMPULSIVELY LOADED STRUCTURES OF RATE-SENSITIVE MATERIALS, Office of Naval Research, Wash., D. C., Journal of Applied Mechanics, ASME, September 1965.
19. Bodner, S. R., and Symonds, P. S., EXPERIMENTAL AND THEORETICAL INVESTIGATION OF THE PLASTIC DEFORMATION OF CANTILEVER BEAMS SUBJECTED TO IMPULSIVE LOADING, Brown University, Journal of Applied Mechanics, December 1962.
20. Hibbit, H. D., et al, A FINITE ELEMENT FORMULATION FOR PROBLEMS OF LARGE STRAIN AND DISPLACEMENT, Brown University, International Journal of Solids and Structures, 1970, Vol. 6, pp. 1069-1086.
21. Toridio, T. G., and Khozeimeh, K., INELASTIC RESPONSE OF FRAMES TO DYNAMIC LOADS, A.S.C.E. Proceedings, Engineering Mechanics Division, June 1971.
22. McDaniel, T. J., DYNAMICS OF STIFFENED CYLINDRICAL SHELLS WITH SPATIALLY VARYING CURVATURE, University of Dayton Research Institute, Air Force Materials Laboratory Report AFML-TR-72-134, Air Force Systems Command, Wright-Patterson Air Force Base, Ohio, July 1972.
23. Bendix Corp., Final Engineering Report, ENERGY ABSORBING CHARACTERISTICS OF CRUSHABLE ALUMINUM STRUCTURES IN A SPACE ENVIRONMENT, NASA-CR-65096, July 1965.

24. Kornhauser, M., STRUCTURAL EFFECTS OF IMPACT, Sparton Books, Inc., Baltimore, Md. 1964.
25. Fisher, L. J., Jr., LANDING-IMPACT-DISSIPATION SYSTEMS, NASA Technical Note D-975, December 1961.
26. McGehee, J. R., A PRELIMINARY EXPERIMENTAL INVESTIGATION OF AN ENERGY-ABSORPTION PROCESS EMPLOYING FRANGIBLE METAL TUBING, NASA Technical Note D-1477, 1962.
27. Kroell, C. K., A SIMPLE, EFFICIENT, ONE SHOT ENERGY ABSORBER, General Motors Research Laboratory, Warren, Michigan, Shock, Vibration and Associated Environments, Part III, Bulletin No. 30, 1962.
28. Weinberg, L. W. T., and Turnbow, J. W., Ph.D., SURVIVABILITY SEAT DESIGN DYNAMIC TEST PROGRAM, Aviation Safety Engineering and Research USAAVLABS Technical Report 65-43, U.S. Army Aviation Materiel Laboratories, Ft. Eustis, Virginia, July, 1965, AD 621718.
29. Langhaar, H. L., THEORETICAL AND EXPERIMENTAL INVESTIGATIONS OF THIN-WEBBED PLATE-GRIDER BEAMS, Transactions of ASME, October 1943.
30. Perry, D. J., AIRCRAFT STRUCTURES, McGraw-Hill Book Co., New York, 1950.
31. Jones, N., et al, THE DYNAMIC PLASTIC BEHAVIOR OF SHELLS, MIT Report 71-6, 1971.
32. Mitchell, B., THE DYNASORB ENERGY ABSORBER, Lockheed Report LR 16735, March 1963.
33. Mitchell, B., SHOCK ABSORPTION WITH ONE SHOT TUBES, Lockheed Report LR 16369, June 1963.
34. Mitchell, B., DESIGN NOTES FOR THE DYNASORB ENERGY ABSORBER, Lockheed Report LR 17201, December 1963.
35. Mitchell, B., ENERGY ABSORPTION AT HIGH SPEED VERTICAL LANDING, Lockheed Report LR 21023, November 1967.
36. Perrone, N., RESPONSE OF RATE SENSITIVE FRAMES TO IMPULSIVE LOAD, ASME Journal of Applied Mechanics, February 1971, pp. 49-62.
37. Jones, Norman, INFLUENCE OF STRAIN-HARDENING AND STRAIN-RATE SENSITIVITY ON THE PERMANENT DEFORMATION OF IMPULSIVELY LOADED RIGID PLASTIC BEAMS, International Journal of Mechanical Science, 1967 Vol. 9, pp. 777-796.
38. Ni, C. M., IMPACT RESPONSE OF CURVED BOX BEAM-COLUMNS WITH LARGE GLOBAL AND LOCAL DEFORMATIONS, Research Laboratories, General Motors Corp., Warren, Michigan, AIAA Paper 73-401.

39. O'Bryan, T. C., and Hatch, H. G., Jr., LIMITED INVESTIGATION OF CRUSH-ABLE STRUCTURES FOR ACCELERATION PROTECTION OF OCCUPANTS OF VEHICLES AT LOW IMPACT SPEEDS, NASA Technical Note D-158, 1958.
40. Jones, Norman, THE INFLUENCE OF LARGE DEFLECTION ON THE BEHAVIOR OF RAPID-PLASTIC CYLINDRICAL SHELLS LOADED IMPULSIVELY, Journal of Applied Mechanics, ASME June 1970, pp. 417-425.
41. D'Amato, R., STATIC POST-FAILURE STRUCTURAL CHARACTERISTICS OF MULTI-WEB BEAMS, WADC TR 59-112, February 1959.
42. Rawlings, B., ENERGY ABSORPTION OF DYNAMICALLY AND STATICALLY TESTED MILD STEEL BEAMS UNDER CONDITIONS OF GROSS DEFORMATION, International Journal of Mechanical Sciences, Pergamon Press, Ltd., 1967, Vol. 9, pp. 633-649, Printed in Great Britain.
43. Thompson, J. E., VEHICLE CRUSH PREDICTION USING FINITE-ELEMENT TECHNIQUES, Chrysler Corp., SAE Paper 73-157, January 1973.
44. Burgmann, J. B., and Rawlings, B., DYNAMIC PLASTIC ANALYSIS OF PIN-JOINTED FRAMES, International Journal of Mechanical Science, Pergamon Press, 1968, Vol. 10, pp. 967-980, Printed in Great Britain. (Received 8 May 1968, and in revised form 30 July 1968).
45. Martin, J. B., MODE APPROXIMATION FOR IMPULSIVELY LOADED STRUCTURES IN THE INELASTIC RANGE, Proceedings of the Southampton 1969 Civil Engineering Material Conference.
46. Jensen, W. R., Falby, W. E., and Prince, N., MATRIX ANALYSIS METHODS FOR ANISOTROPIC INELASTIC STRUCTURES, AFFDL-TR-65-220, April 1966.
47. Isaakson, G., Armen, H., Jr., and Pipko, A., DISCRETE ELEMENT METHODS FOR THE PLASTIC ANALYSIS OF STRUCTURES, NASA CR 803, October 1967.
48. Stricklin, J. A., et al, NONLINEAR DYNAMIC ANALYSIS OF SHELLS OF REVOLUTION BY MATRIX DISPLACEMENT METHOD, AIAA Journal, Vol. 9, No. 4, April 1971, p. 629.
49. Owens, R. H., and Symonds, P. A., PLASTIC DEFORMATION OF A FREE RING UNDER CONCENTRATED DYNAMIC LOADING, ASME Journal of Applied Mechanics, December 1955, p. 524.
50. Lee, H., and Symonds, P. S., LARGE PLASTIC DEFORMATIONS OF BEAM UNDER TRANSVERSE IMPACT, ASME Journal of Applied Mechanics, September 1952, p. 308.
51. Prager, W., A NEW METHOD OF ANALYZING STRESSES AND STRAINS IN WORK-HARDENING PLASTIC SOLIDS, ASME Journal of Applied Mechanics, December 1956, p. 493.

52. Morino, L., Leach, J. W., and Witmer, E. A., AN IMPROVED NUMERICAL CALCULATION TECHNIQUE FOR LARGE ELASTIC-PLASTIC TRANSIENT DEFORMATIONS OF THIN SHELLS, Part 1, ASME Journal of Applied Mechanics, June 1971, p. 423.
53. Pifko, A. Isaakson, A FINITE-ELEMENT METHOD FOR THE PLASTIC BUCKLING ANALYSIS OF PLATES, Grumman Aerospace Corporation, Bethpage, N. Y., AIAA Journal, Vol. 7, No. 10, October 1969.
54. Armen, H., Jr., Pifko, A., and Levine, H. S., FINITE ELEMENT ANALYSIS OF STRUCTURES IN THE PLASTIC RANGE, NASA CR 1649, February 1971.
55. Mallett, R. H., AUTOMATED METHOD FOR THE LARGE DEFLECTION AND INSTABILITY ANALYSIS OF THREE-DIMENSIONAL TRUSS AND FRAME ASSEMBLIES, AFFDL-TR-66-102, December 1966.
56. Przemieniecki, J. S., MATRIX METHODS IN STRUCTURAL MECHANICS, AFFDL-TR-66-80, Conference held October 26-28, 1965.
57. Berke, L., PROCEEDINGS OF THE SECOND CONFERENCE ON MATRIX METHODS IN STRUCTURAL MECHANICS, AFFDL-TR-68-150, Conference held 15-17 October 1968.
58. Symonds, P. S., SURVEY OF METHODS OF ANALYSIS FOR PLASTIC DEFORMATION OF STRUCTURES UNDER DYNAMIC LOADING, BU/NSRDC/1-67, Brown University, Providence, R. I., 1967.
59. Semonian, J. W., and Anderson, R. A., AN ANALYSIS OF THE STABILITY AND ULTIMATE BENDING STRENGTH OF MULTIWEB BEAMS WITH FORMED CHANNEL WEBS, NACA Technical Note TN 3232, 1954.
60. Semonian, J. W., and Peterson, J. P., AN ANALYSIS OF THE STABILITY AND ULTIMATE COMPRESSIVE STRENGTH OF SHORT SHEET STRINGER PANELS WITH SPECIAL REFERENCE TO THE INFLUENCE OF THE RIVETED CONNECTION BETWEEN SHEET STRINGERS, NACA Technical Report TR 1255, 1956.
61. Bleich, F., BUCKLING STRENGTH OF METAL STRUCTURES, McGraw-Hill Publishers, New York, 1952.
62. Newmark, N. M., A METHOD OF COMPUTATION FOR STRUCTURAL DYNAMICS, A.S.C.E. Proceedings, Engineering Mechanics Divisions, Vol. 85, EM-3, pp. 67-94, 1959.
63. Bushnell, D., et al, FINITE-DIFFERENCE ENERGY METHOD FOR NONLINEAR SHELL ANALYSIS, Journal of Computer & Structures, Vol. 1, 1971, pp. 361-387.
64. Koiter, W. T., Ph.D. Thesis, NASA TTF-10, 883, 1967, WPAFB, AFFDL-TR-70-25, 1970.

65. Sewell, M. J., A GENERAL THEORY OF EQUILIBRIUM PATHS THROUGH CRITICAL POINTS, Royal Society Proceedings, Series A, 306, pp. 201-223, 1968.
66. Tong, P., A NOTE ON POST-BUCKLING SOLUTION OF ELASTIC STRUCTURES, ASRL-TR 1444-4, MIT, January 1970.
67. Chu, K. H., and Gonwens, A. J., ELASTIC BUCKLING OF BEAM FLANGE PLATES, EM-4, ASCE, pp. 1324-1336, August 1971.
68. El-Bayoumy, L., BUCKLING OF CLAMPED RECTANGULAR PLATES, ASCE, EM-4, pp. 1277-1288, August 1971.
69. Stein, M., THE PHENOMENON OF CHANGE IN BUCKLE PATTERN IN ELASTIC STRUCTURES, NASA Technical Note TN R-39, 1959.
70. Rhodes, J., and Harvey, J. M., THE LOCAL BUCKLING AND POST-LOCAL BUCKLING BEHAVIOR OF THIN-WALLED BEAMS, Aeronautical Quarterly, 22, 4, pp. 363-388, November 1971.
71. Rhodes, J., THE POST-BUCKLING BEHAVIOR OF THIN FLAT PLATES IN COMPRESSION WITH UNLOADED EDGES ELASTICALLY RESTRAINED AGAINST ROTATION, Journal of Mechanical Engineering Sciences, 13, 2, pp. 82-91, April 1971.
72. Graves, A., Smith, T. R., THE EFFECT OF INITIAL IMPERFECTIONS ON THE STRENGTH OF THIN WALLED BOX COLUMNS, International Journal of Mechanical Sciences, 13, 11, pp. 911-925, November 1971.
73. Hutchinson, S. W., and Koiter, W. T., POST-BUCKLING THEORY, Applied Mechanics Reviews, December 1970.
74. Sewell, M. J., A GENERAL THEORY OF EQUILIBRIUM PATHS THROUGH CRITICAL POINTS, Proceedings of Royal Society, Series A, 306, pp. 201-223, 1968.
75. Cox, H. L., THE BUCKLING OF PLATES AND SHELLS, The MacMillan Company, N. Y., 1963.
76. Chitner, A. H., THIN-WALLED STRUCTURES, John & Wiley & Sons, Inc., New York, 1967.
77. Mau, S. T., and Gallagher, R. H., A FINITE ELEMENT PROCEDURE FOR NON-LINEAR PRE-BUCKLING AND INITIAL POST-BUCKLING ANALYSIS, NASA CR-1936, January 1972.
78. Vos, Robert F., FINITE ELEMENT ANALYSIS OF PLATE BUCKLING AND POST-BUCKLING, Ph.D. Thesis, Rice University, Houston, Texas, 1970.
79. Roberts, T. M., and Ashwell, D. G., THE USE OF FINITE ELEMENT MID-INCREMENT STIFFNESS MATRICES IN THE POST-BUCKLING ANALYSIS OF IMPERFECT STRUCTURES, Journal of Solids and Structures, 7.7, pp. 805-823, July 1971.

80. Greer, D. L., et al, CRASHWORTHY DESIGN PROCEDURES, General Dynamics, CONVAIR, FAA Technical Report ADS-24, Federal Aviation Administration, Washington, D. C., November 1965, AD 623575.
81. Stein, M., LOADS AND DEFORMATIONS OF BUCKLED RECTANGULAR PLATES, NACA Technical Report, TR-R-40, 1959, Washington, D. C.
82. Gatlin, Clifford I., et al, ANALYSIS OF HELICOPTER STRUCTURAL CRASH-WORTHINESS, Dynamic Science, USAAVLABS Technical Report 70-71, 71B, Eustis Directorate, U.S. Army Air Mobility Research and Development Laboratory, Ft. Eustis, Virginia, January 1971, AD880680.
83. Sechler, E. E., and Dunn, L. G., AIRPLANE STRUCTURAL ANALYSIS AND DESIGN, Dover Publishers Inc., 1963, New York, N. Y.
84. Hutchinson, J. W., and Budiansky, B., DYNAMIC BUCKLING ESTIMATES, AIAA Journal 4, pp. 525-530, 1966.
85. Thompson, J.M.T., DYNAMIC BUCKLING UNDER STEP LOADING, IN DYNAMIC STABILITY OF STRUCTURES (Ed. by G. Herrmann), Pergamon Press, 1965.
86. Budiansky, B., DYNAMIC BUCKLING OF ELASTIC STRUCTURES. CRITERIA AND ESTIMATES IN DYNAMIC STABILITY AND STRUCTURES (Ed. by G. Herrmann), Pergamon Press, 1965.
87. Mayer, J., and Budiansky, B., ANALYSIS OF BEHAVIOR OF SIMPLY SUPPORTED FLAT PLATES COMPRESSED BEYOND THE BUCKLING LOAD INTO THE PLASTIC RANGE, NACA TN 3368, Washington, D. C., 1955.
88. Walker, A. C., FLAT RECTANGULAR PLATES SUBJECTED TO LINEARLY-VARYING EDGE COMPRESSIVE LOADING, in Thin-Walled Structures (Ed. by A. H. Chilner), John Wiley & Sons, Inc., New York, 1967.
89. Walker, A. C., THE POST-BUCKLING BEHAVIOR OF SIMPLY SUPPORTED SQUARE PLATES, Aero. Quarterly, Vol. XX, August 1, 1969, pp. 203-222.
90. Masur, E. F., BUCKLING, POST-BUCKLING AND LIMIT ANALYSIS OF COMPLETELY SYMMETRIC ELASTIC STRUCTURES, International Journal of Solids and Structures, 1970, Vol. 6, pp. 587-604.
91. Hutchinson, J. W., and Koiter, W. T., POST-BUCKLING THEORY, Applied Mechanics Review, December 1970, pp. 1353-1366.
92. Needham, R. A., THE ULTIMATE STRENGTH OF ALUMINUM ALLOY FORMED STRUCTURAL SHAPES IN COMPRESSION, Journal of Aeronautical Sciences, Vol. 21, No. 1, 1954, pp. 217-229.
93. Schuette, E. H., OBSERVATIONS OF THE MAXIMUM AVERAGE STRESS OF FLAT PLATES BUCKLED IN EDGE COMPRESSION, NACA TN 1625, 1949.

94. Semonian, J. W., and Peterson, J. P., AN ANALYSIS OF THE STABILITY AND ULTIMATE COMPRESSIVE STRENGTH OF SHORT SHEET-STRINGER PANELS WITH SPECIAL REFERENCE TO THE INFLUENCE OF RIVETED CONNECTION BETWEEN SHEET AND STRINGER, NACA Report 1255, 1956.
95. Stowell, E., A UNIFIED THEORY OF PLASTIC BUCKLING OF COLUMNS AND PLATES, NACA Report 898, 1947.
96. Dow, N. F., and Hickman, W. A., EFFECT OF VARIATION IN RIVET DIAMETER AND PITCH ON THE AVERAGE STRESS AT MAXIMUM LOAD FOR 24S-T3 AND 75S-T6 ALUMINUM-ALLOY, FLAT, Z-STIFFENED PANELS THAT FAIL BY LOCAL INSTABILITY, NACA TN 2139, 1950.
97. Dow, N. F., Hickman, W. A., and Rosen, B. W., EFFECT OF VARIATION IN RIVET STRENGTH ON THE AVERAGE STRESS AT MAXIMUM LOAD FOR ALUMINUM-ALLOY, FLAT, Z-STIFFENED COMPRESSION PANELS THAT FAIL BY LOCAL BUCKLING, NACA TN 2963, 1953.
98. Von Karman, T. L., Sechler, E. E., and Donnel, L. H., THE STRENGTH OF THIN PLATES IN COMPRESSION, Journal of Applied Mechanics Transcripts, A.S.M.E., Vol. 54, No. 2, 1932.
99. Marguerre, K., THE APPARENT WIDTH OF THE PLATE IN COMPRESSION, NACA Technical Memo 933, 1937.
100. Timoshenko, S. P., and Gere, J. M., THEORY OF ELASTIC STABILITY, 2nd Ed., McGraw-Hill Book Co., New York, 1961.
101. Neal, B. G., THE EFFECT OF SHEAR AND NORMAL FORCES ON THE FULLY PLASTIC MOMENT OF A BEAM OF RECTANGULAR CROSS-SECTION, Journal of Applied Mechanics, Vol. 28, 1961, pp. 269-274.
102. Vlasov, V. Z., THIN-WALLED ELASTIC BEAMS, English Translation by National Science Foundation, Second Ed., pp. 286-288, 335-368.
103. Hodge, P. H., Jr., PLASTIC ANALYSIS OF STRUCTURES, McGraw-Hill Book Co., New York, 1959.
104. Neal, B. G., THE PLASTIC METHODS OF STRUCTURAL ANALYSIS, Chapman & Hall, London, 1956.
105. Heimerl, G. J., and Woods, W., EFFECT OF BRAKE FORMING ON THE STRENGTH OF 23S-TITANIUM ALLOY SHEET, NACA Technical Note TN 1072, May 1946.
106. Haley, J., PRELIMINARY DRAFT, COST EFFECTIVENESS OF CRASHWORTHY STRUCTURAL FEATURES IN A 9-13 PLACE HELICOPTER, USAAVS Technical Report 72-1, U. S. Army Agency for Aviation Safety, Ft. Rucker, Alabama, 1972.

107. Walker, A. C., THE POST-BUCKLING BEHAVIOR OF SIMPLY-SUPPORTED SQUARE PLATES, The Aeronautical Quarterly, Vol. XX, 1969, pp. 203-222.
108. Kuhn, P., Petterson, J. P., and Levin, L. R., A SUMMARY OF DIAGONAL TENSION, PART I - METHODS OF ANALYSIS, NACA TN 2661, 1952, Washington, D. C., p. 32.
109. Lowe, W. T., et al, IMPACT BEHAVIOR OF SMALL SCALE MODEL MOTOR COACHES, Proceedings of the Institute of Mechanical Engineers, Vol. 186, No. 36, 1972, pp. 409-419.
110. Saczulski, K. J., and Park, K. C., AN INTERACTIVE HYBRID TECHNIQUE FOR CRASHWORTHY DESIGN OF COMPLEX VEHICLE STRUCTURAL SYSTEMS, To be presented at S.A.E. International Conference of Vehicle Structural Mechanics, Detroit, Michigan, March 26-28, 1974.
111. Hone, C. P., TORSIONAL-FLEXURAL BUCKLING OF AXIALLY-LOADED, THIN-WALLED, ELASTIC STRUTS OF OPEN CROSS-SECTION IN THIN-WALLED STRUCTURES, (Ed. by A. H. Chilver), John Wiley & Sons, Inc., New York, 1967.
112. ENGINEERING DATA, HEXCELL TECHNICAL SERVICE BULLETINS, TB 110 through TB 123, 1960 - 1968, Hexcell Products Inc., Berkeley, California.

LIST OF SYMBOLS

SUBSTRUCTURE ANALYSIS

A	area; for stiffened panel, area of stiffener plus area of sheet corresponding to stiffener spacing, in. ²
a	plate length, in.
A _s	area of stiffener, in. ²
A _w	area of web, in. ²
b _s	width of skin, in.
b _w	width of stiffener web, in.
C	constant (= 3.0)
c	number of cuts
d	rivet diameter, in.
E	modulus of elasticity, psi
e, e _o	distance to neutral axis, in.
f	number of flanges
f _{cr}	crush strength, lb
f _w	effective rivet off-set, in.
G	acceleration level
g	gravity term = 386.2 in./sec ²
H	distance, in.
I	moment of inertia, in. ⁴
I _o	moment of inertia about c.g., in. ⁴
K	eccentricity factor
KE	kinetic energy, in.-lb
K _w	wrinkling coefficient
L, l	one-half column length, in.
L'	effective length = L/\sqrt{C} , in.
M	effective moment, in.-lb

SUBSTRUCTURE ANALYSIS

M_p	plastic hinge moment, in.-lb
N/A	neutral axis
N	effective axial force, lb
N_p	plastic hinge axial force, lb.
p	rivet pitch, in.
P	applied axial force, lb
P_f	monolithic failure load, lb
P_{fw}	wrinkling failure load, lb
q	distributed load, lb/in.
S	stopping distance, core radius of given cross section, in.
s	distance from midcross-section of skin to neutral axis of stiffener, in.
T	ratio of σ/σ_{cy}
T_{cr}	ratio of critical stress (σ_{cr}) to yield stress (σ_{cy})
t_c	minimum core thickness, in.
t_s	skin thickness, in.
\bar{t}_w	stiffener thickness, in.
V	impact velocity, in./sec
W	weight, lb
w_e	effective half width of skin, in.
y	lateral deflection
Z	function of column length and end shortening, in.
α	defined in equation 51
η	plasticity reduction factor
$\bar{\eta}$	cladding reduction factor
π	$P_i = 3.14$
θ	$\tan^{-1} Z$
ρ	radius of gyration = $\sqrt{I/A}$, in.
γ	partial derivative

SUBSTRUCTURE ANALYSIS (Continued)

δu	incremental displacement
δ, Δ, u	displacement, end shortening
ν	Poisson's ratio
x	distance to c.g., in.
σ	compressive stress, psi
σ_b	stress at outermost fiber of sheet skin, psi
σ_{co}	column buckling stress, psi
σ_{cr}	buckling stress, psi
σ_{cy}	compressive yield stress, psi
$\bar{\sigma}_{cy}$	effective compressive yield stress, psi
σ_{cy_w}	effective compressive yield stress for stiffener web, psi
σ_{cy_s}	effective compressive yield stress for skin, psi
σ_e	Euler stress, psi
σ_f	crippling (or failure) stress, psi
σ_{fr}	failure stress of riveted panel, psi
σ_w	failure stress due to wrinkling mode, psi
σ_{zo}	buckling stress at $L'/\rho = 20$

Subscripts:

b	bending
f	filler
i	i^{th} segment
max.	maximum
min.	minimum
p	plastic state
s	skin or sheet
w	stiffener web

SUBSTRUCTURE ANALYSIS (Continued)

Constants (function of stiffened panel configuration)

θ_f

θ_g

g

m

n

PROGRAM "KPASH"

$[A_i]$	rotational transformation matrix from body axes to ground axes
$[A_i]^T$	transpose of A_i matrix
CE	total crash spring (external spring) energy absorbed
DE	total damping energy dissipated
DRI	dynamic response index
E_{TOT}	total system energy
$FD_{ij\ell}$	internal beam damping force (or moment) for the ij^{th} beam in the ℓ^{th} direction
FSP_{ijk}	crash spring forces; spring ij in the k^{th} direction
$F_{ij\ell}$	force (or moment) at point j due to beam ij , in the ℓ^{th} direction
I_{xi}, I_{yi}, I_{zi}	moments of inertia of lumped mass m_i about i^{th} body fixed axes
$I_{xyi}, I_{yzi}, I_{zxi}$	product of inertia of lumped mass m_i about i^{th} body axes
KE	total kinetic energy
l_{ik}	length of vector from m_i to ground contact point C'_{ik}
m_i	i^{th} lumped mass
N	number of lumped masses
PE	total potential energy
SE	total strain energy absorbed
$TERM_{ij}$	term in expression for crash spring energy

PROGRAM "KRASH" (Continued)

W_i	weight of i^{th} lumped mass
x_i, y_i, z_i	ground coordinates of m_i
u_i, v_i, w_i	i^{th} body axes component of absolute translational velocity vector of mass i
p_i, q_i, r_i	i^{th} body axes components of absolute angular velocity vector of mass i
$\begin{pmatrix} X_{ci}, Y_{ci}, Z_{ci} \\ L_{ci}, M_{ci}, N_{ci} \end{pmatrix}$	crash (external) forces and moments, i^{th} body axes
$\Delta x'_i, \Delta y'_i, \Delta z'_i$	$= \{ \Delta v_{ci} \}$, incremental displacement of point i , i^{th} body axes
$\Delta \text{inp}_i, \Delta \text{inq}_i, \Delta \text{inr}_i$	incremental rotation of point i , i^{th} body axes
Δv_{ijl}	incremental displacement vector of point j with respect to point i , due to deformation of beam ij , in the l^{th} direction

USER'S GUIDE AND REFERENCE 1 TERMS

A_i, B_i, C_i	Terms used in Euler's equations of motion
$[A_i]$	Rotation transformation matrix from body axes to ground axes
$[\bar{A}_i]$	Matrix relating $(\phi_i, \theta_i, \psi_i)$ to (p_i, q_i, r_i) in equation (92)
$[\dot{A}_i]$	Time derivative of $[A_i]$
$[A_i'']$	Rotation transformation matrix from ith body axes to c.g. axes
$[A_{ij}]$	Rotation transformation matrix from beam ij axes to body i axes
$[A']$	Rotation transformation matrix from c.g. axes to ground axes
$[\bar{A}']$	Matrix relating (ϕ', θ', ψ') to (p', q', r') in equation (88)
C_{ik}	End point of kth spring on ith mass
C_{ik}^i	Ground contact point of kth spring on ith mass
dvc_{ijk}	Ground axes components of vector from m_i to C_{ik}
dvc_{ijk}^i	Ground axes components of vector for C_{ik}^i to C_{ik}
$[D_i]$	Derivative matrix
$[D']$	Derivative matrix
FM_{ijkl}	Running time sum of ΔFM_{ijkl}
\overline{FM}_{ijkl}	Value of FM_{ijkl} at time of loading reversal
FSP_{ijk}	Body i axes components of spring force at ground contact point C_{ik}
$FSPO_{ik}$	Axial compressive force in kth spring on ith mass
\overline{FSPO}_{ik}	Value of $FSPO_{ik}$ at time of loading reversal




USER'S GUIDE AND REFERENCE 1 TERMS

$FSPO_{fik}$	Final value of $FSPO_{ik}$ in input table of s_{ik} vs. $FSPO_{ik}$
G	Center-of-gravity of total vehicle
H	Origin of helicopter coordinate system (F.S.O, B.L.O, W.L.O)
$He_{xi}, He_{yi}, He_{zi}$	Angular momenta of m_i due to rotation of masses internal to m_i
I_{xi}, I_{yi}, I_{zi}	Moments of inertia of lumped mass m_i , about i th body fixed axes
$I_{xyi}, I_{yzi}, I_{zxi}$	Products of inertia of lumped mass m_i , about i th body fixed axes
ke_{ik}	Linear unloading stiffness for k th spring
$[K_{ij}]$	Six by six linear stiffness matrix for beam ij
$[KR_{ij}]$	Six by six diagonal stiffness reduction matrix for beam ij
l_{ik}	Length of vector from m_i to ground contact point C'_{ik}
$\bar{l}_{xi}, \bar{l}_{yi}, \bar{l}_{zi}(\bar{l}_{ik})$	Free length of k th spring on i th mass
l_{ci}	Aerodynamic lift constant
$LIFT_1$	Aerodynamic lift on m_1 , positive up, in ground axes
m_i	i th lumped mass
μ_{ik}	Ground-spring friction coefficient for k th spring on i th mass
N	Total number of lumped masses
\bar{n}_{ik}	Unit vector triad fixed in i th body coordinate system
$\bar{n}_x, \bar{n}_y, \bar{n}_z$	Unit vector triad fixed in ground coordinate system
O	Origin of ground coordinate system

USER'S GUIDE AND REFERENCE 1 TERMS

p_i, q_i, r_i	i th body axes components of absolute angular velocity vector of mass i
p', q', r'	c.g. axes components of initial($t=0$) vehicle angular velocity vector
$[p_{1i}]$	Contact point velocity matrix used in equation (60)
s_{ik}	Axial external spring compression, k th spring on i th mass
\bar{s}_{ik}	Value of s_{ik} at time of loading reversal
s_{fik}	Final value of s_{ik} in input table of s_{ik} vs. $FSPO_{ik}$
s_{ik}	k th spring axial compression measured relative to current load stroke curve origin
s_{ik}''	Horizontal shift of s_{ik} coordinates with respect to s_{ik} coordinates
t	Time
$[T_{1j}]$	Static balance matrix used in equation (30b)
u_i, v_i, w_i	Body i axes components of absolute translational velocity vector of point m_i
vb_{ij}	x_i, y_i, z_i
vb_{ij}	Running time sum of Δvb_{ij}
\bar{vb}_{ijl}	Value of vb_{ijl} at time of loading reversal
\bar{v}_{ik}	Magnitude of ground plane contact point velocity
vb_{ijl}^i	i th total beam deflection measured relative to current load-stroke curve origin
vb_{ijl}''	Horizontal shift of vb_{ijl}^i coordinates with respect to vb_{ijl} coordinates
vc_{ijk}	Ground coordinates of point C_{ik}

USER'S GUIDE AND REFERENCE 1 TERMS

$\dot{v}_{cp_{ijk}}$	Ground axes components of absolute velocity of ground contact point C_{ik}
	Velocity vector of C'_{ik} with respect to m_1
	Velocity vector of C'_{ik} with respect to ground
	Velocity vector of m_1 with respect to ground
W_i	Weight of i th lumped mass
W_{TOT}	Total vehicle weight
$x_{b_{ij}}, y_{b_{ij}}, z_{b_{ij}}$	Beam ij coordinates
$x_G, y_G, z_G (v_{G_j})$	Ground coordinates of initial ($t = 0$) c.g. position
$\dot{x}_G, \dot{y}_G, \dot{z}_G$	Ground axes components of initial ($t = 0$) c.g. velocity vector
$x''_G, y''_G, z''_G (v_{GPP_{ij}})$	Helicopter axes coordinates of vehicle c.g. (point G)
$x_i, y_i, z_i (v_{a_{ij}})$	Ground coordinates of m_1
$x^i_i, y^i_i, z^i_i (v_{ip_{ij}})$	Coordinates of m_1 in center-of-gravity coordinate system
$x''_i, y''_i, z''_i (v_{ipp_{ij}})$	Coordinates of m_1 in helicopter coordinate system
x_{ij}, y_{ij}, z_{ij}	Ground coordinates of vector from point i to point j
$\dot{x}_{ij}, \dot{y}_{ij}, \dot{z}_{ij}$	i th body coordinates of vector from point i to point j
$\begin{pmatrix} X^i_{ij} & Y^i_{ij} & Z^i_{ij} \\ L^i_{ij} & M^i_{ij} & N^i_{ij} \end{pmatrix}$	Total (summed over time) internal forces and moments at point i due to beam ij , i^{th} body axes
$\begin{pmatrix} X^j_{ji} & Y^j_{ji} & Z^j_{ji} \\ L^j_{ji} & M^j_{ji} & N^j_{ji} \end{pmatrix}$	Total (summed over time) internal forces and moments at point j due to beam ij , j^{th} body axes

USER'S GUIDE AND REFERENCE 1 TERMS

$\begin{pmatrix} X_i, Y_i, Z_i \\ L_i, M_i, N_i \end{pmatrix}$	Total forces and moments on mass i, in ith body axes
X_{Ai}, Y_{Ai}, Z_{Ai}	Aerodynamic forces, i^{th} body axes
$\begin{pmatrix} X_{Ci}, Y_{Ci}, Z_{Ci} \\ L_{Ci}, M_{Ci}, N_{Ci} \end{pmatrix}$	Crash (external) forces and moments, i^{th} body axes
X_{Gi}, Y_{Gi}, Z_{Gi}	Gravity forces, i^{th} body axes
$\begin{pmatrix} X_{Ii}, Y_{Ii}, Z_{Ii} \\ L_{Ii}, M_{Ii}, N_{Ii} \end{pmatrix}$	Internal forces and moments, i^{th} body axes
$XVOC_{ijk}$	Ground axes components of spring force at ground contact point C_{ik} , positive up, left and aft
z_{MAX}	Vertical distance from c.g. to lowest C_{ik}
Δ_i	Determinate expression used in equation (68)
ΔF_{ijk}	Incremental forces and moments at point j due to beam ij
ΔF_{ijk}^{kl}	kth incremental load due to lth incremental deflection for beam ij
$\Delta \phi_{ij}, \Delta \theta_{ij}, \Delta \psi_{ij}$	Incremental rotations of point j with respect to point i, in beam ij axes
$\Delta \phi_i, \Delta \theta_i, \Delta \psi_i$	Incremental change in ith mass Euler angles
Δt	Numerical integration time interval
Δv_{bij}	Six element vector made up of $\Delta x_{bij}, \Delta y_{bij}, \Delta z_{bij}, \Delta \phi_{bij}, \Delta \theta_{bij}, \Delta \psi_{bij}$
$\overrightarrow{\Delta v}_{bij}$	Incremental displacement vector of point j with respect to point i, due to deformation of beam ij
$\overrightarrow{\Delta v}_{dij}$	Incremental displacement vector of point j with respect to point i

USER'S GUIDE AND REFERENCE 1 TERMS

$$\overrightarrow{\Delta v_{ij}}$$

Incremental displacement vector of point j with respect to point i, due to rotation of mass i

$$\Delta x_{bij}, \Delta y_{bij}, \Delta z_{bij}$$

Coordinates of $\overrightarrow{\Delta v_{bij}}$ in beam ij axes

$$\Delta x_i, \Delta y_i, \Delta z_i$$

Incremental displacement of point i, ground axes

$$\Delta x_{ij}, \Delta y_{ij}, \Delta z_{ij}$$

Incremental displacement of point j with respect to point i in ground axes

$$\begin{pmatrix} \Delta X_{ij}, \Delta Y_{ij}, \Delta Z_{ij} \\ \Delta L_{ij}, \Delta M_{ij}, \Delta N_{ij} \end{pmatrix}$$

Incremental internal forces and moments at point j due to beam ij, in beam ij axes (elements of ΔF_{ij} vector)

$$\begin{pmatrix} \Delta X_{ij}^0, \Delta Y_{ij}^0, \Delta Z_{ij}^0 \\ \Delta L_{ij}^0, \Delta M_{ij}^0, \Delta N_{ij}^0 \end{pmatrix}$$

Incremental internal forces and moments at point j due to beam ij, ground axes

$$\begin{pmatrix} \overline{\Delta X_{ij}}, \overline{\Delta Y_{ij}}, \overline{\Delta Z_{ij}} \\ \overline{\Delta L_{ij}}, \overline{\Delta M_{ij}}, \overline{\Delta N_{ij}} \end{pmatrix}$$

Incremental internal forces and moments at point i due to beam ij, ground axes

$$\begin{pmatrix} \Delta X'_{ij}, \Delta Y'_{ij}, \Delta Z'_{ij} \\ \Delta L'_{ij}, \Delta M'_{ij}, \Delta N'_{ij} \end{pmatrix}$$

Incremental internal forces and moments at point i due to beam ij, i^{th} body axes

$$\begin{pmatrix} \Delta X'_{ji}, \Delta Y'_{ji}, \Delta Z'_{ji} \\ \Delta L'_{ji}, \Delta M'_{ji}, \Delta N'_{ji} \end{pmatrix}$$

Incremental internal forces and moments at point j due to beam ij, j^{th} body axes

$$\phi_i, \theta_i, \psi_i$$

Euler angles from ground axes to body axes (time varying)

$$\phi_{ij}, \theta_{ij}, \psi_{ij}$$

Euler angles from i^{th} body axes to beam ij axes (constant)

$$\phi', \theta', \psi'$$

Euler angles from ground axes to c.g. axes (constant); initial ($t=0$) attitude of vehicle

$$\phi''_i, \theta''_i, \psi''_i$$

Euler angles from c.g. axes to i^{th} body axes (constant)